



SATHYABAMA

INSTITUTE OF SCIENCE AND TECHNOLOGY

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SCHOOL OF ELECTRICAL AND ELECTRONICS

DEPARTMENT OF ELECTRICAL AND ELECTRONICS

UNIT – I – Advanced Control Systems – SEEA1602

STATE SPACE ANALYSIS

1.1 INTRODUCTION

The state variable approach is a powerful tool / technique for the analysis and design of control systems. The analysis and design of the following systems can be carried using state space method.

1. Linear system
2. Non-linear system
3. Time invariant system
4. Time varying system
5. Multiple input and multiple output system.

The state space analysis is a modern approach and also easier for analysis using digital computers. The conventional (or old) methods of analysis employs the transfer function of the system. The drawbacks in the transfer function model and analysis are,

1. Transfer function is defined under zero initial conditions.
2. Transfer function is applicable to linear time invariant systems,
3. Transfer function analysis is restricted to single input and single output systems.
4. Does not provides information regarding the internal state of the system.

The state variable analysis can be applied for any type of systems. The analysis can be carried with initial conditions and can be carried on multiple input and multiple output systems. In this method of analysis, it is not necessary that the state variables represent physical quantities of the system, but variables that do not represent physical quantities and those that are neither measurable not observable may be chosen as state variables.

1.2 STATE SPACE FORMULATION

The state of a dynamic system is a minimal set of variable (known as state variables) such that the knowledge of these vairables at $t = t_0$ together with the knowledge of the inputs fo $t \geq t_0$, completely determibnes the behaviour of the sytem for $t > t_0$ (or) A set of vairables which describes the system at any time instant are called state variables.

In the state variable formulation of a system, in general, a system consists of m-inputs, p-outputs and n-state variabls. The state space representation of the system may be visualized in Figure 1.1.

Let, State variables = $x_1(t), x_2(t), x_3(t), \dots, x_n(t)$
 Input variables = $u_1(t), u_2(t), u_3(t), \dots, u_m(t)$
 Output variables = $y_1(t), y_2(t), y_3(t), \dots, y_p(t)$,

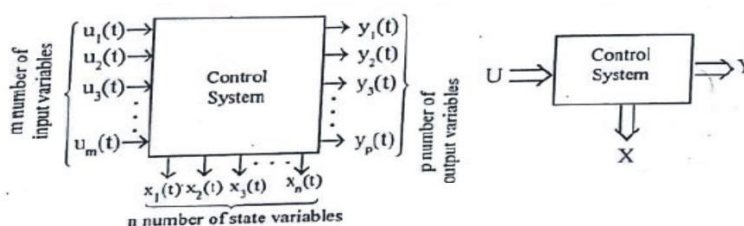


Figure 1.1 State space representation of system

The different variables may be represented by the vectors (column matrix) as shown below.

$$\begin{array}{l} \text{Input vector } U(t) = \begin{bmatrix} u_1(t) \\ u_2(t) \\ \vdots \\ u_m(t) \end{bmatrix} ; \quad \text{Output vector } Y(t) = \begin{bmatrix} y_1(t) \\ y_2(t) \\ \vdots \\ y_p(t) \end{bmatrix} ; \quad \text{State variable vector } X(t) = \begin{bmatrix} x_1(t) \\ x_2(t) \\ \vdots \\ x_n(t) \end{bmatrix} \end{array}$$

STATE EQUATIONS

The state variable representation can be arranged in the form of n number of first order differential equation as shown below.

$$\begin{aligned} \frac{dx_1}{dt} &= \dot{x}_1 = f_1(x_1, x_2, \dots, x_n; u_1, u_2, \dots, u_m) \\ \frac{dx_2}{dt} &= \dot{x}_2 = f_2(x_1, x_2, \dots, x_n; u_1, u_2, \dots, u_m) \\ &\vdots \\ \frac{dx_n}{dt} &= \dot{x}_n = f_n(x_1, x_2, \dots, x_n; u_1, u_2, \dots, u_m) \end{aligned} \quad \dots 1.1$$

The n number of differential equations may be written in vector notation as

$$\dot{X}(t) = f(X(t), U(t)) \quad \dots 1.2$$

The set of all possible values which the input vector $U(t)$ can have (assume) at time t forms the input space of the system. Similarly, the set of all possible values which the output vector $Y(t)$ can assume at time t forms the output space of the system and the set of all possible values which the state vector $X(t)$ can assume at time t forms the state space of the system.

1.3 STATE MODEL OF LINEAR SYSTEM

The state model of a system consist of the state equation and output equation. The state equation of a system is a function of state variables and inputs as defined by equation (1.2). For linear time invariant systems the first derivations of state variable can be expressed as a linear combination of state variables and inputs.

$$\begin{aligned} \dot{x}_1 &= a_{11}x_1 + a_{12}x_2 + \dots + a_{1n}x_n + b_{11}u_1 + b_{12}u_2 + \dots + b_{1m}u_m \\ \dot{x}_2 &= a_{21}x_1 + a_{22}x_2 + \dots + a_{2n}x_n + b_{21}u_1 + b_{22}u_2 + \dots + b_{2m}u_m \\ &\vdots \\ \dot{x}_n &= a_{n1}x_1 + a_{n2}x_2 + \dots + a_{nn}x_n + b_{n1}u_1 + b_{n2}u_2 + \dots + b_{nm}u_m \end{aligned} \quad \dots 1.3$$

where the coefficients a_{ij} and b_{ij} are constants.

In the matrix form the above equations can be expressed as,

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \\ \vdots \\ \dot{x}_n \end{bmatrix} = \begin{bmatrix} a_{11} & a_{12} & \dots & a_{1n} \\ a_{21} & a_{22} & \dots & a_{2n} \\ a_{31} & a_{32} & \dots & a_{3n} \\ \vdots & \vdots & & \vdots \\ a_{n1} & a_{n2} & \dots & a_{nn} \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ \vdots \\ x_n \end{bmatrix} + \begin{bmatrix} b_{11} & b_{12} & \dots & b_{1m} \\ b_{21} & b_{22} & \dots & b_{2m} \\ b_{31} & b_{32} & \dots & b_{3m} \\ \vdots & \vdots & & \vdots \\ b_{n1} & b_{n2} & \dots & b_{nm} \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \\ u_3 \\ \vdots \\ u_m \end{bmatrix} \quad \dots 1.4$$

The matrix equation (1.4) can also be written as, $\dot{X}(t) = A X(t) + B U(t)$...1.5

where, $X(t)$ = State vector of order $(n \times 1)$

$U(t)$ = Input vector of order $(m \times 1)$

A = System matrix of order $(n \times n)$

B = Input matrix of order $(n \times m)$

Note: For convenience the input, output and state variables are denoted as $u_1, u_2, \dots, y_1, y_2, \dots$ and x_1, x_2, \dots ; but actual they are functions of time, t .

The equation, $\dot{X}(t) = A X(t) + B U(t)$ is called the state equation of Linear Time Invariant (LTI) system.

The output at any time are functions of state variables and inputs.

\therefore Output vector, $Y(t) = f(X(t), U(t))$...1.6

Hence the output variables can be expressed as a linear combination of state variables and inputs.

$$\begin{aligned} y_1 &= c_{11}x_1 + c_{12}x_2 + \dots + c_{1n}x_n + d_{11}u_1 + d_{12}u_2 + \dots + d_{1m}u_m \\ y_2 &= c_{21}x_1 + c_{22}x_2 + \dots + c_{2n}x_n + d_{21}u_1 + d_{22}u_2 + \dots + d_{2m}u_m \\ &\vdots \\ y_p &= c_{p1}x_1 + c_{p2}x_2 + \dots + c_{pn}x_n + d_{p1}u_1 + d_{p2}u_2 + \dots + d_{pm}u_m \end{aligned} \quad \dots 1.7$$

where the coefficients c_{ij} and d_{ij} are constants.

In the matrix form the above equations can be expressed as,

$$\begin{bmatrix} y_1 \\ y_2 \\ y_3 \\ \vdots \\ y_p \end{bmatrix} = \begin{bmatrix} c_{11} & c_{12} & \dots & c_{1n} \\ c_{21} & c_{22} & \dots & c_{2n} \\ c_{31} & c_{32} & \dots & c_{3n} \\ \vdots & \vdots & & \vdots \\ c_{p1} & c_{p2} & \dots & c_{pn} \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ \vdots \\ x_n \end{bmatrix} + \begin{bmatrix} d_{11} & d_{12} & \dots & d_{1m} \\ d_{21} & d_{22} & \dots & d_{2m} \\ d_{31} & d_{32} & \dots & d_{3m} \\ \vdots & \vdots & & \vdots \\ d_{p1} & d_{p2} & \dots & d_{pm} \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \\ u_3 \\ \vdots \\ u_m \end{bmatrix} \quad \dots 1.8$$

The matrix equation (1.8) can also be written as, $Y(t) = C X(t) + D U(t)$...1.9

where, $X(t)$ = State vector of order $(n \times 1)$
 $U(t)$ = Input vector of order $(m \times 1)$
 $Y(t)$ = Output vector of order $(p \times 1)$
 C = Output matrix of order $(p \times n)$
 D = Transmission matrix of order $(p \times m)$

The equation $Y(t) = C X(t) + D U(t)$ is called the output equation of Linear Time Invariant (LTI) system.

The state model of a system consists of state equation and output equation. (or) The state equation and output equation together called as state model of the system. Hence the state model of a linear time invariant system (LTI) system is given by the following equations.

$$\begin{aligned}\dot{X}(t) &= A X(t) + B U(t) && \text{..... State equation.} \\ Y(t) &= C X(t) + D U(t) && \text{..... Output equation.}\end{aligned}$$

1.4 STATE DIAGRAM

The pictorial representation of the state model of the system is called state diagram. The state diagram of the system can be either in Block Diagram form or in signal flow graph form.

The state diagram describes the relationships among the state variables and provides physical interpretations of the state variables. The time domain state diagram may be obtained directly from the differential equation governing the system and this diagram can be used for simulation of the system in analog computers.

The s-domain state diagram can be obtained from the transfer function of the system. The state diagram provides a direct relation between time domain and s-domain. [i.e., the time domain equations can be directly obtained from the s-domain state diagram].

The state diagram (Block diagram and signal flow graph) of a state model is constructed using three basic elements, Scalar, Adder and Integrator.

Scalar: The scalar is used to multiply a signal by a constant. The input signal $x(t)$ is multiplied by the scalar a to give the output, $a x(t)$.

Adder: The adder is used to add two or more signals. The output of the adder is the sum of incoming signals.

Integrator: The integrator is used to integrate the signals. They are used to integrate the derivatives of state variables to get the state variables. The initial conditions of the state variable can be added by using an adder after integrator.

The time domain and s-domain elements of block diagram are shown in Table 1.1. The time domain and s-domain elements of signal flow graph are shown in Table 1.2.

Table 1.1 Elements of Block Diagram

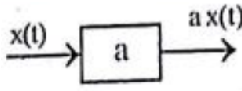
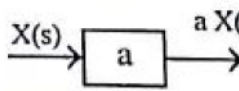
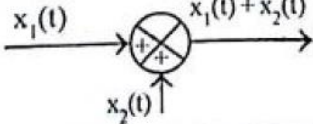
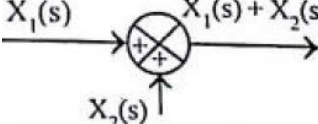
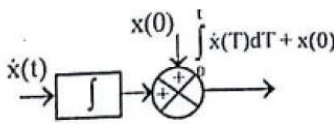
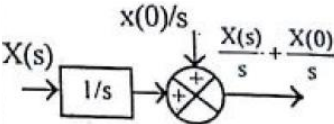
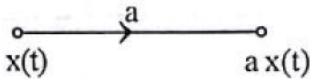
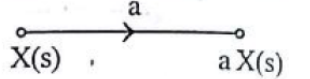
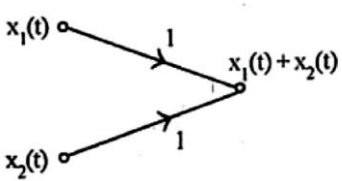
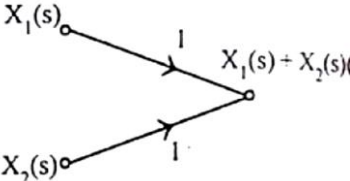
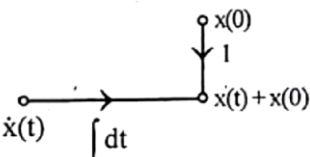
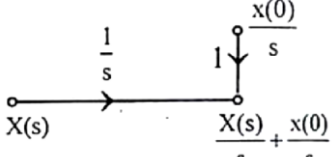
Element	Time domain	s-domain
Scalar		
Adder		
Integrator		

Table 1.2 Elements of Signal Flow Diagram

Element	Time domain	s-domain
Scalar		
Adder		
Integrator		

The state model of linear time invariant system is given by the equations.

$$\begin{aligned}\dot{X}(t) &= A X(t) + B U(t) && \text{..... State equation.} \\ Y(t) &= C X(t) + D U(t) && \text{..... Output equation.}\end{aligned}$$

The block time domain diagram representation of the state model is shown in Figure 1.2 and the time domain signal flow graph representation of the system is shown in Figure 1.3.

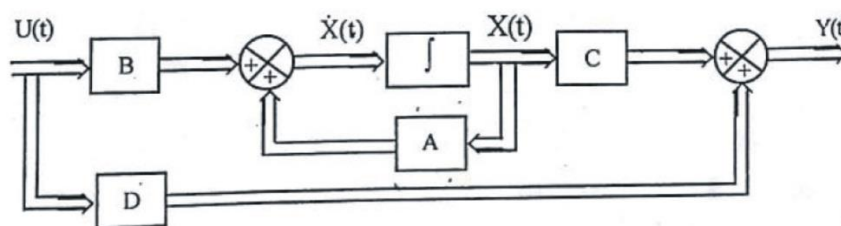
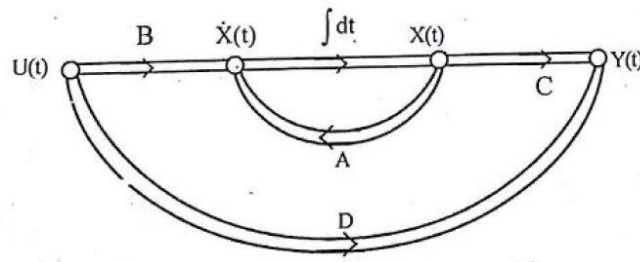


Figure 1.2 Block diagram of state model



CONSTRUCTION OF TIME DOMAIN STATE DIAGRAM

In state space modelling, n -number of first order differential equations are formed for a n^{th} order system. In order to integrate n -numbers of first derivatives, the state diagram requires n -numbers of integrators. Therefore the first step in constructing the state diagram is to draw n -numbers of integrators. Mark the input to the integrators as first derivatives of state variables and so the output of the integrators are state variables. [If initial conditions are given, then they can be added at the output of integrators using adders].

In each state equation, the first derivative of state variable is expressed as a function of state variables and inputs. Therefore from the knowledge of a state equation, the state variables and inputs are multiplied by appropriate scalars and then added to get the first derivative of a state variable. Now, the first derivative of the state variable is given as input to the corresponding integrator. Similarly the input of all other integrators are obtained by considering the state equations one by one.

Each output equation is a function of state variables and inputs. Therefore from the knowledge of an output, equation, the state variables and inputs are multiplied by appropriate scalars and then added to get an output. Similar procedure is followed to generate all other outputs.

1.5 STATE – SPACE REPRESENTATION USING PHYSICAL VARIABLES

In state-space modelling of systems, the choice of state variables is arbitrary. One of the possible choice of state variables is the physical variables. The physical variables of electrical systems are current or voltage in the R , L and C elements. The physical variables of mechanical systems are displacement, velocity and acceleration. The advantages of choosing the physical variables (or quantities) of the system as state variables are the following,

1. The state variables can be utilized for the purpose of feedback.
2. The implementation of design with state variable feedback becomes straight forward.
3. The solution of state equation gives time variation of variables which have direct relevance to the physical system.


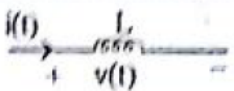
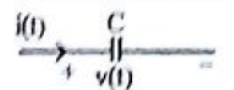
The drawback in choosing the physical quantities as state variables is that the solution of state equation may become a difficult task.

In state space modelling using physical variables, the state equations are obtained from the differential equations governing the system. The differential equations governing a system are obtained from a basic model of the system which is developed using the fundamental elements of the system.

ELECTRICAL SYSTEM

The basic model of a electrical system can be obtained by using the fundamental elements Resistor, Capacitor and Inductor. Using these elements the electrical network or equivalent circuit of the system is drawn. Then the differential equations governing the electrical systems can be formed by writing Kirchoff's current law equations by choosing various nodes in the network or Kirchoff's voltage law by choosing various closed path in the network. The current-voltage relation of the basic elements R, L and C are given in Table 1.3.

Table 1.3

Element	Voltage across the element	Current through the element
	$v(t) = Ri(t)$	$i(t) = \frac{v(t)}{R}$
	$v(t) = L \frac{d}{dt} i(t)$	$i(t) = \frac{1}{L} \int v(t) dt$
	$v(t) = \frac{1}{C} \int i(t) dt$	$i(t) = C \frac{dv(t)}{dt}$

A minimal number of state variables are chosen for obtaining the state model of the system. The best choice of state variables in electrical system are currents and voltages in energy storage elements. The energy storage elements are inductance and capacitance. The physical variables in the differential equations are replaced by state variables and the equations are rearranged as first order differential equations. These set of first order equations constitutes the state equation of the system.

The inputs to the system are exciting voltage sources or current sources. The outputs in electrical system are usually voltages or currents in energy dissipating element. The resistance is energy dissipating element in electrical network. In general the output variables can be any voltage or current in the network.

MECHANICAL TRANSLATIONAL SYSTEM

The basic model of mechanical translational system can be obtained by using three basic elements mass, spring and dash-pot. When a force applied to a mechanical translational system, it is opposed by opposing forces due to mass, friction and elasticity of the system. The forces acting on a body are governed by Newton's second law of motion.

The differential equations governing the system are obtained by writing force balance equations at various nodes in the system. A node is a meeting point of elements. The Table 1.4 shows the force balance equations of idealized elements.

List of symbol used in mechanical translational system are

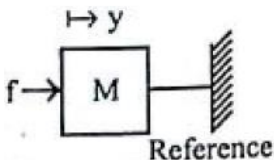
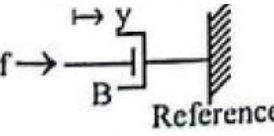
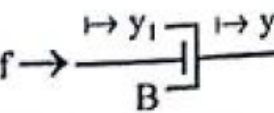
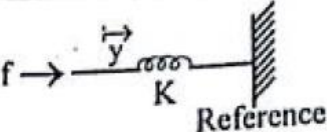
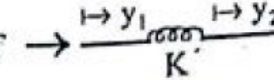
y	=	Displacement, m
v	=	dy/dt = Velocity, m/sec

a	=	$dv/dt = d^2y/dt^2 =$ Acceleration, m/sec^2
f	=	Applied force, N (Newton)
f_m	=	Opposing force offered by mass of the body, N
f_k	=	Opposing force offered by the elasticity of the body (spring), N
f_b	=	Opposing force offered by the friction of the body (dash-pot), N
M	=	Mass, Kg
K	=	Stiffness of spring, N/m
B	=	Viscous friction coefficient, N/(m/sec).

Guidelines to form the state model of mechanical translational systems

1. For each node in the system one differential equation can be framed by equating the sum of applied forces to the sum of opposing forces. Generally, the nodes are mass element.

Table 1.4 Force balance equations of idealized elements

Element	Force balance equations
	$f = f_m = M \frac{d^2y}{dt^2}$
	$f = f_b = B \frac{dy}{dt}$
	$f = f_b = B \frac{d}{dt}(y_1 - y_2)$
	$f = f_k = Ky$
	$f = f_k = K (y_1 - y_2)$

2. Assign a displacement to each node and draw a free body diagram for each node. The free body diagram is obtained by drawing each mass of node separately and then marking all the forces acting on it.
3. In the free body diagram, the opposing forces due to mass, spring and dash-pot are always act in a direction opposite to applied force. The displacement, velocity and acceleration will be in the direction of applied force or in the direction opposite to that of opposing force.
4. For each free body diagram write one differential equation by equating the sum of applied forces to the sum of opposing forces.

5. Choose a minimum number of state variables. The choice of state variables are displacement, velocity or acceleration.
6. The physical variables in differential equations are replaced by state variables and the equations are rearranged as first order differential equations. These set of first order equations constitute the state equation of the system
7. The inputs are the applied forces and the outputs are the displacement, velocity or acceleration of the desired nodes.

MECHANICAL ROTATIONAL SYSTEM

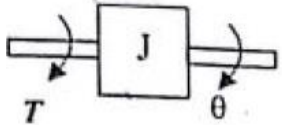
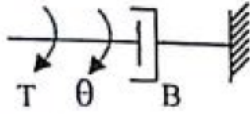
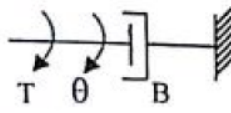
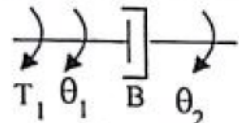
The basic model of mechanical rotational system can be obtained by using three basic elements moment of inertia of mass, rotational dash-pot and rotational spring. When a torque is applied to a mechanical rotational system, it is opposed by opposing torques due to moment of inertia, friction and elasticity of the system. The torque acting on a body are governed by Newton's second law of motion.

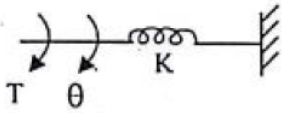
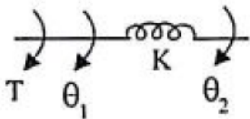
The differential equations governing the system are obtained by writing torque balance equations at various nodes in the system. A node is a meeting point of elements. The Table 1.5 shows the torque balance equations of the idealized elements.

List of symbols used in mechanical rotational system

θ	=	Angular displacement and
$d\theta/dt$	=	Angular velocity, rad/sec
$d^2\theta/dt^2$	=	Angular acceleration, rad/sec
T	=	Applied torque, N-m
J	=	Moment of inertia, Kg-m ² /rad
B	=	Rotational frictional coefficient, N-m/rad/sec)
K	=	Stiffness of the spring, N-m/rad.

Table 1.5 Torque balance equations of idealized elements

Element	Torque balance equations
	$T = T_j = J \frac{d^2\theta}{dt^2}$
	$T = T_b = B \frac{d\theta}{dt}$
	$T = T_b = B \frac{d\theta}{dt}$
	$T = T_b = B \frac{d}{dt}(\theta_1 - \theta_2)$

	$T_k = K\theta$
	$T = T_k = K(\theta_1 - \theta_2)$

Guidelines to form the state model of mechanical rotational systems

1. For each node in the system one differential equation can be framed by equating the sum of applied torques to the sum of opposing torques. Generally the nodes are mass elements but in some cases the nodes may be without mass element.
2. Assign an angular displacement to each node and draw a free body diagram for each node. The free body diagram is obtained by drawing each node separately and then drawing all the torques acting on it.
3. In the free body diagram, the opposing torques due to moment of inertia, spring and dash-pot are always act in a direction opposite to applied force. The angular displacement, velocity and acceleration will be in the direction of applied torque or in the direction opposite to that of opposing torque.
4. For each free body diagram write one differential equation by equating the sum of applied torque to the sum of opposing torques.
5. Choose a minimum number of state variables. The choice of state variables are angular displacement, velocity or acceleration.
6. The physical variables in differential equations are replaced by state variables and the equations are rearranged as first order differential equations. These set of first order equations constitute the state equation of the system.
7. The inputs are the applied torques and the outputs are the angular displacement, velocity or acceleration of the desired nodes.

EXAMPLE 1.1

Obtain the state model of the electrical network shown in Fig 1.1.1 by choosing number of state variables.

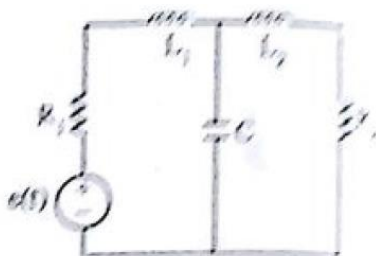


Figure 1.1.1

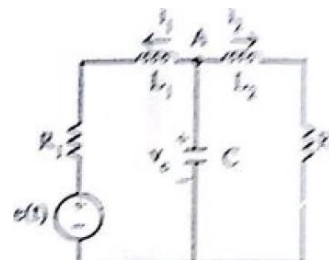


Figure 1.1.2

SOLUTION

Let us choose the current through the inductances i_1 , i_2 and voltage across the capacitor v_s as state variables. The assumed directions of currents and polarity of the voltage are shown in Fig 1.1.2.

[**Note:** The best choice of state variables in electrical network are currents and voltages in energy storage elements].

Let the three state variables x_1 , x_2 and x_3 be related to physical quantities as show below.

$x_1 = i_1 =$ Current through L_1

$x_2 = i_2 =$ Current through L_2

$x_3 = v_o =$ Voltage across capacitor

At node A, by Kirchoff's current law (refer Figure 1.1.3),

$$i_1 + i_2 + C \frac{dv_c}{dt} = 0 \quad \dots 1.1.1$$

On substituting the state variables for physical variables in Eqn. (1.1.1) we get,

$$(i.e., i_1 = x_1, i_2 = x_2 \text{ and } \frac{dv_c}{dt} = \dot{x}_3)$$

$$x_1 + x_2 + C\dot{x}_3 = 0$$

$$C\dot{x}_3 = -x_1 - x_2$$

$$\dot{x}_3 = -\frac{1}{C}x_1 - \frac{1}{C}x_2 \quad \dots 1.1.2$$

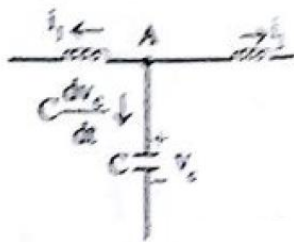


Figure 1.1.3

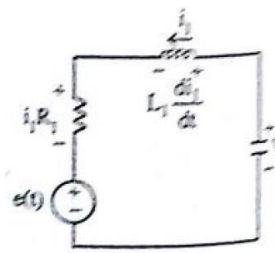


Figure 1.1.4

By Kirchoff's voltage law in the closed path shown in Figure 1.1.4 we get,

$$e(t) + i_1 R_1 + L_1 \frac{di_1}{dt} = v_c \quad \dots 1.1.3$$

On substituting the state variables for physical variables in Eqn (1.1.3) we get,

$$(i.e., i_1 = x_1, \frac{di_1}{dt} = \dot{x}_1 \text{ and } v_c = x_3)$$

$$e(t) + x_1 R_1 + L_1 \dot{x}_1 = x_3$$

Also, let $u(t) = e(t)$ = input to the system

$$\therefore u + x_1 R_1 + L_1 \dot{x}_1 = x_3$$

$$L_1 \dot{x}_1 = x_3 - x_1 R_1 - u$$

$$\dot{x}_1 = -\frac{R_1}{L_1} x_1 + \frac{1}{L_1} x_3 - \frac{1}{L_1} u \quad \dots 1.1.4$$

By Kirchoff's voltage law in the closed path shown in Figure 1.1.5 we get,

$$v_c = L_2 \frac{di_2}{dt} + i_2 R_2 \quad \dots 1.1.5$$

On substituting the state variables for physical variables in Eqn. (1.1.5) we get,

$$(i.e., i_2 = x_2, \frac{di_2}{dt} = \dot{x}_2 \text{ and } v_c = x_3)$$

$$x_3 = L_2 \dot{x}_2 + x_2 R_2$$

$$\therefore L_2 \dot{x}_2 = -x_2 R_2 + x_3$$

$$\dot{x}_2 = -\frac{R_2}{L_2} x_2 + \frac{1}{L_2} x_3$$

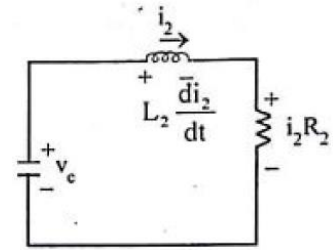


Figure 1.15

...1.1.6

The equations (1.1.2), (1.1.4) and (1.1.6) are the state equations of the system. Hence the state equations of the system are,

$$\dot{x}_1 = -\frac{R_1}{L_1} x_1 + \frac{1}{L_1} x_3 - \frac{1}{L_1} u$$

$$\dot{x}_2 = -\frac{R_2}{L_2} x_2 + \frac{1}{L_2} x_3$$

$$\dot{x}_3 = -\frac{1}{C} x_1 - \frac{1}{C} x_2$$

On arranging the state equations in the matrix form we get,

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} -\frac{R_1}{L_1} & 0 & \frac{1}{L_1} \\ 0 & -\frac{R_2}{L_2} & \frac{1}{L_2} \\ -\frac{1}{C} & -\frac{1}{C} & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} -\frac{1}{L_1} \\ 0 \\ 0 \end{bmatrix} [u]$$

State equation1.1.7

Let us choose the voltage across the resistances as output variables and the output variables are denoted by y_1 and y_2 .

$$\therefore y_1 = i_1 R_1 \quad \dots 1.1.8$$

$$\text{and } y_2 = i_2 R_2 \quad \dots 1.1.9$$

On substituting the state variables in equations (1.1.8) and (1.1.9) we get,

(i.e. $i_1 = x_1$ and $i_2 = x_2$)

$$y_1 = x_1 R_1 \quad ; \quad y_2 = x_2 R_2$$

On arranging the above equations in the matrix form we get

$$\begin{bmatrix} y_1 \\ y_2 \end{bmatrix} = \begin{bmatrix} R_1 & 0 \\ 0 & R_2 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} \quad \text{Output equation} \quad \dots 1.1.10$$

The state equation (Eqn (1.1.7)) and output equation (Eqn (1.1.10)) together constitute the state model of the system.

EXAMPLE 1.2

Obtain the state model of the electrical network shown in Figure 1.2.1 by choosing $v_1(t)$ and $v_2(t)$ as state variables.

SOLUTION

Connect a voltage source at the inputs as shown in Figure 1.2.2. Convert the Voltage source to current source as shown in Figure 1.2.3. At node 1, by Kirchoff's current law we can write (Refer Figure 1.2.4).

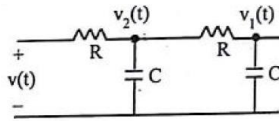


Figure 1.2.1

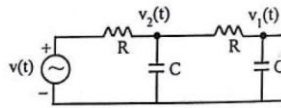


Figure 1.2.2

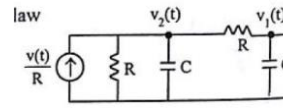


Figure 1.2.3

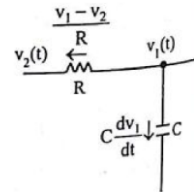


Figure 1.2.4

$$\frac{v_1 - v_2}{R} + C \frac{dv_1}{dt} = 0 \quad 1.2.1$$

At node 2, by Kirchoff's current law, we can write (Refer Figure 1.2.5)

$$\frac{v_2 - v_1}{R} + \frac{v_2}{R} + C \frac{dv_2}{dt} = \frac{v(t)}{R} \quad \dots 1.2.2$$

Let the state variables be x_1 and x_2 and they are related to physical variable as shown below.

$$v_1 = x_1 \text{ and } v_2 = x_2$$

Also, Let $v(t) = u = \text{input}$.

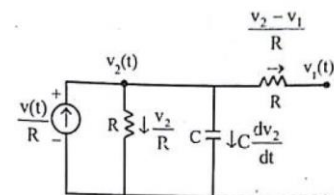


Figure 1.2.5

On substituting the state variables in equation (1.2.1) and (1.2.2) we get,

$$\frac{x_1 - x_2}{R} + C \frac{dx_1}{dt} = 0 \quad \dots 1.2.3$$

$$\frac{x_2 - x_1}{R} + \frac{x_2}{R} + C \frac{dx_2}{dt} = \frac{u}{R} \quad \dots 1.2.4$$

From equation (1.2.3) we get, $\frac{x_1 - x_2}{R} + C\dot{x}_1 = 0$

$$\begin{aligned} \therefore C\dot{x}_1 &= -\frac{x_1}{R} + \frac{x_2}{R} \\ \dot{x}_1 &= -\frac{1}{RC}x_1 + \frac{1}{RC}x_2 \end{aligned} \quad \dots 1.2.5$$

From equation (1.2.4) we get, $\frac{x_2 - x_1}{R} + \frac{x_2}{R} + C\dot{x}_2 = \frac{u}{R}$

$$\begin{aligned} \therefore C\dot{x}_2 &= \frac{x_1}{R} - \frac{x_2}{R} - \frac{x_2}{R} + \frac{u}{R} \\ \dot{x}_2 &= \frac{1}{RC}x_1 - \frac{2}{RC}x_2 + \frac{1}{RC}u \end{aligned} \quad \dots 1.2.6$$

The equation (1.2.5) and (1.2.6) are state equations of the system. Hence the state equations of the system are

$$\begin{aligned} \dot{x}_1 &= -\frac{1}{RC}x_1 + \frac{1}{RC}x_2 \\ \dot{x}_2 &= \frac{1}{RC}x_1 - \frac{2}{RC}x_2 + \frac{1}{RC}u \end{aligned}$$

On arranging the state equations in the matrix form,

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} -\frac{1}{RC} & \frac{1}{RC} \\ \frac{1}{RC} & -\frac{2}{RC} \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{RC} \end{bmatrix} u \quad \dots 1.2.7$$

The output, $y = v_1(t) = x_1$

$$\therefore \text{The output equation is } y = \begin{bmatrix} 1 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} \quad \dots 1.2.8$$

The state equation [Eqn (1.2.7)] and output equation [Eqn (1.2.8)] together constitute the state model of the system.

EXAMPLE 1.3

Construct the state model of mechanical system shown in Figure 1.3.1.

SOLUTION

Free body diagram of M_1 is shown in Figure 1.3.2

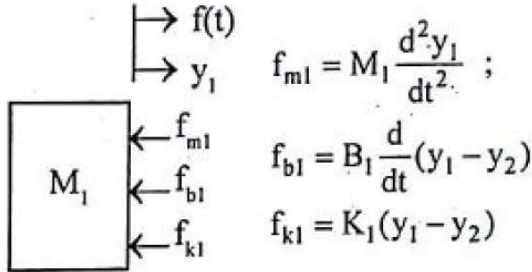


Figure 1.3.2

By Newton's second law, the force balance equation at node M_1 is

$$\begin{aligned}
 f(t) &= f_{m1} + f_{b1} + f_{k1} \\
 f(t) &= M_1 \frac{d^2 y_1}{dt^2} + B_1 \frac{d}{dt} (y_1 - y_2) + K_1 (y_1 - y_2) \\
 f(t) &= M_1 \frac{d^2 y_1}{dt^2} + B_1 \frac{dy_1}{dt} - B_1 \frac{dy_2}{dt} + K_1 y_1 - K_1 y_2
 \end{aligned}
 \quad \dots 1.3.1$$

Free body diagram of M_2 is shown in Figure 1.3.3.

$$\begin{aligned}
 f_{m2} &= M_2 \frac{d^2 y_2}{dt^2} \quad ; \quad f_{b2} = B_2 \frac{dy_2}{dt} \\
 f_{b1} &= B_1 \frac{d}{dt} (y_2 - y_1) \quad ; \quad f_{k2} = K_2 y_2 \\
 f_{k1} &= K_1 (y_2 - y_1)
 \end{aligned}$$

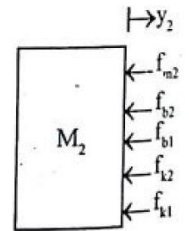


Figure 1.3.3

By Newton's second law, the force balance equation at node M_2 is

$$\begin{aligned}
 f_{m2} + f_{b2} + f_{b1} + f_{k2} + f_{k1} &= 0 \\
 \therefore M_2 \frac{d^2 y_2}{dt^2} + B_2 \frac{dy_2}{dt} + B_1 \frac{d}{dt} (y_2 - y_1) + K_2 y_2 + K_1 (y_2 - y_1) &= 0 \\
 M_2 \frac{d^2 y_2}{dt^2} + B_2 \frac{dy_2}{dt} + B_1 \frac{dy_2}{dt} - B_1 \frac{dy_1}{dt} + K_2 y_2 + K_1 y_2 - K_1 y_1 &= 0
 \end{aligned}
 \quad \dots 1.3.2$$

Let us choose four state variable x_1, x_2, x_3 and x_4 . Also, let the input $f(t) = u$. The state variables are related to physical variables as follows.

$$x_1 = y_1 \quad ; \quad x_2 = y_2 \quad ; \quad x_3 = \frac{dy_1}{dt} \quad ; \quad x_4 = \frac{dy_2}{dt} \quad ; \quad \dot{x}_3 = \frac{d^2 y_1}{dt^2} \quad ; \quad \dot{x}_4 = \frac{d^2 y_2}{dt^2}$$

On substituting, $y_1 = x_1$; $y_2 = x_2$; $\frac{dy_1}{dt} = x_3$; $\frac{dy_2}{dt} = x_4$; $\frac{d^2y_1}{dt^2} = \dot{x}_3$ and $f(t) = u$ in equation (1.3.1) we get,

$$\begin{aligned} u &= M_1 \dot{x}_3 + B_1 x_3 - B_1 x_4 + K_1 x_1 - K_1 x_2 \\ M_1 \dot{x}_3 &= -B_1 x_3 + B_1 x_4 - K_1 x_1 + K_1 x_2 + u \\ \therefore \dot{x}_3 &= -\frac{K_1}{M_1} x_1 + \frac{K_1}{M_1} x_2 - \frac{B_1}{M_1} x_3 + \frac{B_1}{M_1} x_4 + \frac{1}{M_1} u \end{aligned} \quad \dots 1.3.3$$

On substituting $y_1 = x_1$; $y_2 = x_2$; $\frac{dy_1}{dt} = x_3$; $\frac{dy_2}{dt} = x_4$ and $\frac{d^2y_2}{dt^2} = \dot{x}_4$ in equation (1.3.2) we get,

$$\begin{aligned} M_2 \dot{x}_4 + B_2 x_4 + B_1 x_4 - B_1 x_3 + K_1 x_2 + K_2 x_2 - K_1 x_1 &= 0 \\ \therefore M_2 \dot{x}_4 &= -B_2 x_4 - B_1 x_4 + B_1 x_3 - K_2 x_2 - K_1 x_2 + K_1 x_1 \\ &= -(B_2 + B_1) x_4 + B_1 x_3 - (K_2 + K_1) x_2 + K_1 x_1 \\ \therefore \dot{x}_4 &= \frac{K_1}{M_2} x_1 - \frac{(K_1 + K_2)}{M_2} x_2 + \frac{B_1}{M_2} x_3 - \frac{(B_1 + B_2)}{M_2} x_4 \end{aligned} \quad \dots 1.3.4$$

The state variables $x_1 = y_1$.

On differentiating $x_1 = y_1$ with respect to t we get, $\frac{dx_1}{dt} = \frac{dy_1}{dt}$

$$\text{Let } \frac{dx_1}{dt} = \dot{x}_1 \text{ and } \frac{dy_1}{dt} = x_3 ; \therefore \dot{x}_1 = x_3 \quad \dots 1.3.5$$

The state variable, $x_2 = y_2$.

On differentiating $x_2 = y_2$ with respect to t we get, $\frac{dx_2}{dt} = \frac{dy_2}{dt}$

$$\text{Let } \frac{dx_2}{dt} = \dot{x}_2 \text{ and } \frac{dy_2}{dt} = x_4 ; \therefore \dot{x}_2 = x_4 \quad \dots 1.3.6$$

The equations (1.3.3) to (1.3.6) are state equations of the mechanical system. Hence the state equations of the mechanical system are,

$$\begin{aligned} \dot{x}_1 &= x_3 \\ \dot{x}_2 &= x_4 \\ \dot{x}_3 &= -\frac{K_1}{M_1} x_1 + \frac{K_1}{M_1} x_2 - \frac{B_1}{M_1} x_3 + \frac{B_1}{M_1} x_4 + \frac{1}{M_1} u \\ \dot{x}_4 &= \frac{K_1}{M_2} x_1 - \frac{(K_1 + K_2)}{M_2} x_2 + \frac{B_1}{M_2} x_3 - \frac{(B_1 + B_2)}{M_2} x_4 \end{aligned}$$

On arranging the state equations in the matrix form, we get,

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \\ \dot{x}_4 \end{bmatrix} = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ -\frac{K_1}{M_1} & \frac{K_1}{M_1} & -\frac{B_1}{M_1} & \frac{B_1}{M_1} \\ \frac{K_1}{M_2} & -\frac{(K_1 + K_2)}{M_2} & \frac{B_1}{M_2} & -\frac{(B_1 + B_2)}{M_2} \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ \frac{1}{M_1} \\ 0 \end{bmatrix} [u]$$

Let the displacements y_1 and y_2 be the outputs of the system.

$$\therefore y_1 = x_1 \quad \text{and} \quad y_2 = x_2.$$

The output equation in matrix form is given by,

$$\begin{bmatrix} y_1 \\ y_2 \end{bmatrix} = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix}$$

...1.3.8

The state equation [Eqn (1.3.7)] and the output equation [Eqn (1.3.8)] together called state model of the system.

EXAMPLE 1.4

Obtain the state model of the mechanical system shown in Figure 1.4.1 by choosing a minimum of three state variables.

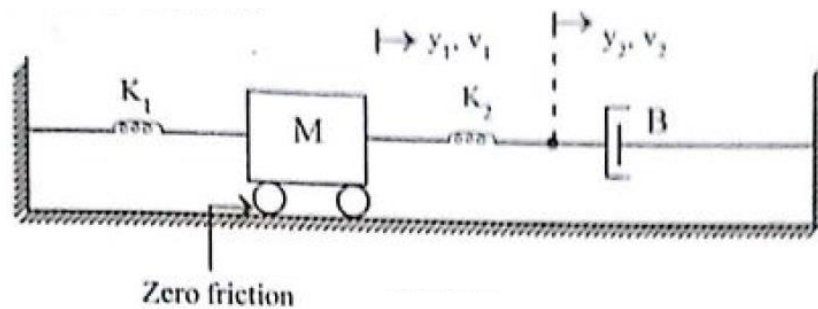


Figure 1.4.1

SOLUTION

Let the three state variables be x_1 , x_2 and x_3 and they are related to physical variables as shown below.

$$x_1 = y_1 ; \quad x_2 = y_2 ; \quad x_3 = \frac{dy_1}{dt} = v_1.$$

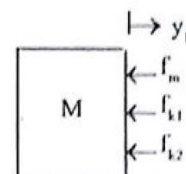


Figure 1.4.2

Free body diagram of mass M is shown in Figure 1.4.2

$$f_m = M \frac{d^2 y_1}{dt^2} ; \quad f_{k1} = K_1 y_1 ; \quad f_{k2} = K_2 (y_1 - y_2)$$

By Newton's second law, the force balance equation at node M is,

$$f_m + f_{k1} + f_{k2} = 0$$

$$M \frac{d^2 y_1}{dt^2} + K_1 y_1 + K_2 (y_1 - y_2) = 0$$

$$M \frac{d^2 y_1}{dt^2} + K_1 y_1 + K_2 y_1 - K_2 y_2 = 0$$

...1.4.1

$$\text{Put } \frac{d^2 y_1}{dt^2} = \dot{x}_3 ; \quad y_1 = x_1, \quad y_2 = x_2 \text{ in equ(1.4.1)}$$

$$M \dot{x}_3 + K_1 x_1 + K_2 x_1 - K_2 x_2 = 0$$

$$M \dot{x}_3 + (K_1 + K_2) x_1 - K_2 x_2 = 0$$

$$\dot{x}_3 = -\frac{K_1 + K_2}{M} x_1 + \frac{K_2}{M} x_2$$

...1.4.2

The free body diagram of node 2 (meeting point of K_2 and B).

$$f_b = B \frac{dy_2}{dt} ; \quad f_{k2} = K_2 (y_2 - y_1)$$

Writing force balance equation at the meeting point of K_2 and B we get,

$$f_b + f_{k2} = 0$$

$$B \frac{dy_2}{dt} + K_2 (y_2 - y_1) = 0$$

$$\therefore \frac{dy_2}{dt} = \frac{K_2}{B} y_1 - \frac{K_2}{B} y_2$$

$$\text{Put } \frac{dy_2}{dt} = \dot{x}_2, \quad y_1 = x_1 \text{ and } y_2 = x_2,$$

$$\therefore \dot{x}_2 = \frac{K_2}{B} x_1 - \frac{K_2}{B} x_2$$

...1.4.3

The state variable, $x_1 = y_1$. On differentiating this expression with respect of t we get

$$\frac{dx_1}{dt} = \frac{dy_1}{dt}$$

$$\text{Let } \frac{dx_1}{dt} = \dot{x}_1 \text{ and } \frac{dy_1}{dt} = x_3 ; \quad \therefore \dot{x}_1 = x_3$$

...1.4.4

The state equations are given by equations (1.4.4), (1.4.3) and (1.4.2).

$$\begin{aligned}\dot{x}_1 &= x_3 \\ \dot{x}_2 &= \frac{K_2}{B}x_1 - \frac{K_2}{B}x_2 \\ \dot{x}_3 &= -\frac{K_1 + K_2}{M}x_1 + \frac{K_2}{M}x_3\end{aligned}$$

On arranging the state equations in the matrix form,

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} 0 & 0 & 1 \\ \frac{K_2}{B} & -\frac{K_2}{B} & 0 \\ -\frac{K_1 + K_2}{M} & 0 & \frac{K_2}{M} \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} \quad \dots 1.4.5$$

If the desired outputs are y_1 and y_2 , then $y_1 = x_1$ and $y_2 = x_2$

The output equation to the matrix form is given by

$$\begin{bmatrix} y_1 \\ y_2 \end{bmatrix} = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} \quad \dots 1.4.6$$

The state equation [Eqn (1.4.5)] and the output equation [Eqn (1.4.6)] together constitute the state model of the system.

EXAMPLE 1.5

Determine the state model of armature controlled dc motor.

SOLUTION

The speed of DC motor is directly proportional to armature voltage and inversely proportional to flux. In armature controlled DC motor the desired speed is obtained by varying the armature voltage. This speed control system is an electro-mechanical control system. The electrical system consists of the armature and the field circuit for analysis purpose. Only the armature circuit is considered because the field circuit but for analysis purpose, only the armature circuit is considered because the field is excited by a constant voltage. The mechanical system consist of the rotating part of the motor and load connected to the shaft of the motor. The armature controlled DC motor speed control system is shown in Figure 1.5.1.

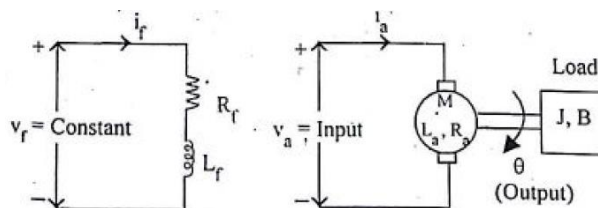


Figure 1.5.1 Armature controlled DC motor

Let	R_a	=	Armature resistance Ω
	L_a	=	Armature inductance, H
	i_a	=	Armature current, A
	v_a	=	Armature voltage, V
	e_b	=	Back emf, V
	K_t	=	Torque constant, N-m/A
	T	=	Torque developed by motor, N-m
	θ	=	Angular displacement of shaft, rad
	ω	=	$d\theta/dt$ = Angular velocity of the shaft, rad/sec
	J	=	Moment of inertia of motor and load, $\text{Kg-m}^2 / \text{rad}$
	B	=	Frictional coefficient of motor and load, N-m/(rad/sec)
	K_b	=	Back emf constant, V/(rad/sec).

The equivalent circuit of armature is shown in Figure 1.5.2.

By Kirchoff's voltage law, we can write

$$i_a R_a + L_a \frac{di_a}{dt} + e_b = v_a \quad \dots 1.5.1$$

Torque of DC motor is proportional to the product of constant in this system, the torque is proportional to i_a alone.

$$T \propto i_a$$

$$\therefore \text{Torque, } T = K_t i_a \quad \dots 1.5.2$$

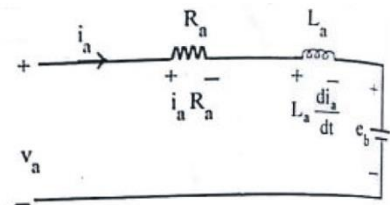


Figure 1.5.2 Equivalent circuit of armature

The mechanical system of the motor is shown in Figure 1.5.3. The differential equation governing the mechanical system of motor is given by

$$J \frac{d^2\theta}{dt^2} + B \frac{d\theta}{dt} = T \quad \dots 1.5.3$$

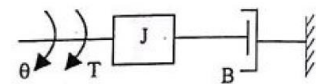


Figure 1.5.3

The back emf of DC machine is proportional to speed (angular velocity) of shaft

$$\therefore e_b \propto \frac{d\theta}{dt} \quad ; \quad \text{Back emf, } e_b = K_b \frac{d\theta}{dt} \quad \dots 1.5.4$$

From Eqn (1.5.1) and (1.5.4) we get,

$$i_a R_a + L_a \frac{di_a}{dt} + K_b \frac{d\theta}{dt} = v_a \quad \dots 1.5.5$$

From Eqn (1.5.2) and (1.5.3) we get,

$$J \frac{d^2\theta}{dt^2} + B \frac{d\theta}{dt} = K_t i_a \quad \dots 1.5.6$$

The equations (1.5.5) and (1.5.6) are the differential equations governing the armature controlled dc motor.

Let us choose i_a , ω and θ as state variables to model the armature controlled dc motor. The physical variables i_a , ω and θ are related to the general notation of state variables x_1 , x_2 and x_3 as shown below.

$$x_1 = i_a ; x_2 = \omega = d\theta/dt \text{ and } x_3 = \theta$$

The input to the motor is the armature voltage, v_a and let $v_a = u$, where u is the general notation for input variable.

On substituting the state variables for the physical variables in equation (1.5.5) we get,

$$\begin{aligned} x_1 R_a + L_a \frac{dx_1}{dt} + K_b x_2 &= u \\ \text{Let } \frac{dx_1}{dt} &= \dot{x}_1, \therefore x_1 R_a + L_a \dot{x}_1 + K_b x_2 = u \\ \dot{x}_1 &= -\frac{R_a}{L_a} x_1 - \frac{K_b}{L_a} x_2 + \frac{1}{L_a} u \end{aligned} \quad \dots 1.5.7$$

On substituting the state variables for physical variables in Eqn (1.5.6) we get,

$$\begin{aligned} J \frac{d^2 x_3}{dt^2} + B \left(\frac{dx_3}{dt} \right) &= K_t x_1 \\ \text{Let } \frac{d^2 x_3}{dt^2} &= \dot{x}_2 \text{ and } \frac{dx_3}{dt} = x_2, \therefore J \dot{x}_2 + B x_2 = K_t x_1 \\ \dot{x}_2 &= \frac{K_t}{J} x_1 - \frac{B}{J} x_2 \end{aligned} \quad \dots 1.5.8$$

The state variable $x_3 = \theta$. On differentiating $x_3 = \theta$ with respect to t we get,

$$\begin{aligned} \frac{dx_3}{dt} &= \frac{d\theta}{dt} \\ \text{Put } \frac{dx_3}{dt} &= \dot{x}_3 \text{ and } \frac{d\theta}{dt} = x_2 \\ \therefore \dot{x}_3 &= x_2 \end{aligned} \quad \dots 1.5.9$$

The equation (1.5.7), (1.5.8) and (1.5.9) are the state equations of the system.

$$\begin{aligned} \dot{x}_1 &= -\frac{R_a}{L_a} x_1 - \frac{K_b}{L_a} x_2 + \frac{1}{L_a} u \\ \dot{x}_2 &= \frac{K_t}{J} x_1 - \frac{B}{J} x_2 \\ \dot{x}_3 &= x_2 \end{aligned}$$

On arranging the state equations in the matrix form,

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} -\frac{R_a}{L_a} & -\frac{K_b}{L_a} & 0 \\ \frac{K_t}{J} & -\frac{B}{J} & 0 \\ 0 & 1 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} \frac{1}{L_a} \\ 0 \\ 0 \end{bmatrix} [u] \quad \dots 1.5.10$$

Let the desired outputs be I_a , ω and θ . Let us equate the desired output quantities to standard notation y_1 , y_2 and y_3 as shown below.

$$y_1 = i_a ; y_2 = \omega = d\theta/dt \text{ and } y_3 = \theta$$

On relating the outputs to state variables we get,

$$y_1 = x_1 ; y_2 = x_2 ; y_3 = x_3$$

\therefore The output equation in the matrix form is

$$\begin{bmatrix} y_1 \\ y_2 \\ y_3 \end{bmatrix} = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} \quad \dots 1.5.11$$

The state equation [Eqn (1.5.10)] and the output equation [Eqn (4.5.11)] together constitute the state model of the armature controlled dc motor.

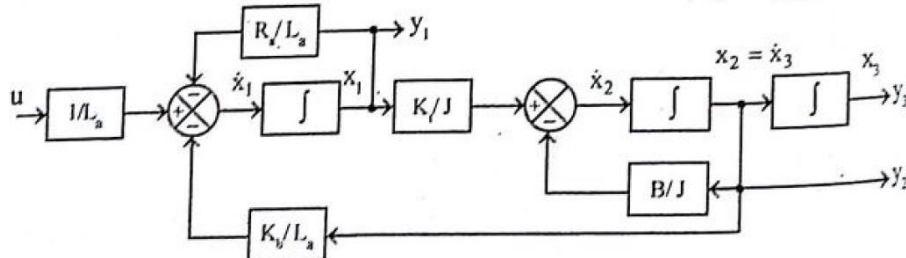


Figure 1.5.4 Block diagram representation of the state model of armature controlled dc motor

EXAMPLE 1.6

Determine the state model of field controlled dc motor.

SOLUTION

The speed of a DC motor is directly proportional to armature voltage and inversely proportional to flux. In field controlled DC motor the armature voltage is kept constant armature the speed is varied by varying the flux of the machine. Since flux is directly proportional to field current, the flux is varied by varying field current. The speed control system is an electromechanical control system. The electrical system consists of armature and field circuit but for analysis purpose, only field circuit is considered because the armature is

excited by a constant voltage. The mechanical system consists of the rotating part of the motor and the load connected to the shaft of the motor. The field controlled DC motor speed control system is shown in Figure 1.6.1.

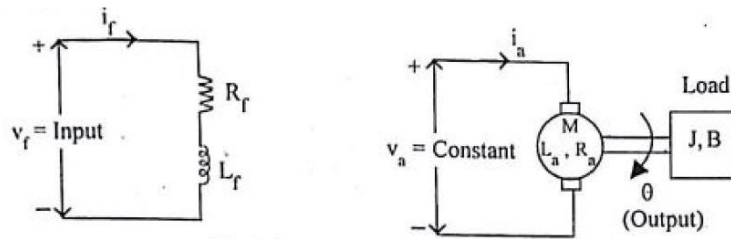


Figure 1.6.1 Field controlled DC motor

Let	R_f	=	Field resistance, Ω
	L_f	=	Field inductance, H
	i_f	=	Field current, A
	v_f	=	Field voltage, V
	θ	=	Angular displacement of the motor shaft, rad
	ω	=	$d\theta/dt$ = Angular velocity of the motor shaft, rad/sec
	T	=	Torque developed by motor, N-m
	K_{tf}	=	Torque constant, N-m/A
	J	=	Moment of inertia of rotor and load, Kg-m ² /rad
	B	=	Frictional coefficient of rotor and load, N-m/(rad/sec).

The equivalent circuit of field is shown in Figure 1.6.2.

By Kirchoff's voltage law, we can write

$$R_f i_f + L_f \frac{di_f}{dt} = v_f$$

...1.6.1

The torque of DC motor is proportional to produce of flux and armature current. Since armature current is constant in this system, the torque is proportional to flux alone, but flux is proportional to field current.

$$\therefore T \propto i_f ; \text{ Torque, } T = K_{tf} i_f$$

...1.6.2

The mechanical system of the motor is shown in Figure 1.6.3 governing the mechanical system of the motor is given by

$$J \frac{d^2\theta}{dt^2} + B \frac{d\theta}{dt} = T$$

...1.6.3

From Eqn (1.6.2) and (1.6.3) we get,

$$J \frac{d^2\theta}{dt^2} + B \frac{d\theta}{dt} = K_{tf} i_f$$

...1.6.4

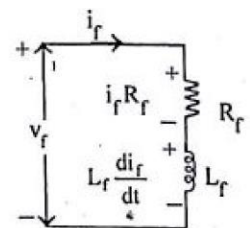


Figure 1.6.2

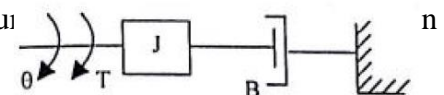


Figure 1.6.3

The equation (1.6.1) and (1.6.4) are the differential equations governing the field controlled dc motor.

Let us choose i_f , ω and θ as state variable to model the field controlled dc motor. The physical variables i_f , ω and θ are related to the general notation of state variables x_1 , x_2 and x_3 as shown in below.

$$x_1 = i_f ; x_2 = \omega = d\theta/dt ; x_3 = \theta$$

The input to the system is the field voltage v_f . Let $v_f = u$, where u is the general notation for input.

On substituting the state variables and input variables for the physical variables in Eqn (1.6.1) we get,

$$\begin{aligned} R_f x_1 + L_f \frac{dx_1}{dt} &= u \\ \text{Let } \frac{dx_1}{dt} &= \dot{x}_1, \quad \therefore R_f x_1 + L_f \dot{x}_1 = u \\ \dot{x}_1 &= -\frac{R_f}{L_f} x_1 + \frac{1}{L_f} u \end{aligned} \quad \dots 1.6.5$$

On substituting the state variables for the physical variables in Eqn (1.6.4) we get,

$$\begin{aligned} J \frac{d^2 x_3}{dt^2} + B \frac{dx_3}{dt} &= K_{tf} x_1 \\ \text{Let } \frac{d^2 x_3}{dt^2} &= \dot{x}_2 \text{ and } \frac{dx_3}{dt} = x_2, \quad \therefore J \dot{x}_2 + B x_2 = K_{tf} x_1 \\ \dot{x}_2 &= \frac{K_{tf}}{J} x_1 - \frac{B}{J} x_2 \end{aligned} \quad \dots 1.6.6$$

The state variable $x_3 = \theta$. On differentiating $x_3 = \theta$ with respect to t we get,

$$\begin{aligned} \frac{dx_3}{dt} &= \frac{d\theta}{dt} \\ \text{put } \frac{dx_3}{dt} &= \dot{x}_3 \text{ and } \frac{d\theta}{dt} = x_2 \quad \therefore \dot{x}_3 = x_2 \end{aligned} \quad \dots 1.6.7$$

The equations (1.6.5), (1.6.6) and (1.6.7) are the state equations of the system.

$$\begin{aligned} \dot{x}_1 &= -\frac{R_f}{L_f} x_1 + \frac{1}{L_f} u \\ \dot{x}_2 &= \frac{K_{tf}}{J} x_1 - \frac{B}{J} x_2 \\ \dot{x}_3 &= x_2 \end{aligned}$$

On arranging the state equations in the matrix form,

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} -\frac{R_f}{L_f} & 0 & 0 \\ \frac{K_{tf}}{J} & -\frac{B}{J} & 0 \\ 0 & 1 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} \frac{1}{L_f} \\ 0 \\ 0 \end{bmatrix} [u] \quad \dots 1.6.8$$

Let the desired output be ω and θ . Let us equate the desired output quantities to standard notation y_1 and y_2 as shown below.

$$y_1 = \omega ; y_2 = \theta$$

On relating the outputs to state variable we get,

$$y_1 = x_2 ; y_2 = x_3$$

The output equation in the matrix for is

$$\begin{bmatrix} y_1 \\ y_2 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} \quad \dots 1.6.9$$

The state equation [Eqn (1.6.8)] and the output equation [Eqn (1.6.9)] together constitute the state model of the system.

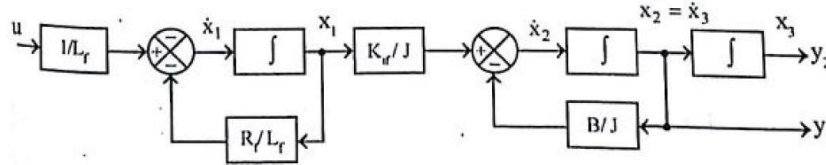


Figure 1.6.4 Block diagram representation of the state model field controlled dc motor

1.6 STATE SPACE REPRESENTATION USING PHASE VARIABLES

The phase variables are defined as those particular state variables which are obtained from one of the system variables and its derivatives. Usually the variable used is the system output and the remaining state variables are then derivatives of the output. The state model using phase variables can be easily determined if the system model is already known in the differential equation or transfer function form. There are three methods of modelling a system using phase variables and they are explained in the following sections.

Method 1

Consider the following n^{th} order linear differential equation relating to the output $y(t)$ to the input $u(t)$ of a system.

$$y^{(n)} + a_1 y^{(n-1)} + a_2 y^{(n-2)} + \dots + a_{n-2} \ddot{y} + a_{n-1} \dot{y} + a_n y = b u \quad \dots 1.10$$

By choosing the output y and their derivatives as state variables, we get,

$$\begin{aligned} x_1 &= y \\ x_2 &= \dot{y} \\ x_3 &= \ddot{y} \\ &\vdots \\ x_n &= y^{(n-1)} ; \quad \therefore \dot{x}_n = y^{(n)} \end{aligned}$$

On substituting the state variables in the differential equation governing the system [Eqn (1.10),] we get,

$$\begin{aligned} \dot{x}_n + a_1 x_n + a_2 x_{n-1} + \dots + a_{n-2} x_3 + a_{n-1} x_2 + a_n x_1 &= b u \\ \therefore \dot{x}_n &= -a_n x_1 - a_{n-1} x_2 - a_{n-2} x_3 - \dots - a_2 x_{n-1} - a_1 x_n + b u \end{aligned}$$

The state equations of the system are

$$\begin{aligned} \dot{x}_1 &= x_2 \\ \dot{x}_2 &= x_3 \\ &\vdots \\ \dot{x}_{n-1} &= x_n \\ \dot{x}_n &= -a_n x_1 - a_{n-1} x_2 - a_{n-2} x_3 - \dots - a_2 x_{n-1} - a_1 x_n + b u \end{aligned}$$

On arranging the above equations in the matrix form we get,

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \\ \vdots \\ \dot{x}_n \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 & 0 & \dots & 0 \\ 0 & 0 & 1 & 0 & \dots & 0 \\ 0 & 0 & 0 & 1 & \dots & 0 \\ \vdots & \vdots & & & & \\ 0 & 0 & 0 & 0 & \dots & 1 \\ -a_n & -a_{n-1} & -a_{n-2} & -a_{n-3} & \dots & -a_1 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ \vdots \\ x_{n-1} \\ x_n \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 0 \\ \vdots \\ 0 \\ b \end{bmatrix} [u] \quad \dots 1.11$$

$$\text{Or } \dot{X} = A X + B U$$

Here the matrix A (system matrix) has a very special form. It has all 1's in the upper off-diagonal, its last row is comprised of the negative of the coefficients of the original differential equation and all other elements are zero. This form of matrix A is known as **Bush form (or) Companion form**.

Also note that B matrix has the speciality that all its elements except the last element are zero. The output being $y = x_1$, the output equation is given by,

$$y = [1 \ 0 \ 0 \ \dots \ 0] \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ \vdots \\ x_n \end{bmatrix} \quad \dots 1.12$$

(or) $Y = C X$

The advantage in using phase variables for state space modelling is that the system state model can be written directly by inspection from the differential equation governing the system.

Method 2

Consider the following n^{th} order differential equation governing the output $y(t)$ to the input $u(t)$ of a system.

$$y^{(n)} + a_1 y^{(n-1)} + \dots + a_{n-1} \dot{y} + a_n y = b_0 \ddot{u} + b_1 \dot{u}^{(m-1)} + \dots + b_{m-1} \dot{u} + b_m u \quad \dots 1.13$$

Let $n = m = 3$

$$\therefore \ddot{y} + a_1 \dot{y} + a_2 y = b_0 \ddot{u} + b_1 \dot{u} + b_2 u \quad \dots 1.14$$

On taking laplace transform of Eqn (1.14) with zero initial conditions we get,

$$\begin{aligned} s^3 Y(s) + a_1 s^2 Y(s) + a_2 s Y(s) + a_3 Y(s) &= b_0 s^3 U(s) + b_1 s^2 U(s) + b_2 s U(s) + b_3 U(s) \\ (s^3 + a_1 s^2 + a_2 s + a_3) Y(s) &= (b_0 s^3 + b_1 s^2 + b_2 s + b_3) U(s) \\ \therefore \frac{Y(s)}{U(s)} &= \frac{b_0 s^3 + b_1 s^2 + b_2 s + b_3}{s^3 + a_1 s^2 + a_2 s + a_3} \\ &= \frac{s^3 \left(b_0 + \frac{b_1}{s} + \frac{b_2}{s^2} + \frac{b_3}{s^3} \right)}{s^3 \left(1 + \frac{a_1}{s} + \frac{a_2}{s^2} + \frac{a_3}{s^3} \right)} = \frac{b_0 + \frac{b_1}{s} + \frac{b_2}{s^2} + \frac{b_3}{s^3}}{1 - \left(-\frac{a_1}{s} - \frac{a_2}{s^2} - \frac{a_3}{s^3} \right)} \quad \dots 1.15 \end{aligned}$$

From the Mason's gain formula, the transfer function of the system is given by

$$T(s) = \frac{1}{\Delta} \sum_K P_K \Delta_K \quad \dots 1.16$$

- Where P_k = path gain of K^{th} forward path.
 Δ = $1 - (\text{sum of loop gain of all individual loops})$
 $\quad + (\text{sum of gain products of all possible combinations of two non-touching loops}) - \dots$
 Δ_k = Δ for that part of the graph which is not touching K^{th} forward path.

The transfer function of a system with four forward paths and with three feedback loops (touching each other) is given by,

$$T(s) = \frac{P_1 + P_2 + P_3 + P_4}{1 - (P_{11} + P_{12} + P_{13})} \quad \dots 1.17$$

On comparing equation (1.15) and (1.17) we get,

$$P_1 = b_0 \quad ; \quad P_2 = \frac{b_1}{s} \quad ; \quad P_3 = \frac{b_2}{s^2} \quad \text{and} \quad P_4 = \frac{b_3}{s^3}$$

$$P_{11} = -\frac{a_1}{s} \quad ; \quad P_{12} = -\frac{a_2}{s^2} \quad \text{and} \quad P_{13} = -\frac{a_3}{s^3}$$

Hence for this system represented by the transfer function as that of equation (1.15), a signal flow graph can be constructed as shown in the Figure 1.4. The signal flow is constructed such that all $\Delta_k = 1$ and all loops are touching loops.

Let us assign state variables at the output of each integrator in the signal flow graph. Hence at the input of each integrator, the first derivative of the state variable will be available. The state equations are formed by summing all the incoming signals to the nodes, whose values corresponds to first derivative of state variables.

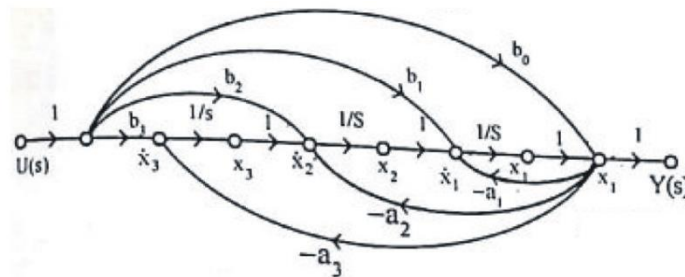
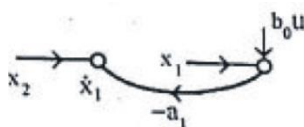


Figure 1.4 Signal flow graph of the system represented by the equation 1.15

By summing up the incoming signals to node \dot{x}_1 we get, (Refer Fig. 1.4a)

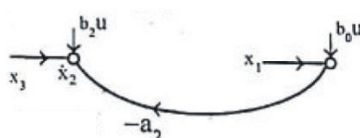


$$\dot{x}_1 = -a_1(x_1 + b_0 u) + x_2 + b_1 u$$

$$\therefore \dot{x}_1 = -a_1 x_1 + x_2 + (b_1 - a_1 b_0) u \quad \dots 1.18$$

Figure 4.4a

By summing up the incoming signals to node \dot{x}_2 we get, (Refer Fig. 1.4b)



$$\therefore \dot{x}_2 = -a_2 x_1 + x_3 + (b_2 - a_2 b_0)u \quad \dots 1.19$$

Figure 4.4b

By summing up the incoming signals to node \dot{x}_3 we get, (Refer Fig. 1.4c)



Figure 4.4c

$$\begin{aligned}\dot{x}_3 &= -a_3(x_1 + b_0u) + b_3u \\ \therefore \dot{x}_3 &= -a_3x_1 + (b_3 - a_3b_0)u \quad \dots 1.20\end{aligned}$$

The output equation is given by the sum of incoming signals to output node.

The output equation is given by the sum of incoming signals to output node.

$$\therefore y = x_1 + b_0 u \quad \dots 1.21$$

On arranging the state equations and the output equations in the matrix form, we get,

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} -a_1 & 1 & 0 \\ -a_2 & 0 & 1 \\ -a_3 & 0 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} b_1 - a_1b_0 \\ b_2 - a_2b_0 \\ b_3 - a_3b_0 \end{bmatrix} [u] \quad \dots 1.22$$

$$y = [1 \quad 0 \quad 0] \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + b_0u \quad \dots 1.23$$

The above results can be generalized for an n^{th} order differential equation, and the general state model for $m = n$ is given below.

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \vdots \\ \dot{x}_{n-1} \\ \dot{x}_n \end{bmatrix} = \begin{bmatrix} -a_1 & 1 & 0 & \dots & 0 \\ -a_2 & 0 & 1 & \dots & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots \\ -a_{n-1} & 0 & 0 & \dots & 1 \\ -a_n & 0 & 0 & \dots & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ \vdots \\ x_{n-1} \\ x_n \end{bmatrix} + \begin{bmatrix} b_1 - a_1b_0 \\ b_2 - a_2b_0 \\ \vdots \\ b_{n-1} - a_{n-1}b_0 \\ b_n - a_nb_0 \end{bmatrix} [u] \quad \dots 1.24$$

$$y = [1 \quad 0 \quad 0 \dots \dots 0] \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ \vdots \\ x_n \end{bmatrix} + b_0u \quad \dots 1.25$$

Method 3

Consider the following n^{th} order differential equation governing the output $y(t)$ to the input $u(t)$ of a system.

$$\ddot{y} + a_1 \dot{y}^{(n-1)} + \dots + a_{n-1} \dot{y} + a_n y = b_0 \ddot{u} + b_1 \dot{u}^{(m-1)} + \dots + b_{m-1} \dot{u} + b_m u \quad \dots 1.26$$

Let $n = m = 3$,

$$\therefore \ddot{y} + a_1 \dot{y} + a_2 y + a_3 y = b_0 \ddot{u} + b_1 \dot{u} + b_2 u + b_3 u \quad \dots 1.27$$

On taking laplace transform of Eqn (1.27) with zero initial conditions, we get.

$$\begin{aligned} s^3 Y(s) + a_1 s^2 Y(s) + a_2 s Y(s) + a_3 Y(s) &= b_0 s^3 U(s) + b_1 s^2 U(s) + b_2 s U(s) + b_3 U(s) \\ (s^3 + a_1 s^2 + a_2 s + a_3) Y(s) &= (b_0 s^3 + b_1 s^2 + b_2 s + b_3) U(s) \\ \therefore \frac{Y(s)}{U(s)} &= \frac{b_0 s^3 + b_1 s^2 + b_2 s + b_3}{s^3 + a_1 s^2 + a_2 s + a_3} \\ \text{Let } \frac{Y(s)}{U(s)} &= \frac{X_1(s)}{U(s)} \cdot \frac{Y(s)}{X_1(s)} \\ \text{Where } \frac{X_1(s)}{U(s)} &= \frac{1}{s^3 + a_1 s^2 + a_2 s + a_3} \end{aligned} \quad \dots 4.28$$

$$\text{and } \frac{Y(s)}{X_1(s)} = b_0 s^3 + b_1 s^2 + b_2 s + b_3 \quad \dots 4.29$$

On cross multiplying the Eqn (1.28) we get,

$$\begin{aligned} X_1(s) [s^3 + a_1 s^2 + a_2 s + a_3] &= U(s) \\ s^3 X_1(s) + a_1 s^2 X_1(s) + a_2 s X_1(s) + a_3 X_1(s) &= U(s) \end{aligned} \quad \dots 4.30$$

On taking inverse laplace transform of Eqn (1.30) we get,

$$\ddot{x}_1 + a_1 \dot{x}_1 + a_2 x_1 + a_3 x_1 = u \quad \dots 1.31$$

Let the state variable be, x_1 , x_2 and x_3

where, $x_2 = \dot{x}_1$

and $x_3 = \ddot{x}_1 = \dot{x}_2$; $\therefore \dot{x}_3 = \ddot{x}_1$

On substituting the state variables in equation (1.31) we get,

$$\begin{aligned} \dot{x}_3 + a_1 x_3 + a_2 x_2 + a_3 x_1 &= u \\ \therefore \dot{x}_3 &= -a_3 x_1 - a_2 x_2 - a_1 x_3 + u \end{aligned}$$

The state equations are,

$$\begin{aligned} \dot{x}_1 &= x_2 \\ \dot{x}_2 &= x_3 \\ \dot{x}_3 &= -a_3 x_1 - a_2 x_2 - a_1 x_3 + u \end{aligned}$$

On cross multiply the Eqn (1.29) we get,

$$Y(s) = b_0 s^3 X_1(s) + b_1 s^2 X_1(s) + b_2 s X_1(s) + b_3 X_1(s) \quad \dots 1.32$$

On taking inverse laplace transform of Eqn (1.32), we get,

$$y = b_0 \ddot{x}_1 + b_1 \ddot{x}_1 + b_2 \dot{x}_1 + b_3 x_1 \quad \dots 1.33$$

On substituting the state variables in Eqn (1.33) we get,

$$y = b_0 \dot{x}_3 + b_1 x_3 + b_2 x_2 + b_3 x_1 \quad \dots 1.34$$

Put $\dot{x}_3 = -a_3 x_1 - a_2 x_2 - a_1 x_3 + u$ in equ(4.34)

$$\begin{aligned} \therefore y &= b_0 (-a_3 x_1 - a_2 x_2 - a_1 x_3 + u) + b_1 x_3 + b_2 x_2 + b_3 x_1 \\ y &= (b_3 - a_3 b_0) x_1 + (b_2 - a_2 b_0) x_2 + (b_1 - a_1 b_0) x_3 + b_0 u \end{aligned} \quad \dots 1.35$$

The equation (1.35) is the output equation.

On arranging the state equations and output equations in the matrix form, we get,

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -a_3 & -a_2 & -a_1 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} [u] \quad \dots 1.36$$

$$y = [(b_3 - a_3 b_0) \quad (b_2 - a_2 b_0) \quad (b_1 - a_1 b_0)] \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + [b_0] u \quad \dots 1.37$$

The above results can be generalized for an n^{th} order differential equation and the general state model for $m = n$ is given below.

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \vdots \\ \dot{x}_{n-1} \\ \dot{x}_n \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 & \dots & 0 \\ 0 & 0 & 1 & \dots & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & 0 & \dots & 1 \\ -a_n & -a_{n-1} & -a_{n-2} & \dots & -a_1 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ \vdots \\ x_{n-1} \\ x_n \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ \vdots \\ 0 \\ 1 \end{bmatrix} [u] \quad \dots 1.38$$

$$y = [(b_n - a_n b_0) \quad (b_{n-1} - a_{n-1} b_0) \quad \dots \quad (b_2 - a_2 b_0) \quad (b_1 - a_1 b_0)] \begin{bmatrix} x_1 \\ x_2 \\ \vdots \\ x_{n-1} \\ x_n \end{bmatrix} + b_0 u \quad \dots 1.39$$

Advantages of Phase Variables

The state space model can be directly formed by inspection from the differential equations governing the system. The phase variables provides a link between the transfer function design approach and time-domain design approach.

Disadvantage of Phase Variables

The phase variables are not physical variables of the system and therefore are not available for measurement and control purposes.

EXAMPLE 1.7

Construct a state model for a system characterized by the differential equation,

$$\frac{d^3y}{dt^3} + 6 \frac{d^2y}{dt^2} + 11 \frac{dy}{dt} + 6y + u = 0.$$

Give the block diagram representation of the state model.

SOLUTION

Let us choose y and their derivatives as state variables. The system is governed by third order differential equation and so the number of state variables are three.

The state variables x_1 , x_2 and x_3 are related to phase variables as follows.

$$x_1 = y$$

$$x_2 = \frac{dy}{dt} = \dot{x}_1$$

$$x_3 = \frac{d^2y}{dt^2} = \dot{x}_2$$

Put $y = x_1$, $\frac{dy}{dt} = x_2$ and $\frac{d^2y}{dt^2} = x_3$ and $\frac{d^3y}{dt^3} = \dot{x}_3$ in the given equation,

$$\therefore \dot{x}_3 + 6x_3 + 11x_2 + 6x_1 + u = 0$$

$$\text{or } \dot{x}_3 = -6x_1 - 11x_2 - 6x_3 - u.$$

The state equations are

$$\dot{x}_1 = x_2$$

$$\dot{x}_2 = x_3$$

$$\dot{x}_3 = -6x_1 - 11x_2 - 6x_3 - u$$

On arranging the state equations in the matrix form we get,

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -6 & -11 & -6 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ -1 \end{bmatrix} [u]$$

Here $y = \text{output}$

But, $y = x_1$

\therefore The output equation is, $y = [1 \ 0 \ 0] \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$

The state equation and output equation, constitutes the state model of the system, The block diagram form of the state diagram of the system is shown in Figure 1.7.1

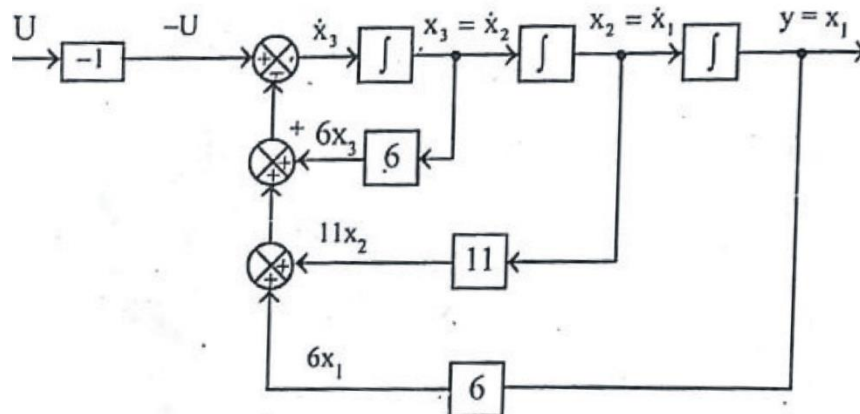


Figure 1.7.1 Block diagram form of state diagram

EXAMPLE 1.8

The state diagram of a system is shown in Figure 1.8.1. Assign state variables and obtain the state model of the system.

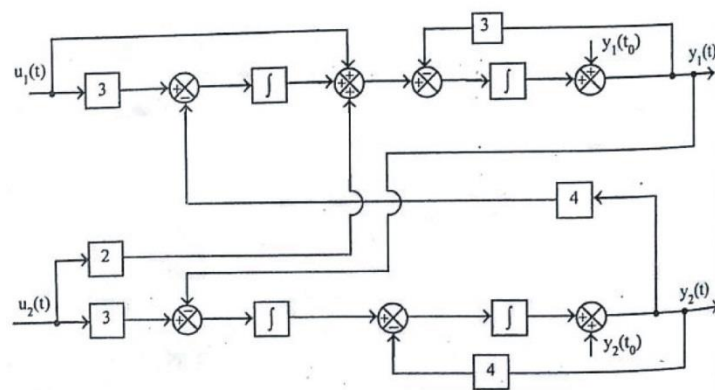


Figure 1.8.1

SOLUTION

Since there are 4-integrators in the state diagram we can assign, 4 state variables. The state variables can be assigned at the output of the integrators as shown in Figure 1.8.2. Hence at the input of the integrator, the first derivative of the state variable will be available. The state equations are formed by summing all the incoming signals to the input of the integrator and equating to the corresponding first derivatives of the state variable.

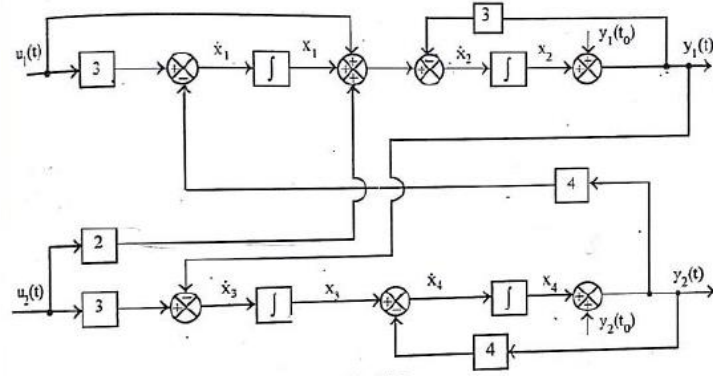


Figure 1.8.2

On adding the signals coming to the 1st integrator we get, (refer Figure 1.8.3).

$$\dot{x}_1 = -4x_4 + 3u_1$$

On adding the signals coming to the 2nd integrator we get, (Refer Figure 1.8.4)

$$\dot{x}_2 = x_1 - 3x_2 + u_1 + 2u_2$$

On adding the signals coming to the 3rd integrator we get, (Refer Figure 1.8.5)

$$\dot{x}_3 = -x_2 + 3u_2$$

On adding the signals coming to the 4th integrator we get, (Refer Figure 1.8.6)

$$\dot{x}_4 = x_3 - 4x_4$$

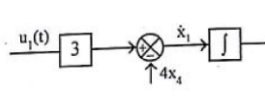


Figure 1.8.3

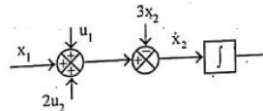


Figure 1.8.4

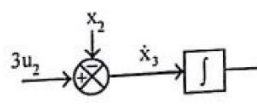


Figure 1.8.5

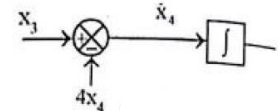


Figure 1.8.6

The state equations are

$$\dot{x}_1 = -4x_4 + 3u_1$$

$$\dot{x}_2 = x_1 - 3x_2 + u_1 + 2u_2$$

$$\dot{x}_3 = -x_2 + 3u_2$$

$$\dot{x}_4 = x_3 - 4x_4$$

The output equations are, $y_1 = x_2$ and $y_2 = x_4$.

The state equations and output equations are arranged in the matrix form as shown below. The state equations and output equations together constitute the state model of the system.

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \\ \dot{x}_4 \end{bmatrix} = \begin{bmatrix} 0 & 0 & 0 & -4 \\ 1 & -3 & 0 & 0 \\ 0 & -1 & 0 & 0 \\ 0 & 0 & 1 & -4 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix} + \begin{bmatrix} 3 & 0 \\ 1 & 2 \\ 0 & 3 \\ 0 & 0 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}$$

$$\begin{bmatrix} y_1 \\ y_2 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix}$$

EXAMPLE 1.9

The state diagram of a linear system is given below. Assign state variables to obtain the state model.

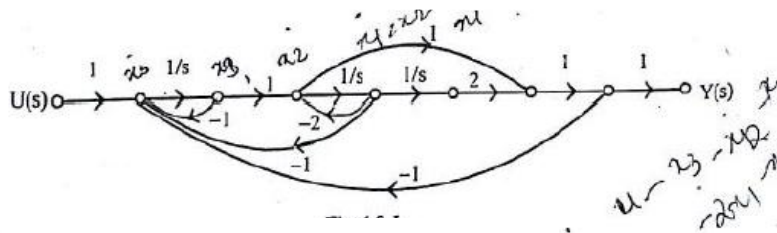


Figure 1.9.1

SOLUTION

Since there are three integrators (1/s) we can assign three state variables. The state variables are assigned at the output of the integrator as shown in Figure 1.9.2. At the input of the integrator we have the first derivative of the state variable. The state equations are formed by summing all the signals at the input of integrator and equating to the corresponding first derivatives of state variable.

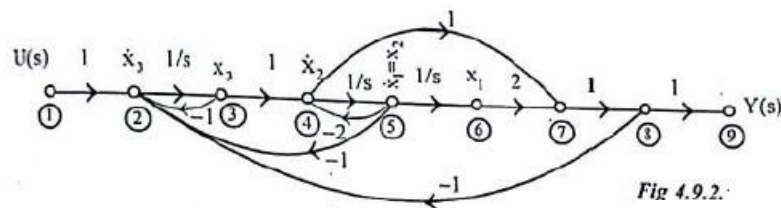


Figure 1.9.2

On adding the signals coming to node-5, we get, (Refer Figure 1.9.3)

$$\dot{x}_1 = x_2$$

On adding the signals coming to node-4, we get, (Refer Figure 1.9.4)

$$\dot{x}_2 = -2x_2 + x_3$$

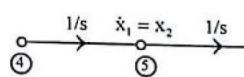


Figure 1.9.3

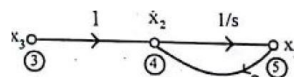


Figure 1.9.4

On adding the signals coming to node-2, we get, (refer Figure 1.9.5).

$$\dot{x}_3 = -(\dot{x}_2 + 2x_1) - x_2 - x_3 + u = -\dot{x}_2 - 2x_1 - x_2 - x_3 + u$$

Put $\dot{x}_2 = -2x_2 + x_3$

$$\therefore \dot{x}_3 = +2x_2 - x_3 - 2x_1 - x_2 - x_3 + u = -2x_1 + x_2 - 2x_3 + u$$

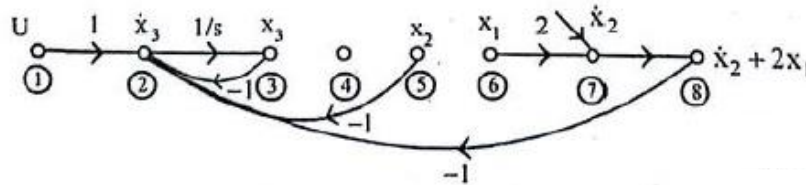


Figure 1.9.5

The state equations are

$$\begin{aligned}\dot{x}_1 &= x_2 \\ \dot{x}_2 &= -2x_2 + x_3 \\ \dot{x}_3 &= -2x_1 + x_2 - 2x_3 + u\end{aligned}$$

The output equation is obtained by adding the signals coming to output node (refer Figure 1.9.6)

$$\begin{aligned}y &= 2x_1 + \dot{x}_2 \\ \text{Put } \dot{x}_2 &= -2x_2 + x_3 \\ y &= 2x_1 + (-2x_2 + x_3) \\ y &= 2x_1 - 2x_2 + x_3\end{aligned}$$

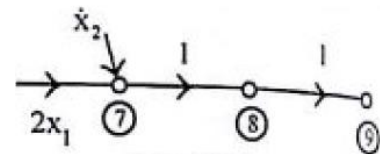


Figure 1.9.6

The state equations and the output equation are arranged in the matrix form as shown below.

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & -2 & 1 \\ -2 & 1 & -2 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} [u]$$

$$y = [2 \quad -2 \quad 1] \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$$

EXAMPLE 1.10

Obtain the state model of the system whose transfer function is given as,

$$\frac{Y(s)}{U(s)} = \frac{10}{s^3 + 4s^2 + 2s + 1}$$

SOLUTION

Method 1

$$\text{Given that, } \frac{Y(s)}{U(s)} = \frac{10}{s^3 + 4s^2 + 2s + 1} \quad \dots 1.10.1$$

On cross multiplying the Eqn (1.10.1) we get,

$$\begin{aligned} Y(s)[s^3 + 4s^2 + 2s + 1] &= 10 U(s) \\ s^3 Y(s) + 4s^2 Y(s) + 2s Y(s) + Y(s) &= 10 U(s) \end{aligned} \quad \dots 1.10.2$$

On taking inverse laplace transform of Eqn (1.10.2) we get,

$$\ddot{y} + 4\ddot{y} + 2\dot{y} + y = 10u. \quad \dots 1.10.3$$

Let us define state variables as follows,

$$x_1 = y \quad ; \quad x_2 = \dot{y} \quad ; \quad x_3 = \ddot{y}$$

Put $\ddot{y} = \dot{x}_3$; $\dot{y} = x_3$; $\dot{x}_2 = x_3$ and $y = x_1$ in the equation (1.10.3)

$$\therefore \dot{x}_3 + 4x_3 + 2x_2 + x_1 = 10u$$

$$\text{or } \dot{x}_3 = -x_1 - 2x_2 - 4x_3 + 10u.$$

The state equations are

$$\dot{x}_1 = x_2 \quad ; \quad \dot{x}_2 = x_3 \quad ; \quad \dot{x}_3 = -x_1 - 2x_2 - 4x_3 + 10u.$$

The output equation is $y = x_1$

The state model in the matrix form is,

$$\begin{aligned} \begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} &= \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -1 & -2 & -4 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 10 \end{bmatrix} [u] \\ y &= [1 \quad 0 \quad 0] \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} \end{aligned}$$

Method 2

$$\begin{aligned} \frac{Y(s)}{U(s)} &= \frac{10}{s^3 + 4s^2 + 2s + 1} = \frac{10}{s^3 \left(1 + \frac{4}{s} + \frac{2}{s^2} + \frac{1}{s^3} \right)} \\ &= \frac{10/s^3}{1 - \left(-\frac{4}{s} - \frac{2}{s^2} - \frac{1}{s^3} \right)} \end{aligned}$$

The signal flow graph for the above transfer function can be constructed as shown in Figure 1.10.1 with a single forward path consisting of three integrators and with path gain $10/s^3$. The graph will have three individual loops with loop gains $-4/2$, $-2/s^2$, and $1/s^3$.

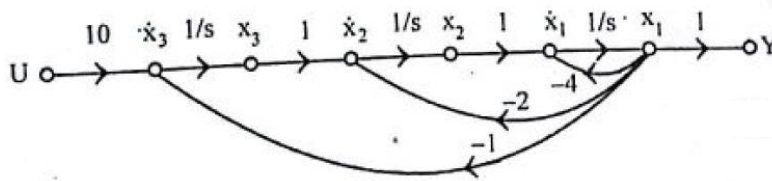


Figure 1.10.1

Assign state variables at the output of the integrator ($1/s$). The state equations are obtained by summing the incoming signals to the input of the integrators and equating them to the corresponding first derivative of the state variable. Refer Figure 1.10.2 to Figure 1.10.4).

The state equations are

$$\dot{x}_1 = -4x_1 + x_2$$

$$\dot{x}_2 = -2x_1 + x_3$$

$$\dot{x}_3 = -x_1 + 10u$$

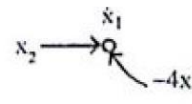


Figure 1.10.2

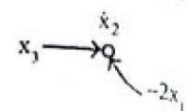


Figure 1.10.3

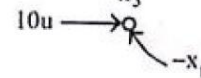


Figure 1.10.4

The output equation is, $y = x_1$

The state model in the matrix form is,

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} -4 & 1 & 0 \\ -2 & 0 & 1 \\ -1 & 0 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 10 \end{bmatrix} [u] \quad ; \quad y = [1 \ 0 \ 0] \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$$

1.7 STATE SPACE REPRESENTATION USING CANONICAL VARIABLES

In canonical form (or normal form) of state model, the system matrix A will be a diagonal matrix. The elements on the diagonal are the poles of the transfer function of the system.

By partial fraction expansion, the transfer function $Y(s)/U(s)$ of the n^{th} order system can be expressed as shown in Eqn (1.40).

$$\frac{\Omega(z)}{\lambda(z)} = p^0 + \frac{z+y^1}{C^1} + \frac{z+y^3}{C^3} + \dots + \frac{z+y^u}{C^u} \quad \dots 1.40$$

where $C_1, C_2, C_3, \dots, C_n$ are residues and $\lambda_1, \lambda_2, \dots, \lambda_n$ are roots of denominator polynomial (or poles of the system).

The equation (1.40) can be rearranged as shown below.

$$\begin{aligned}\frac{Y(s)}{U(s)} &= b_0 + \frac{C_1}{s(1+\frac{\lambda_1}{s})} + \frac{C_2}{s(1+\frac{\lambda_2}{s})} + \dots + \frac{C_n}{s(1+\frac{\lambda_n}{s})} \\ &= b_0 + \frac{C_1/s}{1+\lambda_1/s} + \frac{C_2/s}{1+\lambda_2/s} + \dots + \frac{C_n/s}{1+\lambda_n/s}\end{aligned}$$

$$\begin{aligned}\therefore Y(s) &= b_0 U(s) + \left[\frac{1/s}{1+(1/s) \times \lambda_1} \times C_1 \right] U(s) + \left[\frac{1/s}{1+(1/s) \times \lambda_2} \times C_2 \right] U(s) + \\ &\quad \dots + \left[\frac{1/s}{1+(1/s) \times \lambda_n} \times C_n \right] U(s)\end{aligned} \quad \dots 1.41$$

The equation (1.41) can be represented by a block diagram as shown in Figure 1.5.

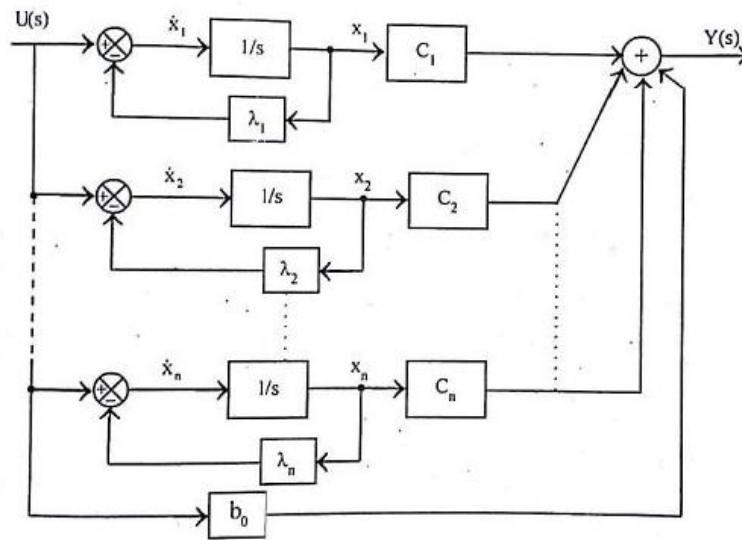


Figure 1.5 Block diagram of canonical state model

Assign state variables at the output of integrator. The input of the integrator will be first derivative of state variable. The state equations are formed by adding the incoming signals to the integrator and equating to first derivative of state variable. The state equations are,

$$\begin{aligned}\dot{x}_1 &= -\lambda_1 x_1 + u \\ \dot{x}_2 &= -\lambda_2 x_2 + u \\ &\vdots \\ \dot{x}_n &= -\lambda_n x_n + u\end{aligned}$$

The output equation is, $y = C_1 x_1 + C_2 x_2 + \dots + C_n x_n + b_0 u$.

The canonical form of state model in the matrix form is given below.

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \\ \vdots \\ \dot{x}_n \end{bmatrix} = \begin{bmatrix} -\lambda_1 & 0 & 0 & \dots & 0 \\ 0 & -\lambda_2 & 0 & \dots & 0 \\ 0 & 0 & -\lambda_3 & \dots & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & 0 & \dots & -\lambda_n \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ \vdots \\ x_n \end{bmatrix} + \begin{bmatrix} 1 \\ 1 \\ 1 \\ \vdots \\ 1 \end{bmatrix} [u] \quad \dots 1.41$$

$$y = [C_1 \ C_2 \ C_3 \ \dots \ C_n] \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ \vdots \\ x_n \end{bmatrix} + [b_0] [u] \quad \dots 1.42$$

The advantage of canonical form is that the state equations are independent of each other. The disadvantage is that the canonical variables are not physical variables and so they are not available for measurement and control.

When a pole of the transfer function has multiplicity, the canonical state model will be in a special form called Jordan canonical form. In this form the system matrix A will have a Jordan block of size $q \times q$, correspond to a pole of value λ_1 with multiplicity q . In the Jordan block the diagonal element will be the poles and the element just above the diagonal is one.

Consider a system with poles $\lambda_1, \lambda_1, \lambda_1, \lambda_4, \lambda_5, \dots, \lambda_n$ where λ_1 has multiplicity of three. The input matrix (B) and system matrix for this case will be as shown in Eqn (1.41a). The system matrix is also denoted as J.

$$B = \begin{bmatrix} 0 \\ 0 \\ 1 \\ 1 \\ \vdots \\ 1 \end{bmatrix} ; \quad A = J = \begin{bmatrix} -\lambda_1 & 1 & 0 & 0 & \dots & 0 \\ 0 & -\lambda_1 & 1 & 0 & \dots & 0 \\ 0 & 0 & -\lambda_1 & 0 & \dots & 0 \\ 0 & 0 & 0 & -\lambda_4 & \dots & 0 \\ \vdots & \vdots & \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & 0 & 0 & \dots & -\lambda_n \end{bmatrix} \quad \text{Jordan block of size } 3 \times 3 \quad \dots 1.41a$$

The transfer function of the system for this case is given by Eqn (1.40a) and the block diagram is shown in Figure 1.5a.

$$\frac{Y(s)}{U(s)} = b_0 + \frac{C_1}{(s + \lambda_1)^3} + \frac{C_2}{(s + \lambda_1)^2} + \frac{C_3}{s + \lambda_1} + \frac{C_4}{s + \lambda_4} + \dots + \frac{C_n}{s + \lambda_n} \quad \dots 1.40a$$

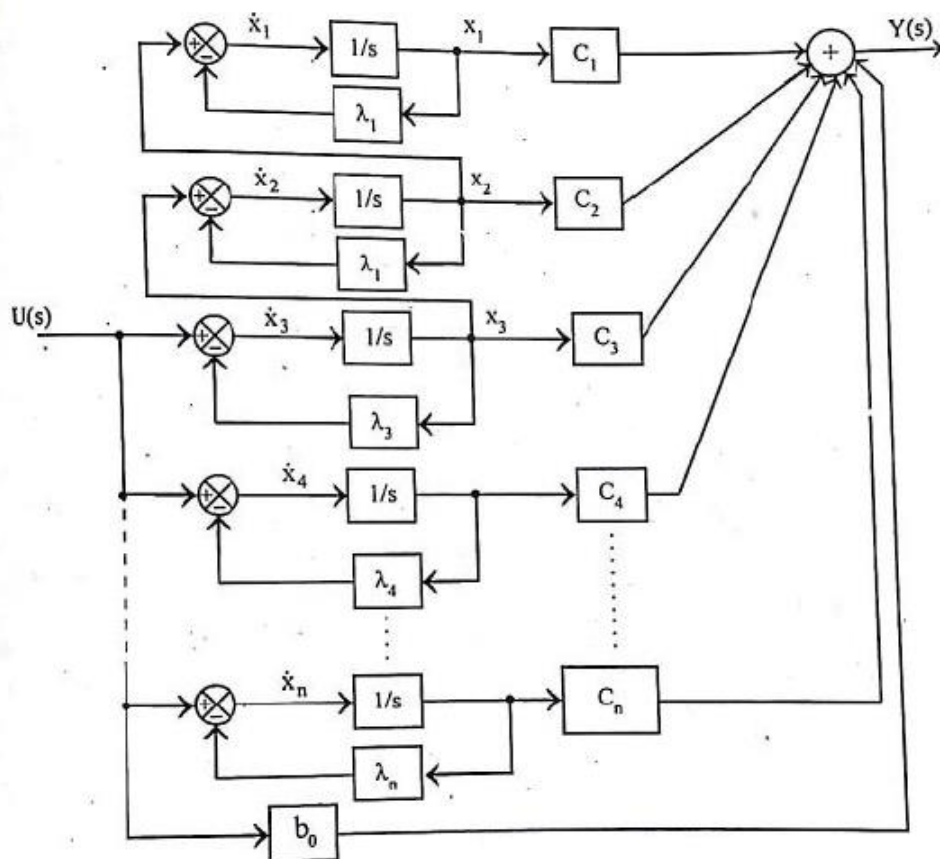


Figure 1.5a Block diagram of Jordan canonical state model

EXAMPLE 1.11

A feedback system has a closed-loop transfer function

$$\frac{Y(s)}{U(s)} = \frac{10(s+4)}{s(s+1)(s+3)}$$

Construct three different state models for this system and give block diagram representation for each state model.

SOLUTION

Mode 1

$$\frac{Y(s)}{U(s)} = \frac{10(s+4)}{s(s+1)(s+3)} = \frac{10s+40}{s(s^2+4s+3)} = \frac{10s+40}{s^3+4s^2+3s} = \frac{10s+40}{s^3 \left(1 + \frac{4}{s} + \frac{3}{s^2}\right)} = \frac{\frac{10}{s^2} + \frac{40}{s^3}}{1 - \left(-\frac{4}{s} - \frac{3}{s^2}\right)}$$

A signal flow graph for the above transfer function can be constructed as shown Figure 1.11.1 with two forward paths and two individual loops. The forward path gains are $10/s^2$ and $40/s^3$. The loop gains are $-4/s$ and $-3/s^2$.

Assign state variables at the output of integrator as shown in FIGURE 1.11.1 and so the input of integrator is first derivative of state variable. The state equation are obtained by summing all the incoming signals to the integrator and equating to the corresponding first derivative of the state variable. [Refer Figure 1.11.2 to 1.11.3]

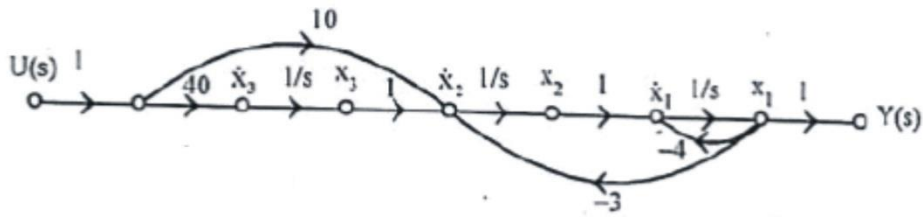


Figure 1.11.1

The state equations are

$$\begin{aligned}\dot{x}_1 &= -4x_1 + x_2 \\ \dot{x}_2 &= -3x_1 + x_3 + 10u \\ \dot{x}_3 &= 40u\end{aligned}$$

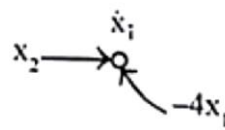


Figure 1.11.2

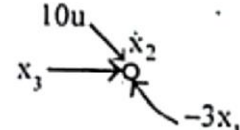


Figure 1.11.3

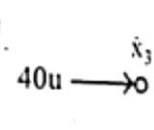


Figure 1.11.4

The output equation is, $y = x_1$

The state model is obtained by arranging the state equations and the output equation in the matrix form as shown below. The block diagram representative of this state model is shown in Figure 1.11.5.

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} -4 & 1 & 0 \\ -3 & 0 & 1 \\ 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 10 \\ 40 \end{bmatrix} [u] \quad ; \quad y = [1 \ 0 \ 0] \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$$

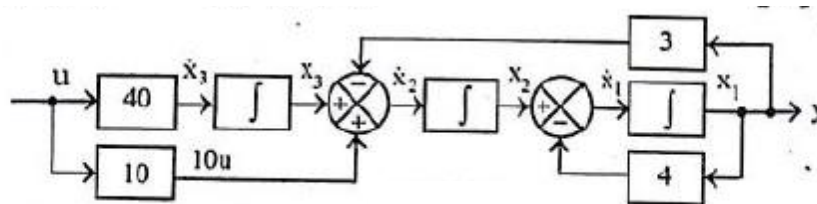


Figure 1.11.5

Model 2

Give that, $\frac{Y(s)}{U(s)} = \frac{10(s+4)}{s(s+1)(s+3)}$

Let $\frac{Y(s)}{U(s)} = \frac{X_1(s)}{U(s)} \cdot \frac{Y(s)}{X_1(s)} = \frac{10(s+4)}{s(s+1)(s+3)}$

Let $\frac{X_1(s)}{U(s)} = \frac{1}{s(s+1)(s+3)}$ and $\frac{Y(s)}{X_1(s)} = 10(s+4)$

$$\frac{X_1(s)}{U(s)} = \frac{1}{s(s+1)(s+3)} = \frac{1}{s(s^2+4s+3)} \quad ; \quad \therefore \frac{X_1(s)}{U(s)} = \frac{1}{s^3+4s^2+3s} \quad \dots 1.11.1$$

On cross multiplying the Eqn (1.11.1) we get,

$$\begin{aligned} X_1(s)[s^3+4s^2+3s] &= U(s) \\ \therefore s^3 X_1(s) + 4s^2 X_1(s) + 3s X_1(s) &= U(s) \end{aligned} \quad \dots 1.11.2$$

On taking inverse laplace transform of Eqn (1.11.2) we get,

$$\ddot{x}_1 + 4\ddot{x}_1 + 3\dot{x}_1 = u \quad \dots 1.11.3$$

Let the state variables be x_1 , x_2 and x_3 ; where $x_2 = \dot{x}_1$ and $x_3 = \dot{x}_2$.

Put $\dot{x}_1 = x_2$, $\dot{x}_2 = x_3$ and $\ddot{x}_1 = \dot{x}_3$ in Eqn (1.11.3),

$$\therefore \dot{x}_3 + 4x_3 + 3x_2 = u \quad (\text{or}) \quad \dot{x}_3 = -3x_2 - 4x_3 + u$$

The state equations are

$$\dot{x}_1 = x_2 \quad ; \quad \dot{x}_2 = x_3 \quad ; \quad \dot{x}_3 = -3x_2 - 4x_3 + u$$

Consider the second part of transfer function,

$$\frac{Y(s)}{X_1(s)} = 10(s+4) = 10s+40 \quad \dots 1.11.4$$

On cross multiplying Eqn (1.11.4) we get,

$$Y(s) = 10s X_1(s) + 40 X_1(s) \quad \dots 1.11.5$$

On taking inverse laplace transform of Eqn (1.11.5) we get,

$$\begin{aligned} y &= 10\dot{x}_1 + 40x_1 \\ \text{Put } \dot{x}_1 &= x_2, \quad \therefore y = 10x_2 + 40x_1 = 40x_1 + 10x_2 \end{aligned}$$

Here, $y = 40x_1 + 10x_2$ is the output equation. The state model in the matrix form is shown below. The block diagram representation of this state model is shown in Figure 1.11.6.

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ 0 & -3 & -4 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} [u] \quad ; \quad y = [40 \quad 10 \quad 0] \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$$

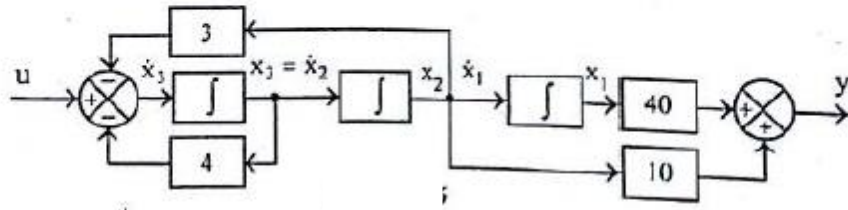


Figure 1.11.6

Model 3

$$\frac{Y(s)}{U(s)} = \frac{10(s+4)}{s(s+1)(s+3)}$$

By partial fraction expansion $Y(s) / U(s)$ can be expressed as,

$$\frac{Y(s)}{U(s)} = \frac{10(s+4)}{s(s+1)(s+3)} = \frac{A}{s} + \frac{B}{s+1} + \frac{C}{s+3}$$

$$A = \frac{10(s+4)}{(s+1)(s+3)} \Big|_{s=0} = \frac{10 \times 4}{1 \times 3} = \frac{40}{3}$$

$$B = \frac{10(s+4)}{s(s+3)} \Big|_{s=-1} = \frac{10(-1+4)}{-1(-1+3)} = \frac{10 \times 3}{-1 \times 2} = -15$$

$$C = \frac{10(s+4)}{s(s+1)} \Big|_{s=-3} = \frac{10(-3+4)}{-3(-3+1)} = \frac{10 \times 1}{-3 \times (-2)} = \frac{5}{3}$$

$$\frac{Y(s)}{U(s)} = \frac{40/3}{s} - \frac{15}{s+1} + \frac{5/3}{s+3}$$

...1.11.6

The equation (1.11.6) can be rearranged as shown below

$$\frac{Y(s)}{U(s)} = \frac{40/3}{s} - \frac{15}{s(1+1/s)} + \frac{5/3}{s(1+3/s)}$$

$$\therefore Y(s) = \left[\frac{1}{s} \times \frac{40}{3} \right] U(s) - \left[\frac{1/s}{1 + \frac{1}{s} \times 1} \times 15 \right] U(s) + \left[\frac{1/s}{1 + \frac{1}{s} \times 3} \times \frac{5}{3} \right] U(s)$$

...1.11.7

The block diagram of the Eqn (1.11.7) is shown in Figure 1.11.7

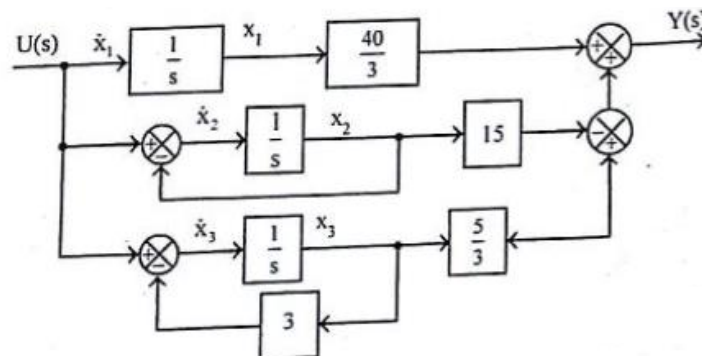


Figure 1.11.7

Assign state variables at the output of the integrator as shown in Figure 1.11.7. At the input of the integrator, the first derivative of the state variables will be available. The state equations are obtained by adding incoming signals to the integrator and equating to the corresponding first derivative of the state variable.

The state equations are

$$\begin{aligned}\dot{x}_1 &= u \\ \dot{x}_2 &= -x_2 + u \\ \dot{x}_3 &= -3x_3 + u\end{aligned}$$

The output equation is $y = \frac{40}{3}x_1 - 15x_2 + \frac{5}{3}x_3$

The state model in the matrix form is shown below. The Figure 1.11.7 is the block diagram representation of this state model.

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} 0 & 0 & 0 \\ 0 & -1 & 0 \\ 0 & 0 & -3 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 1 \\ 1 \\ 1 \end{bmatrix} [u] ; \quad y = \begin{bmatrix} \frac{40}{3} & -15 & \frac{5}{3} \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$$

EXAMPLE 1.12

Determine the canonical state model of the system, whose transfer function is $T(s) = 2(s+5)/[(s+2)(s+3)(s+4)]$

SOLUTION

By partial fraction expansion,

$$\begin{aligned}\frac{Y(s)}{U(s)} &= \frac{2(s+5)}{(s+2)(s+3)(s+4)} = \frac{A}{s+2} + \frac{B}{s+3} + \frac{C}{s+4} \\ A &= \left. \frac{2(s+5)}{(s+3)(s+4)} \right|_{s=-2} = \frac{2(-2+5)}{(-2+3)(-2+4)} = \frac{2 \times 3}{1 \times 2} = 3 \\ B &= \left. \frac{2(s+5)}{(s+2)(s+4)} \right|_{s=-3} = \frac{2(-3+5)}{(-3+2)(-3+4)} = \frac{2 \times 2}{-1 \times 1} = -4 \\ C &= \left. \frac{2(s+5)}{(s+2)(s+3)} \right|_{s=-4} = \frac{2(-4+5)}{(-4+2)(-4+3)} = \frac{2 \times 1}{-2 \times (-1)} = 1 \\ \therefore \frac{Y(s)}{U(s)} &= \frac{3}{s+2} - \frac{4}{s+3} + \frac{1}{s+4}\end{aligned}$$

...1.12.1

The equation (1.12.1) can be rearranged as shown below.

$$\begin{aligned}\frac{Y(s)}{U(s)} &= \frac{3}{s(1+2/s)} - \frac{4}{s(1+3/s)} + \frac{1}{s(1+4/s)} \\ \therefore Y(s) &= \left[\frac{\frac{1}{s}}{1+\frac{1}{s} \times 2} \times 3 \right] U(s) - \left[\frac{\frac{1}{s}}{1+\frac{1}{s} \times 3} \times 4 \right] U(s) + \left[\frac{\frac{1}{s}}{1+\frac{1}{s} \times 4} \right] U(s)\end{aligned}$$

...1.12.2

The equation (1.12.2) can be represented by the block diagram in Figure 1.12.1

Assigns state variables at the output of the integrators as shown in Figure 4.12.1. At the input of the integrators we have first derivative of the state variables. The state equations are formed by adding all the incoming signals to the integrator and equating to the corresponding first derivative of state variable.

The state equations are

$$\dot{x}_1 = -2x_1 + u \quad ; \quad \dot{x}_2 = -3x_2 + u \quad ; \quad \dot{x}_3 = -4x_3 + u$$

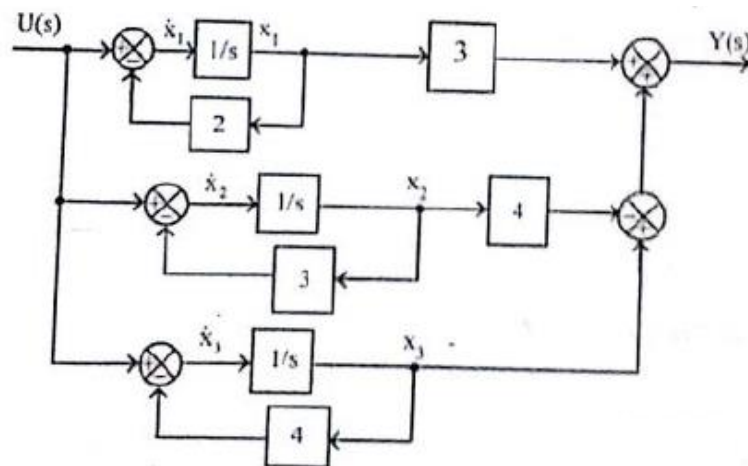


Figure 1.12.1

The output equation is, $y = 3x_1 - 4x_2 + x_3$

The state model in matrix form is given below.

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} -2 & 0 & 0 \\ 0 & -3 & 0 \\ 0 & 0 & -4 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 1 \\ 1 \\ 1 \end{bmatrix} [u] \quad ; \quad y = \begin{bmatrix} 3 & -4 & 1 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$$

1.8 SOLUTION OF STATE EQUATIONS

SOLUTION OF HOMOGENEOUS STATE EQUATIONS

(Solution of State Equations without input or excitation)

Consider a first order differential equation, with initial condition, $x(0) = x_0$.

$$\frac{dx}{dt} = ax \quad ; \quad x(0) = x_0 \quad \dots 1.43$$

On rearranging Eqn (1.43) we get, $\frac{dx}{x} = a dt \quad \dots 1.44$

On integrating Eqn (1.44) we get,

$$\log x = at + C$$

$$\therefore x = e^{(at+C)} = e^{at} \cdot e^C \quad \dots 1.45$$

When $t = 0$, from Eqn (1.45) we get, $x = x(0) = e^C$

Given that $x(0) = x_0$; $\therefore e^C = x_0$

On substituting the initial condition in Eqn (1.45), we get the solution of first order differential equation as

$$x = e^{at} x_0. \quad \dots 1.46$$

$$\text{We know that, } e^x = \left[1 + x + \frac{1}{2!}x^2 + \dots + \frac{1}{n!}x^n + \dots \right] \quad \dots 1.47$$

From Eqn (1.46) and (1.47) we get,

$$x = e^{at} x_0 = \left(1 + at + \frac{1}{2!}a^2t^2 + \frac{1}{3!}a^3t^3 + \dots + \frac{1}{i!}a^it^i + \dots \right) x_0 \quad \dots 1.48$$

Consider the state equations without input vector, (i.e., homogeneous state equation)

$$\dot{X}(t) = A X(t) \quad ; \quad X(0) = X_0 \quad \dots 1.49$$

Where $X(0)$ is the initial condition vector.

By analog of the solution of first order differential equation [Eqn (4.48)], the solution of the matrix or vector equation can be assumed as shown in Eqn (1.50).

$$X(t) = A_0 + A_1t + A_2t^2 + A_3t^3 + \dots + A_it^i + \dots \quad \dots 1.50$$

Where $A_0, A_1, A_2, \dots, A_i \dots$ are matrices and the elements of the matrices are constants.

On differentiating the Eqn (1.5) we get,

$$\dot{X}(t) = A_1 + 2A_2t + 3A_3t^2 + \dots + iA_it^{i-1} + \dots \quad \dots 1.51$$

On multiplying Equation (1.50) by A , we get,

$$A X(t) = A \left[A_0 + A_1t + A_2t^2 + A_3t^3 + \dots + A_it^i + \dots \right] \quad \dots 1.52$$

From Eqn (1.49), we know that $\dot{X}(t) = A X(t)$. Therefore we can equate the coefficients of equal powers of t in equations (1.51) and (1.52) as shown below.

On equating constants we get,

$$A_1 = A A_0$$

On equating coefficients of t we get,

$$2A_2 = A A_1$$

$$\therefore A_2 = \frac{1}{2} A A_1$$

$$\text{Put } A_1 = A A_0$$

$$\therefore A_2 = \frac{1}{2} A A A_0$$

$$A_2 = \frac{1}{2} A^2 A_0$$

On equating coefficients of t^2 we get,

$$3A_3 = A A_2$$

$$\therefore A_3 = \frac{1}{3} A A_2$$

$$\text{Put } A_2 = \frac{1}{2} A^2 A_0$$

$$\therefore A_3 = \frac{1}{3} A \times \frac{1}{2} A^2 A_0$$

$$A_3 = \frac{1}{3!} A^3 A_0$$

Similarly, on equating coefficient of t^i we get

$$A_i = \frac{1}{i!} A^i A_0$$

In the above analysis, the matrices A_1, A_2, A_3 , etc., are expressed in terms of A and A_1 . Hence replace the matrices $A_1, A_2, A_3, \dots, A_i$ in the assumed solution of $X(t)$ [i.e., Eqn (1.50)] by the equivalent terms of obtained above.

$$\begin{aligned} \therefore X(t) &= A_0 + A A_0 t + \frac{1}{2!} A^2 A_0 t^2 + \frac{1}{3!} A^3 A_0 t^3 + \dots + \frac{1}{i!} A^i A_0 t^i + \dots \\ &= \left[I + A t + \frac{1}{2!} A^2 t^2 + \frac{1}{3!} A^3 t^3 + \dots + \frac{1}{i!} A^i t^i + \dots \right] A_0 \end{aligned} \quad \dots 1.53$$

where I is the unit matrix.

$$\text{It is given that, when } t = 0, \quad X(t) = X(0) = X_0 \quad \dots 1.54$$

From Eqn (1.53) when $t = 0$, we get

$$X(t)|_{t=0} = X(0) = A_0 \quad \dots 1.55$$

From Equations (1.54) and (1.55) we get,

$$A_0 = X_0 \quad \dots 1.56$$

On substituting for A_0 from Eqn (1.56) in Eqn (1.53) we get,

$$X(t) = \left[I + A t + \frac{1}{2!} A^2 t^2 + \frac{1}{3!} A^3 t^3 + \dots + \frac{1}{i!} A^i t^i + \dots \right] X_0 \quad \dots 1.57$$

Each of the term inside the brackets is an $n \times n$ matrix. Because of the similarity of the entity inside the bracket with a scalar exponential of e^{at} , we call it a matrix exponential, which may be written as,

$$e^{At} = I + At + \frac{1}{2!} A^2 t^2 + \frac{1}{3!} A^3 t^3 + \dots + \frac{1}{i!} A^i t^i + \dots \quad \dots 1.58$$

Hence the solution of the state equation is

$$X(t) = e^{At} X_0 \quad \dots 1.59$$

The matrix e^{at} is called state transition matrix and denoted by $\phi(t)$. From the solution of the state equations it is observed that the initial state X_0 at $t = 0$, is driven to state $X(t)$ at time t by state transition matrix.

SOLUTION OF NON HOMOGENEOUS STATE EQUATIONS

(Solution of state equations with input or excitation)

The state equation of n^{th} order system is given by

$$\dot{X}(t) = A X(t) + B U(t) \quad ; \quad X(0) = X_0 \quad \dots 1.60$$

where X_0 is initial condition vector.

The state equation of Eqn (1.60) can be rearrangement as shown below.

$$\dot{X}(t) - A X(t) = B U(t) \quad \dots 1.61$$

Premultiply both sides of Eqn (1.61) by e^{-At}

$$\begin{aligned} e^{-At} [\dot{X}(t) - A X(t)] &= e^{-At} B U(t) \\ e^{-At} \dot{X}(t) + e^{-At} (-A) X(t) &= e^{-At} B U(t) \end{aligned} \quad \dots 1.62$$

Consider the differential of $e^{-At} X(t)$

$$\frac{d}{dt} (e^{-At} X(t)) = e^{-At} \dot{X}(t) + e^{-At} (-A) X(t) \quad \dots 1.63$$

On comparing equations (1.62) and (1.63) we can write,

$$\begin{aligned} \frac{d}{dt} (e^{-At} X(t)) &= e^{-At} B U(t) \\ d(e^{-At} X(t)) &= e^{-At} B U(t) dt \end{aligned} \quad \dots 1.64$$

On integrating the equation (1.64) between limits 0 to t we get,

$$e^{-At} X(t) = X_0 + \int_0^t e^{-A\tau} B U(\tau) d\tau \quad \dots 1.65$$

where X_0 = Initial condition vector = Integral constant

τ = Dummy variable substituted for t .

Premultiply both sides of Eqn (1.65) by e^{At} ,

$$\begin{aligned} e^{At} e^{-At} X(t) &= e^{At} X_0 + e^{At} \int_0^t e^{-A\tau} B U(\tau) d\tau \\ X(t) &= e^{At} X_0 + e^{At} \int_0^t e^{-A\tau} B U(\tau) d\tau \end{aligned} \quad \dots 1.66$$

The term e^{At} independent of the integral variable τ , and so e^{At} can be brought inside the integral function.

$$\begin{aligned} \therefore X(t) &= e^{At} X_0 + \int_0^t e^{At} \cdot e^{-A\tau} B U(\tau) d\tau \\ X(t) &= e^{At} X_0 + \int_0^t e^{A(t-\tau)} B U(\tau) d\tau \end{aligned} \quad \dots 1.67$$

The equation (1.67) is the solution of state equation, when the initial conditions are known at $t = 0$. If initial conditions are known at $t = t_0$ then the solution of state equations is given by Eqn (1.68).

$$X(t) = e^{A(t-t_0)} X(t_0) + \int_{t_0}^t e^{A(t-\tau)} B U(\tau) d\tau \quad \dots 1.68$$

The state transition matrix e^{At} is denoted by the symbol $\phi(t)$, i.e., $\phi(t) = e^{At}$

Hence, $e^{A(t-t_0)}$ can be expressed as, $e^{A(t-t_0)} = \phi(t-t_0)$...1.69

and, $e^{A(t-\tau)}$ can be expressed as, $e^{A(t-\tau)} = \phi(t-\tau)$...1.70

The equation (1.67) and (1.68) can also be expressed as

$$X(t) = \phi(t) X(0) + \int_0^t \phi(t-\tau) B U(\tau) d\tau \quad \text{if the initial conditions are known } t = 0 \quad \dots 1.71$$

$$X(t) = \phi(t-t_0) X(t_0) + \int_{t_0}^t \phi(t-\tau) B U(\tau) d\tau \quad \text{if the initial conditions are known } t = t_0 \quad \dots 1.72$$

PROPERTIES OF STATE TRANSITION MARIX

1. $\phi(0) = e^{A \times 0} = I$ (unit matrix)
2. $\phi(t) = e^{At} = (e^{-At})^{-1} = [\phi(-t)]^{-1}$ or $\phi^{-1}(t) = \phi(-t)$
3. $\phi(t_1 + t_2) = e^{A(t_1+t_2)} = (e^{At_1}) \cdot (e^{At_2}) = \phi(t_1) \phi(t_2) = \phi(t_2) \cdot \phi(t_1)$

COMPUTATION OF STATE TRANSITION MATRIX

- Method 1: Computation of e^{At} using matrix exponential.
- Method 2: Computation of e^{At} using laplace transform.
- Method 3: Computation of e^{At} by canonical transformation.
- Method 4: Computation of e^{At} using Sylvester's interpolation formula (or computation based on Cayley-Hamilton theorem).

The computation of state transition matrix using matrix exponential and laplace transform are presented in this section.

Computation of state transition matrix using matrix exponential

In this method, the e^{At} is computed using the matrix exponential of Eqn (1.58), which is also given below,

$$e^{At} = I + At + \frac{1}{2!} A^2 t^2 + \frac{1}{3!} A^3 t^3 + \dots + \frac{1}{l!} A^l t^l + \dots$$

where, e^{At} = State transition matrix of order $n \times n$

A = System matrix of order $n \times n$

I = Unit matrix of order $n \times n$.

The disadvantage in this method is that each term of e^{At} will be an infinite series and the convergence of the infinite series are obtained by trial and error.

Computation of State Transition Matrix by Laplace Transform Method

Consider the state equation without input vector, $\dot{X}(t) = A X(t)$...1.73

On taking laplace transform of equation (1.73) we get,

$$\begin{aligned} s X(s) - X(0) &= A X(s) \\ s X(s) - A X(s) &= X(0) \\ (sI - A) X(s) &= X(0), \end{aligned}$$

where I is a unit matrix.

Premultiply both sides by $(sI - A)^{-1}$

$$X(s) = (sI - A)^{-1} X(0)$$

On taking inverse laplace transform we get,

$$X(t) = \mathbf{L}^{-1} [(sI - A)^{-1} X(0)] \quad ; \quad X(t) = \mathbf{L}^{-1} [(sI - A)^{-1}] X(0) \quad \dots 1.74$$

On computing Eqn (1.74) with the solution of state equation, $X(t) = e^{At} X(0)$ we get

$$e^{At} = \mathbf{L}^{-1} [(sI - A)^{-1}] \quad \text{or} \quad \mathbf{L} [e^{At}] = (sI - A)^{-1} \quad \dots 1.75$$

We know that, $\mathbf{e}_{yt} = \phi(t)$, $\therefore \mathbf{F}[\mathbf{e}_{yt}] = \mathbf{F}[\phi(t)] = \phi(z)$... (1.76)

where, $\phi(s) = (sI-A)^{-1}$ and it is called resolvent matrix.

From the system matrix, A the resolvent matrix, $\phi(s)$ can be computed. By taking inverse laplace transform of resolvent matrix, the state transition matrix is computed, from which the solution of state equation is obtained.

The solution of state equation is given by

$$\begin{aligned} \mathbf{X}(t) &= e^{At} \mathbf{X}(0) \\ \therefore \mathbf{X}(t) &= \mathbf{L}^{-1}[\phi(s)] \mathbf{X}(0) \end{aligned} \quad \dots (1.77)$$

where, $\phi(s) = (sI-A)^{-1}$

Consider the state equation with forcing function (input or excitation)

$$\dot{\mathbf{X}} = \mathbf{A}\mathbf{X} + \mathbf{B}\mathbf{U} \quad \dots 1.78$$

On taking laplace transform of Eqn (1.78), we get

$$\begin{aligned} s \mathbf{X}(s) - \mathbf{X}(0) &= \mathbf{A} \mathbf{X}(s) + \mathbf{B} \mathbf{U}(s) \\ sI \mathbf{X}(s) - \mathbf{A} \mathbf{X}(s) &= \mathbf{X}(0) + \mathbf{B} \mathbf{U}(s), \\ (sI - \mathbf{A}) \mathbf{X}(s) &= \mathbf{X}(0) + \mathbf{B} \mathbf{U}(s) \quad \text{where I is the unit matrix.} \end{aligned} \quad \dots 1.79$$

Premultiply the Eqn (1.79) by $(sI-A)^{-1}$

$$\begin{aligned} \therefore \mathbf{X}(s) &= (sI-A)^{-1} \mathbf{X}(0) + (sI-A)^{-1} \mathbf{B} \mathbf{U}(s) \\ &= \phi(s) \mathbf{X}(0) + \phi(s) \mathbf{B} \mathbf{U}(s) \end{aligned} \quad \dots 1.80$$

On taking inverse Laplace transform of Eqn (1.80) we get,

$$\mathbf{X}(t) = \phi(t) \mathbf{X}(0) + \mathbf{L}^{-1}[\phi(s) \cdot \mathbf{B} \cdot \mathbf{U}(s)] \quad \dots 1.81$$

The equation (1.81) is the solution of state equation with forcing function.

EXAMPLE 1.13

Consider the matrix A, Compute e^{At} by two methods.

$$A = \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix}$$

SOLUTION

Method 1

$$e^{At} = \left[1 + At + \frac{1}{2!} A^2 t^2 + \frac{1}{3!} A^3 t^3 + \dots \right]$$

$$A = \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix}$$

$$A^2 = A \cdot A = \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix} \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix} = \begin{bmatrix} -2 & -3 \\ 6 & 7 \end{bmatrix}$$

$$A^3 = A^2 \cdot A = \begin{bmatrix} -2 & -3 \\ 6 & 7 \end{bmatrix} \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix} = \begin{bmatrix} 6 & 7 \\ -14 & -15 \end{bmatrix}$$

$$A^4 = A^3 \cdot A = \begin{bmatrix} 6 & 7 \\ -14 & -15 \end{bmatrix} \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix} = \begin{bmatrix} -14 & -15 \\ 30 & 31 \end{bmatrix}$$

$$e^{At} = I + At + \frac{1}{2!} A^2 t^2 + \frac{1}{3!} A^3 t^3 + \frac{1}{4!} A^4 t^4 + \dots$$

$$= \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} + \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix} t + \frac{1}{2} \begin{bmatrix} -2 & -3 \\ 6 & 7 \end{bmatrix} t^2 + \frac{1}{6} \begin{bmatrix} 6 & 7 \\ -14 & -15 \end{bmatrix} t^3 + \frac{1}{24} \begin{bmatrix} -14 & -15 \\ 30 & 31 \end{bmatrix} t^4 + \dots$$

$$= \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} + \begin{bmatrix} 0 & t \\ -2t & -3t \end{bmatrix} + \begin{bmatrix} -t^2 & -\frac{3}{2}t^2 \\ 3t^2 & \frac{7}{2}t^2 \end{bmatrix} + \begin{bmatrix} t^3 & \frac{7}{6}t^3 \\ -\frac{7}{3}t^3 & -\frac{5}{2}t^3 \end{bmatrix} + \begin{bmatrix} -\frac{7}{12}t^4 & -\frac{5}{8}t^4 \\ \frac{5}{4}t^4 & \frac{31}{24}t^4 \end{bmatrix} + \dots$$

$$= \begin{bmatrix} 1 - t^2 + t^3 - \frac{7}{12}t^4 + \dots & t - \frac{3}{2}t^2 + \frac{7}{6}t^3 - \frac{5}{8}t^4 + \dots \\ -2t + 3t^2 - \frac{7}{3}t^3 + \frac{5}{4}t^4 + \dots & 1 - 3t + \frac{7}{2}t^2 - \frac{5}{2}t^3 + \frac{31}{24}t^4 + \dots \end{bmatrix}$$

The each term in the matrix is an expansion of e^{at} . The convergence of series obtained by trial and error. Consider the expansion of e^{-1} and e^{-2t} .

$$e^{-t} = 1 - t + \frac{1}{2!}t^2 - \frac{1}{3!}t^3 + \frac{1}{4!}t^4 \dots = 1 - t + \frac{1}{2}t^2 - \frac{1}{6}t^3 + \frac{1}{24}t^4 \dots$$

$$e^{-2t} = 1 - 2t + \frac{1}{2!}2^2t^2 - \frac{1}{3!}2^3t^3 + \frac{1}{4!}2^4t^4 \dots = 1 - 2t + 2t^2 - \frac{4}{3}t^3 + \frac{2}{3}t^4 \dots$$

$$\begin{aligned} 2e^{-t} - e^{-2t} &= 2 - 2t + t^2 - \frac{1}{3}t^3 + \frac{1}{12}t^4 \dots - 1 + 2t - 2t^2 + \frac{4}{3}t^3 - \frac{2}{3}t^4 + \dots \\ &= 1 - t^2 + t^3 - \frac{7}{12}t^4 + \dots \end{aligned}$$

$$\begin{aligned} e^{-t} - e^{-2t} &= 1 - t + \frac{1}{2}t^2 - \frac{1}{6}t^3 + \frac{1}{24}t^4 \dots - 1 + 2t - 2t^2 + \frac{4}{3}t^3 - \frac{2}{3}t^4 + \dots \\ &= t - \frac{3}{2}t^2 + \frac{7}{6}t^3 - \frac{5}{8}t^4 + \dots \end{aligned}$$

$$\begin{aligned} -2e^{-t} + 2e^{-2t} &= -2 + 2t - t^2 + \frac{1}{3}t^3 - \frac{1}{12}t^4 + \dots + 2 - 4t + 4t^2 - \frac{8}{3}t^3 + \frac{4}{3}t^4 \dots \\ &= -2t + 3t^2 - \frac{7}{3}t^3 + \frac{5}{4}t^4 \dots \end{aligned}$$

$$\begin{aligned} -e^{-t} + 2e^{-2t} &= -1 + t - \frac{1}{2}t^2 + \frac{1}{6}t^3 - \frac{1}{24}t^4 + \dots + 2 - 4t + 4t^2 - \frac{8}{3}t^3 + \frac{4}{3}t^4 \dots \\ &= 1 - 3t + \frac{7}{2}t^2 - \frac{5}{2}t^3 + \frac{31}{24}t^4 + \dots \end{aligned}$$

$$\therefore e^{At} = \begin{bmatrix} 2e^{-t} - e^{-2t} & e^{-t} - e^{-2t} \\ -2e^{-t} + 2e^{-2t} & -e^{-t} + 2e^{-2t} \end{bmatrix}$$

Method 2

$$A = \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix}$$

$$e^{At}c = \phi(t) = L^{-1}[(sI - A)^{-1}]$$

$$sI - A = s \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} - \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix} = \begin{bmatrix} s & 0 \\ 0 & s \end{bmatrix} - \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix} = \begin{bmatrix} s & -1 \\ 2 & s+3 \end{bmatrix}$$

Let, $\Delta = |sI - A|$ = determinant of $(sI - A)$

$$\therefore \Delta = |sI - A| = \begin{vmatrix} s & -1 \\ 2 & s+3 \end{vmatrix} = s(s+3) + 2 = s^2 + 3s + 2 = (s+2)(s+1)$$

$$\phi(s) = [sI - A]^{-1} = \frac{[\text{Cofactor of } (sI - A)]^T}{\text{determinant of } (sI - A)} = \frac{[\text{Cofactor of } (sI - A)]^T}{\Delta}$$

$$\therefore \phi(s) = \frac{1}{(s+1)(s+2)} \begin{bmatrix} s+3 & 1 \\ -2 & s \end{bmatrix}$$

$$\phi(s) = \begin{bmatrix} \frac{s+3}{(s+1)(s+2)} & \frac{1}{(s+1)(s+2)} \\ \frac{-2}{(s+1)(s+2)} & \frac{s}{(s+1)(s+2)} \end{bmatrix}$$

By partial fraction expansion, $\phi(s)$ can be written as,

$$\phi(s) = \begin{bmatrix} \frac{A_1}{s+1} + \frac{B_1}{s+2} & \frac{A_2}{s+1} + \frac{B_2}{s+2} \\ \frac{A_3}{s+1} + \frac{B_3}{s+2} & \frac{A_4}{s+1} + \frac{B_4}{s+2} \end{bmatrix}$$

$\frac{s+3}{(s+1)(s+2)} = \frac{A_1}{s+1} + \frac{B_1}{s+2}$ $A_1 = \left. \frac{s+3}{s+2} \right _{s=-1} = 2$ $B_1 = \left. \frac{s+3}{s+1} \right _{s=-2} = -1$	$\frac{1}{(s+1)(s+2)} = \frac{A_2}{s+1} + \frac{B_2}{s+2}$ $A_2 = \left. \frac{1}{s+2} \right _{s=-1} = 1$ $B_2 = \left. \frac{1}{s+1} \right _{s=-2} = -1$
$\frac{-2}{(s+1)(s+2)} = \frac{A_3}{s+1} + \frac{B_3}{s+2}$ $A_3 = \left. \frac{-2}{s+2} \right _{s=-1} = -2$ $B_3 = \left. \frac{-2}{s+1} \right _{s=-2} = 2$	$\frac{s}{(s+1)(s+2)} = \frac{A_4}{s+1} + \frac{B_4}{s+2}$ $A_4 = \left. \frac{s}{s+2} \right _{s=-1} = -1$ $B_4 = \left. \frac{s}{s+1} \right _{s=-2} = 2$

$$\therefore \phi(s) = \begin{bmatrix} \frac{2}{s+1} - \frac{1}{s+2} & \frac{1}{s+1} - \frac{1}{s+2} \\ \frac{-2}{s+1} + \frac{2}{s+2} & \frac{-1}{s+1} + \frac{2}{s+2} \end{bmatrix}$$

On taking inverse Laplace transform $\phi(s)$ we get $\phi(t)$, where $\phi(t) = e^{At}$

$$\therefore e^{At} = \phi(t) = \begin{bmatrix} 2e^{-t} - e^{-2t} & e^{-t} - e^{-2t} \\ -2e^{-t} + 2e^{-2t} & -e^{-t} + 2e^{-2t} \end{bmatrix}$$

It is observed that the results of both the methods are same.

EXAMPLE 1.14

Given that $A_1 = \begin{bmatrix} \sigma & 0 \\ 0 & \sigma \end{bmatrix}$; $A_2 = \begin{bmatrix} 0 & \omega \\ -\omega & 0 \end{bmatrix}$; $A = \begin{bmatrix} \sigma & \omega \\ -\omega & \sigma \end{bmatrix}$ compute e^{At} .

SOLUTION

Here $A = A_1 + A_2$.

$$\therefore e^{At} = e^{(A_1 + A_2)t} = e^{A_1 t} \cdot e^{A_2 t}$$

$$sI - A_1 = s \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} - \begin{bmatrix} \sigma & 0 \\ 0 & \sigma \end{bmatrix} = \begin{bmatrix} s & 0 \\ 0 & s \end{bmatrix} - \begin{bmatrix} \sigma & 0 \\ 0 & \sigma \end{bmatrix} = \begin{bmatrix} s - \sigma & 0 \\ 0 & s - \sigma \end{bmatrix}$$

$$\Delta_1 = |sI - A_1| = \begin{vmatrix} s - \sigma & 0 \\ 0 & s - \sigma \end{vmatrix} = (s - \sigma)^2$$

$$[sI - A_1]^{-1} = \frac{1}{\Delta_1} \begin{bmatrix} s - \sigma & 0 \\ 0 & s - \sigma \end{bmatrix} = \frac{1}{(s - \sigma)^2} \begin{bmatrix} s - \sigma & 0 \\ 0 & s - \sigma \end{bmatrix} = \begin{bmatrix} \frac{1}{s - \sigma} & 0 \\ 0 & \frac{1}{s - \sigma} \end{bmatrix}$$

$$e^{A_1 t} = \mathbf{L}^{-1}[(sI - A_1)^{-1}] = \begin{bmatrix} e^{-\sigma t} & 0 \\ 0 & e^{-\sigma t} \end{bmatrix}$$

$$sI - A_2 = s \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} - \begin{bmatrix} 0 & \omega \\ -\omega & 0 \end{bmatrix} = \begin{bmatrix} s & 0 \\ 0 & s \end{bmatrix} - \begin{bmatrix} 0 & \omega \\ -\omega & 0 \end{bmatrix} = \begin{bmatrix} s & -\omega \\ \omega & s \end{bmatrix}$$

$$\Delta_2 = |sI - A_2| = \begin{vmatrix} s & -\omega \\ \omega & s \end{vmatrix} = s^2 + \omega^2$$

$$[sI - A_2]^{-1} = \frac{1}{\Delta_2} \begin{bmatrix} s & \omega \\ -\omega & s \end{bmatrix} = \frac{1}{s^2 + \omega^2} \begin{bmatrix} s & \omega \\ -\omega & s \end{bmatrix} = \begin{bmatrix} \frac{s}{s^2 + \omega^2} & \frac{\omega}{s^2 + \omega^2} \\ \frac{-\omega}{s^2 + \omega^2} & \frac{s}{s^2 + \omega^2} \end{bmatrix}$$

$$e^{A_2 t} = \mathbf{L}^{-1}[(sI - A_2)^{-1}] = \begin{bmatrix} \cos \omega t & \sin \omega t \\ -\sin \omega t & \cos \omega t \end{bmatrix}$$

$$e^{At} = e^{A_1 t} \cdot e^{A_2 t} = \begin{bmatrix} e^{-\sigma t} & 0 \\ 0 & e^{-\sigma t} \end{bmatrix} \begin{bmatrix} \cos \omega t & \sin \omega t \\ -\sin \omega t & \cos \omega t \end{bmatrix}$$

$$\therefore e^{At} = \begin{bmatrix} e^{-\sigma t} \cos \omega t & e^{-\sigma t} \sin \omega t \\ -e^{-\sigma t} \sin \omega t & e^{-\sigma t} \cos \omega t \end{bmatrix}$$

EXAMPLE 1.15

For a system represented by state equation $\dot{X}(t) = A X(t)$

$$\text{The response is } X(t) = \begin{bmatrix} e^{-2t} \\ -2e^{-2t} \end{bmatrix} \text{ when } X(0) = \begin{bmatrix} 1 \\ -2 \end{bmatrix}$$

$$\text{and } X(t) = \begin{bmatrix} e^{-t} \\ -e^{-t} \end{bmatrix} \text{ when } X(0) = \begin{bmatrix} 1 \\ -1 \end{bmatrix}$$

Determine the system matrix A and the state transition matrix

SOLUTION

$$\text{The Solution of State equation is, } X(t) = e^{At} X(0) \quad \dots 1.15.1$$

Premultiply the Eqn (1.15.1) by e^{-At}

$$\begin{aligned} e^{-At} X(t) &= e^{-At} e^{At} X(0) \\ \therefore e^{-At} X(t) &= X(0) \end{aligned} \quad \dots 1.15.2$$

$$\text{One of the response is } X(t) = \begin{bmatrix} e^{-2t} \\ -2e^{-2t} \end{bmatrix} \text{ and } X(0) = \begin{bmatrix} 1 \\ -2 \end{bmatrix}.$$

On substituting the response in equation (1.15.2) we get,

$$e^{-At} \begin{bmatrix} e^{-2t} \\ -2e^{-2t} \end{bmatrix} = \begin{bmatrix} 1 \\ -2 \end{bmatrix} \quad \dots 1.15.3$$

$$\text{Let } e^{-At} = \begin{bmatrix} c_{11} & c_{12} \\ c_{21} & c_{22} \end{bmatrix} \quad \dots 1.15.4$$

From equation (1.15.3) and (1.15.4) we can write

$$\begin{bmatrix} c_{11} & c_{12} \\ c_{21} & c_{22} \end{bmatrix} \begin{bmatrix} e^{-2t} \\ -2e^{-2t} \end{bmatrix} = \begin{bmatrix} 1 \\ -2 \end{bmatrix} \quad \dots 1.15.5$$

On multiplying the equation (1.15.5) we get the following two equations.

$$c_{11} e^{-2t} - 2c_{12} e^{-2t} = 1 \quad \dots 1.15.6$$

$$c_{21} e^{-2t} - 2c_{22} e^{-2t} = -2 \quad \dots 1.15.7$$

The second solution of state equation is $X(t) = \begin{bmatrix} e^{-t} \\ -e^{-t} \end{bmatrix}$ and $X(0) = \begin{bmatrix} 1 \\ -1 \end{bmatrix}$

On substituting this solution in equation (1.15.2) we get,

$$e^{-At} \begin{bmatrix} e^{-t} \\ -e^{-t} \end{bmatrix} = \begin{bmatrix} 1 \\ -1 \end{bmatrix} \quad \dots 1.15.8$$

From Eqn (1.15.4) and (1.15.8) we can write,

$$\begin{bmatrix} c_{11} & c_{12} \\ c_{21} & c_{22} \end{bmatrix} \begin{bmatrix} e^{-t} \\ -e^{-t} \end{bmatrix} = \begin{bmatrix} 1 \\ -1 \end{bmatrix} \quad \dots 1.15.9$$

On multiplying the equation (1.15.9) we get the following two equations,

$$c_{11} e^{-t} - c_{12} e^{-t} = 1 \quad \dots 1.15.10$$

$$c_{21} e^{-t} - c_{22} e^{-t} = -1 \quad \dots 1.15.11$$

$$\text{equation (1.15.10)} \quad \times e^{-t} \Rightarrow c_{11} e^{-2t} - c_{12} e^{-2t} = e^{-t}$$

$$\text{equation (1.15.11)} \quad \times 1 \Rightarrow \begin{array}{ccc} c_{11} e^{-2t} & -2c_{12} e^{-2t} & = 1 \\ (-) & (+) & (-) \end{array}$$

$$\text{On subtracting} \quad \underline{c_{12} e^{-2t} = e^{-t} - 1}$$

From Eqn (1.15.12) we get

$$e_{12} = \frac{e^{-t} - 1}{e^{-2t}} = \frac{e^{-t}}{e^{-2t}} - \frac{1}{e^{-2t}} = e^t - e^{2t} \quad \dots 1.15.13$$

$$\text{From equation (1.15.6), } e_{11} = \frac{1 + 2e_{12}e^{-2t}}{e^{-2t}}$$

$$\begin{aligned} \text{Put } e_{12} &= e^t - e^{2t}, \quad \therefore e_{11} = \frac{1 + 2(e^t - e^{2t})e^{-2t}}{e^{-2t}} = \frac{1 + 2e^{-t} - 2}{e^{-2t}} = \frac{2e^{-t} - 1}{e^{-2t}} \\ &= \frac{2e^{-t}}{e^{-2t}} - \frac{1}{e^{-2t}} = 2e^t - e^{2t} \end{aligned}$$

$$\text{equation (4.15.11)} \times e^{-t} \Rightarrow e_{21} e^{-2t} - e_{22} e^{-2t} = -e^{-t}$$

$$\text{equation (4.15.7)} \times 1 \Rightarrow \underset{(-)}{e_{21} e^{-2t}} - \underset{(+)}{2e_{22} e^{-2t}} = \underset{(+)}{-2}$$

On subtracting

$$e_{22} e^{-2t} = 2 - e^{-t}$$

...1.15.14

From Eqn (1.15.14) we get,

$$e_{22} = \frac{-e^{-t} + 2}{e^{-2t}} = \frac{-e^{-t}}{e^{-2t}} + \frac{2}{e^{-2t}} = -e^t + 2e^{2t}$$

$$\text{From Equation (1.15.11), } e_{21} = \frac{-1 + e_{22} e^{-t}}{e^{-t}}$$

$$\begin{aligned} \text{Put } e_{22} &= -e^t + 2e^{2t}, \quad \therefore e_{21} = \frac{-1 + (-e^t + 2e^{2t})e^{-t}}{e^{-t}} \\ e_{21} &= \frac{-1 - 1 + 2e^t}{e^{-t}} = \frac{-2}{e^{-t}} + \frac{2e^t}{e^{-t}} = -2e^t + 2e^{2t} \end{aligned}$$

$$\begin{aligned} \therefore e^{-At} &= \begin{bmatrix} e_{11} & e_{12} \\ e_{21} & e_{22} \end{bmatrix} = \begin{bmatrix} 2e^t - e^{2t} & e^t - e^{2t} \\ -2e^t + 2e^{2t} & -e^t + 2e^{2t} \end{bmatrix} \\ \therefore e^{At} &= \begin{bmatrix} 2e^{-t} - e^{-2t} & e^{-t} - e^{-2t} \\ -2e^{-t} + 2e^{-2t} & -e^{-t} + 2e^{-2t} \end{bmatrix} \end{aligned}$$

e^{At} is the state transition matrix.

We know that, $\mathbf{L}[e^{At}] = \phi(s)$

Where $\phi(s) = (s\mathbf{I} - \mathbf{A})^{-1}$; $\therefore \phi(s)^{-1} = (s\mathbf{I} - \mathbf{A})$ or $\mathbf{A} = s\mathbf{I} - \phi(s)^{-1}$

$$\phi(s) = \mathbf{L}[e^{At}] = \begin{bmatrix} \frac{2}{s+1} - \frac{1}{s+2} & \frac{1}{s+1} - \frac{1}{s+2} \\ \frac{-2}{s+1} + \frac{2}{s+2} & \frac{-1}{s+1} + \frac{2}{s+2} \end{bmatrix}$$

$$\begin{aligned}
&= \begin{bmatrix} \frac{2(s+2)-(s+1)}{(s+1)(s+2)} & \frac{(s+2)-(s+1)}{(s+1)(s+2)} \\ \frac{-2(s+2)+2(s+1)}{(s+1)(s+2)} & \frac{-(s+2)+2(s+1)}{(s+1)(s+2)} \end{bmatrix} \\
&= \begin{bmatrix} \frac{s+3}{(s+1)(s+2)} & \frac{1}{(s+1)(s+2)} \\ \frac{-2}{(s+1)(s+2)} & \frac{s}{(s+1)(s+2)} \end{bmatrix}
\end{aligned}$$

$$\begin{aligned}
\text{Determinant of } \phi(s) &= \frac{s(s+3)+2}{(s+1)^2(s+2)^2} = \frac{s^2+3s+2}{(s+1)^2(s+2)^2} \\
&= \frac{(s+1)(s+2)}{(s+1)^2(s+2)^2} = \frac{1}{(s+1)(s+2)}
\end{aligned}$$

$$\phi(s)^{-1} = \frac{1}{(s+1)(s+2)} \begin{bmatrix} \frac{s}{(s+1)(s+2)} & \frac{-1}{(s+1)(s+2)} \\ \frac{2}{(s+1)(s+2)} & \frac{s+3}{(s+1)(s+2)} \end{bmatrix} = \begin{bmatrix} s & -1 \\ 2 & s+3 \end{bmatrix}$$

$$\Lambda = sI - \phi(s)^{-1} = s \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} - \begin{bmatrix} s & -1 \\ 2 & s+3 \end{bmatrix} = \begin{bmatrix} s & 0 \\ 0 & s \end{bmatrix} - \begin{bmatrix} s & -1 \\ 2 & s+3 \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix}$$

RESULT

$$\Lambda = \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix} ; \quad e^{\Lambda t} = \begin{bmatrix} 2e^{-t} - e^{-2t} & e^{-t} - e^{-2t} \\ -2e^{-t} + 2e^{-2t} & -e^{-t} + 2e^{-2t} \end{bmatrix}$$

EXAMPLE 1.16

A linear time-invariant system is characterized by homogenous state equation.

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} 1 & 0 \\ 1 & 1 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix}$$

Compute the solution of the homogenous equation, assuming the initial state vector.

$$X_0 = \begin{bmatrix} 1 \\ 0 \end{bmatrix}$$

SOLUTION

$$\text{Here } A = \begin{bmatrix} 1 & 0 \\ 1 & 1 \end{bmatrix} ; \quad sI - A = s \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} - \begin{bmatrix} 1 & 0 \\ 1 & 1 \end{bmatrix} = \begin{bmatrix} s & 0 \\ 0 & s \end{bmatrix} - \begin{bmatrix} 1 & 0 \\ 1 & 1 \end{bmatrix} = \begin{bmatrix} s-1 & 0 \\ -1 & s-1 \end{bmatrix}$$

$$|sI - A| = \begin{vmatrix} s-1 & 0 \\ -1 & s-1 \end{vmatrix} = (s-1)^2 - 0 = (s-1)^2$$

$$(sI - A)^{-1} = \frac{1}{(s-1)^2} \begin{bmatrix} s-1 & 0 \\ 1 & s-1 \end{bmatrix} = \begin{bmatrix} \frac{1}{s-1} & 0 \\ \frac{1}{(s-1)^2} & \frac{1}{s-1} \end{bmatrix}$$

$$e^{At} = \phi(t) = \mathbf{L}^{-1}[(\phi(s))] = \mathbf{L}^{-1}[(sI - A)^{-1}] = \begin{bmatrix} e^t & 0 \\ te^t & e^t \end{bmatrix}$$

$$\text{The solution of the state equation is, } \mathbf{X}(t) = e^{At} \mathbf{X}_0 = \begin{bmatrix} e^t & 0 \\ te^t & e^t \end{bmatrix} \begin{bmatrix} 1 \\ 0 \end{bmatrix} = \begin{bmatrix} e^t \\ te^t \end{bmatrix}$$

1.9 STATE SPACE REPRESENTATION OF DISCRETE TIME SYSTEMS

The state variable analysis techniques of continuous time systems can be extended to the discrete-time system. The discrete form of state space representation is quite analogue to the continuous form.

In the state variable formulation of a discrete time system, in general, a system consists of m-inputs, p-outputs and n-state variables. The state space representation of discrete-time system may be visualized as shown in Figure 1.6.

Let, State variables = $x_1(k), x_2(k), x_3(k), \dots, x_n(k)$
 Input variables = $u_1(k), u_2(k), u_3(k), \dots, u_m(k)$
 Output variables = $y_1(k), y_2(k), y_3(k), \dots, y_p(k)$

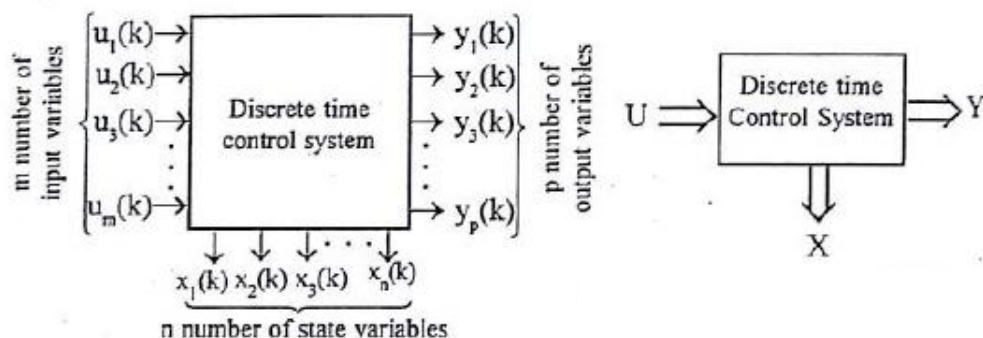


Figure 1.6 State space representation of discrete time system

The different variables may be represented by the vectors (column matrix) as shown below.

$$\begin{array}{l} \text{Input} \\ \text{vector} \end{array} U(k) = \begin{bmatrix} u_1(k) \\ u_2(k) \\ \vdots \\ u_m(k) \end{bmatrix} ; \quad \begin{array}{l} \text{Output} \\ \text{vector} \end{array} Y(k) = \begin{bmatrix} y_1(k) \\ y_2(k) \\ \vdots \\ y_p(k) \end{bmatrix} ; \quad \begin{array}{l} \text{State variable} \\ \text{vector} \end{array} X(k) = \begin{bmatrix} x_1(k) \\ x_2(k) \\ \vdots \\ x_n(k) \end{bmatrix}$$

Note: The simplified notation $x(k)$, $y(k)$ and $u(k)$ are used to denote $x(kT)$, $y(kT)$ and $u(kT)$ respectively. Also for convenience the variables are denoted, x_1, x_2, x_3, \dots ; y_1, y_2, y_3 , and u_1, u_2, u_3, \dots .

The state equation of a discrete time system is a set of n -numbers of first order difference equations.

$$\begin{aligned} x_1(k+1) &= f_1(x_1, x_2, \dots, x_n ; u_1, u_2, \dots, u_m) \\ x_2(k+1) &= f_2(x_1, x_2, \dots, x_n ; u_1, u_2, \dots, u_m) \\ &\vdots \\ x_n(k+1) &= f_n(x_1, x_2, \dots, x_n ; u_1, u_2, \dots, u_m) \end{aligned}$$



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DEPARTMENT OF ELECTRICAL AND ELECTRONICS

UNIT – II – Advanced Control Systems – SEEA1602

ANALYSIS AND DESIGN OF CONTROL SYSTEM IN STATE SPACE

2.1 DEFINITIONS OF INVOLVING MATRICES

Matrix: A matrix is an ordered array of elements which may be real numbers, complex numbers, functions or operators. In general the array consists of m rows and n columns. When $m = n$, the matrix is called square matrix. When $n = 1$, the matrix is called column matrix or vector. When $m = 1$, the matrix is called row matrix or vector.

Diagonal matrix: It is a square matrix whose elements other than main diagonal are all zeros.

Unit matrix: It is a diagonal matrix whose diagonal elements are all equal to unity. The elements other than diagonal are all zeros. It is denoted by I .

Transpose: If the rows and columns of an $m \times n$ matrix A are interchanged, then the resulting $n \times m$ matrix is called the transpose of A . The transpose of A is denoted by A^T .

Determinant: A determinant consisting of the elements of a square matrix (in the order given in the matrix) is called the determinant of the matrix.

Symmetric matrix: A square matrix is symmetric if it is equal to its transpose, i.e., $A^T = A$. If A is a square matrix, then $A + A^T$ is a symmetric matrix.

Skew-symmetric matrix: A square matrix is skew-symmetric if it is equal to the negative of its transpose, i.e., $A^T = -A$. If A is a square matrix then $A - A^T$ is a skew symmetric matrix.

Orthogonal Matrix: A matrix A is called an orthogonal matrix if it is real and satisfies the relationship $A^T A = AA^T = I$.

Minor: If the i^{th} row and j^{th} column of determinant A are deleted, the remaining $(n-1)$ rows and columns form a determinant M_{ij} . This determinant is called the minor of the element a_{ij} .

Cofactor: The cofactor C_{ij} of element a_{ij} of the matrix A is defined as $C_{ij} = (-1)^{(i+j)} M_{ij}$, where M_{ij} is the minor of a_{ij} .

Adjoint matrix: The adjoint matrix of a square matrix A is found by replacing each element a_{ij} of matrix A by its cofactor C_{ij} and then transposing.

Singular matrix: A square matrix is called singular if its associated determinant is zero. If the determinant of the matrix is non zero then the matrix is non singular.

Rank of matrix: A matrix A is said to have a rank r if there exists an $r \times r$ submatrix of A which is non singular and all other $q \times q$ submatrices are singular, where $q \geq (r+1)$.

Conjugate matrix: The conjugate of a matrix A is the matrix in which each element is the complex conjugate of the corresponding element of A . The conjugate of A is denoted by A^* .

Real matrix: If all the elements of a matrix are real then the matrix is called real matrix. A real matrix is equal to its conjugate.

2.2 EIGENVALUES AND EIGENVECTORS

A nonzero column vector X is an eigenvector of a square matrix A , if there exists a scalar λ such that $AX = \lambda X$, then λ is an eigenvalue (or characteristic value) of A . Eigenvalue may be zero but the corresponding vector may not be a zero vector.

The characteristic equation of $n \times n$ matrix A is the n^{th} degree polynomial of equation. $|\lambda I - A| = 0$, where I is the unit matrix. Solving the characteristic equation for λ gives the eigenvalues of A . The eigenvalues may be real, complex or multiples of each other.

Once an eigenvalue is determined it may be substituted into $AX = \lambda X$ and then that equation may be solved for the corresponding eigenvector.

PROPERTIES OF EIGENVALUES AND EIGENVECTORS

1. The sum of the eigenvalues of a matrix is equal to its trace, which is the sum of the elements on its main diagonal.
2. Eigenvectors corresponding to different eigenvalues are linearly independent.
3. A matrix is singular if and only if it has a zero eigenvalue.
4. If X is an eigenvector of A corresponding to the eigenvalue of λ and A is invertible, then X is an eigenvector of A^{-1} corresponding to its eigenvalue $1/\lambda$.
5. If X is an eigenvector of a matrix then KX is also an eigenvector for any nonzero constant K . Here both X and KX correspond to the same eigenvalue.
6. A matrix and its transpose have the same eigenvalues.
7. The eigenvalues of an upper or lower triangular matrix are the elements on its main diagonal.
8. The product of the eigenvalues (counting multiplicities) of the matrix equals the determinant of the matrix.
9. If X is an eigenvector of A corresponding to eigenvalue of λ , then X is an eigenvector of $A - CI$ corresponding to the eigenvalue $\lambda - C$ for any scalar C .

DETERMINATION OF EIGENVECTORS

Case 1: Distinct eigenvalues

If the eigenvalues of A are all distinct, then we have only one independent eigenvector corresponding to any particular eigenvalue λ_i . The eigenvector corresponding to λ_i may be obtained by taking cofactors of matrix $(\lambda_i I - A)$ along any row.

Let, m_i = Eigenvector corresponding to λ_i

Now the eigenvector m_i is given by

$$m_i = \begin{bmatrix} C_{k1} \\ C_{k2} \\ \vdots \\ C_{kn} \end{bmatrix} ; k = 1 \text{ or } 2 \text{ or } \dots n \quad \dots 2.1$$

where $C_{k1}, C_{k2}, \dots, C_{kn}$ are cofactors of matrix $(\lambda_i I - A)$ along k^{th} row.

Case ii: Multiple eigenvalues

In this case the eigenvectors corresponding to the distinct eigenvalues are evaluated as mentioned in case (i).

If the matrix has repeated eigenvalues with multiplicity “q”, then there exists only one independent eigenvector corresponding to that repeated eigenvalue. If λ_i is a repeated eigenvalue, then the independent vector corresponding to λ_i can be evaluated by taking the cofactor of matrix $(\lambda_i I - A)$ along any row as mentioned in case (1). The remaining (q-1) eigenvectors can be obtained as shown in Eqn (2.2).

Let, $m_p = p^{\text{th}}$ eigenvector corresponding to repeated eigenvalue λ_i .

$$m_p = \begin{bmatrix} \frac{1}{p!} \frac{d^p}{d\lambda_i^p} C_{k1} \\ \frac{1}{p!} \frac{d^p}{d\lambda_i^p} C_{k2} \\ \vdots \\ \frac{1}{p!} \frac{d^p}{d\lambda_i^p} C_{kn} \end{bmatrix} ; p = 1, 2, 3, \dots (q-1) \quad \dots 2.2$$

where $C_{k1}, C_{k2}, C_{k3}, \dots, C_{kn}$ are cofactors of matrix $(\lambda_i I - A)$ along k^{th} row

2.3 SIMILARITY TRANSFORMATION

The square matrices A and B are said to be similar if a non singular matrix P exists such that

$$P^{-1} A P = B \quad \dots 2.3$$

The process of transformation is called similarity transformation and it is a linear transformation. The matrix P is called transformation matrix. Also the matrix, A can be obtained from B by a similarity transformation with a transformation matrix P^{-1} ,

$$\text{i.e., } A = P B P^{-1} \quad \dots 2.4$$

The similarity transformation can be used for diagonalization of a square matrix. If an $n \times n$ matrix has n linearly independent eigenvectors (i.e., with distinct eigenvalues) then it can

be diagonalized by a similarity transformation. If a matrix has multiple eigenvalues then it will not have a complete set of n linearly independent eigenvectors and so it cannot be diagonalized. However such a matrix can be transformed into a Jordan matrix (Jordan canonical form).

The transformation matrix for diagonalization or converting to Jordan form can be obtained from eigenvectors. For a system with n state variables we can find numbers of eigenvectors $m_1, m_2, m_3, \dots, m_n$. The eigenvectors are column vectors of order $(n \times 1)$. The transformation matrix is obtained by arranging the eigenvectors columnwise as shown in Eqn (2.5). This transformation matrix is also called Modal matrix and denoted by M .

$$\text{Modal matrix, } M = [m_1 \ m_2 \ m_3 \ \dots \ M_n] \quad \dots 2.5$$

The similarity transformation will not alter certain properties of the matrix. A property of a matrix is said to be invariant if it is possessed by all similar matrices. The determinant, characteristic equation and trace of a matrix are invariant under a similarity transformation. Since the characteristic equation is invariant the eigenvalues are also invariant under a linear or similarity transformation.

PROOF FOR INVARIANCE OF DETERMINANT

Let A and B are similar matrices and P be the transformation matrix which transforms A to B by a similarity transformation, $P^{-1} A P = B$.

$$\therefore B = P^{-1} A P \quad \dots 2.6$$

On taking determinant of Eqn (2.6) we get,

$$|B| = |P^{-1} A P| \quad \dots 2.7$$

Since the determinant of a product of two or more square matrices is equal to the product of their individual determinants, the Eqn (2.7) can be written as,

$$\begin{aligned} |B| &= |P^{-1}| |A| |P| = |A| |P^{-1}| |P| \\ &= |A| |P^{-1} P| = |A| |I| & (\because P^{-1} P = I) \\ &= |A| & (\because |I| = 1) \end{aligned}$$

From the above analysis it is evident that the determinant of a matrix is invariant under a similarity transformation.

PROOF FOR INVARIANCE OF CHARACTERISTIC EQUATION AND EIGENVALUES

Let A and B are similar matrices and P be the transformation matrix which transforms A to B by a similarity transformation, $P^{-1} A P = B$.

The characteristic equation of matrix B is given by

$$|\lambda I - B| = 0 \quad \dots 2.8$$

On substituting $B = P^{-1} A P$ in Eqn (2.8) we get,

$$\begin{aligned}
|\lambda I - B| &= |\lambda I - P^{-1}AP| \\
&= |\lambda P^{-1}P - P^{-1}AP| \quad (\because P^{-1}P = I) \\
&= |P^{-1}(\lambda I - A)P|
\end{aligned}
\tag{2.9}$$

Since the determinant of a product is the product of the determinant, the Eqn (2.9) can be written as,

$$\begin{aligned}
|\lambda I - B| &= |P^{-1}| |\lambda I - A| |P| = |\lambda I - A| |P^{-1}| |P| \\
&= |\lambda I - A| |P^{-1}P| \\
&= |\lambda I - A| |I| \quad (\because P^{-1}P = I) \\
&= |\lambda I - A| \quad (\because |I| = 1)
\end{aligned}$$

From the above analysis it is clear that the characteristic equations of A and B are identical. Since the characteristic equations are identical, the eigenvalues of A and B are identical. Hence the eigenvalues are invariant under a similarity (linearity) transformation.

PROOF FOR INVARIANCE OF TRACE OF A MATRIX

Let A and B are similar matrices and P be the transformation matrix which transforms A to B by a similarity transformation, $P^{-1}AP = B$.

$$\therefore \text{tr } B = \text{tr } P^{-1}AP \tag{2.10}$$

For an $n \times m$ matrix C and $m \times n$ matrix D, regardless of whether $CD = DC$ or $CD \neq DC$, we have,

$$\text{tr } (CD) = \text{tr } (DC) \tag{2.11}$$

Using the property of Eqn (2.11), the Eqn (2.10) can be written as,

$$\begin{aligned}
\text{tr } B &= \text{tr } AP P^{-1} \\
&= \text{tr } AI = \text{tr } A \quad (\because PP^{-1} = I \text{ and } AI = A)
\end{aligned}$$

From the above analysis it is clear that the trace of a matrix is invariant under a similarity transformation.

2.4 CAYLEY – HAMILTON THEOREM

The Cayley – Hamilton theorem states that every square matrix satisfies its own characteristic equation.

Consider an $n \times n$ matrix A and its characteristic equation [Eqn (2.12)].

$$|\lambda I - A| = \lambda^n + a_1 \lambda^{n-1} + \dots + a_{n-1} \lambda + a_n = 0 \tag{2.12}$$

By Cayley-Hamilton theorem, the matrix A has to satisfy its characteristic equation, hence Eqn (2.12) can be written as,

$$A^n + a_1 A^{n-1} + \dots + a_{n-1} A + a_n I = 0 \tag{2.13}$$

PROOF OF CAY-LEY HAMILTON THEOREM

Let A be a square matrix. The characteristic equation of A is given by

$$|\lambda I - A| = \lambda^n + a_1 \lambda^{n-1} + a_2 \lambda^{n-2} + \dots + a_{n-2} \lambda^2 + a_{n-1} \lambda + a_n = 0 \quad \dots 2.14$$

We have to prove that A satisfies the characteristic equation,

$$\text{i.e., } A^n + a_1 A^{n-1} + a_2 A^{n-2} + \dots + a_{n-2} A^2 + a_{n-1} A + a_n I = 0 \quad \dots 2.15$$

where I is the unit matrix of order (n x n).

Consider the matrix $(\lambda I - A)$. Let the matrix B be adjoint of $(\lambda I - A)$.

$$\therefore B = \text{adj}(\lambda I - A) \quad \dots 2.16$$

The elements of $\text{adj}(\lambda I - A)$ are the cofactors of the elements of $(\lambda I - A)$. Therefore each element of B will be a polynomial in λ of degree (n-1) or less. We know that every matrix whose elements are ordinary polynomials can be written as matrix polynomial. Hence the matrix B can be written as a matrix polynomial as shown in Eqn (2.17).

$$\therefore B = B_1 \lambda^{n-1} + B_2 \lambda^{n-2} + B_3 \lambda^{n-3} + \dots + B_{n-2} \lambda^2 + B_{n-1} \lambda + B_n \quad \dots 2.17$$

From equations (2.16) and (2.17) we get,

$$\text{adj}(\lambda I - A) = B_1 \lambda^{n-1} + B_2 \lambda^{n-2} + B_3 \lambda^{n-3} + \dots + B_{n-2} \lambda^2 + B_{n-1} \lambda + B_n \quad \dots 2.18$$

We know that, $(\lambda I - A)(\lambda I - A)^{-1} = I$

$$\text{But, } (\lambda I - A)^{-1} = \frac{\text{adj}(\lambda I - A)}{|\lambda I - A|}$$

$$\therefore (\lambda I - A) \frac{\text{adj}(\lambda I - A)}{|\lambda I - A|} = I$$

$$(\lambda I - A) \text{adj}(\lambda I - A) = |\lambda I - A| I$$

Using equations (2.12) and (5.18), the equation (5.19) can be written as,

$$\begin{aligned} (\lambda I - A) (B_1 \lambda^{n-1} + B_2 \lambda^{n-2} + B_3 \lambda^{n-3} + \dots + B_{n-2} \lambda^2 + B_{n-1} \lambda + B_n) \\ = (\lambda^n + a_1 \lambda^{n-1} + a_2 \lambda^{n-2} + \dots + a_{n-2} \lambda^2 + a_{n-1} \lambda + a_n) I \end{aligned}$$

$$\begin{aligned} (B_1 \lambda^n + B_2 \lambda^{n-1} + B_3 \lambda^{n-2} + \dots + B_{n-2} \lambda^3 + B_{n-1} \lambda^2 + B_n \lambda) \\ - AB_1 \lambda^{n-1} - AB_2 \lambda^{n-2} - AB_3 \lambda^{n-3} - \dots - AB_{n-2} \lambda^2 - AB_{n-1} \lambda - AB_n \\ = I \lambda^n + a_1 I \lambda^{n-1} + a_2 I \lambda^{n-2} + \dots + a_{n-2} I \lambda^2 + a_{n-1} I \lambda + a_n I \end{aligned} \quad \dots 2.19$$

On equating the coefficient of like powers of λ in Eqn (2.18) we get the following (n+1) equations

$$\left. \begin{array}{ll} B_1 = I & \dots\dots (1) \\ B_2 - AB_1 = a_1 I & \dots\dots (2) \\ B_3 - AB_2 = a_2 I & \dots\dots (3) \\ \vdots & \vdots \\ B_{n-1} - AB_{n-2} = a_{n-2} I & \dots\dots (n-1) \\ B_n - AB_{n-1} = a_{n-1} I & \dots\dots (n) \\ -AB_n = a_n I & \dots\dots (n+1) \end{array} \right\} (n+1) \text{ equations}$$

On premultiplying both sides of equations (1), (2), (3),... (n-1), (n) and (n+1) by $A^n, A^{n-1}, A^{n-2}, \dots, A^2, A$ and I respectively we get the following (n+1) equations.

$$\left. \begin{array}{ll} A^n \times \text{equ}(1) \Rightarrow & A^n B_1 = A^n I \\ A^{n-1} \times \text{equ}(2) \Rightarrow & A^{n-1} B_2 - A^n B_1 = a_1 A^{n-1} I \\ A^{n-2} \times \text{equ}(3) \Rightarrow & A^{n-2} B_3 - A^{n-1} B_2 = a_2 A^{n-2} I \\ \vdots & \vdots \\ A^2 \times \text{equ}(n-1) \Rightarrow & A^2 B_{n-1} - A^3 B_{n-2} = a_{n-2} A^2 I \\ A \times \text{equ}(n) \Rightarrow & A B_n - A^2 B_{n-1} = a_{n-1} A I \\ I \times \text{equ}(n+1) \Rightarrow & -A B_n = a_n I \end{array} \right\} (n+1) \text{ equations}$$

On adding the above (n+1) equations we get, (i.e., all the left hand side terms gets cancelled and becomes zero),

$$\begin{aligned} 0 &= A^n I + a_1 A^{n-1} I + a_2 A^{n-2} I + \dots\dots + a_{n-2} A^2 I + a_{n-1} A I + a_n I \\ \therefore A^n + a_1 A^{n-1} + a_2 A^{n-2} + \dots\dots + a_{n-2} A^2 + a_{n-1} A + a_n I &= 0 \end{aligned} \quad \dots 2.20$$

The Eqn (5.20) shows that the matrix A satisfies its characteristic equation. Thus Cayley-Hamilton theorem is proved.

COMPUTATION OF THE FUNCTION OF A MATRIX USING CAYLEY-HAMILTON THEOREM

The Cayley-Hamilton theorem provides a simple procedure for evaluating the function of a matrix. Consider a matrix A of order $(n \times n)$ with eigenvalues $\lambda_1, \lambda_2, \lambda_3, \dots, \lambda_n$. The characteristic equation $q(\lambda)$ of matrix A will be as shown in Eqn (2.21)

$$\therefore q(\lambda) = |\lambda I - A| = \lambda^n + a_1 \lambda^{n-1} + \dots\dots + a_{n-1} \lambda + a_n = 0 \quad \dots 2.21$$

Let $f(A)$ be a function of matrix A and $f(A)$ can be expressed as a matrix polynomial. Let $f(\lambda)$ be a scalar polynomial obtained from $f(A)$ after substituting A by λ .

On dividing $f(\lambda)$ by $q(\lambda)$, we get

$$\frac{f(\lambda)}{q(\lambda)} = Q(\lambda) + \frac{R(\lambda)}{q(\lambda)} \quad \dots 2.22$$

where $Q(\lambda)$ = Quotient polynomial
and $R(\lambda)$ = Remainder polynomial

$$\therefore f(\lambda) = Q(\lambda) q(\lambda) + R(\lambda) \quad \dots 2.23$$

If we evaluate the Eqn (5.23) using the eigenvalues $\lambda_1, \lambda_2, \lambda_3, \dots, \lambda_n$ then from Eqn (5.23) we get, $q(\lambda) = 0$ and we have,

$$f(\lambda_i) = R(\lambda_i) \quad \dots 2.24$$

where $i = 1, 2, 3, \dots, n$

The remainder polynomial $R(\lambda)$ will be in the form of Eqn (2.25) shown below.

$$R(\lambda) = \alpha_0 + \alpha_1 \lambda + \alpha_2 \lambda^2 + \dots + \alpha_{n-1} \lambda^{n-1} \quad \dots 2.25$$

where $\alpha_0, \alpha_1, \alpha_2, \dots, \alpha_{n-1}$ are constants

From equations (2.24) and (2.25) when $\lambda = \lambda_i$ we get,

$$f(\lambda_i) = \alpha_0 + \alpha_1 \lambda_i + \alpha_2 \lambda_i^2 + \dots + \alpha_{n-1} \lambda_i^{n-1} \quad \dots 2.26$$

where $i = 1, 2, 3, \dots, n$

On substituting the n number of eigenvalues in Eqn (2.27), one by one, we get n number of equations. These equations can be solved to find the constants $\alpha_0, \alpha_1, \dots, \alpha_{n-1}$.

$$f(A) = Q(A) q(A) + R(A) \quad \dots 2.27$$

The Cayley-Hamilton theorem says that every square matrix satisfies its characteristic equation and so $q(A) = 0$. Therefore the Eqn (2.27) can be written as,

$$f(A) = R(A) \quad \dots 2.28$$

From Eqn (2.28) we get,

$$R(A) = \alpha_0 I + \alpha_1 A + \alpha_2 A^2 + \dots + \alpha_{n-1} A^{n-1} \quad \dots 2.29$$

From equations (2.28) and (2.29) we get,

$$f(A) = \alpha_0 I + \alpha_1 A + \alpha_2 A^2 + \dots + \alpha_{n-1} A^{n-1} \quad \dots 2.30$$

The Eqn (2.30) can be used to evaluate the function $f(A)$. On substituting the eigenvalues in Eqn (2.26) we get n-number of linear equations. The constants $\alpha_0, \alpha_1, \alpha_2, \alpha_3, \dots, \alpha_{n-1}$ are obtained by solving these n-number of linear equations.

The Eqn (2.26) can be used to form n-number of independent equations only when all the eigenvalues are distinct. If the matrix A have a multiple eigenvalue with multiplicity in then only one independent equation can be obtained by substituting the multiple eigenvalue in Eqn. (2.26). The remaining (m-1) equations are obtained by differentiating Eqn (2.26) after replacing λ_i by λ and then evaluating with $\lambda = \lambda_p$ where λ_p is the multiple eigenvalue, as shown in Eqn (5.31). [The equations corresponding to distinct eigenvalues are obtained by substituting the eigenvalues in Eqn (2.26)].

$$\left. \frac{d^j}{d\lambda^j} [f(\lambda)] \right|_{\lambda=\lambda_p} = \left. \frac{d^j}{d\lambda^j} [\alpha_0 + \alpha_1\lambda + \alpha_2\lambda^2 + \dots + \alpha_{n-1}\lambda^{n-1}] \right|_{\lambda=\lambda_p} \quad \dots 2.31$$

where $j = 1, 2, 3, \dots, (n-1)$

The equation (2.30) can also be used to compute the state transition matrix of continuous time system e^{At} by taking $f(A) = e^{At}$ and the state transition matrix of discrete time system A^k by taking $f(A) = A^k$.

Note: In order to solve $f(A) = e^{At}$, when the eigenvalues are distinct the equations (2.26) and (2.30) can also be obtained by using the sylvester's interpolation formula given below

$$\begin{vmatrix} 1 & \lambda_1 & \lambda_1^2 & \dots & \lambda_1^{n-1} & e^{\lambda_1 t} \\ 1 & \lambda_2 & \lambda_2^2 & \dots & \lambda_2^{n-1} & e^{\lambda_2 t} \\ \vdots & \vdots & \vdots & & \vdots & \vdots \\ 1 & \lambda_n & \lambda_n^2 & \dots & \lambda_n^{n-1} & e^{\lambda_n t} \\ I & A & A^2 & \dots & A^{n-1} & e^{At} \end{vmatrix} = 0$$

EXAMPLE 2.1

Find $f(A) = A^7$ for $A = \begin{bmatrix} 0 & 3 \\ -2 & -5 \end{bmatrix}$

SOLUTION

$$\begin{aligned} [\lambda I - A] &= \lambda \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} - \begin{bmatrix} 0 & 3 \\ -2 & -5 \end{bmatrix} = \begin{bmatrix} \lambda & -3 \\ 2 & \lambda + 5 \end{bmatrix} \\ |\lambda I - A| &= \begin{vmatrix} \lambda & -3 \\ 2 & \lambda + 5 \end{vmatrix} = \lambda(\lambda + 5) + 3 \times 2 = \lambda^2 + 5\lambda + 6 = (\lambda + 2)(\lambda + 3) \end{aligned}$$

The characteristic equation is given by

$$|\lambda I - A| = 0, \quad \therefore (\lambda + 2)(\lambda + 3) = 0$$

The eigenvalues λ_1, λ_2 are roots of characteristic equation.

$$\therefore \lambda_1 = -2, \quad \lambda_2 = -3,$$

Given that, $f(A) = A^7, \therefore f(\lambda) = \lambda^7$

$$\text{when } \lambda_i = \lambda_1 = -2 \quad ; \quad f(\lambda_1) = f(-2) = (-2)^7 = -128 \quad \dots 2.1.1$$

$$\text{when } \lambda_i = \lambda_2 = -3 \quad ; \quad f(\lambda_2) = f(-3) = (-3)^7 = -2187 \quad \dots 2.1.2$$

We know that, $f(\lambda_i) = \alpha_0 + \alpha_1 \lambda_i + \alpha_2 \lambda_i^2 + \dots + \alpha_{n-1} \lambda_i^{n-1}$

$$\text{Here } n = 2, \quad \therefore f(\lambda_i) = \alpha_0 + \alpha_1 \lambda_i \quad \dots 2.1.3$$

From Eqn (2.1.1) and (2.1.3) when $\lambda_i = \lambda_1 = -2$, we get,

$$\begin{aligned} f(\lambda_1) &= \alpha_0 + \alpha_1 \lambda_1 \\ -128 &= \alpha_0 + \alpha_1 (-2) \\ \therefore \alpha_0 &= 2\alpha_1 - 128 \end{aligned} \quad \dots 2.1.4$$

From Eqn (2.1.2) and (2.1.3) when $\lambda_i = \lambda_2 = -3$ we get,

$$\begin{aligned} f(\lambda_2) &= \alpha_0 + \alpha_1 \lambda_2 \\ -2187 &= \alpha_0 + \alpha_1 (-3) \\ 3\alpha_1 &= \alpha_0 + 2187 \\ \therefore \alpha_1 &= \frac{\alpha_0 + 2187}{3} = \frac{1}{3}\alpha_0 + 729 \end{aligned} \quad \dots 2.1.5$$

On substituting for α_1 from Eqn (2.1.5) in Eqn (2.1.4) we get,

$$\begin{aligned} \alpha_0 &= 2 \left(\frac{1}{3} \alpha_0 + 729 \right) - 128 \\ \alpha_0 &= \frac{2}{3} \alpha_0 + 1458 - 128 \\ \alpha_0 - \frac{2}{3} \alpha_0 &= 1330 \\ \frac{1}{3} \alpha_0 &= 1330 \quad ; \quad \therefore \alpha_0 = 3 \times 1330 = 3990 \end{aligned}$$

On substituting the value of α_0 in Eqn (2.1.5) we get,

$$\alpha_1 = \frac{1}{3} (3990) + 729 = 1330 + 729 = 2059$$

We know that, $f(A) = \alpha_0 I + \alpha_1 A$

$$\begin{aligned} \therefore f(A) &= 3990 \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} + 2059 \begin{bmatrix} 0 & 3 \\ -2 & -5 \end{bmatrix} \\ &= \begin{bmatrix} 3990 & 0 \\ 0 & 3990 \end{bmatrix} + \begin{bmatrix} 0 & 6177 \\ -4118 & -10295 \end{bmatrix} = \begin{bmatrix} 3990 & 6177 \\ -4118 & -6305 \end{bmatrix} \\ \therefore A^7 &= \begin{bmatrix} 3990 & 6177 \\ -4118 & -6305 \end{bmatrix} \end{aligned}$$

ALTERNATE METHOD

$$A^2 = A \cdot A = \begin{bmatrix} 0 & 3 \\ -2 & -5 \end{bmatrix} \begin{bmatrix} 0 & 3 \\ -2 & -5 \end{bmatrix} = \begin{bmatrix} -6 & -15 \\ 10 & 19 \end{bmatrix}$$

$$A^3 = A^2 \cdot A = \begin{bmatrix} -6 & -15 \\ 10 & 19 \end{bmatrix} \begin{bmatrix} 0 & 3 \\ -2 & -5 \end{bmatrix} = \begin{bmatrix} 30 & 57 \\ -38 & -65 \end{bmatrix}$$

$$A^4 = A^3 \cdot A = \begin{bmatrix} 30 & 57 \\ -38 & -65 \end{bmatrix} \begin{bmatrix} 0 & 3 \\ -2 & -5 \end{bmatrix} = \begin{bmatrix} -114 & -195 \\ 130 & 211 \end{bmatrix}$$

$$\therefore A^7 = A^4 \cdot A^3 = \begin{bmatrix} -114 & -195 \\ 130 & 211 \end{bmatrix} \begin{bmatrix} 30 & 57 \\ -38 & -65 \end{bmatrix} = \begin{bmatrix} 3990 & 6177 \\ -4118 & -6305 \end{bmatrix}$$

EXAMPLE 1.2

For $A = \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix}$

Compute the state transition matrix e^{At} using Cayley-Hamilton theorem.

SOLUTION

Given that, $A = \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix}$

$$[\lambda I - A] = \lambda \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} - \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix} = \begin{bmatrix} \lambda & -1 \\ 2 & \lambda + 3 \end{bmatrix}$$

$$|\lambda I - A| = \begin{vmatrix} \lambda & -1 \\ 2 & \lambda + 3 \end{vmatrix} = \lambda(\lambda + 3) + 1 \times 2 = \lambda^2 + 3\lambda + 2$$

$$= (\lambda + 1)(\lambda + 2)$$

The characteristic equation is given by

$$|\lambda I - A| = 0, \quad \therefore (\lambda + 1)(\lambda + 2) = 0$$

The eigenvalues λ_1, λ_2 are roots of characteristic equation.

$$\therefore \lambda_1 = -1, \quad \lambda_2 = -2,$$

$$\text{Let } f(A) = e^{At}; \quad \therefore f(\lambda) = e^{\lambda t}$$

$$\text{when } \lambda_i = \lambda_1 = -1; \quad f(\lambda_1) = f(-1) = e^{-t} \quad \dots 2.2.1$$

$$\text{when } \lambda_i = \lambda_2 = -2; \quad f(\lambda_2) = f(-2) = e^{-2t} \quad \dots 2.2.2$$

$$\text{We know that, } f(\lambda_i) = \alpha_0 + \alpha_1 \lambda_i + \alpha_2 \lambda_i^2 + \dots + \alpha_{n-1} \lambda_i^{n-1}$$

$$\text{Here } n = 2, \quad \therefore f(\lambda_i) = \alpha_0 + \alpha_1 \lambda_i \quad \dots 2.2.3$$

From Eqn (2.2.1) and (2.2.3) when $\lambda_i = \lambda_1 = -1$, we get,

$$\begin{aligned}
f(\lambda_1) &= \alpha_0 + \alpha_1 \lambda_1 \\
e^{-t} &= \alpha_0 + \alpha_1 (-1) \\
\therefore \alpha_0 &= \alpha_1 + e^{-t}
\end{aligned}$$

...2.2.4

From Eqn (2.2.2) and (2.2.3) when $\lambda_1 = \lambda_2 = -2$, we get,

$$\begin{aligned}
f(\lambda_2) &= \alpha_0 + \alpha_1 \lambda_2 \\
e^{-2t} &= \alpha_0 + \alpha_1 (-2) \\
2\alpha_1 &= \alpha_0 - e^{-2t} \\
\therefore \alpha_1 &= \frac{1}{2}\alpha_0 - \frac{1}{2}e^{-2t}
\end{aligned}$$

...2.2.5

On substitutign for α_1 from Eqn (2.2.5) in Eqn (2.2.4) we get,

$$\begin{aligned}
\alpha_0 &= \frac{1}{2}\alpha_0 - \frac{1}{2}e^{-2t} + e^{-t} \\
\alpha_0 - \frac{1}{2}\alpha_0 &= -\frac{1}{2}e^{-2t} + e^{-t} \\
\frac{1}{2}\alpha_0 &= e^{-t} - \frac{1}{2}e^{-2t} \\
\therefore \alpha_0 &= 2e^{-t} - e^{-2t}
\end{aligned}$$

On substituting the vaue of α_0 in Eqn (2.2.5) we get,

$$\begin{aligned}
\alpha_1 &= \frac{1}{2}(2e^{-t} - e^{-2t}) - \frac{1}{2}e^{-2t} \\
\alpha_1 &= e^{-t} - \frac{1}{2}e^{-2t} - \frac{1}{2}e^{-2t} = e^{-t} - e^{-2t}
\end{aligned}$$

By Cayley-Hamilton theorem,

$$\begin{aligned}
f(A) &= \alpha_0 I + \alpha_1 A \\
\therefore f(A) &= (2e^{-t} - e^{-2t}) \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} + (e^{-t} - e^{-2t}) \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix} \\
&= \begin{bmatrix} 2e^{-t} - e^{-2t} & 0 \\ 0 & 2e^{-t} - e^{-2t} \end{bmatrix} + \begin{bmatrix} 0 & e^{-t} - e^{-2t} \\ -2e^{-t} + 2e^{-2t} & -3e^{-t} + 3e^{-2t} \end{bmatrix} \\
&= \begin{bmatrix} 2e^{-t} - e^{-2t} & e^{-t} - e^{-2t} \\ -2e^{-t} + 2e^{-2t} & -e^{-t} + 2e^{-2t} \end{bmatrix}
\end{aligned}$$

$$\therefore \text{State transititon matrix, } e^{At} = \begin{bmatrix} 2e^{-t} - e^{-2t} & e^{-t} - e^{-2t} \\ -2e^{-t} + 2e^{-2t} & -e^{-t} + 2e^{-2t} \end{bmatrix}$$

EXAMPLE 2.3

The system matrix A of as discrete time systme is given by $A = \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix}$

Compute the state transition matrix A^k using the Cayley-Hamilton theorem.

SOLUTION

$$\text{Given that, } A = \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix}$$

$$[\lambda I - A] = \lambda \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} - \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix} = \begin{bmatrix} \lambda & -1 \\ 2 & \lambda + 3 \end{bmatrix}$$

$$|\lambda I - A| = \begin{vmatrix} \lambda & -1 \\ 2 & \lambda + 3 \end{vmatrix} = \lambda(\lambda + 3) + 1 \times 2 = \lambda^2 + 3\lambda + 2$$

$$= (\lambda + 1)(\lambda + 2)$$

The characteristic equation is given by

$$|\lambda I - A| = 0, \quad \therefore (\lambda + 1)(\lambda + 2) = 0$$

The eigenvalues λ_1, λ_2 are roots of characteristic equation.

$$\therefore \lambda_1 = -1, \quad \lambda_2 = -2,$$

$$\text{Let } f(A) = A^k \quad ; \quad \therefore f(\lambda) = \lambda^k$$

$$\text{when } \lambda_i = \lambda_1 = -1 \quad ; \quad f(\lambda_1) = f(-1) = (-1)^k \quad \dots 2.3.1$$

$$\text{when } \lambda_i = \lambda_2 = -2 \quad ; \quad f(\lambda_2) = f(-2) = (-2)^k \quad \dots 2.3.2$$

$$\text{We know that, } f(\lambda_i) = \alpha_0 + \alpha_1 \lambda_i + \alpha_2 \lambda_i^2 + \dots + \alpha_{n-1} \lambda_i^{n-1}$$

$$\text{Here } n = 2, \quad \therefore f(\lambda_i) = \alpha_0 + \alpha_1 \lambda_i \quad \dots 2.3.3$$

From Eqn (2.3.1) and (2.3.3) when $\lambda_i = \lambda_1 = -1$ we get,

$$f(\lambda_1) = \alpha_0 + \alpha_1 \lambda_1$$

$$(-1)^k = \alpha_0 + \alpha_1 (-1)$$

$$\therefore \alpha_0 = \alpha_1 + (-1)^k \quad \dots 2.3.4$$

From Eqn (2.3.2) and (2.3.3) when $\lambda_i = \lambda_2 = -2$, we get,

$$f(\lambda_2) = \alpha_0 + \alpha_1 \lambda_2$$

$$(-2)^k = \alpha_0 + \alpha_1 (-2)$$

$$2\alpha_1 = \alpha_0 - (-2)^k$$

$$\therefore \alpha_1 = \frac{1}{2}\alpha_0 - \frac{1}{2}(-2)^k \quad \dots 2.3.5$$

On substituting for α_1 from Eqn (2.3.5) in Eqn (2.3.4) we get,

$$\begin{aligned}\alpha_0 &= \frac{1}{2}\alpha_0 - \frac{1}{2}(-2)^k + (-1)^k \\ \alpha_0 - \frac{1}{2}\alpha_0 &= (-1)^k - \frac{1}{2}(-2)^k \\ \frac{1}{2}\alpha_0 &= (-1)^k - \frac{1}{2}(-2)^k \\ \therefore \alpha_0 &= 2(-1)^k - (-2)^k\end{aligned}$$

On substituting the value of α_0 in Eqn (2.3.5) we get,

$$\begin{aligned}\alpha_1 &= \frac{1}{2}(2(-1)^k - (-2)^k) - \frac{1}{2}(-2)^k \\ \alpha_1 &= (-1)^k - \frac{1}{2}(-2)^k - \frac{1}{2}(-2)^k \\ \therefore \alpha_1 &= (-1)^k - (-2)^k\end{aligned}$$

By Cayley-Hemilton theorem,

$$\begin{aligned}f(A) &= \alpha_0 I + \alpha_1 A \\ \therefore f(A) &= (2(-1)^k - (-2)^k) \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} + ((-1)^k - (-2)^k) \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix} \\ &= \begin{bmatrix} 2(-1)^k - (-2)^k & 0 \\ 0 & 2(-1)^k - (-2)^k \end{bmatrix} + \begin{bmatrix} 0 & (-1)^k - (-2)^k \\ -2(-1)^k + 2(-2)^k & -3(-1)^k + 3(-2)^k \end{bmatrix} \\ &= \begin{bmatrix} 2(-1)^k - (-2)^k & (-1)^k - (-2)^k \\ -2(-1)^k + 2(-2)^k & -(-1)^k + 2(-2)^k \end{bmatrix} \\ \therefore \text{State transition matrix of discrete time system} \} A^k &= \begin{bmatrix} 2(-1)^k - (-2)^k & (-1)^k - (-2)^k \\ -2(-1)^k + 2(-2)^k & -(-1)^k + 2(-2)^k \end{bmatrix}\end{aligned}$$

2.5 TRANSFORMATION OF STATE MODEL

The state model of a system is not unique and it can be formed using physical variables, phase variables or canonical variables. The physical variables are useful from an application point of view because they can be measured and used for control purposes. However, the state model using physical variables is not convenient for investigation of system properties and evaluation of time response. But the canonical state model is most convenient for time domain analysis. In canonical model the system matrix A will be a diagonal matrix. Therefore each component state variable equation is a first order equation and is decoupled from all other component state variable equations.

When a non diagonal system matrix A has distinct eigenvalues, it can be converted to a diagonal matrix by a similarity transformation using a modal matrix, M . Due to this the state model is transformed to canonical form.

When a non diagonal system matrix has multiple eigenvalues, it can be converted to Jordan matrix by a similarity transformation using modal matrix, M. Due to this the state model is transformed to Jordan canonical form.

CANONICAL FORM OF STATE MODEL

Consider the state equation of a system, $\dot{X} = AX + BU$, where X is the state variable vector of order $n \times 1$. Let us define a new state variable vector Z, such that $X = MZ$, where M is the Modal Matrix or Diagonalization matrix.

The state model of the n^{th} order system is given by

$$\dot{X} = AX + BU$$

$$Y = CX + DU$$

On substituting $X = MZ$ in the state model of the system, we get

$$\dot{X} = AMZ + BU \quad \dots 2.32$$

$$Y = CMZ + DU \quad \dots 2.33$$

Premultiply Eqn (2.32) by M^{-1}

$$\therefore M^{-1}\dot{X} = M^{-1}AMZ + M^{-1}BU \quad \dots 2.34$$

The relation governing X and Z is, $X = MZ$2.35

On differentiating Eqn (2.35), we get, $\dot{X} = M\dot{Z}$...2.36

On premultiplying the Eqn (2.36) by M^{-1} we get

$$M^{-1}\dot{X} = \dot{Z} \quad \dots 2.37$$

From Eqn (2.34) and (2.37), we get,

$$\dot{Z} = M^{-1}AMZ + M^{-1}BU \quad \dots 2.38$$

$$L_1, M^{-1}AM = \tilde{A} \text{ (called grammian matrix)} \quad \dots 2.39$$

$$M^{-1}B = \dot{X} = \tilde{B} \quad \dots 2.40$$

$$CM = \tilde{C} \quad \dots 2.41$$



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DEPARTMENT OF ELECTRICAL AND ELECTRONICS

UNIT – III – Advanced Control Systems – SEEA1602

CONCEPTS OF CONTROLLABILITY AND OBSERVABILITY

CONTROLLABILITY

The controllability verifies the usefulness of a state variable. In the controllability test we can find, whether the state variable can be controlled to achieve the desired output. The choice of state variables is arbitrary while forming the state model. After determining the state model, the controllability of the state variable is verified. If the state variable is not controllable then we have to go for another choice of state variable.

Definition of controllability

A system is said to be completely state controllable if it is possible to transfer the system state from any initial state $X(t_0)$ to any other desired state $X(t_0)$ in specified finite time by a control vector $U(t)$.

The controllability of a state model can be tested by Kalman's test or Gilbert's test.

Gilbert's method of testing controllability

Case (i): When the system matrix has distinct eigenvalues

In this case the system matrix can be diagonalized and the state model can be converted to canonical form.

Consider the state model of the system,

$$\begin{aligned}\dot{X} &= AX + BU \\ Y &= CX + DU\end{aligned}$$

The state model can be converted to canonical form by a transformation, $X = MZ$, where M is the modal matrix and Z is the transformed state variable vector.

The transformed state model is given by

$$\begin{aligned}\dot{Z} &= \Lambda Z + \tilde{B}U \\ Y &= \tilde{C}Z + DU\end{aligned}$$

where $\Lambda = M^{-1}AM$

$$\tilde{B} = M^{-1}B$$

$$\tilde{C} = CM$$

In this case the necessary and sufficient condition for complete controllability is that, the matrix \tilde{B} must have no rows with all zeros. If any row of the matrix \tilde{B} is zero then the corresponding state variable is uncontrollable.

Case (ii): When the system matrix has repeated eigenvalues

In this case, the system matrix cannot be diagonalized but can be transformed to Jordan canonical form.

Consider the state model of the system,

$$\begin{aligned}\dot{X} &= AX + BU \\ Y &= CX + DU\end{aligned}$$

The state model can be transformed to Jordan canonical form by a transformation. $X = MZ$, where M is model matrix and Z is the transformed state variable vector.

The transformed state model is given by,

$$\begin{aligned}\dot{Z} &= JZ + \tilde{B}U \\ Y &= \tilde{C}Z + DU\end{aligned}$$

where

$$\begin{aligned}J &= M^{-1}AM \\ \tilde{B} &= M^{-1}B \\ \tilde{C} &= CM\end{aligned}$$

In this case, the system is completely controllable if the elements of any row of B that correspond to the last row of each Jordan block are not all zero and the rows corresponding to other state variables must not have all zeros.

Kalman's method of testing controllability

Consider a system with state equation, $\dot{X} = AX + BU$. For this system, a composite matrix, Q_c can be formed such that,

$$Q_c = [B \quad AB \quad A^2B \quad \dots \quad A^{n-1}B] \quad \dots 3.1$$

where n is the order of the system (n is also equal to number of state variables)

In this case the system is completely state controllable if the rank of the composite matrix, Q_c is n .

The rank of the matrix is n , if the determinant of $(n \times n)$ composite matrix Q_c is non-zero. i.e., if $|Q_c| \neq 0$, then rank of $Q_c = n$ and the system is completely state controllable.

The advantage is kalman's test is that the calculations are simpler. But the disadvantage in kalman's test is that, we can't find the state variable which is uncontrollable. But in Gilbert's method we can find the uncontrollable state variable which is the state variable corresponding to the row of \tilde{B} which has all zeros.

Condition for complete state controllability in the s-plane

A necessary and sufficient condition for complete state controllability is that no cancellation of poles and zeros occurs in the transfer function of the system. If cancellation occurs then the system cannot be controlled in the direction of the cancelled mode.

OBSERVABILITY

In observability test we can find whether the state variable is observable or measurable. The concept of observability is useful in solving the problem of reconstructing unmeasurable

state variables from measurable ones in the minimum possible length of time. In state feedback control the estimation of unmeasurable state variables is essential in order to construct the control signals.

Definition of observability

A system is said to be completely observable if every state $X(t)$ can be completely identified by measurements of the output $Y(t)$ over a finite time interval. The observability of a system can be tested by either Gilbert's method or Kalman's method.

Gilber's method of testing observability

Consider a state model of n^{th} order system, $\dot{X} = AX + BU$; $Y = CX + DC$

The state model can be transformed to a canonical or Jordan canonical form by a transformation, $X = MZ$, where M is the modal matrix and Z is the transformed state variable vector.

The transformed state model is,

$$\begin{aligned} \dot{Z} &= \Lambda Z + \tilde{B}U \\ Y &= \tilde{C}Z + DU \end{aligned} \quad (\text{or}) \quad \begin{aligned} \dot{Z} &= JZ + \tilde{B}U \\ Y &= \tilde{C}Z + DU \end{aligned}$$

where $\Lambda = M^{-1}AM$; if eigenvalues are distinct ; $\tilde{B} = M^{-1}B$
 $J = M^{-1}AM$; if eigenvalues have multiplicity; $\tilde{C} = CM$

The necessary and sufficient condition for complete observability is that none of the columns of the matrix \tilde{C} be zero. If any of the column's of \tilde{C} has all zeros then the corresponding state variable is not observable.

Kalman's Test for observability

Consider a system with state model, $\dot{X} = AX + BU$; $Y = CX + DU$

For this system, a composite matrix, Q_0 can be formed such that,

$$Q_0 = \begin{bmatrix} C^T & A^T C^T & (A^T)^2 C^T & (A^T)^3 C^T & \dots & (A^T)^{n-1} C^T \end{bmatrix} \quad \dots 3.2$$

where n is the order of the system (n is also equal to number of state variables)

In this case, the system is completely observable if the rank of composite matrix, Q_0 is n . The rank of the matrix is n , if the determinant of $n \times n$ composite matrix Q_0 is non-zero. The disadvantage is Kalman's test is that, the non observable state variables cannot be determined.

Condition for complete observability in the s-plane

The necessary and sufficient condition for complete observability is that no cancellation of poles and zeros occurs in the transfer function. If cancellation occurs, the cancelled mode cannot be observed in the output.

RELATIONSHIPS BETWEEN CONTROLLABILITY, OBSERVABILITY & TRANSFER FUNCTIONS

The concepts of controllability and observability play an important role in the design of control system in state space. They govern the existence of a complete solution to the control system design problem. The solution to this problem may not exist if the system considered is not controllable.

It is important to note that all physical systems are controllable and observable. However, the mathematical models of these systems may not possess the property of the controllability or observability. Then it is necessary to know the conditions under which a system is controllable and observable and the designer can seek another state model which is controllable and observable.

Duality property

The concepts of controllability and observability are dual concepts and it is proposed by Kalman as principle of duality. The principle of duality states that a system is completely state controllable if and only if its dual system is completely observable or vice-versa. [i.e., if the system is observable then its dual is controllable]. Using the principle of duality, the observability of a given system can be checked by testing the state controllability of its dual or vice-versa.

Consider the system S_1 , described by the state model shown below.

$$\dot{X} = AX + BU$$

$$Y = CX$$

Let the dual of system S_1 be denoted as S_2 and the dual system S_2 is described by the following state model.

$$\dot{Z} = A^T Z + C^T V$$

$$N = B^T Z$$

where, Z = State vector of dual system

V = Input vector of dual system

N = Output vector of dual system

For the system S_1 the composite matrix, Q_{c1} for controllability is given by Eqn (3.3) and the composite matrix, Q_{o1} for observability is given by Eqn (3.4).

$$Q_{c1} = [B \quad AB \quad A^2B \quad \dots \quad A^{n-1}B] \quad \dots 3.3$$

$$Q_{o1} = [C^T \quad A^T C^T \quad (A^T)^2 C^T \quad \dots \quad (A^T)^{n-1} C^T] \quad \dots 3.4$$

For the dual system S_2 the composite matrix, Q_{c2} for controllability is given by Eqn (3.5) and the composite matrix Q_{o2} for observability is given by Eqn (3.6).

$$Q_{c2} = [C^T \quad A^T C^T \quad (A^T)^2 C^T \quad \dots \quad (A^T)^{n-1} C^T] \quad \dots 3.5$$

$$Q_{o2} = [A \quad AB \quad A^2B \quad \dots \quad A^{n-1}B] \quad \dots 3.6$$

From equations (3.3) and (3.4) we get $Q_{c1} = Q_{c2}$, hence if the system S_1 is controllable then its dual system S_2 is observable.

From equations (3.5) and (3.6) we get $Q_{c1} = Q_{c2}$, hence if the system S_1 is observable then its dual system S_2 is controllable.

Effect of pole-zero cancellation in transfer function

The concepts of controllability and observability are closely related to the properties of the transfer function. Consider an n^{th} order system with distinct eigenvalues. The transfer function of the system can be expressed as a ratio of the two polynomials as shown in Eqn. (3.7).

$$T(s) = \frac{Y(s)}{U(s)} = \frac{b_0 s^m + b_1 s^{m-1} + \dots + b_{m-1} s + b_m}{s^n + a_1 s^{n-1} + \dots + a_{n-1} s + a_n} ; m < n \quad \dots 3.7$$

$$= \frac{K(s + \beta_1)(s + \beta_2) \dots (s + \beta_m)}{(s + \lambda_1)(s + \lambda_2) \dots (s + \lambda_n)}$$

By partial fraction expansion technique the Eqn (3.8) can be written as,

$$\frac{Y(s)}{U(s)} = \frac{C_1}{s + \lambda_1} + \frac{C_2}{s + \lambda_2} + \dots + \frac{C_i}{s + \lambda_i} + \dots + \frac{C_n}{s + \lambda_n} \quad \dots 3.8$$

where $C_1, C_2, C_3, \dots, C_n$ are residues.

If the transfer function has identical pair of pole and zero at $\beta_i = \lambda_i$, then $C_i = 0$. The effect of this cancellation on controllability and observability properties depends on the choice of state variables [or depends on the method of forming state model].

In one method of state space modelling using canonical of variables, the $C_i = 0$, will appear in input (control) vector B and the state x_i is uncontrollable. In another method of state space modelling using canonical variables, the $C_i = 0$, will appear in output vector C and the state x_i is shielded from observation.

From the above discussion we can conclude that if cancellation of pole-zero occurs in the transfer function of a system, then the system will be either not state controllable or unobservable, depending on how the state variables are defined (or chosen). If the transfer function does not have pole-zero cancellation, the system can always be represented by completely controllable and observable state model.

EXAMPLE 1.6

Write the state equations for the system shown in Figure 3.1 in which x_1, x_2 and x_3 constitute the state vector. Determine whether the system is completely controllable and observable.

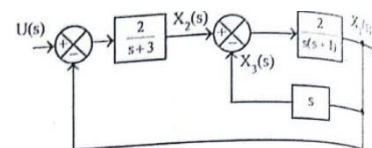


Figure 3.1

SOLUTION

To find state model

The state equations are obtained by writing equations for the output of each block and then taking inverse Laplace transform.

With reference to Figure 3.2 we can write,

$$X_1(s) = [X_2(s) - X_3(s)] \left[\frac{2}{s(s+1)} \right]$$

$$s(s+1) X_1(s) = 2X_2(s) - 2X_3(s)$$

$$s^2 X_1(s) + s X_1(s) = 2X_2(s) - 2X_3(s)$$

On taking inverse laplace transform

$$\ddot{x}_1 + \dot{x}_1 = 2x_2 - 2x_3 \quad \dots 3.6.1$$

With reference to Figure 3.3, we can write,

$$X_3(s) = sX_1(s)$$

On taking inverse laplace transform

$$x_3 = \dot{x}_1 \quad \dots 3.6.2$$

With reference to Figure 3.6.4 we can write

$$X_2(s) = [U(s) - X_1(s)] \left[\frac{2}{s+3} \right]$$

$$X_2(s) (s+3) = 2U(s) - 2X_1(s)$$

$$sX_2(s) + 3X_2(s) = 2U(s) - 2X_1(s)$$

On taking inverse Laplace transform

$$\dot{x}_2 + 3x_2 = 2u - 2x_1$$

$$\dot{x}_2 = -2x_1 - 3x_2 + 2u \quad \dots 3.6.3$$

From Eqn (2.6.2), $\dot{x}_1 = x_3$; $\therefore \ddot{x}_1 = \dot{x}_3$

Put $\dot{x}_1 = x_3$ and $\ddot{x}_1 = \dot{x}_3$ in equation (3.6.1)

$$\therefore \dot{x}_3 + x_3 = 2x_2 - 2x_3$$

$$\dot{x}_3 = 2x_2 - 2x_3 - x_3$$

$$\dot{x}_3 = 2x_2 - 3x_3$$

The state equation are given by equations (3.6.2), (3.6.3), and (3.6.4)

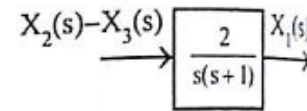


Figure 3.2

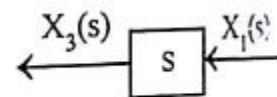


Figure 3.3

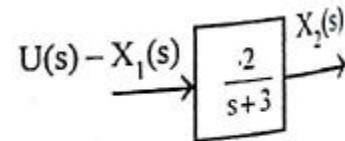


Figure 3.4

$$\begin{aligned}\dot{x}_1 &= x_3 \\ \dot{x}_2 &= -2x_1 - 3x_2 + 2u \\ \dot{x}_3 &= 2x_2 - 3x_3\end{aligned}$$

The output equation is $y = x_1$

The state model in the matrix form is

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 2 \\ 0 \end{bmatrix} u \quad ; \quad y = \begin{bmatrix} 1 & 0 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$$

To find eigenvalues

Here the system matrix, $A = \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix}$

The characteristic equation is $|\lambda I - A| = 0$

$$\begin{aligned}(\lambda I - A) &= \lambda \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} - \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix} = \begin{bmatrix} \lambda & 0 & -1 \\ 2 & \lambda+3 & 0 \\ 0 & -2 & \lambda+3 \end{bmatrix} \\ |\lambda I - A| &= \begin{vmatrix} \lambda & 0 & -1 \\ 2 & \lambda+3 & 0 \\ 0 & -2 & \lambda+3 \end{vmatrix} = \lambda(\lambda+3)^2 - 1(-4) = \lambda(\lambda^2 + 6\lambda + 9) + 4\end{aligned}$$

$$\begin{aligned}&= \lambda^3 + 6\lambda^2 + 9\lambda + 4 = (\lambda+1)(\lambda^2 + 5\lambda + 4) \\ &= (\lambda+1)(\lambda+1)(\lambda+4) = (\lambda+1)^2(\lambda+4)\end{aligned}$$

$\lambda = -1$	1	6	9	4
	↓	-1	-5	-4
		1	5	4
				0

The eigenvalues are $\lambda_1 = -1$, $\lambda_2 = -1$, and $\lambda_3 = -4$,

To find eigenvectors

$$(\lambda_1 I - A) = \lambda_1 \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} - \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix} = \begin{bmatrix} \lambda_1 & 0 & -1 \\ 2 & \lambda_1+3 & 0 \\ 0 & -2 & \lambda_1+3 \end{bmatrix}$$

Let C_{11} , C_{12} and C_{13} be cofactors along 1st row of the matrix $(\lambda_1 I - A)$

$$C_{11} = (+1) \begin{vmatrix} \lambda_1+3 & 0 \\ -2 & \lambda_1+3 \end{vmatrix} = (\lambda_1+3)^2 = \lambda_1^2 + 6\lambda_1 + 9$$

$$C_{12} = (-1) \begin{vmatrix} 2 & 0 \\ 0 & \lambda_1+3 \end{vmatrix} = -(2(\lambda_1+3)) = -2\lambda_1 - 6$$

$$C_{13} = (+1) \begin{vmatrix} 2 & \lambda_1+3 \\ 0 & -2 \end{vmatrix} = -4$$

$$m_1 = \begin{bmatrix} C_{11} \\ C_{12} \\ C_{13} \end{bmatrix} = \begin{bmatrix} \lambda_1^2 + 6\lambda_1 + 9 \\ -2\lambda_1 - 6 \\ -4 \end{bmatrix} = \begin{bmatrix} 1 - 6 + 9 \\ 2 - 6 \\ -4 \end{bmatrix} = \begin{bmatrix} 4 \\ -4 \\ -4 \end{bmatrix}$$

$$m_2 = \begin{bmatrix} \frac{d}{d\lambda_1} C_{11} \\ \frac{d}{d\lambda_1} C_{12} \\ \frac{d}{d\lambda_1} C_{13} \end{bmatrix} = \begin{bmatrix} 2\lambda_1 + 6 \\ -2 \\ 0 \end{bmatrix} = \begin{bmatrix} -2 + 6 \\ -2 \\ 0 \end{bmatrix} = \begin{bmatrix} 4 \\ -2 \\ 0 \end{bmatrix}$$

$$(\lambda_3 I - A) = (-4) \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} - \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix} = \begin{bmatrix} -4 & 0 & -1 \\ 2 & -1 & 0 \\ 0 & -2 & -1 \end{bmatrix}$$

Let C_{11} , C_{12} and C_{13} be the cofactors along 1st row of the matrix $(\lambda_3 I - A)$:

$$C_{11} = (+1) \begin{vmatrix} -1 & 0 \\ -2 & -1 \end{vmatrix} = 1 ; C_{12} = (-1) \begin{vmatrix} 2 & 0 \\ 0 & -1 \end{vmatrix} = 2 ; C_{13} = (+1) \begin{vmatrix} 2 & -1 \\ 0 & -2 \end{vmatrix} = -4$$

$$\therefore m_3 = \begin{bmatrix} C_{11} \\ C_{12} \\ C_{13} \end{bmatrix} = \begin{bmatrix} 1 \\ 2 \\ -4 \end{bmatrix}$$

To find canonical form of state model

The modal matrix, M is given by

$$M = [m_1 \ m_2 \ m_3] = \begin{bmatrix} 4 & 4 & 1 \\ -4 & -2 & 2 \\ -4 & 0 & -4 \end{bmatrix}$$

$$M^{-1} = \frac{[\text{Cofactor of } M]^T}{\text{Determinant of } M} = \frac{M_{\text{cof}}^T}{\Delta_M}$$

$$\Delta_M = \begin{vmatrix} 4 & 4 & 1 \\ -4 & -2 & 2 \\ -4 & 0 & -4 \end{vmatrix} = 4(8) - 4(24) + 1(-8) = 32 - 96 - 8 = -72$$

$$M_{\text{cof}}^T = \begin{bmatrix} 8 & -24 & -8 \\ 16 & -12 & -16 \\ 10 & -12 & 8 \end{bmatrix}^T = \begin{bmatrix} 8 & 16 & 10 \\ -24 & -12 & -12 \\ -8 & -16 & 8 \end{bmatrix}$$

$$M^{-1} = \frac{1}{-72} \begin{bmatrix} 8 & 16 & 10 \\ -24 & -12 & -12 \\ -8 & -16 & 8 \end{bmatrix} = \frac{1}{18} \begin{bmatrix} -2 & -4 & -2.5 \\ 6 & 3 & 3 \\ 2 & 4 & -2 \end{bmatrix}$$

$$\begin{aligned}
J = M^{-1}AM &= \frac{1}{18} \begin{bmatrix} -2 & -4 & -2.5 \\ 6 & 3 & 3 \\ 2 & 4 & -2 \end{bmatrix} \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix} \begin{bmatrix} 4 & 4 & 1 \\ -4 & -2 & 2 \\ -4 & 0 & -4 \end{bmatrix} \\
&= \frac{1}{18} \begin{bmatrix} 8 & 7 & 5.5 \\ -6 & -3 & -3 \\ -8 & -16 & 8 \end{bmatrix} \begin{bmatrix} 4 & 4 & 1 \\ -4 & -2 & 2 \\ -4 & 0 & -4 \end{bmatrix} \\
&= \frac{1}{18} \begin{bmatrix} -18 & 18 & 0 \\ 0 & -18 & 0 \\ 0 & 0 & -72 \end{bmatrix} = \begin{bmatrix} -1 & 1 & 0 \\ 0 & -1 & 0 \\ 0 & 0 & -4 \end{bmatrix} \quad \text{Jordan block} \\
\tilde{B} = M^{-1}B &= \frac{1}{18} \begin{bmatrix} -2 & -4 & -2.5 \\ 6 & 3 & 3 \\ 2 & 4 & -2 \end{bmatrix} \begin{bmatrix} 0 \\ 2 \\ 0 \end{bmatrix} = \begin{bmatrix} -8/18 \\ 6/18 \\ 8/18 \end{bmatrix} = \begin{bmatrix} -4/9 \\ 3/9 \\ 4/9 \end{bmatrix} \\
\tilde{C} = CM &= \begin{bmatrix} 1 & 0 & 0 \end{bmatrix} \begin{bmatrix} 4 & 4 & 1 \\ -4 & -2 & 2 \\ -4 & 0 & -4 \end{bmatrix} = \begin{bmatrix} 4 & 4 & 1 \end{bmatrix}
\end{aligned}$$

The Jordan canonical form of state model is shown below.

$$\begin{aligned}
\dot{Z} &= JZ + \tilde{B}U ; \quad Y = \tilde{C}Z + DU \quad (\text{Here } DU \text{ is not defined}) \\
\begin{bmatrix} \dot{z}_1 \\ \dot{z}_2 \\ \dot{z}_3 \end{bmatrix} &= \begin{bmatrix} -1 & 1 & 0 \\ 0 & -1 & 0 \\ 0 & 0 & -4 \end{bmatrix} \begin{bmatrix} z_1 \\ z_2 \\ z_3 \end{bmatrix} + \begin{bmatrix} -4/9 \\ 3/9 \\ 4/9 \end{bmatrix} [u] ; \quad Y = \begin{bmatrix} 4 & 4 & 1 \end{bmatrix} \begin{bmatrix} z_1 \\ z_2 \\ z_3 \end{bmatrix}
\end{aligned}$$

CONCLUSION

It is observed that the elements of the rows of \tilde{B} are not all zeros. Hence the system is completely controllable (or state controllable).

It is observed that the elements of the columns of \tilde{C} are not all zeros. Hence the system is completely observable [i.e., all the state variables are observable].

ALTERNATE METHOD

KALMAN'S TEST FOR CONTROLLABILITY

$$A^2 = A.A = \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix} \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix} = \begin{bmatrix} 0 & 2 & -3 \\ 6 & 9 & -2 \\ -4 & -12 & 9 \end{bmatrix}$$

$$A.B = \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix} \begin{bmatrix} 0 \\ 2 \\ 0 \end{bmatrix} = \begin{bmatrix} 0 \\ -6 \\ 4 \end{bmatrix}$$

$$A^2.B = \begin{bmatrix} 0 & 2 & -3 \\ 6 & 9 & -2 \\ -4 & -12 & 9 \end{bmatrix} \begin{bmatrix} 0 \\ 2 \\ 0 \end{bmatrix} = \begin{bmatrix} 4 \\ 18 \\ -24 \end{bmatrix}$$

The composite matrix for controllability, $Q_c = [B \ AB \ A^2B]$

$$= \begin{bmatrix} 0 & 0 & 4 \\ 2 & -6 & 18 \\ 0 & 4 & -24 \end{bmatrix}$$

Determinant of $Q_c = \begin{vmatrix} 0 & 0 & 4 \\ 2 & -6 & 18 \\ 0 & 4 & -24 \end{vmatrix} = 4 \times 8 = 32$; Since $|Q_c| \neq 0$, the rank of $Q_c = 3$.

Hence the system is completely state controllable.

KALMAN'S TEST FOR OBSERVABILITY

$$A^T = \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix}^T = \begin{bmatrix} 0 & -2 & 0 \\ 0 & -3 & 2 \\ 1 & 0 & -3 \end{bmatrix}$$

$$C^T = [1 \ 0 \ 0]^T = \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix}$$

$$(A^T)^2 = \begin{bmatrix} 0 & -2 & 0 \\ 0 & -3 & 2 \\ 1 & 0 & -3 \end{bmatrix} \begin{bmatrix} 0 & -2 & 0 \\ 0 & -3 & 2 \\ 1 & 0 & -3 \end{bmatrix} = \begin{bmatrix} 0 & 6 & -4 \\ 2 & 9 & -12 \\ -3 & -2 & 9 \end{bmatrix}$$

$$A^T C^T = \begin{bmatrix} 0 & -2 & 0 \\ 0 & -3 & 2 \\ 1 & 0 & -3 \end{bmatrix} \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix}$$

$$(A^T)^2 C^T = \begin{bmatrix} 0 & 6 & -4 \\ 2 & 9 & -12 \\ -3 & -2 & 9 \end{bmatrix} \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix} = \begin{bmatrix} 0 \\ 2 \\ -3 \end{bmatrix}$$

The composite matrix for observability $Q_o = [C^T \ A^T C^T \ (A^T)^2 C^T] = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 0 & 2 \\ 0 & 1 & -3 \end{bmatrix}$

Determinant of $Q_o = \begin{vmatrix} 1 & 0 & 0 \\ 0 & 0 & 2 \\ 0 & 1 & -3 \end{vmatrix} = 1 \times -2 = -2$; Since $|Q_o| \neq 0$, the rank of $Q_o = 3$

Hence the system is completely observable (or all the state variables of the system are observable).

3.7 CONTROLLABLE PHASE VARIABLE FORM OF STATE MODEL

A controllable system can be represented by a modified state model called controllable phase variable form by transforming the system matrix, A into phase variable form (Bush form or companion form).

Consider the state model of n^{th} order system with single-input and single output as shown below.

$$\dot{X} = AX + Bu \quad \dots 3.9$$

$$y = CX + Du \quad \dots 3.10$$

Let us choose a transformation, $Z = P_c X$ to transform the state model to controllable phase variable form.

Here Z = Transformed state vector of order $(n \times 1)$

P_c = Transformation matrix of order $(n \times n)$

$$Z = \begin{bmatrix} z_1 \\ z_2 \\ z_3 \\ \vdots \\ z_n \end{bmatrix} \quad \text{and} \quad P_c = \begin{bmatrix} p_{11} & p_{12} & p_{13} & \dots & p_{1n} \\ p_{21} & p_{22} & p_{23} & \dots & p_{2n} \\ p_{31} & p_{32} & p_{33} & \dots & p_{3n} \\ \vdots & \vdots & \vdots & \dots & \vdots \\ p_{n1} & p_{n2} & p_{n3} & \dots & p_{nn} \end{bmatrix}$$

On premultiplying the equation $Z = P_c X$ by P_c^{-1} we get

$$P_c^{-1}Z = P_c^{-1} P_c X$$

$$\therefore X = P_c^{-1} Z$$

On differentiating the equation $X = P_c^{-1}Z$ we get,

$$\dot{X} = P_c^{-1}\dot{Z}$$

On substituting $X = P_c^{-1}Z$ and $\dot{X} = P_c^{-1}\dot{Z}$ in the state model (equation (3.9) and (3.10)) of the system we get,

$$P_c^{-1} \dot{Z} = AP_c^{-1}Z + Bu \quad \dots 3.11$$

$$y = C P_c^{-1} Z + Du \quad \dots 3.12$$

On premultiplying the equations (3.11) by P_c we get,

$$\dot{Z} = P_c A P_c^{-1} Z + P_c B u$$

$$y = C P_c^{-1} Z + Du$$

Let, $P_c A P_c^{-1} = A_c$; $P_c B = B_c$ and $C P_c^{-1} = C_c$

$$\therefore \dot{Z} = A_c Z + B_c u \quad \dots 3.13$$

$$y = C_c Z + D u \quad \dots 3.14$$

The equations (3.13) and (3.14) are called the controllable phase variable form of state model of the system.

Note: In controllable phase variable form of state model the matrices A_c , B_c and C_c will be as shown below.

$$A_c = \begin{bmatrix} 0 & 1 & 0 & \dots & 0 \\ 0 & 0 & 1 & \dots & 0 \\ 0 & 0 & 0 & \dots & 0 \\ \vdots & \vdots & \vdots & \dots & \vdots \\ 0 & 0 & 0 & \dots & 1 \\ -a_n & -a_{n-1} & -a_{n-2} & \dots & -a_1 \end{bmatrix} \quad B_c = \begin{bmatrix} 0 \\ 0 \\ 0 \\ \vdots \\ 0 \\ 1 \end{bmatrix} \quad ; \quad C_c = [c_{11} \ c_{12} \ c_{13} \ \dots \ c_{1n}]$$

Determination of transformation matrix, P_c

The $n \times n$ transformation matrix, P_c and be expressed as n -numbers of row vectors (Matrices) as shown below.

$$P_c = \begin{bmatrix} p_{11} & p_{12} & p_{13} & \dots & p_{1n} \\ p_{21} & p_{22} & p_{23} & \dots & p_{2n} \\ p_{31} & p_{32} & p_{33} & \dots & p_{3n} \\ \vdots & \vdots & \vdots & \dots & \vdots \\ p_{n1} & p_{n2} & p_{n3} & \dots & p_{nn} \end{bmatrix} = \begin{bmatrix} P_1 \\ P_2 \\ P_3 \\ \vdots \\ P_n \end{bmatrix} \quad \dots 3.15$$

Where

$$\begin{aligned} P_1 &= [p_{11} \ p_{12} \ p_{13} \ \dots \ p_{1n}] \\ P_2 &= [p_{21} \ p_{22} \ p_{23} \ \dots \ p_{2n}] \\ P_3 &= [p_{31} \ p_{32} \ p_{33} \ \dots \ p_{3n}] \\ &\vdots \\ P_n &= [p_{n1} \ p_{n2} \ p_{n3} \ \dots \ p_{nn}] \end{aligned}$$

The transformation $Z = P_c X$ can be written in the expanded form as,

$$\begin{bmatrix} z_1 \\ z_2 \\ z_3 \\ \vdots \\ z_n \end{bmatrix} = \begin{bmatrix} p_{11} & p_{12} & p_{13} & \dots & p_{1n} \\ p_{21} & p_{22} & p_{23} & \dots & p_{2n} \\ p_{31} & p_{32} & p_{33} & \dots & p_{3n} \\ \vdots & \vdots & \vdots & \dots & \vdots \\ p_{n1} & p_{n2} & p_{n3} & \dots & p_{nn} \end{bmatrix} \cdot \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ \vdots \\ x_n \end{bmatrix} \quad \dots 3.16$$

From equation (3.16) we get,

$$\begin{aligned} z_1 &= p_{11} x_1 + p_{12} x_2 + p_{13} x_3 + \dots + p_{1n} x_n \\ &= [p_{11} \ p_{12} \ p_{13} \ \dots \ p_{1n}] \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ \vdots \\ x_n \end{bmatrix} \end{aligned}$$

$$\therefore z_1 = P_1 X \quad \dots 3.17$$

On differentiating equation (3.17) we get

$$\dot{z}_1 = P_1 \dot{X} \quad \dots 3.18$$

On substituting for \dot{X} from equation (3.9) in equation (3.18) we get

$$\dot{z}_1 = P_1 \dot{X} = P_1 (AX + Bu) = P_1 AX + P_1 Bu$$

Since the transformed state variables are functions of state variables alone, the term $P_1 B$ will be zero (i.e., $P_1 B = 0$)

$$\therefore \dot{z}_1 = P_1 AX \quad \dots 3.19$$

We know that, $\dot{z}_1 = z_2$

$$\therefore z_2 = \dot{z}_1 = P_1 AX \quad \dots 3.20$$

On differentiating equation (3.20) we get

$$\dot{z}_2 = P_1 A \dot{X} \quad \dots 3.21$$

On substituting for \dot{X} for equation (3.9) in equation (3.21) we get,

$$\begin{aligned} \dot{z}_2 &= P_1 A (AX + Bu) = P_1 A^2 X + P_1 A B u \\ &= P_1 A^2 X \quad (\because P_1 A B = 0) \end{aligned}$$

We know that, $\dot{z}_2 = z_3$

$$\therefore z_3 = P_1 A^2 X \quad \dots 3.22$$

Similarly the k^{th} transformed state variable z_k can be expressed as

$$z_k = P_1 A^{(k-1)} X \quad \text{and} \quad P_1 A^{(k-2)} B = 0$$

Hence the n-numbers of transformed state variables can be expressed as shown below.

$$\begin{aligned}
\therefore z_1 &= P_1 X \\
z_2 &= P_1 A X \\
z_3 &= P_1 A^2 X \\
&\vdots \\
z_k &= P_1 A^{(k-1)} X \\
&\vdots \\
z_n &= P_1 A^{(n-1)} X
\end{aligned}$$

On arranging the above equations in the matrix form we get

$$\begin{bmatrix} z_1 \\ z_2 \\ z_3 \\ \vdots \\ z_n \end{bmatrix} = \begin{bmatrix} P_1 \\ P_1 A \\ P_1 A^2 \\ \vdots \\ P_1 A^{(n-1)} \end{bmatrix} X \quad \dots 3.23$$

Providing $P_1 B = P_1 A B = \dots = P_1 A^{(n-2)} B = 0$ and $P_1 A^{(n-1)} B = 1$.

The equation (3.23) is same as $Z = P_c X$ we can write,

$$P_c = \begin{bmatrix} P_1 \\ P_1 A \\ P_1 A^2 \\ \vdots \\ P_1 A^{(n-1)} \end{bmatrix} \quad \dots 3.24$$

On arranging the elements $P_1 B, P_1 A B, P_1 A^2 B, \dots, P_1 A^{(n-1)} B$ as column vector we get

$$\begin{aligned}
\begin{bmatrix} P_1 B & P_1 A B & P_1 A^2 B & \dots & P_1 A^{(n-2)} B & P_1 A^{(n-1)} B \end{bmatrix} &= \begin{bmatrix} 0 & 0 & 0 & \dots & 0 & 1 \end{bmatrix} \\
P_1 \begin{bmatrix} B & A B & A^2 B & \dots & A^{(n-2)} B & A^{(n-1)} B \end{bmatrix} &= \begin{bmatrix} 0 & 0 & 0 & \dots & 0 & 1 \end{bmatrix} \\
P_1 Q_c &= \begin{bmatrix} 0 & 0 & 0 & \dots & 0 & 1 \end{bmatrix} \\
P_1 &= \begin{bmatrix} 0 & 0 & 0 & \dots & 0 & 1 \end{bmatrix} Q_c^{-1} \quad \dots 3.25
\end{aligned}$$

$$\text{where, } Q_c = [B \ A B \ A^2 B \ \dots \ A^{(n-2)} B \ A^{(n-1)} B] \quad \dots 3.26$$

Using the equation (3.24), (3.25) and (3.26), the transformation matrix, P_c can be evaluated.

Alternate method to find transformation matrix, P_c

Let A be the system matrix of original state model. Now the characteristic equation governing the system is given by Eqn (3.27).

$$|\lambda I - A| = \lambda^n + a_1 \lambda^{n-1} + a_2 \lambda^{n-2} + \dots + a_{n-1} \lambda + a_n = 0 \quad \dots 3.27$$

Using the coefficients $a_1, a_2, \dots, a_{n-2}, a_{n-1}$ of characteristics equation [Eqn (3.27)] we can form a matrix, W as shown in Eqn (3.28).

$$W = \begin{bmatrix} a_{n-1} & a_{n-2} & \dots & a_2 & a_1 & 1 \\ a_{n-2} & a_{n-3} & \dots & a_1 & 1 & 0 \\ \vdots & \vdots & & \vdots & \vdots & \vdots \\ a_1 & 1 & \dots & 0 & 0 & 0 \\ 1 & 0 & \dots & 0 & 0 & 0 \end{bmatrix} \quad \dots 3.28$$

Now the transformation matrix, P_c is given by

$$P_c = (Q W)^{-1} \quad \dots 3.29$$

$$(\text{or}) \quad P_c^{-1} = (Q_c W) \quad \dots 3.30$$

Where, $Q = [B \ AB \ A^2B \ \dots \ A^{(n-2)}B \ A^{(n-1)}B]$

EXAMPLE 1.7

The state model of a system is given by

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 2 \\ 0 \end{bmatrix} [u] ; \quad y = [1 \ 0 \ 0] \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$$

Convert the state model to controllable phase variable form

SOLUTION

The given state model can be transformed to controllable phase variable form, only if the system is completely state controllable. Hence check for controllability.

Kalman's test for controllability

From the given state model we get,

$$A = \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix} \text{ and } B = \begin{bmatrix} 0 \\ 2 \\ 0 \end{bmatrix}$$

$$A^2 = A \cdot A = \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix} \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix} = \begin{bmatrix} 0 & 2 & -3 \\ 6 & 9 & -2 \\ -4 & -12 & 9 \end{bmatrix}$$

$$A.B = \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix} \begin{bmatrix} 0 \\ 2 \\ 0 \end{bmatrix} = \begin{bmatrix} 0 \\ -6 \\ 4 \end{bmatrix}$$

$$A^2.B = \begin{bmatrix} 0 & 2 & -3 \\ 6 & 9 & -2 \\ -4 & -12 & 9 \end{bmatrix} \begin{bmatrix} 0 \\ 2 \\ 0 \end{bmatrix} = \begin{bmatrix} 4 \\ 18 \\ -24 \end{bmatrix}$$

The composite matrix for controllability, $Q_c = [B \ AB \ A^2B] = \begin{bmatrix} 0 & 0 & 4 \\ 2 & -6 & 18 \\ 0 & 4 & -24 \end{bmatrix}$

Determinant of $Q_c = \Delta_{QC} = \begin{vmatrix} 0 & 0 & 4 \\ 2 & -6 & 18 \\ 0 & 4 & -24 \end{vmatrix} = 4 \times 8 = 32.$

Since, $\Delta_{QC} \neq 0$, the rank of $Q_c = 3$. Hence the system is completely state controllable.

To find transformation matrix P_c

The system state model can be converted to controllable phase variable form by choosing a transformation matrix, P_c .

Where $P_c = \begin{bmatrix} P_1 \\ P_1 A \\ P_1 A^2 \end{bmatrix}$ and $P_1 = [0 \ 0 \ 1] Q_c^{-1}$

$$Q_c^{-1} = \frac{[\text{Cofactor of } Q_c]^T}{\text{Determinant of } Q_c} = \frac{Q_{c, \text{cof}}^T}{\Delta_{QC}}$$

$$Q_{c, \text{cof}}^T = \begin{bmatrix} 72 & 48 & 8 \\ 16 & 0 & 0 \\ 24 & 8 & 0 \end{bmatrix}^T = \begin{bmatrix} 72 & 16 & 24 \\ 48 & 0 & 8 \\ 8 & 0 & 0 \end{bmatrix}$$

$$Q_c^{-1} = \frac{1}{32} \begin{bmatrix} 72 & 16 & 24 \\ 48 & 0 & 8 \\ 8 & 0 & 0 \end{bmatrix} = \begin{bmatrix} 2.25 & 0.5 & 0.75 \\ 1.5 & 0 & 0.25 \\ 0.25 & 0 & 0 \end{bmatrix}$$

$$P_1 = [0 \ 0 \ 1] Q_c^{-1} = [0 \ 0 \ 1] \begin{bmatrix} 2.25 & 0.5 & 0.75 \\ 1.5 & 0 & 0.25 \\ 0.25 & 0 & 0 \end{bmatrix} = [0.25$$

$$P_1 A = [0.25 \ 0 \ 0] \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix} = [0 \ 0 \ 0.25]$$

$$P_1 A^2 = [0.25 \ 0 \ 0] \begin{bmatrix} 0 & 2 & -3 \\ 6 & 9 & -2 \\ -4 & -12 & 9 \end{bmatrix} = [0 \ 0.5 \ -0.75]$$

$$\therefore \text{Transformation matrix, } P_c = \begin{bmatrix} P_1 \\ P_1 A \\ P_1 A^2 \end{bmatrix} = \begin{bmatrix} 0.25 & 0 & 0 \\ 0 & 0 & 0.25 \\ 0 & 0.5 & -0.75 \end{bmatrix}$$

To determine the controllable phase variable form of state model

The controllable phase variable form of state model is given by,

$$\dot{Z} = A_c Z + B_c u$$

$$y = C_c Z \quad (\text{Here } D \text{ is not given})$$

Where $A_c = P_c A P_c^{-1}$; $B_c = P_c B$ and $C_c = C P_c^{-1}$

$$\text{The transformation matrix, } P_c = \begin{bmatrix} 0.25 & 0 & 0 \\ 0 & 0 & 0.25 \\ 0 & 0.5 & -0.75 \end{bmatrix}$$

$$\therefore P_c^{-1} = \frac{[\text{Cofactor of } P_c]^T}{\text{Determinant of } P_c} = \frac{P_{c, \text{cof}}^T}{\Delta_{P_c}}$$

$$\Delta_{P_c} = \begin{vmatrix} 0.25 & 0 & 0 \\ 0 & 0 & 0.25 \\ 0 & 0.5 & -0.75 \end{vmatrix} = 0.25 \times (-0.5 \times 0.25) = -0.03125$$

$$P_{c, \text{cof}}^T = \begin{bmatrix} -0.125 & 0 & 0 \\ 0 & -0.1875 & -0.125 \\ 0 & -0.0625 & 0 \end{bmatrix}^T = \begin{bmatrix} -0.125 & 0 & 0 \\ 0 & -0.1875 & -0.0625 \\ 0 & -0.125 & 0 \end{bmatrix}$$

$$\therefore P_c^{-1} = \frac{1}{-0.03125} \begin{bmatrix} -0.125 & 0 & 0 \\ 0 & -0.1875 & -0.0625 \\ 0 & -0.125 & 0 \end{bmatrix} = \begin{bmatrix} 4 & 0 & 0 \\ 0 & 6 & 2 \\ 0 & 4 & 0 \end{bmatrix}$$

$$\begin{aligned} A_c = P_c A P_c^{-1} &= \begin{bmatrix} 0.25 & 0 & 0 \\ 0 & 0 & 0.25 \\ 0 & 0.5 & -0.75 \end{bmatrix} \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix} \begin{bmatrix} 4 & 0 & 0 \\ 0 & 6 & 2 \\ 0 & 4 & 0 \end{bmatrix} \\ &= \begin{bmatrix} 0 & 0 & 0.25 \\ 0 & 0.5 & -0.75 \\ -1 & -3 & 2.25 \end{bmatrix} \begin{bmatrix} 4 & 0 & 0 \\ 0 & 6 & 2 \\ 0 & 4 & 0 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -4 & -9 & -6 \end{bmatrix} \end{aligned}$$

$$B_c = P_c B = \begin{bmatrix} 0.25 & 0 & 0 \\ 0 & 0 & -0.25 \\ 0 & 0.5 & -0.75 \end{bmatrix} \begin{bmatrix} 0 \\ 2 \\ 0 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix}$$

$$C_c = C P_c^{-1} = [1 \ 0 \ 0] \begin{bmatrix} 4 & 0 & 0 \\ 0 & 6 & 2 \\ 0 & 4 & 0 \end{bmatrix} = [4 \ 0 \ 0]$$

The controllable phase variable form of state model is given by,

$$\begin{bmatrix} \dot{z}_1 \\ \dot{z}_2 \\ \dot{z}_3 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -4 & -9 & -6 \end{bmatrix} \begin{bmatrix} z_1 \\ z_2 \\ z_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} u$$

$$y = [4 \ 0 \ 0] \begin{bmatrix} z_1 \\ z_2 \\ z_3 \end{bmatrix}$$

Alternate method of find P_c

From the given state model we get,

$$A = \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix}$$

$$[\lambda I - A] = \lambda \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} - \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix} = \begin{bmatrix} \lambda & 0 & -1 \\ 2 & \lambda + 3 & 0 \\ 0 & -2 & \lambda + 3 \end{bmatrix}$$

$$|\lambda I - A| = \begin{vmatrix} \lambda & 0 & -1 \\ 2 & \lambda + 3 & 0 \\ 0 & -2 & \lambda + 3 \end{vmatrix} = \lambda(\lambda + 3)^2 - 1(-4) = \lambda(\lambda^2 + 6\lambda + 9) + 4$$

$$= \lambda^3 + 6\lambda^2 + 9\lambda + 4$$

The characteristic equation is $\lambda^3 + 6\lambda^2 + 9\lambda + 4 = 0$

The standard form of characteristic equation when $n = 3$ is given by

$$\lambda^3 + a_1\lambda^2 + a_2\lambda + a_3 = 0$$

On comparing the characteristic equation of the system with standard form we get,

$$a_1 = 6, \ a_2 = 9 \text{ and } a_3 = 4$$

$$\therefore W = \begin{bmatrix} a_2 & a_1 & 1 \\ a_1 & 1 & 0 \\ 1 & 0 & 0 \end{bmatrix} = \begin{bmatrix} 9 & 6 & 1 \\ 6 & 1 & 0 \\ 1 & 0 & 0 \end{bmatrix}$$

$$\therefore P_c^{-1} = Q_c W = \begin{bmatrix} 0 & 0 & 4 \\ 2 & -6 & 18 \\ 0 & 4 & -24 \end{bmatrix} \begin{bmatrix} 9 & 6 & 1 \\ 6 & 1 & 0 \\ 1 & 0 & 0 \end{bmatrix} = \begin{bmatrix} 4 & 0 & 0 \\ 0 & 6 & 2 \\ 0 & 4 & 0 \end{bmatrix}$$

$$\therefore P_c = (P_c^{-1})^{-1} = \begin{bmatrix} 4 & 0 & 0 \\ 0 & 6 & 2 \\ 0 & 4 & 0 \end{bmatrix}^{-1}$$

$$\text{Let } T = \begin{bmatrix} 4 & 0 & 0 \\ 0 & 6 & 2 \\ 0 & 4 & 0 \end{bmatrix} ; \therefore P_c = T^{-1}$$

$$T^{-1} = \frac{[\text{Cofactor of } T]^T}{\text{Determinant of } T} = \frac{T_{\text{cof}}^T}{\Delta_T} ; \Delta_T = \begin{vmatrix} 4 & 0 & 0 \\ 0 & 6 & 2 \\ 0 & 4 & 0 \end{vmatrix} = 4(-4 \times 2) = -32$$

$$T_{\text{cof}}^T = \begin{bmatrix} -8 & 0 & 0 \\ 0 & 0 & -16 \\ 0 & -8 & 24 \end{bmatrix}^T = \begin{bmatrix} -8 & 0 & 0 \\ 0 & 0 & -8 \\ 0 & -16 & 24 \end{bmatrix}$$

$$\therefore P_c = T^{-1} = \frac{T_{\text{cof}}^T}{\Delta_T} = \frac{1}{-32} \begin{bmatrix} -8 & 0 & 0 \\ 0 & 0 & -8 \\ 0 & -16 & 24 \end{bmatrix} = \begin{bmatrix} 0.25 & 0 & 0 \\ 0 & 0 & 0.25 \\ 0 & 0.5 & -0.75 \end{bmatrix}$$

1.8 CONTROL SYSTEM DESIGN VIA POLE PLACEMENT BY STATE FEEDBACK

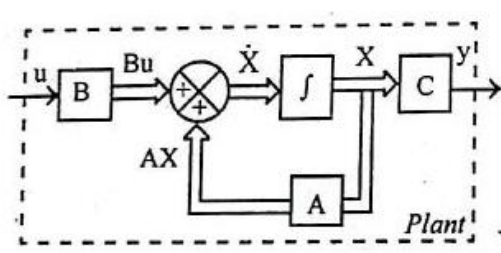
In the conventional approach to the design of a single-input, single-output control system, a controller or compensator is designed such that the dominant closed-loop poles have a desired damping ratio, ζ and undamped natural frequency, ω_n . In the compensated system the output alone is used as feedback signal to achieve desired performance. In state space design any inner parameter or variable of a system can be used for feedback. If the state variables (inner parameters or variables of the system) are used for feedback, then the system can be optimized for satisfying a desired performance index.

In control system design by pole placement or pole assignment technique, the state variables are used for feedback, to achieve desired closed loop poles. The advantage in this system is that the closed loop poles may be placed at any desired locations by means of state feedback through an appropriate state feedback gain matrix, K . The necessary and sufficient condition to be satisfied by the system for arbitrary pole placement is that the system be completely state controllable.

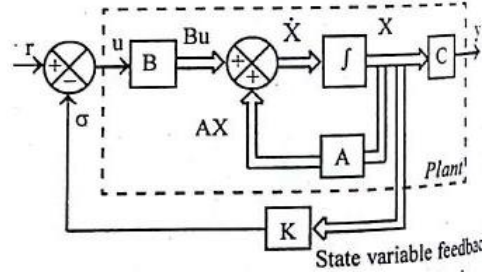
Consider the n^{th} order single – input single-output system with and without state variable feedback as shown in Figure 3.5. The state model of the system without state feedback is given by.

$$\dot{X} = AX + Bu \quad \dots 3.31$$

$$Y = CX \quad \dots 3.32$$



Figure(a) System without state feedback



Figure(b) System with state feedback

Figure 3.5 The n^{th} order single – input single – output system

Let r = System input when state variable feedback is employed.
 σ = Feedback signal obtained from state variables.
 U = Plant input.

The feedback signal, σ is obtained from state feedback and it is related to the state variables by the equation,

$$\sigma = KX \quad \dots 3.33$$

where K = State feedback gain matrix of order $(1 \times n)$ and

$$K = (k_1 \ k_2 \ k_3 \ \dots \ k_n) \quad \dots 3.34$$

In system employing state variable feedback, the plant input, u is the difference between system input, r and feedback input, σ .

$$\therefore \text{Plant input, } u = r - \sigma \quad \dots 3.35$$

On substituting, $\sigma = KX$ in equation (3.35) we get,

$$u = r - KX \quad \dots 3.36$$

The equation (3.36) is called control law.

The state equation of the system with state variable feedback is obtained by substituting the expression for u , from equation (3.36) in equation (3.31).

$$\begin{aligned} \therefore \dot{X} &= AX + Bu = AX + B(r - KX) \\ &= AX + Br - BKX = (A - BK)X + Br \end{aligned}$$

Therefore, the state model of the system with state variable feedback is given by the following equations [Eqn (3.32) and (3.33)].

$$\dot{X} = (A - BK)X + Br \quad \dots 3.32$$

$$y = CX \quad \dots 3.33$$

where, $K = [k_1, k_2, k_3 \dots k_n]$

and $r = u + KX$

This design technique starts with the determination of desired closed-loop poles to satisfy transient response and/or frequency response requirements. By choosing an appropriate gain matrix, K for state feedback, it is possible to force the system to have closed loop poles at the desired locations, provided that the original system is completely state controllable. In this design technique it is assumed that all state variable are measurable and are available for feedback.

DETERMINATION OF STATE FEEDBACK GAIN MATRIX, K

The state feedback gain matrix can be determined by three methods. In all the three methods, the system has to be first checked for complete state controllability.

The state model of the original n^{th} order system is given by

$$\dot{X} = AX + Bu$$

$$Y = CX$$

To check for controllability of original system, determine the composite matrix for controllability Q_c .

$$\text{Where, } Q_c = [B \ AB \ A^2B \ \dots \ A^{n-1}B]$$

Then calculate the determinant of Q_c . If the determinant of Q_c is not equal to zero, then the rank of Q_c is n and so the system is completely state controllable. (Here n is the order of the system). If the rank is not equal to n then arbitrary pole placement is not possible. When the system is completely state controllable any one of the following method can be used to find K .

METHOD – I

1. Determine the characteristic polynomial of original system. The characteristic polynomial is given by $|\lambda I - A| = 0$.

$$\text{Let, } |\lambda I - A| = \lambda^n + a_1 \lambda^{n-1} + a_2 \lambda^{n-2} + \dots + a_{n-1} \lambda + a_n.$$

2. Determine the desired characteristic polynomial from the specified closed loop poles. Let the specified or desired closed loop poles be $\mu_1, \mu_2, \mu_3, \dots, \mu_n$.

Now the desired characteristic polynomial is given by

$$(\lambda - \mu_1)(\lambda - \mu_2)(\lambda - \mu_3) \dots (\lambda - \mu_n) = \lambda^n + \alpha_1 \lambda^{n-1} + \alpha_2 \lambda^{n-2} + \dots + \alpha_{n-1} \lambda + \alpha_n.$$

3. Determine the transformation matrix, P_c which transforms the original state model to controllable phase variable form.

$$\text{The transformation matrix, } P_c = \begin{bmatrix} P_1 \\ P_1 A \\ \vdots \\ P_1 A^{n-1} \end{bmatrix}$$

and, $P_1 = [0 \ 0 \ \dots \ 0 \ 1] Q_c^{-1}$

4. Determine the state feedback gain matrix, from the following equation.

$$K = [\alpha_n - a_n \quad \alpha_{n-1} - a_{n-1} \quad \dots \quad \alpha_2 - a_2 \quad \alpha_1 - a_1] P_c.$$

Note: If the given system state modal is in controllable phase variable form then $P_c = 1$, unit matrix.

METHOD – II

1. Determine the characteristic polynomial of the system with state feedback, which is given by, $|\lambda I - (A - BK)| = 0$.

Here take, $K = [k_1, k_2, k_3 \dots k_n]$

$$\text{Let } |\lambda I - (A - BK)| = |\lambda I - A + BK| = \lambda^n + b_1 \lambda^{n-1} + b_2 \lambda^{n-2} + \dots + b_{n-1} \lambda + b_n.$$

The coefficients of this polynomial $b_1, b_2, b_3, \dots, b_n$ will be functions of $k_1, k_2, k_3, \dots, k_n$.

2. Determine the desired characteristic polynomial from the specified closed loop poles. Let the specified on desired closed loop poles be $\mu_1, \mu_2, \mu_3, \dots, \mu_n$. Now the desired characteristic polynomial is given by,

$$(\lambda - \mu_1)(\lambda - \mu_2)(\lambda - \mu_3) \dots (\lambda - \mu_n) = \lambda^n + \alpha_1 \lambda^{n-1} + \alpha_2 \lambda^{n-2} + \dots + \alpha_{n-1} \lambda + \alpha_n.$$

3. By equating the coefficients of polynomials obtained in step-1 and step-2, we get n-number of equations.

i.e., $b_1 = \alpha_1$; $b_2 = \alpha_2$; \dots ; $b_{n-1} = \alpha_{n-1}$ and $b_n = \alpha_n$.

On solving these equations we get the elements k_1, k_2, \dots, k_n of state feedback gain matrix, K .

Note: Method – II is suitable only for low values of n (i.e. for 2nd and 3rd order systems) otherwise calculations will be tedious.

METHOD – III

1. Determine the desired characteristic polynomial from the specified closed loop poles. Let the specified or desired closed loop poles be $\mu_1, \mu_2, \mu_3, \dots, \mu_n$.

Now the desired characteristic polynomial is given by,

$$(\lambda - \mu_1)(\lambda - \mu_2)(\lambda - \mu_3) \dots (\lambda - \mu_n) = \lambda^n + \alpha_1 \lambda^{n-1} + \alpha_2 \lambda^{n-2} + \dots + \alpha_{n-1} \lambda + \alpha_n.$$

2. Determine the matrix $\phi(A)$ using the coefficients of desired characteristic polynomial.

$$\phi(A) = A^n + \alpha_1 A^{n-1} + \alpha_2 A^{n-2} + \dots + \alpha_{n-1} A + \alpha_n I.$$

3. Calculate the state feedback gain matrix, K , using the Ackermann's formula given below.

$$K = [0 \ 0 \ \dots \ 0 \ 1] Q_c^{-1} \phi(A)$$

$$\text{Where, } Q_c = [B \ AB \ A^2B \ \dots \ A^{n-1}B]$$

EXAMPLE 1.8

Consider a linear system described by the transfer function

$$\frac{Y(s)}{U(s)} = \frac{10}{s(s+1)(s+2)}$$

Design a feedback controller with a state feedback so that the closed loop poles are placed at $-2, -1 \pm j1$

SOLUTION

To determine the state equation of the system

$$\text{Given that, } \frac{Y(s)}{U(s)} = \frac{10}{s(s+1)(s+2)} \quad \dots 3.8.1$$

On cross multiplying the equation (3.8.1) we get,

$$\begin{aligned} Y(s) [s(s+1)(s+2)] &= 10 U(s) \\ Y(s) [s(s^2+3s+2)] &= 10 U(s) \\ Y(s) [s^3+3s^2+2s] &= 10 U(s) \\ \therefore s^3 Y(s) + 3s^2 Y(s) + 2s Y(s) &= 10 U(s) \end{aligned} \quad \dots 3.8.2$$

On taking inverse laplace transform of equation (3.8.2) we get,

$$\ddot{y} + 3\ddot{y} + 2\dot{y} = 10u \quad \dots 3.8.3$$

Let us define state variables as follows,

$$x_1 = y; \ x_2 = \dot{y}; \ x_3 = \ddot{y}$$

$$\text{Put } \ddot{y} = \dot{x}_3; \ \ddot{y} = x_3; \ \dot{y} = x_2 \text{ and } y = x_1 \text{ in equation (3.8.3)}$$

$$\begin{aligned} \therefore \dot{x}_3 + 3x_3 + 2x_2 &= 10u \\ \text{or } \dot{x}_3 &= -2x_2 - 3x_3 + 10u \end{aligned}$$

The state equations governing the system are

$$\begin{aligned}\dot{x}_1 &= x_2 \\ \dot{x}_2 &= x_3 \\ \dot{x}_3 &= -2x_2 - 3x_3 + 10u\end{aligned}$$

The state equation in the matrix form is

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ 0 & -2 & -3 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 10 \end{bmatrix} u$$

...3.8.4

Check for controllability

$$\text{Given that, } A = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ 0 & -2 & -3 \end{bmatrix} \text{ and } B = \begin{bmatrix} 0 \\ 0 \\ 10 \end{bmatrix}$$

$$A^2 = A.A = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ 0 & -2 & -3 \end{bmatrix} \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ 0 & -2 & -3 \end{bmatrix} = \begin{bmatrix} 0 & 0 & 1 \\ 0 & -2 & -3 \\ 0 & 6 & 7 \end{bmatrix}$$

...3.8.5

$$AB = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ 0 & -2 & -3 \end{bmatrix} \begin{bmatrix} 0 \\ 0 \\ 10 \end{bmatrix} = \begin{bmatrix} 0 \\ 10 \\ -30 \end{bmatrix}$$

$$A^2B = \begin{bmatrix} 0 & 0 & 1 \\ 0 & -2 & -3 \\ 0 & 6 & 7 \end{bmatrix} \begin{bmatrix} 0 \\ 0 \\ 10 \end{bmatrix} = \begin{bmatrix} 10 \\ -30 \\ 70 \end{bmatrix}$$

$$\text{Composite matrix for controllability, } Q_c = [B \ AB \ A^2B] = \begin{bmatrix} 0 & 0 & 10 \\ 0 & 10 & -30 \\ 10 & -30 & 70 \end{bmatrix}$$

$$\text{Determinant of } Q_c = \Delta_{Q_c} = \begin{vmatrix} 0 & 0 & 10 \\ 0 & 10 & -30 \\ 10 & -30 & 70 \end{vmatrix} = 10(-10 \times 10) = -1000$$

Since, $\Delta_{Q_c} \neq 0$, the system is completely state controllable.

To find Q_c^{-1}

From equation (3.8.6) and (3.8.7) we get

$$Q_c = \begin{bmatrix} 0 & 0 & 10 \\ 0 & 10 & -30 \\ 10 & -30 & 70 \end{bmatrix} \text{ and } \Delta_{Q_c} = -1000.$$

$$Q_c^{-1} = \frac{[\text{cofactor of } Q_c]^T}{\text{Determinant of } Q_c} = \frac{1}{\Delta_{Q_c}} \begin{bmatrix} 610 & -300 & -100 \\ 300 & -100 & 0 \\ -100 & 0 & 0 \end{bmatrix}^T$$

$$= \frac{1}{-1000} \begin{bmatrix} 610 & 300 & -100 \\ -300 & -100 & 0 \\ -100 & 0 & 0 \end{bmatrix} = \begin{bmatrix} -0.61 & 0.3 & 0.1 \\ -0.3 & 0.1 & 0 \\ 0.1 & 0 & 0 \end{bmatrix}$$

To find desired characteristic polynomial

The desired closed loop poles are

$$\mu_1 = -2, \mu_2 = -1 + j1 \text{ and } \mu_3 = -1 - j1$$

Hence the desired characteristic polynomial is

$$\begin{aligned} (\lambda - \mu_1) (\lambda - \mu_2) (\lambda - \mu_3) &= (\lambda + 2) (\lambda + 1 - j1) (\lambda + 1 + j1) \\ &= (\lambda + 2) ((\lambda + 1)^2 - (j1)^2) \\ &= (\lambda + 2) (\lambda^2 + 2\lambda + 1 + 1) \\ &= (\lambda + 2) (\lambda^2 + 2\lambda + 2) \\ &= \lambda^3 + 2\lambda^2 + 2\lambda + 2\lambda^2 + 4\lambda + 4 \\ &= \lambda^3 + 4\lambda^2 + 6\lambda + 4 \end{aligned}$$

The desired characteristic polynomial is

$$\lambda^3 + 4\lambda^2 + 6\lambda + 4 = 0$$

...3.8.9

To determine the state variable feedback matrix, K

Method – I

Characteristic polynomial of original system is given by $|\lambda I - A| = 0$

$$[\lambda I - A] = \lambda \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} - \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ 0 & -2 & -3 \end{bmatrix} = \begin{bmatrix} \lambda & -1 & 0 \\ 0 & \lambda & -1 \\ 0 & 2 & \lambda + 3 \end{bmatrix}$$

$$|\lambda I - A| = \begin{vmatrix} \lambda & -1 & 0 \\ 0 & \lambda & -1 \\ 0 & 2 & \lambda + 3 \end{vmatrix} = \lambda [\lambda (\lambda + 3) + 2] = \lambda^2 (\lambda + 3) + 2\lambda = \lambda^3 + 3\lambda^2 + 2\lambda.$$

The characteristic polynomial of original system is,

$$\lambda^3 + 3\lambda^2 + 2\lambda = 0 \quad \dots 3.8.10$$

From Eqn (3.8.9) we get the desired characteristic polynomial as

$$\lambda^3 + 4\lambda^2 + 6\lambda + 4 = 0 \quad \dots 3.8.11$$

From equation (3.8.8.) we get,

$$Q_c^{-1} = \begin{bmatrix} -0.61 & 0.3 & 0.1 \\ -0.3 & 0.1 & 0 \\ 0.1 & 0 & 0 \end{bmatrix}$$

$$P_1 = [0 \ 0 \ 1] Q_c^{-1} = [0 \ 0 \ 1] \begin{bmatrix} -0.61 & 0.3 & 0.1 \\ -0.3 & 0.1 & 0 \\ 0.1 & 0 & 0 \end{bmatrix} = [0.1 \ 0 \ 0]$$

$$P_1 A = [0.1 \ 0 \ 0] \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ 0 & -2 & -3 \end{bmatrix} = [0 \ 0.1 \ 0]$$

$$P_1 A^2 = [0.1 \ 0 \ 0] \begin{bmatrix} 0 & 0 & 1 \\ 0 & -2 & -3 \\ 0 & 6 & 7 \end{bmatrix} = [0 \ 0 \ 0.1]$$

$$\therefore P_c = \begin{bmatrix} P_1 \\ P_1 A \\ P_1 A^2 \end{bmatrix} = \begin{bmatrix} 0.1 & 0 & 0 \\ 0 & 0.1 & 0 \\ 0 & 0 & 0.1 \end{bmatrix}$$

The state feedback gain matrix, $K = [\alpha_3 - a_3 \ \alpha_2 - a_2 \ \alpha_1 - a_1] P_c$

From equation (3.8.11) we get, $\alpha_3 = 4$; $\alpha_2 = 6$; $\alpha_1 = 4$

From equation (3.8.10) we get, $a_3 = 0$; $a_2 = 2$; $a_1 = 3$

$$\therefore K = [4-0 \ 6-2 \ 4-3] P_c$$

$$= [4 \ 4 \ 1] \begin{bmatrix} 0.1 & 0 & 0 \\ 0 & 0.1 & 0 \\ 0 & 0 & 0.1 \end{bmatrix} = [0.4 \ 0.4 \ 0.1]$$

Method – II

From the given state model we get,

$$A = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ 0 & -2 & -3 \end{bmatrix} \quad \text{and} \quad B = \begin{bmatrix} 0 \\ 0 \\ 10 \end{bmatrix}$$

Let, $K = [k_1 \ k_2 \ k_3]$

The characteristic polynomial of the system with state feedback is given by,

$$|\lambda I - (A - BK)| = |\lambda I - A + BK| = 0$$

$$\begin{aligned} [\lambda I - A + BK] &= \lambda \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} - \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ 0 & -2 & -3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 10 \end{bmatrix} \begin{bmatrix} k_1 & k_2 & k_3 \end{bmatrix} \\ &= \begin{bmatrix} \lambda & 0 & 0 \\ 0 & \lambda & 0 \\ 0 & 0 & \lambda \end{bmatrix} - \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ 0 & -2 & -3 \end{bmatrix} + \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & 0 \\ 10k_1 & 10k_2 & 10k_3 \end{bmatrix} \\ &= \begin{bmatrix} \lambda & -1 & 0 \\ 0 & \lambda & -1 \\ 10k_1 & 2 + 10k_2 & \lambda + 3 + 10k_3 \end{bmatrix} \end{aligned}$$

$$\begin{aligned} |\lambda I - A + BK| &= \begin{vmatrix} \lambda & -1 & 0 \\ 0 & \lambda & -1 \\ 10k_1 & 2 + 10k_2 & \lambda + 3 + 10k_3 \end{vmatrix} \\ &= \lambda [\lambda(\lambda + 3 + 10k_3) + 2 + 10k_2] + 1[10k_1] \\ &= \lambda^2(\lambda + 3 + 10k_3) + (2 + 10k_2)\lambda + 10k_1 \\ &= \lambda^3 + (3 + 10k_3)\lambda^2 + (2 + 10k_2)\lambda + 10k_1 \end{aligned}$$

The characteristic polynomial of the system with state feedback is

$$\lambda^3 + (3 + 10k_3)\lambda^2 + (2 + 10k_2)\lambda + 10k_1 = 0 \quad \dots 3.8.12$$

From equation (3.8.12) we get the desired characteristic polynomial as,

$$\lambda^3 + 4\lambda^2 + 6\lambda + 4 = 0 \quad \dots 3.8.13$$

On equating the coefficients of λ^0 term (constant) is equations (3.8.12) and (3.8.13) we get,

$$10k_1 = 4 \quad ; \quad \therefore k_1 = \frac{4}{10} = 0.4$$

On equating the coefficient of λ^1 term in equations (3.8.12) and (3.8.13) we get,

$$2 + 10k_3 = 6; \therefore k_3 = \frac{6-2}{10} = 0.4$$

On equating the coefficient of λ^2 term in equations (3.8.12) and (3.8.13) we get,

$$3 + 10k_3 = 4; \therefore k_3 = \frac{4-3}{10} = 0.1$$

The state feedback gain matrix, $K = [k_1 \ k_2 \ k_3] = [0.4 \ 0.4 \ 0.1]$

Method – III

From equation (3.8.9) we get the desired characteristic polynomial as,

$$\lambda^3 + 4\lambda^2 + 6\lambda + 4 = 0 \quad \dots 3.8.14$$

Here, $\phi(A) = A^3 + \alpha_1 A^2 + \alpha_2 A + \alpha_3 I$

From equation (3.8.14) we get, $\alpha_1 = 4; \alpha_2 = 6; \alpha_3 = 4$.

From the given state equation and equation (3.8.5) we get,

$$\begin{aligned} A &= \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ 0 & -2 & -3 \end{bmatrix} \text{ and } A^2 = \begin{bmatrix} 0 & 0 & 1 \\ 0 & -2 & -3 \\ 0 & 6 & 7 \end{bmatrix} \\ \therefore A^3 &= A^2 A = \begin{bmatrix} 0 & 0 & 1 \\ 0 & -2 & -3 \\ 0 & 6 & 7 \end{bmatrix} \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ 0 & -2 & -3 \end{bmatrix} = \begin{bmatrix} 0 & -2 & -3 \\ 0 & 6 & 7 \\ 0 & -14 & -15 \end{bmatrix} \\ \therefore \phi(A) &= A^3 + \alpha_1 A^2 + \alpha_2 A + \alpha_3 I \\ &= \begin{bmatrix} 0 & -2 & -3 \\ 0 & 6 & 7 \\ 0 & -14 & -15 \end{bmatrix} + 4 \begin{bmatrix} 0 & 0 & 1 \\ 0 & -2 & -3 \\ 0 & 6 & 7 \end{bmatrix} + 6 \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ 0 & -2 & -3 \end{bmatrix} + 4 \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} \\ &= \begin{bmatrix} 0 & -2 & -3 \\ 0 & 6 & 7 \\ 0 & -14 & -15 \end{bmatrix} + \begin{bmatrix} 0 & 0 & 4 \\ 0 & -8 & -12 \\ 0 & 24 & 28 \end{bmatrix} + \begin{bmatrix} 0 & 6 & 0 \\ 0 & 0 & 6 \\ 0 & -12 & -18 \end{bmatrix} + \begin{bmatrix} 4 & 0 & 0 \\ 0 & 4 & 0 \\ 0 & 0 & 4 \end{bmatrix} = \begin{bmatrix} 4 & 4 & 1 \\ 0 & 2 & 1 \\ 0 & -2 & -1 \end{bmatrix} \end{aligned}$$

$$\text{From equation (3.8.8) we get, } Q_c^{-1} = \begin{bmatrix} -0.61 & 0.3 & 0.1 \\ -0.3 & 0.1 & 0 \\ 0.1 & 0 & 0 \end{bmatrix}$$

From Ackermann's formula we get,

$$\begin{aligned}
K &= [0 \ 0 \ 1] Q_c^{-1} \phi(A) \\
&= [0 \ 0 \ 1] \begin{bmatrix} -0.61 & 0.3 & 0.1 \\ -0.3 & 0.1 & 0 \\ 0.1 & 0 & 0 \end{bmatrix} \begin{bmatrix} 4 & 4 & 1 \\ 0 & 2 & 1 \\ 0 & -2 & -1 \end{bmatrix} \\
&= [0.1 \ 0 \ 0] \begin{bmatrix} 4 & 4 & 1 \\ 0 & 2 & 1 \\ 0 & -2 & -1 \end{bmatrix} = [0.4 \ 0.4 \ 0.1]
\end{aligned}$$

The state feedback gain matrix $K = [0.4 \ 0.4 \ 0.1]$

Note: It is observed that the values of k_1, k_2, k_3 obtained by all the three methods are same. Because for a given set of poles the values of k_1, k_2, k_3, \dots will be unique.

EXAMPLE 1.9

A single input system is described by the following state equations.

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} -1 & 0 & 0 \\ 1 & -2 & 0 \\ 2 & 1 & -3 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 10 \\ 1 \\ 0 \end{bmatrix} u$$

Design a state feedback controller which will give closed-loop poles at $-1 \pm j2, -6$.

SOLUTION

Check for controllability

$$\text{Given that } A = \begin{bmatrix} -1 & 0 & 0 \\ 1 & -2 & 0 \\ 2 & 1 & -3 \end{bmatrix} \text{ and } B = \begin{bmatrix} 10 \\ 1 \\ 0 \end{bmatrix}$$

$$A^2 = A \cdot A = \begin{bmatrix} -1 & 0 & 0 \\ 1 & -2 & 0 \\ 2 & 1 & -3 \end{bmatrix} \begin{bmatrix} -1 & 0 & 0 \\ 1 & -2 & 0 \\ 2 & 1 & -3 \end{bmatrix} = \begin{bmatrix} 1 & 0 & 0 \\ -3 & 4 & 0 \\ -7 & -5 & 9 \end{bmatrix}$$

$$AB = \begin{bmatrix} -1 & 0 & 0 \\ 1 & -2 & 0 \\ 2 & 1 & -3 \end{bmatrix} \begin{bmatrix} 10 \\ 1 \\ 0 \end{bmatrix} = \begin{bmatrix} -10 \\ 8 \\ 21 \end{bmatrix}$$

$$A^2B = \begin{bmatrix} 1 & 0 & 0 \\ -3 & 4 & 0 \\ -7 & -5 & 9 \end{bmatrix} \begin{bmatrix} 10 \\ 1 \\ 0 \end{bmatrix} = \begin{bmatrix} 10 \\ -26 \\ -75 \end{bmatrix}$$

$$\left. \begin{array}{l} \text{Composite matrix} \\ \text{for controllability} \end{array} \right\} Q_c = [B \ AB \ A^2B] = \begin{bmatrix} 10 & -10 & 10 \\ 1 & 8 & -26 \\ 0 & 21 & -75 \end{bmatrix}$$

$$\text{Determinant of } Q_c = \Delta_{QC} = \begin{vmatrix} 10 & -10 & 10 \\ 1 & 8 & -26 \\ 0 & 21 & -75 \end{vmatrix}$$

$$\begin{aligned} \therefore \Delta_{QC} &= 10 [8 \times (-75) - 21 \times (-26)] + 10 [-75] + 10 [21] \\ &= -540 - 750 + 210 \\ &= -1080 \end{aligned}$$

Since, $\Delta_{QC} \neq 0$, The system is completely state controllable.

To find Q_c^{-1}

From equations (3.9.2) and (3.9.3) we get,

$$Q_c = \begin{bmatrix} 10 & -10 & 10 \\ 1 & 8 & -26 \\ 0 & 21 & -75 \end{bmatrix} \text{ and } \Delta_{QC} = -1080.$$

$$\begin{aligned} Q_c^{-1} &= \frac{[\text{cofactor of } Q_c]^T}{\text{Determinant of } Q_c} = \frac{1}{\Delta_{QC}} \begin{bmatrix} -54 & 75 & 21 \\ -540 & -750 & -210 \\ 180 & 270 & 90 \end{bmatrix}^T \\ &= \frac{1}{-1080} \begin{bmatrix} -54 & -540 & 180 \\ 75 & -750 & 270 \\ 21 & -210 & 90 \end{bmatrix} = \begin{bmatrix} 0.05 & 0.5 & -0.1667 \\ -0.0694 & 0.6944 & -0.25 \\ -0.0194 & 0.1944 & -0.0833 \end{bmatrix} \end{aligned}$$

...3.9.4

To find desired characteristic polynomial

The desired closed loop poles are,

$$+j2, \mu_2 = -1-j2 \text{ and } \mu_3 = -6$$

Hence the desired characteristic polynomial is,

$$\begin{aligned} (\lambda - \mu_1)(\lambda - \mu_2)(\lambda - \mu_3) &= (\lambda + 1 - j2)(\lambda + 1 + j2)(\lambda + 6) \\ &= ((\lambda + 1)^2 - (j2)^2)(\lambda + 6) \\ &= (\lambda^2 + 2\lambda + 1 + 4)(\lambda + 6) \\ &= (\lambda^2 + 2\lambda + 5)(\lambda + 6) \\ &= \lambda^3 + 2\lambda^2 + 5\lambda + 6\lambda^2 + 12\lambda + 30 \\ &= \lambda^3 + 8\lambda^2 + 17\lambda + 30 \end{aligned}$$

The desired characteristic polynomial is $\lambda^3 + 8\lambda^2 + 17\lambda + 30 = 0$

To determine the state variable feedback matrix, K

Method – I

The characteristics equation of original system is given by,

$$[\lambda I - A] = \lambda \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} - \begin{bmatrix} -1 & 0 & 0 \\ 1 & -2 & 0 \\ 2 & 1 & -3 \end{bmatrix} = \begin{bmatrix} \lambda+1 & 0 & 0 \\ -1 & \lambda+2 & 0 \\ -2 & -1 & \lambda+3 \end{bmatrix}$$

$$|\lambda I - A| = \begin{vmatrix} \lambda+1 & 0 & 0 \\ -1 & \lambda+2 & 0 \\ -2 & -1 & \lambda+3 \end{vmatrix} = (\lambda+1)(\lambda+2)(\lambda+3)$$

$$= (\lambda+1)(\lambda^2+5\lambda+6) = \lambda^3+5\lambda^2+6\lambda+\lambda^2+5\lambda+6 \\ = \lambda^3+6\lambda^2+11\lambda+6$$

The characteristic polynomial of original system is.

$$\lambda^3+6\lambda^2+11\lambda+6=0 \quad \dots 3.9.6$$

From equation (3.9.5) we get the desired characteristic polynomial as

$$\lambda^3+8\lambda^2+17\lambda+30=0 \quad \dots 3.9.7$$

From equation (3.9.4) we get,

$$Q_c^{-1} = \begin{bmatrix} 0.05 & 0.5 & -0.1667 \\ -0.0694 & 0.6944 & -0.25 \\ -0.0194 & 0.1944 & -0.0833 \end{bmatrix}$$

$$P_1 = [0 \ 0 \ 1] Q_c^{-1} = [0 \ 0 \ 1] \begin{bmatrix} 0.05 & 0.5 & -0.1667 \\ -0.0694 & 0.6944 & -0.25 \\ -0.0194 & 0.1944 & -0.0833 \end{bmatrix} \\ = [-0.0194 \ 0.1944 \ -0.0833]$$

$$P_1 A = [-0.0194 \ 0.1944 \ -0.0833] \begin{bmatrix} -1 & 0 & 0 \\ 1 & -2 & 0 \\ 2 & 1 & -3 \end{bmatrix} \\ = [0.0472 \ -0.4721 \ 0.2499]$$

$$P_1 A^2 = [-0.0194 \ 0.1944 \ -0.0833] \begin{bmatrix} 1 & 0 & 0 \\ -3 & 4 & 0 \\ -7 & -5 & 9 \end{bmatrix} \\ = [-0.0195 \ 1.1941 \ -0.7497]$$

$$P_C = \begin{bmatrix} P_1 \\ P_1 A \\ P_1 A^2 \end{bmatrix} = \begin{bmatrix} -0.0194 & 0.1944 & -0.0833 \\ 0.0472 & -0.4721 & 0.2499 \\ -0.0195 & 1.1941 & -0.7497 \end{bmatrix}$$

The state feedback gain matrix, $K = [\alpha_3 - a_3 \quad \alpha_2 - a_2 \quad \alpha_1 - a_1] P_C$

From equation (3.9.7) we get, $\alpha_3 = 30$; $\alpha_2 = 17$; $\alpha_1 = 8$

From equation (3.9.6) we get, $a_3 = 6$; $a_2 = 11$; $a_1 = 6$

$$\begin{aligned} \therefore K &= [30-6 \quad 17-11 \quad 8-6] P_C \\ &= [24 \quad 6 \quad 2] \begin{bmatrix} -0.0194 & 0.1944 & -0.0833 \\ 0.0472 & -0.4721 & 0.2499 \\ -0.0195 & 1.1941 & -0.7497 \end{bmatrix} \\ &= [-0.22 \quad 4.22 \quad -2] \end{aligned}$$

Method – II

From the given state model we get

$$A = \begin{bmatrix} -1 & 0 & 0 \\ 1 & -2 & 0 \\ 2 & 1 & -3 \end{bmatrix} \quad \text{and} \quad B = \begin{bmatrix} 10 \\ 1 \\ 0 \end{bmatrix}$$

$$\text{Let, } K = [k_1 \quad k_2 \quad k_3]$$

The characteristic polynomial of the systems with state feedback is given by,

$$|\lambda I - (A - BK)| = |\lambda I - A + BK| = 0$$

$$\begin{aligned} [\lambda I - A + BK] &= \lambda \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} - \begin{bmatrix} -1 & 0 & 0 \\ 1 & -2 & 0 \\ 2 & 1 & -3 \end{bmatrix} + \begin{bmatrix} 10 \\ 1 \\ 0 \end{bmatrix} [k_1 \quad k_2 \quad k_3] \\ &= \begin{bmatrix} \lambda & 0 & 0 \\ 0 & \lambda & 0 \\ 0 & 0 & \lambda \end{bmatrix} - \begin{bmatrix} -1 & 0 & 0 \\ 1 & -2 & 0 \\ 2 & 1 & -3 \end{bmatrix} + \begin{bmatrix} 10k_1 & 10k_2 & 10k_3 \\ k_1 & k_2 & k_3 \\ 0 & 0 & 0 \end{bmatrix} \\ &= \begin{bmatrix} \lambda + 1 + 10k_1 & 10k_2 & 10k_3 \\ -1 + k_1 & \lambda + 2 + k_2 & k_3 \\ -2 & -1 & \lambda + 3 \end{bmatrix} \end{aligned}$$

$$|\lambda I - A + BK| = \begin{vmatrix} \lambda + 1 + 10k_1 & 10k_2 & 10k_3 \\ -1 + k_1 & \lambda + 2 + k_2 & k_3 \\ -2 & -1 & \lambda + 3 \end{vmatrix}$$

$$\begin{aligned}
&= (\lambda + 1 + 10k_1) [(\lambda + 2 + k_2)(\lambda + 3) + k_3] - 10k_2 [(-1 + k_1)(\lambda + 3) + 2k_3] \\
&\quad + 10k_3 [(-1 + k_1) + 2(\lambda + 2 + k_2)] \\
&= [\lambda + (1 + 10k_1)] [\lambda^2 + 3\lambda + 2\lambda + 6 + \lambda k_2 + 3k_2 + k_3] \\
&\quad - 10k_2 [-\lambda - 3 + \lambda k_1 + 3k_1 + 2k_3] + 10k_3 [1 - k_1 + 2\lambda + 4 + 2k_2] \\
&= [\lambda + (1 + 10k_1)] [\lambda^2 + (5 + k_2)\lambda + (6 + 3k_2 + k_3)] \\
&\quad - 10k_2 [(-1 + k_1)\lambda + (-3 + 3k_1 + 2k_3)] + 10k_3 [2\lambda + (5 - k_1 + 2k_2)] \\
&= \lambda^3 + (5 + k_2)\lambda^2 + (6 + 3k_2 + k_3)\lambda \\
&\quad + (1 + 10k_1)\lambda^2 + (1 + 10k_1)(5 + k_2)\lambda + (1 + 10k_1)(6 + 3k_2 + k_3) \\
&\quad - 10k_2(-1 + k_1)\lambda - 10k_2(-3 + 3k_1 + 2k_3) + 20k_3\lambda + 10k_3(5 - k_1 + 2k_2) \\
&= \lambda^3 + (6 + 10k_1 + k_2)\lambda^2 + (6 + 3k_2 + k_3 + 5 + k_2 + 50k_1 + 10k_1k_2 \\
&\quad + 10k_2 - 10k_1k_2 + 20k_3)\lambda + (6 + 3k_2 + k_3 + 60k_1 + 30k_1k_2 \\
&\quad + 10k_1k_3 + 30k_2 - 30k_1k_2 - 20k_2k_3 + 50k_3 - 10k_1k_3 + 20k_2k_3) \\
&= \lambda^3 + (6 + 10k_1 + k_2)\lambda^2 + (11 + 50k_1 + 14k_2 + 21k_3)\lambda \\
&\quad + (6 + 60k_1 + 33k_2 + 51k_3)
\end{aligned}$$

The characteristic polynomial of system with state feedback is

$$\lambda^3 + (6 + 10k_1 + k_2)\lambda^2 + (11 + 50k_1 + 14k_2 + 21k_3)\lambda + (6 + 60k_1 + 33k_2 + 51k_3) = 0 \quad \dots 3.9.8$$

From equation (3.9.5) we get the desired characteristic polynomial as,

$$\lambda^3 + 8\lambda^2 + 17\lambda + 30 = 0 \quad \dots 3.9.9$$

On equating the coefficients of λ^2 term in equations (3.9.8) and (3.9.9) we get,

$$\begin{aligned}
6 + 10k_1 + k_2 &= 8 \\
10k_1 + k_2 &= 8 - 6 \\
\therefore 10k_1 + k_2 &= 2 \quad \dots 3.9.10
\end{aligned}$$

On equating the coefficients of λ^1 term in equations (3.9.8) and (3.9.9) we get,

$$\begin{aligned}
11 + 50k_1 + 14k_2 + 21k_3 &= 17 \\
50k_1 + 14k_2 + 21k_3 &= 17 - 11 \\
\therefore 50k_1 + 14k_2 + 21k_3 &= 6 \quad \dots 3.9.11
\end{aligned}$$

On equating the coefficients of λ^0 term (constant) in equation (3.9.8) and (3.9.9) we get,

$$6 + 60k_1 + 33k_2 + 51k_3 = 30$$

$$60k_1 + 33k_2 + 51k_3 = 30 - 6$$

$$\therefore 60k_1 + 33k_2 + 51k_3 = 24$$

...3.9.12

The equations (3.9.10), (3.9.11) and (3.9.12) can be arranged in the matrix form and k_1 , k_2 and k_3 are solved using cramer's rule as shown below.

$$\begin{bmatrix} 10 & 1 & 0 \\ 50 & 14 & 21 \\ 60 & 33 & 51 \end{bmatrix} \begin{bmatrix} k_1 \\ k_2 \\ k_3 \end{bmatrix} = \begin{bmatrix} 2 \\ 6 \\ 24 \end{bmatrix}$$

$$\Delta = \begin{vmatrix} 10 & 1 & 0 \\ 50 & 14 & 21 \\ 60 & 33 & 51 \end{vmatrix} = 10(14 \times 51 - 33 \times 21) - 1(50 \times 51 - 60 \times 21) \\ = 210 - 1290 = -1080$$

$$\Delta_1 = \begin{vmatrix} 2 & 1 & 0 \\ 6 & 14 & 21 \\ 24 & 33 & 51 \end{vmatrix} = 2(14 \times 51 - 33 \times 21) - 1(6 \times 51 - 24 \times 21) \\ = 42 + 198 = 240$$

$$\Delta_2 = \begin{vmatrix} 10 & 2 & 0 \\ 50 & 6 & 21 \\ 60 & 24 & 51 \end{vmatrix} = 10(6 \times 51 - 24 \times 21) - 2(50 \times 51 - 60 \times 21) \\ = -1980 - 2580 = -4560$$

$$\Delta_3 = \begin{vmatrix} 10 & 1 & 2 \\ 50 & 14 & 6 \\ 60 & 33 & 24 \end{vmatrix} = 10(14 \times 24 - 33 \times 6) - 1(50 \times 24 - 60 \times 6) \\ + 2(50 \times 33 - 60 \times 14) \\ = 1380 - 840 + 1620 = 2160$$

$$k_1 = \frac{\Delta_1}{\Delta} = \frac{240}{-1080} = -0.22$$

$$k_2 = \frac{\Delta_2}{\Delta} = \frac{-4560}{-1080} = 4.22$$

$$k_3 = \frac{\Delta_3}{\Delta} = \frac{2160}{-1080} = -2$$

The state feedback gain matrix, $K = [k_1 \ k_2 \ k_3] = [-0.22 \ 4.22 \ -2]$

Method – III

From equation (3.9.5) we get the desired characteristic polynomial as,

$$\lambda^3 + 8\lambda^2 + 17\lambda + 30 = 0 \quad \dots 3.9.13$$

Here, $\phi(A) = A^3 + \alpha_1 A^2 + \alpha_2 A + \alpha_3 I$

From equation (3.9.13) we get, $\alpha_1 = 8$; $\alpha_2 = 17$; $\alpha_3 = 30$

From the given state equation and equation (3.9.1) we get,

$$A = \begin{bmatrix} -1 & 0 & 0 \\ 1 & -2 & 0 \\ 2 & 1 & -3 \end{bmatrix} \text{ and } A^2 = \begin{bmatrix} 1 & 0 & 0 \\ -3 & 4 & 0 \\ -7 & -5 & 9 \end{bmatrix}$$

$$\therefore A^3 = A^2 \cdot A = \begin{bmatrix} 1 & 0 & 0 \\ -3 & 4 & 0 \\ -7 & -5 & 9 \end{bmatrix} \begin{bmatrix} -1 & 0 & 0 \\ 1 & -2 & 0 \\ 2 & 1 & -3 \end{bmatrix} = \begin{bmatrix} -1 & 0 & 0 \\ 7 & -8 & 0 \\ 20 & 19 & -27 \end{bmatrix}$$

$$\therefore \phi(A) = A^3 + \alpha_1 A^2 + \alpha_2 A + \alpha_3 I$$

$$= \begin{bmatrix} -1 & 0 & 0 \\ 7 & -8 & 0 \\ 20 & 19 & -27 \end{bmatrix} + 8 \begin{bmatrix} 1 & 0 & 0 \\ -3 & 4 & 0 \\ -7 & -5 & 9 \end{bmatrix} + 17 \begin{bmatrix} -1 & 0 & 0 \\ 1 & -2 & 0 \\ 2 & 1 & -3 \end{bmatrix} + 30 \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

$$= \begin{bmatrix} -1 & 0 & 0 \\ 7 & -8 & 0 \\ 20 & 19 & -27 \end{bmatrix} + \begin{bmatrix} 8 & 0 & 0 \\ -24 & 32 & 0 \\ -56 & -40 & 72 \end{bmatrix} + \begin{bmatrix} -17 & 0 & 0 \\ 17 & -34 & 0 \\ 34 & 17 & -51 \end{bmatrix} + \begin{bmatrix} 30 & 0 & 0 \\ 0 & 30 & 0 \\ 0 & 0 & 30 \end{bmatrix}$$

$$= \begin{bmatrix} 20 & 0 & 0 \\ 0 & 20 & 0 \\ -2 & -4 & 24 \end{bmatrix}$$

From equation (3.9.4) we get,

$$Q_C^{-1} = \begin{bmatrix} 0.05 & 0.5 & -0.1667 \\ -0.0694 & 0.6944 & -0.25 \\ -0.0194 & 0.1944 & -0.0833 \end{bmatrix}$$

From Ackermann's formula we get,

$$\begin{aligned} K &= [0 \ 0 \ 1] Q_C^{-1} \phi(A) \\ &= [0 \ 0 \ 1] \begin{bmatrix} 0.05 & 0.5 & -0.1667 \\ -0.0694 & 0.6944 & -0.25 \\ -0.0194 & 0.1944 & -0.0833 \end{bmatrix} \begin{bmatrix} 20 & 0 & 0 \\ 0 & 20 & 0 \\ -2 & -4 & 24 \end{bmatrix} \\ &= [-0.0194 \ 0.1944 \ -0.0833] \begin{bmatrix} 20 & 0 & 0 \\ 0 & 20 & 0 \\ -2 & -4 & 24 \end{bmatrix} = [-0.22 \ 4.22 \ -2] \end{aligned}$$

The state feedback gain matrix, $K = [-0.22 \ 4.22 \ -2]$

Note: The result obtained from all the three methods are same.

1.9 OBSERVABLE PHASE VARIABLE FORM OF STATE MODEL

An observable system can be represented by a modified state model called observable phase variable form by transforming the system matrix A into the transpose of bush of companion form as shown in equation (3.34)

$$A_o = \begin{bmatrix} 0 & 0 & \dots & 0 & -a_n \\ 1 & 0 & \dots & 0 & -a_{n-1} \\ \vdots & \vdots & & \vdots & \vdots \\ 0 & 0 & \dots & 1 & -a_1 \end{bmatrix} \quad \dots 3.34$$

Consider the state model of a n^{th} order system with single-input and single-output as shown below.

$$\dot{X} = AX + Bu \quad \dots 3.35$$

$$y = CX + Du \quad \dots 3.36$$

Let us choose a transformation $Z = P_o X$ to transform the state model of observable phase variable form.

Here, Z = Transformed state vector of order $(n \times 1)$

P_o = Transformed matrix of order $(n \times n)$

On premultiplying the equation, $Z = P_o X$ by P_o^{-1} we get,

$$P_o^{-1} Z = P_o^{-1} P_o X$$

$$\therefore X = P_o^{-1} Z$$

On differentiating the equation $X = P_o^{-1} Z$ we get,

$$\dot{X} = P_o^{-1} \dot{Z}$$

On substituting $X = P_o^{-1} Z$ and $\dot{X} = P_o^{-1} \dot{Z}$ in the state model [equations (3.35) and (3.36)] of the system we get,

$$P_o^{-1} \dot{Z} = A P_o^{-1} Z + B u \quad \dots 3.37$$

$$y = C P_o^{-1} Z + D u \quad \dots 3.38$$

On premultiplying the equation by P_o we get,

$$\begin{aligned} \dot{Z} &= P_o A P_o^{-1} Z + P_o B u \\ y &= C P_o^{-1} Z + D u \end{aligned}$$

Let $P_o A P_o^{-1} = A_o$; $P_o B = B_o$ and $C P_o^{-1} = C_o$,

$$\therefore \dot{Z} = A_o Z + B_o u \quad \dots 3.39$$

$$y = C_o Z + D u \quad \dots 3.40$$

The equation (3.39) and (3.40) are called observable phase variable form of state model of the system.

Note: In observable phase variable form of state model the matrix A.

$$B_o = \begin{bmatrix} \beta_r \\ \beta_{r-1} \\ \vdots \\ \beta_1 \end{bmatrix} ; \quad C_o = [0 \quad 0 \quad \dots \quad 0 \quad 1]$$

DETERMINATION OF THE TRANSFORMATION MATRIX P_o

Let A be the system matrix of original state model. Now the characteristic equation governing the system is given by equation (3.41).

$$[\lambda I - A] = \lambda^n + a_1 \lambda^{n-1} + a_2 \lambda^{n-2} + \dots + a_{n-1} \lambda + a_n = 0 \quad \dots 3.41$$

Using the coefficients $a_1, a_2, \dots, a_{n-2}, a_{n-1}$ of characteristic equation. [equation 3.41] we can form a matrix was shown in equation (3.42).

$$W = \begin{bmatrix} a_{n-1} & a_{n-2} & \dots & a_2 & a_1 & 0 \\ a_{n-2} & a_{n-3} & \dots & a_1 & 1 & 0 \\ \vdots & \vdots & & \vdots & \vdots & \vdots \\ a_1 & 1 & \dots & 0 & 0 & 0 \\ 1 & 0 & \dots & 0 & 0 & 0 \end{bmatrix} \quad \dots 3.42$$

Now the transformation matrix P_o is given by

$$P_o = W Q_o^T \quad \dots 3.43$$

Where $Q_o = [C^T \ A^T C^T \ (A^T)^2 C^T \ \dots \ (A^T)^{n-1} C^T]$

EXAMPLE 1.10

The state model of a system is given by

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 2 \\ 0 \end{bmatrix} [u] ; \quad y = [1 \ 0 \ 0] \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$$

Convert the state model to observable phase variable form.

SOLUTION

The given state model can be transformed to observable phase variable form, only if the system is completely observable. Hence check for observability.

Kalman's test for observability

From the given state model we get,

$$A = \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix} \text{ and } C = [1 \ 0 \ 0]$$

$$A^T = \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix}^T = \begin{bmatrix} 0 & -2 & 0 \\ 0 & -3 & 2 \\ 1 & 0 & -3 \end{bmatrix}$$

$$(A^T)^2 = \begin{bmatrix} 0 & -2 & 0 \\ 0 & -3 & 2 \\ 1 & 0 & -3 \end{bmatrix} \begin{bmatrix} 0 & -2 & 0 \\ 0 & -3 & 2 \\ 1 & 0 & -3 \end{bmatrix} = \begin{bmatrix} 0 & 6 & -4 \\ 2 & 9 & -12 \\ -3 & -2 & 9 \end{bmatrix}$$

$$C^T = [1 \ 0 \ 0]^T = \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix}$$

$$A^T C^T = \begin{bmatrix} 0 & -2 & 0 \\ 0 & -3 & 2 \\ 1 & 0 & -3 \end{bmatrix} \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix}; \quad (A^T)^2 C^T = \begin{bmatrix} 0 & 6 & -4 \\ 2 & 9 & -12 \\ -3 & -2 & 9 \end{bmatrix} \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix} = \begin{bmatrix} 0 \\ 2 \\ -3 \end{bmatrix}$$

The composite matrix for observability $\left\{ \begin{array}{l} Q_o = [C^T \ A^T C^T \ (A^T)^2 C^T] = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 0 & 2 \\ 0 & 1 & -3 \end{bmatrix} \end{array} \right.$

$$\text{Determinant of } Q_o = \Delta_{Q_o} = \begin{vmatrix} 1 & 0 & 0 \\ 0 & 0 & 2 \\ 0 & 1 & -3 \end{vmatrix} = 1(-2) = -2$$

Since $\Delta_{Q_o} \neq 0$, the rank of $Q_o = 3$. Hence the system is completely observable.

To find transformation matrix, P_o

From the given state model we get, $A = \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix}$

$$[\lambda I - A] = \lambda \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} - \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix} = \begin{bmatrix} \lambda & 0 & -1 \\ 2 & \lambda+3 & 0 \\ 0 & -2 & \lambda+3 \end{bmatrix}$$

$$|\lambda I - A| = \begin{vmatrix} \lambda & 0 & -1 \\ 2 & \lambda+3 & 0 \\ 0 & -2 & \lambda+3 \end{vmatrix} = \lambda(\lambda+3)^2 - 1(-4) = \lambda(\lambda^2 + 6\lambda + 9) + 4$$

$$= \lambda^3 + 6\lambda^2 + 9\lambda + 4$$

The characteristic equation is,

$$\lambda^3 + 6\lambda^2 + 9\lambda + 4 = 0$$

The standard form of characteristic equation when $n = 3$ is given by,

$$\lambda^3 + a_1\lambda^2 + a_2\lambda + a_3 = 0$$

On comparing the characteristic equation of the system with standard form we get,

$$a_1 = 6, \quad a_2 = 9 \quad \text{and} \quad a_3 = 4$$

$$\therefore W = \begin{bmatrix} a_2 & a_1 & 1 \\ a_1 & 1 & 0 \\ 1 & 0 & 0 \end{bmatrix} = \begin{bmatrix} 9 & 6 & 1 \\ 6 & 1 & 0 \\ 1 & 0 & 0 \end{bmatrix}$$

$$\therefore P_o = WQ_o^T = \begin{bmatrix} 9 & 6 & 1 \\ 6 & 1 & 0 \\ 1 & 0 & 0 \end{bmatrix} \begin{bmatrix} 1 & 0 & 0 \\ 0 & 0 & 2 \\ 0 & 1 & -3 \end{bmatrix}^T = \begin{bmatrix} 9 & 6 & 1 \\ 6 & 1 & 0 \\ 1 & 0 & 0 \end{bmatrix} \begin{bmatrix} 1 & 0 & 0 \\ 0 & 0 & 1 \\ 0 & 2 & -3 \end{bmatrix} = \begin{bmatrix} 9 & 2 & 3 \\ 6 & 0 & 1 \\ 1 & 0 & 0 \end{bmatrix}$$

$$P_o^{-1} = \frac{[\text{Cofactor of } P_o]^T}{\text{Determinant of } P_o} = \frac{P_{o, \text{cof}}^T}{\Delta_{P_o}}$$

$$\Delta_{P_o} = \begin{vmatrix} 9 & 2 & 3 \\ 6 & 0 & 1 \\ 1 & 0 & 0 \end{vmatrix} = -2(-1) = 2$$

$$P_{o, \text{cof}}^T = \begin{bmatrix} 0 & 1 & 0 \\ 0 & -3 & 2 \\ 2 & 9 & -12 \end{bmatrix}^T = \begin{bmatrix} 0 & 0 & 2 \\ 1 & -3 & 9 \\ 0 & 2 & -12 \end{bmatrix}$$

$$P_o^{-1} = \frac{1}{\Delta_{P_o}} P_{o, \text{cof}}^T = \frac{1}{2} \begin{bmatrix} 0 & 0 & 2 \\ 1 & -3 & 9 \\ 0 & 2 & -12 \end{bmatrix} = \begin{bmatrix} 0 & 0 & 1 \\ 0.5 & -1.5 & 4.5 \\ 0 & 1 & -6 \end{bmatrix}$$

To determine the observable phase variable form of state model

The observable phase variable form of state model is given by,

$$\dot{Z} = A_o Z + B_o u$$

$$Y = C_o Z$$

(Here D is not given)

Where, $A_o = P_o A P_o^{-1}$; $B_o = P_o B$ and $C_o = C P_o^{-1}$

$$P_o A P_o^{-1} = \begin{bmatrix} 9 & 2 & 3 \\ 6 & 0 & 1 \\ 1 & 0 & 0 \end{bmatrix} \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix} \begin{bmatrix} 0 & 0 & 1 \\ 0.5 & -1.5 & 4.5 \\ 0 & 1 & -6 \end{bmatrix}$$



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UNIT – IV – Advanced Control Systems – SEEA1602

SAMPLED DATA CONTROL SYSTEMS

4.1 INTRODUCTION

When the signal or information at any or some points in a system is in the form of discrete pulses, then the system is called discrete data system. In control engineering the discrete data system is popularly known as sampled data system.

The control system becomes a sampled data system in any one of the following situations.

1. When a digital computer or microprocessor or digital device is employed as a part of the control loop.
2. When the control components are used on time sharing basis.
3. When the control signals are transmitted by pulse modulation.
4. When the output or input of a component in the system is a digital or discrete signals.

The controllers are provided in control systems to modify the errors signal for better control action. If the controllers are constructed using analog elements then they are called analog controllers and their input and output are analog signals, which are continuous functions of time. The analog controllers are complex, costlier and once fabricated it is difficult to alter the controllers.

A digital controller can be employed to implement complex or time shared control functions. [In time shared controller, a single controller will perform more than one function]. The digital controller are simple, versatile, programmable, fast acting and less costlier than analog controllers.

The digital controller can be a special purpose computer (microprocessor based system) or a general purpose computer or it is constructed using non-programmable digital devices. When computer or microprocessor is involved then the controller becomes programmable and its easier to alter the control functions by modifying the program instructions.

A sampled-data control system using digital controller is shown in Figure 4.1. The input and output signal in a digital computer will be digital signals, but the error signal (input to the controller) to be modified by the controller and the control signal (output of the controller) to drive the plant are analog in nature. Hence a sampler and an analog-to-digital converter (ADC) are provided at the computer input. A digital to analog converter (DAC) and a hold circuit are provided at the computer output.

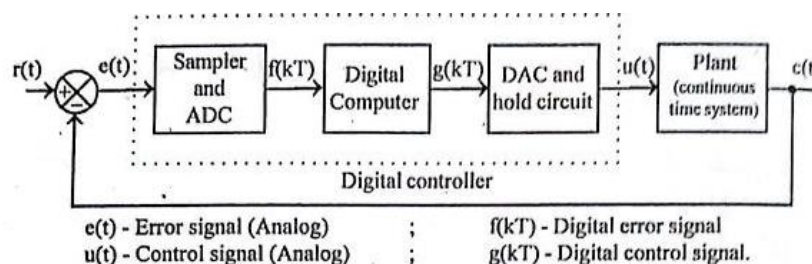


Figure 4.1 Sampled-data control system

The sampler converts the continuous time-error signal into a sequence of pulses and ADC produces a binary code (binary number) for each sample. These codes are the input data to the digital computer which processes the binary codes and produces another stream of binary codes as output. The DAC and hold circuit converts the output binary codes to continuous time signal (Analog signal) called control signal. This output control signal is used to drive the plant.

ADVANTAGES OF DIGITAL CONTROLLERS

1. The digital controllers can perform large and complex computation with any desired degree of accuracy at very high speed. In analog controllers the cost of controllers increases rapidly with the increase in complexity of computation and desired accuracy.
2. The digital controllers are easily programmable and so they are more versatile.
3. Digital controllers have better resolution.

ADVANTAGES OF SAMPLED DATA CONTROL SYSTEMS

1. The sampled data systems are highly accurate, fast and flexible.
2. Use of time sharing concept of digital computer results in economical cost and space.
3. Digital transducers used in the system have better resolution.
4. The digital components used in the system are less affected by noise, non linearities and transmission errors of noisy channel.
5. The sampled data system requires low power instruments which can be built to have high sensitivity.
6. Digital coded signals can be stored, transmitted, retransmitted, detected, analysed or processed as desired.
7. The system performance can be modified by compensation techniques.

4.2 SAMPLING PROCESS

Sampling is the conversion of a continuous-time signal (or analog signal) into a discrete-time signal obtained by taking samples of the continuous time signal (or analog signal) at discrete time instants. Thus if $f(t)$ is the input to the sampler as shown in Figure 4.2, the output is $f(kT)$ where T is called the sampling interval or sampling period. The reciprocal of T , i.e., $1/T = F_s$ is called the sampling rate (or samples per second or sampling frequency). This type of sampling is called periodic sampling, since samples are obtained uniformly at intervals of T seconds.

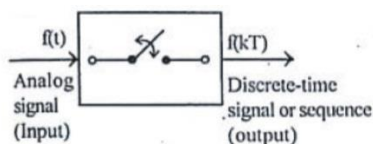


Fig. a Sampler

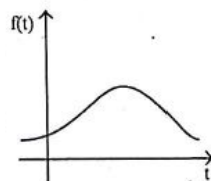


Fig.b Analog signal

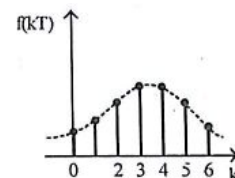


Fig.c Discrete signal or sequence

Figure 4.2 Periodic sampling of an analog signal

(In this book only periodic sampling of signals is considered, because periodic sampling is most widely used in practice. The other forms of sampling are multiple-order sampling, multiple-rate sampling and Random sampling.

Multiple-order sampling: A particular sampling pattern is repeated periodically.

Multiple-rate sampling: In this method two simultaneous sampling operations with different time periods are carried out on the signal to produce the sampled output.

Random sampling: In this case the sampling instants are random.

The sampling frequency $F_s (=1/T)$ must be selected large enough such that the sampling process will not result in any loss of spectral information. (i.e. if the spectrum of the analog signal can be recovered from the spectrum of the discrete – time signal, there is no loss of information). A guideline for choosing the sampling frequency is given sampling theorem given below.

SAMPLING THEOREM: A band limited continuous time signal with highest frequency (bandwidth) f_m hertz, can be uniquely recovered from its samples provided that the sampling rate F_s is greater than or equal to $2f_m$ samples per second.

From the sampling theorem we can infer that the knowledge of frequency content of a signal is essential while choosing the sampling frequency.

For processing the sampled signals by digital means, it has to be converted to binary codes and this conversion process is called quantization and coding. The process of converting a discrete time continuous valued signal into a discrete time discrete valued signal is called quantization. In quantization the value of each signal sample is represented by a value selected from a finite set of possible values called quantization levels. The difference between the unquantized sample and the quantized output is called the quantization error. The coding is the process of representing each discrete value by an n-bit binary sequence (or code or number). The process of sampling, quantization and coding are performed by sample/hold circuit and ADC.

1.3 ANALYSIS OF SAMPLING PROCESS IN FREQUENCY DOMAIN

The sampling process explained in the previous section is equivalent to multiplying the analog signal, $f(t)$ with a impulse train, $\delta_T(t)$ to produce the sampled signal, $f_s(t)$. Let the impulse train consists of pulses of area, Δ . Hence the impulse sampled signal, $f_s(t)$ can be expressed as,

$$f_s(t) = f(t) \Delta \delta_T(t) \quad \dots 4.1$$

Mathematically, the impulse train, $\delta_T(t)$ can be expressed as,

$$\delta_T(t) = \sum_{k=-\infty}^{+\infty} \delta(t - kT) \quad \dots 4.2$$

$$\therefore f_s(t) = \Delta f(t) \cdot \sum_{k=-\infty}^{+\infty} \delta(t - kT) \quad \dots 4.3$$

where T is the sampling period.

A typical analog signal, $f(t)$ [Fig a]; the impulse train, $\delta_T(t)$ [Fig b] and the impulse sampled signal, $f_s(t)$ [Fig c] are shown in Figure 4.3.

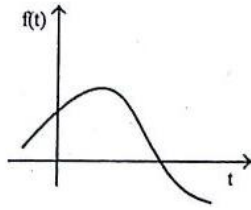


Fig.a. Analog signal

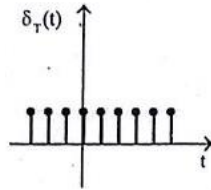


Fig.b. Impulse train

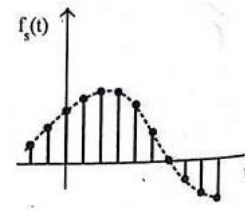


Fig.c. Impulse sampled analog signal

Figure 4.3 Impulse sampling of an analog signals

The frequency content (frequency response) of a signal can be obtained from the fourier transform of the signal [i.e., Fourier transform converts the time domain signal to frequency domain signal]. Hence the frequency response of the impulse sampled signal can be obtained by taking fourier transform of Eqn (4.3).

The fourier transform of a single-valued function, $f(t)$ is defined as

$$\mathcal{F}\{f(t)\} = F(\omega) = \int_{-\infty}^{\infty} f(t) e^{-j\omega t} dt \quad \dots 4.4$$

On taking fourier transform of $f_s(t)$ using the definition of fourier transform we get,

$$\begin{aligned} \mathcal{F}\{f_s(t)\} = F_s(\omega) &= \int_{-\infty}^{\infty} f_s(t) e^{-j\omega t} dt \\ &= \int_{-\infty}^{\infty} \Delta f(t) \sum_{k=-\infty}^{+\infty} \delta(t - kT) e^{-j\omega t} dt \end{aligned} \quad \dots 4.5$$

Mathematically the Eqn (4.5) represents, the convolution of two signals, $f(t)$ and $\delta(t - kT)$. The convolution theorem of fourier transform says that, the convolution of two time domain signals is equivalent to the product of their individual fourier transforms. Therefore, fourier transform of $f_s(t)$ can be expressed as a product of fourier transform of $f(t)$ and $\delta(t - kT)$.

$$\therefore F_s(\omega) = \frac{\Delta}{2\pi} \mathcal{F}\{f(t)\} \cdot \mathcal{F}\left\{\sum_{k=-\infty}^{+\infty} \delta(t - kT)\right\} \quad \dots 4.6$$

$$\text{Let, } \mathcal{F}\{f(t)\} = F(\omega) \quad \dots 4.7$$

$$\mathcal{F}\left\{\sum_{k=-\infty}^{+\infty} \delta(t - kT)\right\} = \omega_s \sum_{k=-\infty}^{+\infty} \delta(\omega - k\omega_s) \quad \dots 4.8$$

where, $\omega_s = 2\pi/T = \text{sampling frequency in rad/sec.}$

Using equations (4.7) and (4.8), the equ (4.6) can be written as,

$$F_s(\omega) = \frac{\Delta}{2\pi} \times F(\omega) \times \omega_s \sum_{k=-\infty}^{+\infty} \delta(\omega - k\omega_s) = \frac{\Delta}{2\pi} \frac{2\pi}{T} \sum_{k=-\infty}^{+\infty} F(\omega) \delta(\omega - k\omega_s)$$

Since $F(\omega) \delta(\omega - k\omega_s) = F(\omega - k\omega_s)$

$$F_s(\omega) = \frac{\Delta}{T} \sum_{k=-\infty}^{+\infty} F(\omega - k\omega_s) \quad \dots 4.9$$

The equation (4.9) gives the frequency spectrum of the impulse sampled signal.

Let $f(\omega)$ be a band-limited signal with a maximum frequency of ω_m . The frequency spectrum of $F(\omega)$ is shown in Figure 4.4(a), which is a plot of $|F(\omega)|$ Vs ω . The frequency spectrum of impulse sampled signal, i.e., $|F_s(\omega)|$ Vs ω , is shown in Figure 4.4(b), when $\omega_s > 2\omega_m$ and in Figure 4.4(c), when $\omega_s < 2\omega_m$.

In Figure 4.4(b) the frequency spectrum of original signal is repeated periodically with period ω_s and there is no overlapping of original spectrum. In Figure 4.4(c) the periodic repetition of original spectrum overlaps.

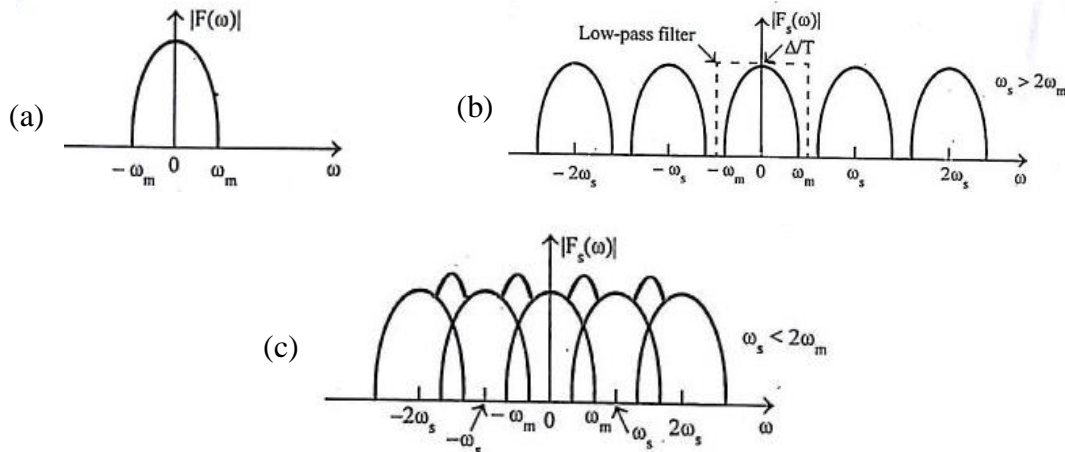


Figure 4.4 Fourier spectra of input signal and its impulse sampled version

From fig 4.4 it is observed that, as long as $\omega_s \geq \omega_m$, the original spectrum is preserved (since there is no overlapping) in the sampled signal and can be extracted from it by low-pass filtering. This fact was proposed as shanon's sampling theorem, which states that the information contained in a signal is fully preserved in the sampled version as long as the sampling frequency is at least twice the maximum frequency in the signal.

4.4 RECONSTRUCTION OF SAMPLED SIGNALS USING HOLD CIRCUITS

The hold circuits are popularly used in the process of analog-to-digital conversion (ADC) and digital-to-analog conversion (DAC). In ADC process the hold circuit is used to hold the sample until the quantization and coding for the current sample is complete.

In DAC process various types of hold circuits are used to convert the discrete time signal to analog signal. The simplest hold circuit is the zero order hold (ZOH). In zero order hold circuits the signal is reconstructed such that the value of reconstructed signal for a sampling period is same as the value of last received sample. The schematic diagram of sampler and zero order hold (ZOH) is shown in Fig 4.5. The signal reconstruction by zero order hold (ZOH) circuit is illustrated in Fig 4.6.

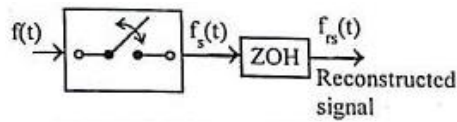


Figure 4.5 Sampler and ZOH

The high frequencies present in the reconstructed signal are easily filtered out by the various elements of the control system, because the control system is basically a low-pass filter.

In a first-order hold, the last two signal samples (current and previous sample) are used to reconstruct the signal for the current sampling period. Similarly higher order hold circuits can be devised. First or higher-order hold circuits offer no particular advantage over the zero order hold. In sampled data control systems, the zero-order hold when used in conjunction with a high sampling rate provides a satisfactory performance. An ideal sample / hold circuit introduces no distortion in the conversion process. However, in practical sample / hold circuits the following problems may be encountered.

1. Errors in the periodicity of sampling process.
2. Non linear variations in the duration of sampling aperture.
3. Droop (changes) in the voltage held during conversion.

4.5 DISCRETE SEQUENCE (DISCRETE TIME SIGNAL)

A discrete sequence or discrete time signal, $f(k)$, is a function of an independent variable, k , which is an integer. It is important to note that a Discrete time signal is not defined at instants between two successive samples. Also, it is incorrect to think that $f(k)$ is equal to zero if k is not an integer. Simply the signal $f(k)$ is not defined for non-integer values of k . A discrete-time signal is defined for every integer value of k in the range $-\infty < k < \infty$. Since a digital signal is represented by a set of numbers it is also called a sequence. (i.e., the terms signal and sequence refers the digital or discrete time signal).

METHODS OF REPRESENTING A DISCRETE TIME SIGNAL OR SEQUENCE

1. Functional representation

$$\begin{aligned} f(k) &= 1 && k = 1, 3 \\ &= 4 && k = 2 \\ &= 0 && \text{other } k \end{aligned}$$

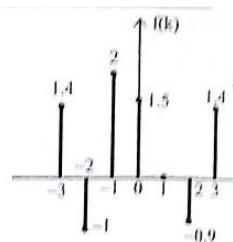


Figure 4.7 Graphical representation of a discrete time signal

2. Graphical representation

The graphical representation of a discrete sequence is shown in Figure 4.7.

3. Tabular representation

k	-2	-1	0	1	2
$f(k)$	0	6	0	1	4	..

4. Sequence representation

An infinite duration signal or sequence with the time origin ($k=0$) indicated by the symbol \uparrow is represented as

$$f(k) \{ \dots 1, 2, 1, 4, 1, 0, 0 \dots \}$$

\uparrow

An infinite sequence $f(k)$, which is zero for $k < 0$, may be represented as

$$f(k) = \{2, 1, 4, 1, 0, 0 \dots\} \text{ (or) } f(k) = \{2, 1, 4, 1 \dots\}$$

\uparrow

An finite duration sequence with the time origin ($k=0$), indicated by the symbol \uparrow is represented as

$$f(k) = \{3, -1, -2, 5, 0, 4 \dots\}$$

\uparrow

A finite duration sequence that satisfies the condition $f(k) = 0$ for $k < 0$ may be represented as

$$f(k) = \{2, 1, 4, 1\} \text{ (or) } f(k) = \{2, 1, 4, 1\}$$

\uparrow

SOME ELEMENTARY DISCRETE TIME SIGNALS

1. Digital impulse signal or unit sample sequence

$$\delta(k) = \begin{cases} 1 & ; \quad k = 0 \\ 0 & ; \quad k \neq 0 \end{cases}$$

An impulse delayed by k_0 ,

$$\delta_1(k) = \delta(k - k_0) = \begin{cases} 1 & ; \quad k = k_0 \\ 0 & ; \quad k \neq k_0 \end{cases}$$

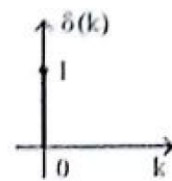


Figure 4.8 Digital impulse signal

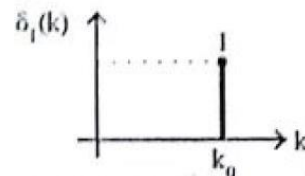


Figure 4.9 Delayed impulse signal

2. Unit step signal

$$u(k) = \begin{cases} 1 & ; \quad k \geq 0 \\ 0 & ; \quad k < 0 \end{cases}$$

An unit step signal delayed by k_0

$$u_1(k) = u(k - k_0) = \begin{cases} 0 & ; \quad k < k_0 \\ 1 & ; \quad k \geq k_0 \end{cases}$$

The unit step is related to digital impulse by the summation relation

$$u(k) = \sum_{m=0}^{\infty} \delta(k - m)$$

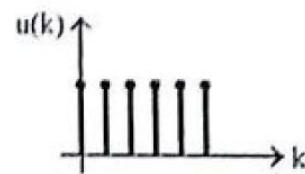


Figure 4.10 Unit step signal

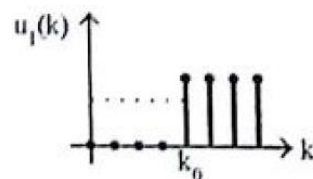


Figure 4.11 Delayed unit step signal

3. Ramp signal

$$u_r(k) = \begin{cases} k & ; k \geq 0 \\ 0 & ; k < 0 \end{cases}$$

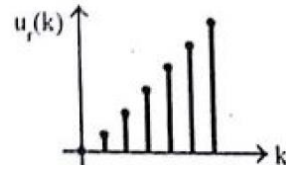


Figure 4.12 Ramp signal

4. Exponential signal

$$g(k) = \begin{cases} a^k & ; k \geq 0 \\ 0 & ; k < 0 \end{cases}$$

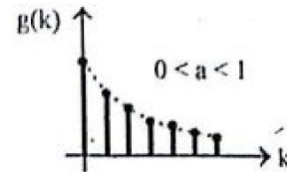


Figure 4.13 Exponential signal

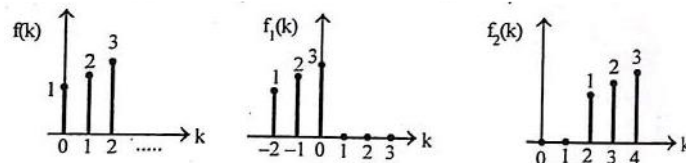
MATHEMATICAL OPERATIONS ON DISCRETE TIME SIGNALS

1. Shifting in time

A signal $f(k)$ may be shifted in time by replacing the independent variable k by $(k-m)$, where m is an integer. If m is a positive integer, the time shift results in a delay by m units of time. If m is a negative integer, the time shift results in an advance of the signal by $|m|$ units in time. The delay results in shifting each sample of $f(k)$ to right. The advance results in shifting each sample of $f(k)$ to left.

Example

$$\begin{array}{l} \text{Let, } f(k) = 1 \quad k=0 \\ \quad \quad \quad 2 \quad k=1 \\ \quad \quad \quad 3 \quad k=2 \end{array} \quad \begin{array}{l} \text{Now, } f_1(k) = f(k+2) = 1 \quad k=-2 \\ \quad \quad \quad 2 \quad k=-1 \\ \quad \quad \quad 3 \quad k=0 \end{array} \quad \begin{array}{l} \text{and } f_2(k) = f(k-2) = 1 \quad k=2 \\ \quad \quad \quad 2 \quad k=3 \\ \quad \quad \quad 3 \quad k=4 \end{array}$$

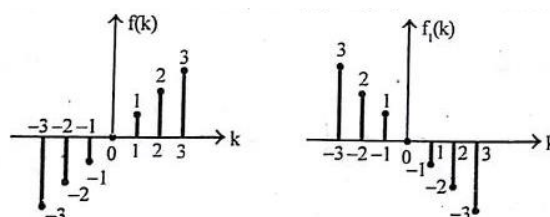


2. Folding or reflection or Transpose

The folding of a signal $f(k)$ is performed by changing the sign of the time base k in the signal $f(k)$. The folding operation produces a signal $f(-k)$ which is mirror image of $f(k)$ with respect to time origin $k=0$.

Example

$$\text{Let } f(k) = k \quad -3 \leq k \leq 3 ; f_1(k) = f(-k) = -k \quad -3 \leq k \leq 3$$



3. Amplitude scaling or scalar multiplication

Amplitude scaling of a signal by a constant A is accomplished by multiplying the value of every signal sample by A .

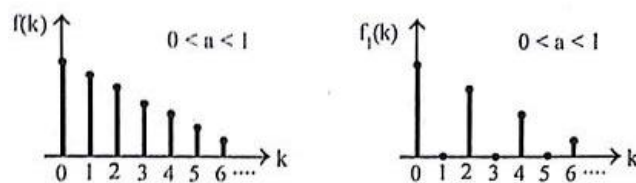
Let $c(k)$ be amplitude scaled signal of $f(k)$, then $c(k) = Af(k)$

$$\begin{array}{ll} \text{Let } f(k) = 20 & k = 0 \\ 36 & k = 1 \\ 40 & k = 2 \\ -15 & k = 3 \end{array} \quad \text{and } A = 0.1 \quad ; \quad \begin{array}{ll} c(k) = 2.0 & k = 0 \\ 3.6 & k = 1 \\ 4.0 & k = 2 \\ -1.5 & k = 3 \end{array}$$

4. Time scaling or down sampling

In a signal, $f(k)$, if k is replaced by μk , where μ is an integer, then it is called time scaling or down sampling.

Example: If $f(k) = a^k$; $k \geq 0$, then $f_1(k) = f(2k) = a^k$ for even values of k



5. Signal (or vector) addition

The sum of two signals $f_1(k)$ and $f_2(k)$ is a signal $c(k)$, whose value at any instant is equal to the sum of the samples of these two signals at that instant.

$$\text{i.e. } c(k) = f_1(k) + f_2(k); \quad -\infty < k < \infty.$$

Example

$$\text{Let } f_1(k) = \{1, 2, -1, 2\} \text{ and } f_2(k) = \{-2, 1, 3, 1\}$$

$$c(k) = f_1(k) + f_2(k) = \{-1, 3, 2, 3\}$$

6. Signal (or vector) multiplication

Signal multiplication results in the product of two signals on a sample-by-sample basis. The product of two signals $f_1(k)$ and $f_2(k)$ is a signal $c(k)$, whose value at any instant is equal to the product of the sample of these two signals at that instant. The product is also called modulation.

Example

$$\text{Let } f_1(k) = \{1, 2, -1, 2\} \text{ and } f_2(k) = \{-2, 1, 3, 1\}$$

$$c(k) = f_1(k) \cdot f_2(k) = \{-2, 2, -3, 2\}$$

1.6 z-TRANSFORM

Transform techniques are an important tool in the analysis of signals and linear time invariant systems. The Laplace transforms are popularly used for analysis of continuous time signals and systems. Similarly z-transform plays an important role in analysis and representation of linear discrete time systems. The z-transform provides a method for the analysis of discrete time systems in the frequency domain which is generally more efficient than its time domain analysis.

DEFINITION OF Z-TRANSFORM

Let, $f(k)$ = Discrete time signal or sequence

$$F(z) = z\{f(k)\} = \text{z-transform of } f(k)$$

The z-transform of a discrete time signal or sequence is defined as the power series

$$F(z) = \sum_{k=-\infty}^{\infty} f(k) z^{-k} \quad \dots 4.10$$

where z is a complex variable.

The sequence of equ (4.10) is considered to be two sided and the transform is called two sided z-transform, since the time index k is defined for both positive and negative values. If the sequence $f(k)$ is one sided sequence, (i.e. $f(k)$ is defined only for positive value of k) then the z-transform is called one sided z-transform.

The one sided z-transform of $f(k)$ is defined as,

$$F(z) = \sum_{k=0}^{\infty} f(k) z^{-k}$$

REGION OF CONVERGENCE

Since the z-transform is an infinite power series, it exists only for those values for z for which the series converges. The region of convergence, (ROC) of $F(z)$ is the set of all values of z for the which $F(z)$ attains a finite value. The ROC of a finite-duration signal is the entire z -plane, except possibly the point $z = 0$ and / or $z = \infty$. These points are excluded, because z^k (when $k > 0$) becomes unbounded for $z = \infty$ and z^{-k} (when $k > 0$) becomes unbounded for $z = 0$.

The complex variables z can be expressed in the polar form as,

$$Z = r e^{j\theta} \quad \dots 4.11$$

where $r = |z|$ and $\theta = \angle z$

On substituting for z from equ (4.11) in equ (4.10) we get,

$$F(z) = \sum_{k=-\infty}^{\infty} f(k) z^{-k} = \sum_{k=-\infty}^{\infty} f(k) (r e^{j\theta})^{-k} = \sum_{k=-\infty}^{\infty} f(k) r^{-k} e^{-j\theta k} \quad \dots 4.12$$

$$\text{Now, } |F(z)| = \sum_{k=-\infty}^{\infty} |f(k) r^{-k}|$$

In the ROC of $F(z)$, $|F(z)| < \infty$.

From equ (4.13) we observe that $|F(z)|$ is finite, if the sequence $f(k) r^{-k}$ is absolutely summable.

To find the ROC, the equ (4.13) can be expressed as,

$$\begin{aligned} |F(z)| &= \sum_{k=-\infty}^{\infty} |f(k) r^{-k}| = \sum_{k=-\infty}^{-1} |f(k) r^{-k}| + \sum_{k=0}^{\infty} |f(k) r^{-k}| \\ &= \sum_{k=1}^{\infty} |f(-k) r^k| + \sum_{k=0}^{\infty} \left| \frac{f(k)}{r^k} \right| \end{aligned} \quad \dots 4.14$$

If $F(z)$ converges in some region of the complex plane, both summations in equ (4.14) must be finite.

If the first sum of equ (4.14) converges, there must exist values of r small enough for $f(-k)r^k$ to be absolutely summable. Hence the ROC for the first sum consists of all points in a circle of radius, r_1 as shown in Figure 4.14, where $r_1 > r$.

If the second sum of equ (4.14) converges, there must exist large values of r for which $f(k) / r^k$ is absolutely summable. Hence the ROC for the second sum consists of all points in a circle of radius, r_2 as shown in Figure 4.15, where $r_2 < r$.

Therefore, the ROC of $F(z)$ is the region inbetween two circles of radius r_1 and r_2 as shown in Figure 4.16. where $r_2 < r < r_1$.

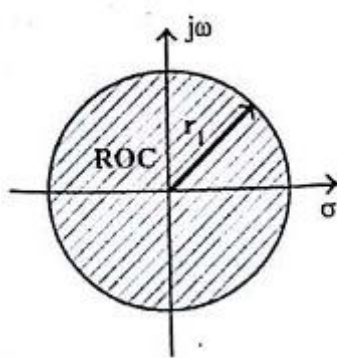


Figure 4.14 ROC for

$$\sum_{k=1}^{\infty} |f(-k) r^k|$$

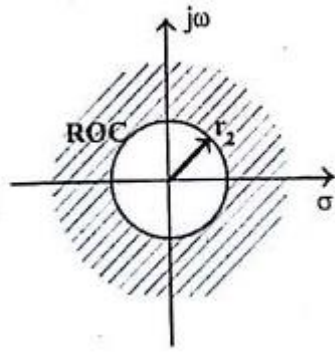


Figure 4.15 ROC for

$$\sum_{k=0}^{\infty} |f(k) / r^k|$$

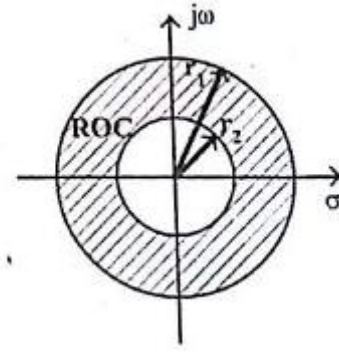


Figure 4.16 ROC for $F(z)$

Table 4.1 Characteristic families of signals with their corresponding ROC

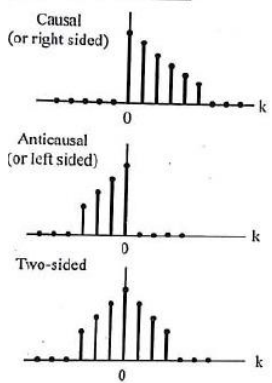
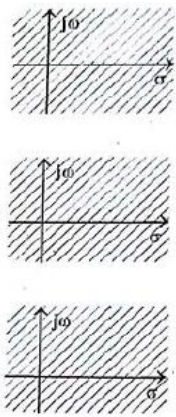
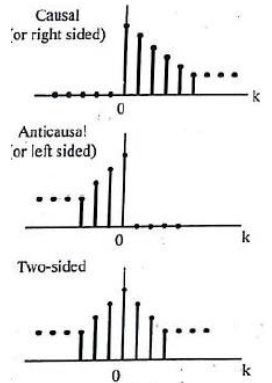
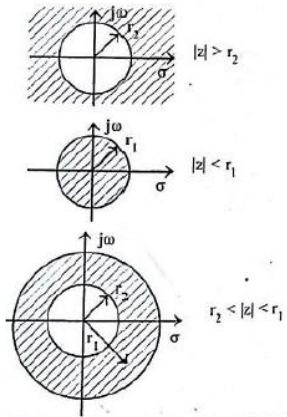
SIGNAL	ROC
Finite-Duration Signals 	 <p>Entire z-plane except $z = 0$</p> <p>Entire z-plane except $z = \infty$</p> <p>Entire z-plane except $z = 0$ and $z = \infty$</p>
Infinite-Duration Signals 	 <p>$z > r_2$</p> <p>$z < r_1$</p> <p>$r_2 < z < r_1$</p>

Table 4.2 Properties of one-sided Z-transform

Notations $F(z) = \mathcal{Z}\{f(k)\}$; $F_1(z) = \mathcal{Z}\{f_1(k)\}$; $F_2(z) = \mathcal{Z}\{f_2(k)\}$		
Property	Discrete sequence	z-transform
Linearity	$a_1 f_1(k) + a_2 f_2(k)$	$a_1 F_1(z) + a_2 F_2(z)$
Shifting, $m \geq 0$	$f(k+m)$ $f(k-m)$	$z^m F(z) - \sum_{i=0}^{m-1} f(i)z^{m-i}$ $z^{-m} F(z)$
Multiplication by k^m (or differentiation in z-domain)	$K^m f(k)$	$\left(-z \frac{d}{dz}\right)^m F(z)$
Scaling in z-domain (or multiplication by a^k)	$a^k f(k)$	$F(a^{-1}z)$
Time reversal	$f(-k)$	$F(z^{-1})$
Conjugation	$f^*(k)$	$F^*(z)$
Convolution	$\sum_{m=0}^k h(k-m) r(m)$	$H(z)R(z)$

Initial value	$f(0) = \lim_{z \rightarrow \infty} F(z)$	
Final value	$f(\infty) = \lim_{z \rightarrow 1} (1 - z^{-1}) F(z)$ $= \lim_{z \rightarrow 1} (z - 1) F(z)$ if $F(z)$ is analytic for $ z > 1$	

Table 4.3 Some Common one side Z-transform

$f(t) : t \geq 0$	$f(k)$ or $f(kT) ; k \geq 0$	$F(z)$
	$\delta(k)$	1
	$u(k)$ or 1	$z/(z-1)$
	a^k	$z/(z-a)$
	$k a^k$	$\frac{az(z+a)}{(z-a)^3}$
	$(k+i) a^k$	$\frac{z^2}{(z-a)^2}$
	$\frac{(k+1)(k+2)}{2!} a^k$	$\frac{z^3}{(z-a)^3}$
	$\frac{(k+1)(k+2)(k+3)}{3!} a^k$	$\frac{z^4}{(z-a)^4}$
	$\frac{a^k}{k!}$	$e^{az^{-1}}$
t	kT	$\frac{Tz}{(z-1)^2}$
t^2	$(kT)^2$	$\frac{T^2 z(z+1)}{(z-1)^3}$
e^{-at}	$kT e^{-aT}$	$\frac{z T e^{-aT}}{(z - e^{-aT})^2}$
$\sin \omega t$	$\sin \omega kT$	$\frac{z \sin \omega T}{z^2 - 2z \cos \omega T + 1}$
$\cos \omega t$	$\cos \omega kT$	$\frac{z(z - \cos \omega T)}{z^2 - 2z \cos \omega T + 1}$

Note: Two sided sequence can be converted to one sided sequence by multiplying by

GEOMETRIC SERIES

A geometric series is a series in which consecutive elements differ by a constant ratio. Such a series can be written in the form,

$$f(k) = \sum_{k=M_1}^{M_2} C \quad \dots 4.17$$

where C is a constant and M_1 and M_2 are any two numbers.

If C is a complex number, where $|C| < 1$, then by Taylor's series expansion we can write,

$$\frac{1}{1-C} = 1 + C + C^2 + \dots = \sum_{k=0}^{\infty} C^k \quad \dots 4.18$$

Applying the result in the reverse direction yields the infinite geometric series sum formula

$$\therefore \sum_{k=0}^{\infty} C^k = \frac{1}{1-C} \quad \dots 4.19$$

The equ (4.19) is the infinite geometric series sum formula.

We can also compute the sum of a finite number of elements in a geometric series. Let us consider the following sum,

$$1 + C + C^2 + \dots + C^{M-1} = \sum_{k=0}^{M-1} C^k \quad \dots 4.20$$

The sum of the finite duration sequence in equ (4.20) can be expressed as the difference between the sum of two infinite duration sequence as shown in equ (4.21).

$$\sum_{k=0}^{M-1} C^k = \sum_{k=0}^{\infty} C^k - \sum_{k=M}^{\infty} C^k \quad \dots 4.21$$

$$\begin{aligned} \text{Now, } \sum_{k=M}^{\infty} C^k &= C^M + C^{M+1} + C^{M+2} + \dots \\ &= C^M + C^M C + C^M C^2 + \dots = C^M (1 + C + C^2 + C^3 + \dots) \\ &= C^M \left(\sum_{k=0}^{\infty} C^k \right) \end{aligned} \quad \dots 4.22$$

From equations (4.21) and (4.22) we can write,

$$\begin{aligned} \sum_{k=0}^{M-1} C^k &= \sum_{k=0}^{\infty} C^k - C^M \sum_{k=0}^{\infty} C^k = (1 - C^M) \sum_{k=0}^{\infty} C^k \\ &= (1 - C^M) \left(\frac{1}{1-C} \right) = \frac{1 - C^M}{1 - C} = \frac{C^M - 1}{C - 1} \quad \text{except } C = 1 \end{aligned} \quad \dots 4.23$$

$$\text{when } C = 1, \sum_{k=0}^{M-1} C^k = M \quad \dots 4.24$$

The equation (4.23) and (4.24) are finite geometric series sum formula.

Note: The infinite geometric series sum formula requires that the magnitude of C be strictly less than unity, but the finite geometric series sum formula is valid for any value of C .

EXAMPLE 4.1

Determine the z-transform and their ROC of the following discrete sequence

$$(a) f(k) = \{3, 2, 5, 7\} \quad (b) f(k) = \{2, 4, 5, 7, 3\}$$

↑

SOLUTION

- (a) Given that, $f(k) = \{3, 2, 5, 7\}$
i.e., $f(0) = 3$; $f(1) = 2$; $f(2) = 5$; $f(3) = 7$
and $f(k) = 0$ for $k < 0$ and for $k > 3$
By the definition of z-transform

$$\mathcal{Z}\{f(k)\} = F(z) = \sum_{k=-\infty}^{\infty} f(k) z^{-k}$$

The given sequence is a finite duration sequence, hence the limits of summation can be changed as $k = 0$ to $k = 3$.

$$\therefore F(z) = \sum_{k=0}^3 f(k) z^{-k}$$

On expanding the summation we get,

$$\begin{aligned} F(z) &= f(0) z^0 + f(1) z^{-1} + f(2) z^{-2} + f(3) z^{-3} \\ &= 3 + 2z^{-1} + 5z^{-2} + 7z^{-3} \end{aligned}$$

Here $F(z)$ is bounded (i.e., finite) except when $z = 0$, therefore the ROC is entire z-plane except $z = 0$.

- (b) Given that, $f(k) = \{2, 4, 5, 7, 3\}$
↑
i.e., $f(-2) = 2$; $f(-1) = 4$; $f(0) = 5$; $f(1) = 7$; $f(2) = 3$
and $f(k) = 0$ for $k < -2$ and for $k > 2$
By the definition of z-transform

$$\mathcal{Z}\{f(k)\} = F(z) = \sum_{k=-\infty}^{\infty} f(k) z^{-k}$$

The given sequence is a finite duration sequence, hence the limits of summation can be changed as $k = -2$ to $k = 2$.

$$\therefore F(z) = \sum_{k=-2}^2 f(k) z^{-k}$$

On expanding the summation we get,

$$\begin{aligned} F(z) &= f(-2) z^2 + f(-1) z^1 + f(0) z^0 + f(1) z^{-1} + f(2) z^{-2} \\ &= 2z^2 + 4z + 5 + 7z^{-1} + 3z^{-2} \end{aligned}$$

Here $F(z)$ is bounded (i.e., finite) except when $z = 0$ and $z = \infty$, therefore the ROC is entire z -plane except $z = 0$ and $z = \infty$.

EXAMPLE 4.2

$$(a) f(k) = u(k) \quad (b) f(k) = (1/2)^k u(k) \quad (c) f(k) = a^k u(-k-1)$$

SOLUTION

(a) Given that, $f(k) = u(k)$

$u(k)$ is a discrete unit step sequence, which is defined as

$$u(k) = 1 \text{ for } k \geq 0 \\ = 0 \text{ for } k < 0$$

By the definition of z -transform,

$$\begin{aligned} \mathcal{Z}\{f(k)\} = F(z) &= \sum_{k=-\infty}^{\infty} f(k) z^{-k} \\ &= \sum_{k=0}^{\infty} u(k) z^{-k} = \sum_{k=0}^{\infty} z^{-k} = \sum_{k=0}^{\infty} (z^{-1})^k \end{aligned}$$

Here, $F(z)$ is an infinite geometric series and it converges if $|z| > 1$ (i.e., $|z^{-1}| < 1$). Using infinite geometric series sum formula we get,

$$F(z) = \frac{1}{1 - z^{-1}} = \frac{1}{1 - 1/z} = \frac{z}{z - 1}$$

(b) Given that, $f(k) = (1/2)^k u(k)$

$u(k)$ is a discrete unit step sequence, which is defined as

$$u(k) = 1 \text{ for } k \geq 0 \\ = 0 \text{ for } k < 0$$

$$\therefore f(k) = (1/2)^k \text{ for } k \geq 0 \\ = 0 \text{ for } k < 0$$

By the definition of z -transform,

$$\begin{aligned} \mathcal{Z}\{f(k)\} = F(z) &= \sum_{k=-\infty}^{\infty} f(k) z^{-k} \\ &= \sum_{k=0}^{\infty} (1/2)^k z^{-k} = \sum_{k=0}^{\infty} \left(\frac{1}{2} z^{-1}\right)^k \end{aligned}$$

Here, $F(z)$ is an infinite geometric series and it converges if $|z| > 1$ (i.e., $|z^{-1}| < 1$). Using infinite geometric series sum formula we get,

$$F(z) = \frac{1}{1 - \frac{1}{2} z^{-1}} = \frac{1}{1 - \frac{1}{2} \cdot \frac{1}{z}} = \frac{2z}{2z - 1}$$

- (c) Give that $f(k) = \alpha^k u(-k-1)$
 $u(-k-1)$ is a discrete unit step sequence, which is defined as
 $u(-k-1) = 0$ for $k \geq 0$
 $= 1$ for $k \leq -1$
 $\therefore f(k) = 0$ for $k \geq 0$
 $= \alpha^k$ for $k \leq -1$

By the definition of z-transform,

$$\begin{aligned} Z\{f(k)\} &= F(z) = \sum_{k=-\infty}^{\infty} f(k) z^{-k} \\ &= \sum_{k=-\infty}^{-1} (\alpha^k) z^{-k} = \sum_{k=1}^{\infty} \alpha^{-k} z^k \\ &= \sum_{k=1}^{\infty} (\alpha^{-1}z)^k = \sum_{k=0}^{\infty} (\alpha^{-1}z)^k - 1 \end{aligned}$$

Using infinite geometric series sum formula we get,

$$\begin{aligned} F(z) &= \frac{1}{1 - \alpha^{-1}z} - 1 = \frac{1}{1 - \frac{z}{\alpha}} - 1 = \frac{\alpha}{\alpha - z} - 1 \\ &= \frac{\alpha - \alpha + z}{\alpha - z} = \frac{z}{\alpha - z} \end{aligned}$$

EXAMPLE 4.3

Find the one sided z-transform of the following discrete sequences.

- (a) $f(k) = k a^{(k-1)}$ (b) $f(k) = k^2$

SOLUTION

- (a) Given that $f(k) = k a^{(k-1)}$

The one sided z-transform of a^k is given by

$$Z\{a^k\} = \sum_{k=0}^{\infty} a^k z^{-k} = \sum_{k=0}^{\infty} (az^{-1})^k \quad \dots 4.3.1$$

Using infinite geometric series sum formula,

$$Z\{a^k\} = \frac{1}{1 - az^{-1}} = \frac{1}{1 - a/z} = \frac{z}{z - a} \quad \dots 4.3.2$$

From equation (4.3.1) and (4.3.2) we get

$$\sum_{k=0}^{\infty} a^k z^{-k} = \frac{z}{z - a}$$

On expanding the summation in the above equation, we get,

$$1 + az^{-1} + a^2 z^{-2} + a^3 z^{-3} + \dots = \frac{z}{z - a} \quad \dots 4.3.3$$

On differentiating the equation (4.3.3) we get,

$$\begin{aligned} -az^{-2} - 2a^2z^{-3} - 3a^3z^{-4} - \dots &= \frac{(z-a) \times 1 - z \times 1}{(z-a)^2} \\ -az^{-2} - 2a^2z^{-3} - 3a^3z^{-4} - \dots &= \frac{-a}{(z-a)^2} \end{aligned} \quad \dots 4.3.4$$

On multiplying the equation (4.3.4) by $-(z/a)$ we get,

$$z^{-1} + 2az^{-2} + 3a^2z^{-3} + \dots = \frac{z}{(z-a)^2} \quad \dots 4.3.5$$

The infinite series on the left hand side on the equ (4.3.5) can be expressed as a simulation and the equ (4.3.5) is written as shown below.

$$\sum_{k=1}^{\infty} k a^{(k-1)} z^{-k} = \frac{z}{(z-a)^2} \quad \dots 4.3.6$$

By definition of z-transform, the one sided z-transform of $k a^{(k-1)}$ is given by,

$$\mathcal{Z}\{k a^{(k-1)}\} = \sum_{k=0}^{\infty} k a^{(k-1)} z^{-k} = \sum_{k=1}^{\infty} k a^{(k-1)} z^{-k} \quad \dots 4.3.7$$

(Because, $k a^{(k-1)} = 0$ when $k = 0$)

On comparing equations (4.3.6) and (4.3.7) we get,

$$\mathcal{Z}\{k a^{(k-1)}\} = \frac{z}{(z-a)^2}$$

(b) Given that, $f(k) = k^2$

Let us multiply the given discrete sequence by a discrete unit step sequence,

$$\therefore f(k) = k^2 u(k)$$

Note: Multiplying a one sided sequence by $u(k)$ will not alter its value.

By the property of z-transform, we get,

$$\mathcal{Z}\{k^m u(k)\} = \left(-z \frac{d}{dz}\right)^m U(z)$$

$$\text{where, } U(z) = \mathcal{Z}\{u(k)\} = \frac{z}{z-1}$$

$$\therefore -z \frac{d}{dz} U(z) = -z \left[\frac{d}{dz} \left(\frac{z}{z-1} \right) \right] = -z \left[\frac{z-1-z}{(z-1)^2} \right] = \frac{z}{(z-1)^2}$$

$$\left(-z \frac{d}{dz}\right)^2 U(z) = -z \frac{d}{dz} \left[-z \frac{d}{dz} U(z) \right]$$

$$\begin{aligned}
&= -z \frac{d}{dz} \left(\frac{z}{(z-1)^2} \right) = -z \left(\frac{(z-1)^2 - z \times 2(z-1)}{(z-1)^4} \right) \\
&= -z \left(\frac{(z-1)(z-1-2z)}{(z-1)^4} \right) = -z \left(\frac{-(z+1)}{(z-1)^3} \right) = \frac{z(z+1)}{(z-1)^3} \\
\therefore \mathcal{Z}\{f(k)\} = \mathcal{Z}\{k^2 u(k)\} &= \left(-z \frac{d}{dz} \right)^2 U(z) = \frac{z(z+1)}{(z-1)^3}
\end{aligned}$$

EXAMPLE 4.4

Find the one sided z-transform of the discrete sequence generated by mathematically sampling the following continuous time functions

- (a) t^2 (b) $\sin \omega t$ (c) $\cos \omega t$

SOLUTION

- (a) Given that, $f(t) = t^2$

The discrete sequence is generated by replacing t by kT , where T is the sampling time period.

$$\therefore f(k) = (kT)^2 = k^2 T^2 = k^2 g(k)$$

where, $g(k) = T^2$

By the definition of one sided z-transform we get,

$$G(z) = \mathcal{Z}\{g(k)\} = \mathcal{Z}\{T^2\} = \sum_{k=0}^{\infty} T^2 z^{-k} = T^2 \sum_{k=0}^{\infty} (z^{-1})^k = T^2 \left(\frac{1}{1-z^{-1}} \right) = \frac{T^2 z}{z-1}$$

By the property of z-transform we get,

$$\begin{aligned}
\mathcal{Z}\{f(k)\} = F(z) &= \left(-z \frac{d}{dz} \right)^2 G(z) = -z \frac{d}{dz} \left(-z \frac{d}{dz} G(z) \right) \\
&= -z \frac{d}{dz} \left(-z \frac{d}{dz} \frac{T^2 z}{z-1} \right) = -z \frac{d}{dz} \left(-z \times \frac{(z-1) T^2 - T^2 z}{(z-1)^2} \right) \\
&= -z \frac{d}{dz} \left(\frac{z T^2}{(z-1)^2} \right) = -z \times \frac{(z-1)^2 T^2 - z T^2 \times 2(z-1)}{(z-1)^4} \\
&= -z \times \frac{(z-1)(z T^2 - T^2 - 2z T^2)}{(z-1)^4} = -z \times \frac{-z T^2 - T^2}{(z-1)^3} = \frac{z T^2 (z+1)}{(z-1)^3}
\end{aligned}$$

- (b) Given that, $f(t) = \sin \omega t$

The discrete sequence is generated by replacing t by kT , where T is the sampling time period.

$$\therefore f(k) = \sin(\omega kT)$$

By the definition of one sided z-transform.

$$\mathcal{Z}\{f(k)\} = F(z) = \sum_{k=0}^{\infty} f(k) z^{-k} = \sum_{k=0}^{\infty} \sin \omega kT \times z^{-k}$$

We know that, $\sin \theta = (e^{j\theta} - e^{-j\theta})/2j$

$$\therefore F(z) = \sum_{k=0}^{\infty} \frac{e^{j\omega kT} - e^{-j\omega kT}}{2j} z^{-k} = \frac{1}{2j} \sum_{k=0}^{\infty} e^{j\omega kT} z^{-k} - \frac{1}{2j} \sum_{k=0}^{\infty} e^{-j\omega kT} z^{-k}$$

We know that, $\mathcal{Z}\{e^{\pm a k T}\} = \sum_{k=0}^{\infty} e^{\pm a k T} z^{-k} = \frac{z}{z - e^{\pm a T}}$

$$\begin{aligned} \therefore F(z) &= \frac{1}{2j} \frac{z}{z - e^{j\omega T}} - \frac{1}{2j} \frac{z}{z - e^{-j\omega T}} \\ &= \frac{z(z - e^{-j\omega T}) - z(z - e^{j\omega T})}{2j(z - e^{j\omega T})(z - e^{-j\omega T})} = \frac{z^2 - z e^{-j\omega T} - z^2 + z e^{j\omega T}}{2j(z^2 - z e^{-j\omega T} - z e^{j\omega T} + e^{j\omega T} \cdot e^{-j\omega T})} \\ &= \frac{z(e^{j\omega T} - e^{-j\omega T})/2j}{z^2 - z(e^{j\omega T} + e^{-j\omega T}) + 1} \end{aligned}$$

We know that, $\sin \theta = (e^{j\theta} - e^{-j\theta})/2j$ and $\cos \theta = (e^{j\theta} + e^{-j\theta})/2$

$$\therefore F(z) = \frac{z \sin \omega T}{z^2 - 2z \cos \omega T + 1}$$

(c) Given that, $f(t) = \cos \omega t$

The discrete sequence is generated by replacing t by kT , where T is the sampling time period.

$$\therefore f(k) = \cos(\omega k T)$$

By the definition of one sided z - transform,

$$\mathcal{Z}\{f(k)\} = F(z) = \sum_{k=0}^{\infty} f(k) z^{-k} = \sum_{k=0}^{\infty} \cos \omega k T \times z^{-k}$$

We know that, $\cos \theta = (e^{j\theta} + e^{-j\theta})/2$

$$\therefore F(z) = \sum_{k=0}^{\infty} \frac{e^{j\omega kT} + e^{-j\omega kT}}{2} z^{-k} = \frac{1}{2} \sum_{k=0}^{\infty} e^{j\omega kT} z^{-k} + \frac{1}{2} \sum_{k=0}^{\infty} e^{-j\omega kT} z^{-k}$$

We know that $\mathcal{Z}\{e^{\pm a k T}\} = \sum_{k=0}^{\infty} e^{\pm a k T} z^{-k} = \frac{z}{z - e^{\pm a T}}$

$$\begin{aligned} \therefore F(z) &= \frac{1}{2} \frac{z}{z - e^{j\omega T}} + \frac{1}{2} \frac{z}{z - e^{-j\omega T}} \\ &= \frac{z(z - e^{-j\omega T}) + z(z - e^{j\omega T})}{2(z - e^{j\omega T})(z - e^{-j\omega T})} = \frac{z^2 - z e^{-j\omega T} + z^2 - z e^{j\omega T}}{2(z^2 - z e^{-j\omega T} - z e^{j\omega T} + e^{j\omega T} \cdot e^{-j\omega T})} \\ &= \frac{2z^2 - z(e^{j\omega T} + e^{-j\omega T})}{2[z^2 - z(e^{j\omega T} + e^{-j\omega T}) + 1]} = \frac{z^2 - z(e^{j\omega T} + e^{-j\omega T})/2}{z^2 - z(e^{j\omega T} + e^{-j\omega T}) + 1} \end{aligned}$$

We know that, $\cos \theta = (e^{j\theta} - e^{-j\theta})/2$

$$\therefore F(z) = \frac{z(z - \cos \omega T)}{z^2 - 2z \cos \omega T + 1}$$

EXAMPLE 4.5

Find the one sided z-transform of the discrete sequence generated by mathematically sampling the following continuous time function,

(a) $e^{-at} \cos \omega t$ (b) $e^{-at} \sin \omega t$

SOLUTION

(a) Given that, $f(t) = e^{-at} \cos \omega t$

The discrete sequence is generated by replacing t by kT , where T is the sampling time period.

$$\therefore f(k) = e^{-at} \cos \omega kT$$

By the definition of one sided z-transform we get,

$$\begin{aligned} F(z) = \mathcal{Z}\{f(k)\} &= \sum_{k=0}^{\infty} e^{-akT} \cos \omega kT z^{-k} = \sum_{k=0}^{\infty} e^{-akT} \left(\frac{e^{j\omega kT} + e^{-j\omega kT}}{2} \right) z^{-k} \\ &= \frac{1}{2} \sum_{k=0}^{\infty} \left(e^{-aT} e^{j\omega T} z^{-1} \right)^k + \frac{1}{2} \sum_{k=0}^{\infty} \left(e^{-aT} e^{-j\omega T} z^{-1} \right)^k \end{aligned}$$

From infinite geometric sum series formula we know that, $\sum_{k=0}^{\infty} C^k = \frac{1}{1-C}$

$$\begin{aligned} \therefore F(z) &= \frac{1}{2} \frac{1}{1 - e^{-aT} e^{j\omega T} z^{-1}} + \frac{1}{2} \frac{1}{1 - e^{-aT} e^{-j\omega T} z^{-1}} \\ &= \frac{1}{2} \frac{1}{1 - e^{j\omega T} / z e^{+aT}} + \frac{1}{2} \frac{1}{1 - e^{-j\omega T} / z e^{+aT}} \\ &= \frac{1}{2} \left[\frac{z e^{aT}}{z e^{aT} - e^{j\omega T}} + \frac{z e^{aT}}{z e^{aT} - e^{-j\omega T}} \right] \\ &= \frac{1}{2} \left[\frac{z e^{aT} (z e^{aT} - e^{-j\omega T}) + z e^{aT} (z e^{aT} - e^{j\omega T})}{(z e^{aT} - e^{j\omega T})(z e^{aT} - e^{-j\omega T})} \right] \\ &= \frac{z e^{aT}}{2} \left[\frac{z e^{aT} - e^{-j\omega T} + z e^{aT} - e^{j\omega T}}{(z e^{aT})^2 - z e^{aT} e^{-j\omega T} - z e^{aT} e^{j\omega T} + e^{j\omega T} e^{-j\omega T}} \right] \\ &= \frac{z e^{aT}}{2} \left[\frac{2z e^{aT} - (e^{j\omega T} + e^{-j\omega T})}{z^2 e^{2aT} - z e^{aT} (e^{j\omega T} + e^{-j\omega T}) + 1} \right] \\ &= \left[\frac{z e^{aT} (z e^{aT} - \cos \omega T)}{z^2 e^{2aT} - 2z e^{aT} \cos \omega T + 1} \right] \quad \left(\because \cos \theta = \frac{e^{j\theta} + e^{-j\theta}}{2} \right) \end{aligned}$$

(b) Given that, $f(t) = e^{-at} \sin \omega t$

The discrete sequence $f(k)$ is generated by replacing t by kT , where T is the sampling time period.

$$\therefore f(k) = e^{-akt} \sin \omega kT$$

By the definition of one sided z-transform we get,

$$\begin{aligned} F(z) &= \mathcal{Z}\{f(k)\} = \sum_{k=0}^{\infty} e^{-akT} \sin \omega kT z^{-k} \\ &= \sum_{k=0}^{\infty} e^{-akT} \left(\frac{e^{j\omega kT} - e^{-j\omega kT}}{2j} \right) z^{-k} \\ &= \frac{1}{2j} \sum_{k=0}^{\infty} \left(e^{-aT} e^{j\omega T} z^{-1} \right)^k - \frac{1}{2j} \sum_{k=0}^{\infty} \left(e^{-aT} e^{-j\omega T} z^{-1} \right)^k \end{aligned}$$

From infinite geometric sum series formula we know that, $\sum_{k=0}^{\infty} C^k = \frac{1}{1-C}$

$$\begin{aligned} \therefore F(z) &= \frac{1}{2j} \frac{1}{1 - e^{-aT} e^{j\omega T} z^{-1}} - \frac{1}{2j} \frac{1}{1 - e^{-aT} e^{-j\omega T} z^{-1}} \\ &= \frac{1}{2j} \frac{1}{1 - e^{j\omega T} / z e^{aT}} - \frac{1}{2j} \frac{1}{1 - e^{-j\omega T} / z e^{aT}} \\ &= \frac{1}{2j} \frac{z e^{aT}}{z e^{aT} - e^{j\omega T}} - \frac{1}{2j} \frac{z e^{aT}}{z e^{aT} - e^{-j\omega T}} \\ &= \frac{1}{2j} \left[\frac{z e^{aT} (z e^{aT} - e^{-j\omega T}) - z e^{aT} (z e^{aT} - e^{j\omega T})}{(z e^{aT} - e^{j\omega T})(z e^{aT} - e^{-j\omega T})} \right] \\ &= \frac{1}{2j} \left[\frac{(z e^{aT}) [z e^{aT} - e^{-j\omega T} - z e^{aT} + e^{j\omega T}]}{(z e^{aT})^2 - z e^{aT} e^{-j\omega T} - z e^{aT} e^{j\omega T} + e^{j\omega T} e^{-j\omega T}} \right] \\ &= \left[\frac{z e^{aT} [e^{j\omega T} - e^{-j\omega T}]}{z^2 e^{2aT} - z e^{aT} (e^{j\omega T} + e^{-j\omega T}) + 1} \right] \\ &= \frac{z e^{aT} \sin \omega T}{z^2 e^{2aT} - 2z e^{aT} \cos \omega T + 1} \end{aligned}$$

INVERSE z-TRANSFORM

The following methods are employed to recover the original discrete sequence from its z-transform.

1. Direct evaluation by contour integration (or) complex inversion integral.
2. Partial fraction expansion.
3. Power series expansion.

The inverse z-transform by partial fraction expansion method and power series expansion method are presented in this section. The inverse z-transform by contour integration is beyond the scope of the book.

PARTIAL FRACTION EXPANSION METHOD

Let $f(k)$ = Discrete sequence

and $F(z) = Z\{f(k)\} = z$ -Transform of $f(k)$.

The function $F(z)$ can be expressed as a ratio of two polynomials in z as shown below.

$$F(z) = \frac{b_0 z^m + b_1 z^{m-1} + b_2 z^{m-2} + \dots + b_{m-1} z + b_m}{z^n + a_1 z^{n-1} + a_2 z^{n-2} + \dots + a_{n-1} z + a_n} ; \text{ where } m \leq n$$

The function $F(z)$ can be expressed as a series of sum terms by partial fraction expansion technique.

$$\therefore F(z) = A_0 + \sum_{i=1}^n \frac{A_i}{(z + p_i)} \quad \dots 4.25$$

where A_0 is a constant, A_1, A_2, \dots, A_n are residues and p_1, p_2, \dots, p_n are poles of $F(z)$.

Note: Sometimes it will be convenient to express $F(z)/z$ as a series of sum terms instead of $F(z)$.

Once the function $F(z)$ is expressed as a series of sum terms, the inverse z -transform of $F(z)$ is given by sum of inverse z -transform of each term in equ (4.25); [The inverse z -transform of each term of equ (4.25) can be obtained from standard z -transform pairs.

The coefficients of the polynomials of $F(z)$ are assumed real and so the roots of the polynomial are real and/or complex conjugate pairs) i.e., complex roots will occur only in conjugate pairs). Hence on factorizing the denominator polynomial we get the following cases. (The roots of the denominator polynomial are poles of $F(z)$).

Case (i) : When roots (or poles) are real and distinct

Case (ii) : When roots (or poles) have multiplicity

Case (iii) : When roots (or poles) are complex conjugate.

Case (i) : When roots (or poles) are real and distinct

In this case $F(z)$ can be expressed as,

$$F(z) = \frac{b_0 z^m + b_1 z^{m-1} + b_2 z^{m-2} + \dots + b_{m-1} z + b_m}{(z + p_1)(z + p_2) \dots (z + p_n)}$$

$$= A_0 + \frac{A_1}{(z + p_1)} + \frac{A_2}{(z + p_2)} + \dots + \frac{A_n}{(z + p_n)}$$

where A_0 is a constant ; A_1, A_2, \dots, A_n are residues and P_1, P_2, \dots, P_n are poles.

The constant A_0 is present when $m = n$ (i.e., when the order of numerator and denominator polynomial are equal). The value of A_0 is obtained by dividing the numerator polynomial by denominator polynomial.

The residue A_i is evaluated by multiplying both sides of $H(z)$ by $(z+p_i)$ and letting $z = -p_i$.

$$\therefore A_i = (z + p_i) F(z) \Big|_{z=-p_i}$$

Case (ii) When roots (or poles) have multiplicity

Let one of pole has a multiplicity of q . (i.e., repeats q times). In this case $F(z)$ can be expressed as,

$$\begin{aligned} F(z) &= \frac{b_0 z^m + b_1 z^{m-1} + b_2 z^{m-2} + \dots + b_{m-1} z + b_m}{(z + p_1)(z + p_2) \dots (z + p_x)^q \dots (z + p_n)} \\ &= A_0 + \frac{A_1}{(z + p_1)} + \frac{A_2}{(z + p_2)} + \dots + \frac{A_{x0}}{(z + p_x)^q} + \frac{A_{x1}}{(z + p_x)^{q-1}} + \dots \\ &\quad \dots + \frac{A_{x(q-1)}}{(z + p_x)} + \dots + \frac{A_n}{(z + p_n)} \end{aligned}$$

where $A_{x0}, A_{x1}, \dots, A_{x(q-1)}$ are residues of repeated root (or pole), $z = -p_x$.

The constant A_0 and residues of distinct real roots are evaluated as explained in case(i).

The residue A_{xr} of repeated root is obtained as shown below.

$$A_{xr} = \frac{1}{r!} \frac{d^r}{dz^r} [(z + p_x)^q F(z)] \Big|_{z=-p_x} ; \text{ where } r = 0, 1, 2, \dots, (q-1)$$

Case (iii) When roots (or poles) are complex conjugate

Let $F(z)$ has one pair of complex conjugate pole. In this case $F(z)$ can be expressed as,

$$\begin{aligned} F(z) &= \frac{b_0 z^m + b_1 z^{m-1} + b_2 z^{m-2} + \dots + b_{m-1} z + b_m}{(z + p_1)(z + p_2) \dots (z^2 + az + b) \dots (z + p_n)} \\ &= A_0 + \frac{A_1}{z + p_1} + \frac{A_2}{z + p_2} + \dots + \frac{A_x}{z + \sigma + j\omega} + \frac{A_x^*}{z + \sigma - j\omega} + \dots + \frac{A_n}{z + p_n} \end{aligned}$$

The constant A_0 and residues of real and non-repeated roots are evaluated as explained in case (i).

The residue A_x is evaluated as that of case(i) and the residue A_x^* is conjugate of A_x .

POWER SERIES EXPANSION METHOD

Let $f(k)$ = Discrete sequence

and $F(z) = Z\{f(k)\}$ = z-transform of $f(k)$.

By the definition of z-transform we get,

$$F(z) = \sum_{k=-\infty}^{\infty} f(k) z^{-k}$$

On expanding the summation we get,

$$F(z) = \dots f(-3) z^{-(-3)} + f(-2) z^{-(-2)} + f(-1) z^{-(-1)} + f(0) z^0 \\ + f(1) z^{-1} + f(2) z^{-2} + f(3) z^{-3} + \dots \quad \dots 4.26$$

In the given function, $F(z)$ can be expressed as a power series of z by long division then on comparing the coefficients of z with that of equ (4.26), the samples of $f(k)$ are determined. [i.e. the coefficient of z^1 is the i^{th} sample $f(i)$ of the sequence $f(k)$].

Note: The different method of evaluation of inverse z -transform of a function $F(z)$ will result in different type of mathematical expressions. But on evaluating the expressions for each value of k , we may get a same sequence.

EXAMPLE 4.6

Determine the inverse z -transform of the following function,

$$\begin{array}{ll} \text{(a)} & F(z) = \frac{1}{1 - 1.5z^{-1} + 0.5z^{-2}} \\ \text{(b)} & F(z) = \frac{z^2}{z^2 - z + 0.5} \\ \text{(c)} & F(z) = \frac{1 + z^{-1}}{1 - z^{-1} + 0.5z^{-2}} \\ \text{(d)} & F(z) = \frac{1}{(1 + z^{-1})(1 - z^{-1})^2} \end{array}$$

SOLUTION

(a) Given that, $F(z) = \frac{1}{1 - 1.5z^{-1} + 0.5z^{-2}}$

$$\begin{aligned} F(z) &= \frac{1}{1 - 1.5z^{-1} + 0.5z^{-2}} = \frac{1}{1 - \frac{1.5}{z} + \frac{0.5}{z^2}} \\ &= \frac{z^2}{z^2 - 1.5z + 0.5} = \frac{z^2}{(z-1)(z-0.5)} \\ \therefore \frac{F(z)}{z} &= \frac{z}{(z-1)(z-0.5)} \end{aligned}$$

By partial fraction expansion, $F(z) / z$ can be expressed as

$$\begin{aligned} \frac{F(z)}{z} &= \frac{A_1}{z-1} + \frac{A_2}{z-0.5} \\ A_1 &= \left. \frac{F(z)}{z} (z-1) \right|_{z=1} = \left. \frac{z}{(z-1)(z-0.5)} (z-1) \right|_{z=1} \\ &= \left. \frac{z}{(z-0.5)} \right|_{z=1} = \frac{1}{1-0.5} = 2 \\ A_2 &= \left. \frac{F(z)}{z} (z-0.5) \right|_{z=0.5} = \left. \frac{z}{(z-1)(z-0.5)} (z-0.5) \right|_{z=0.5} \\ &= \frac{0.5}{0.5-1} = -1 \\ \therefore \frac{F(z)}{z} &= \frac{2}{z-1} - \frac{1}{z-0.5} \\ F(z) &= \frac{2z}{z-1} - \frac{z}{z-0.5} \end{aligned}$$

We know that $\mathcal{Z}\{a^k\} = \frac{z}{z-a}$ and $\mathcal{Z}\{u(k)\} = \frac{z}{z-1}$

On taking inverse z-transform of F(z) we get,

$$f(k) = 2 u(k) + (0.5)^k; \quad k \geq 0$$

(Here we consider only one sided z-transform)

(b) Given that, $F(z) = \frac{z^2}{z^2 - z + 0.5}$

$$F(z) = \frac{z^2}{z^2 - z + 0.5} = \frac{z^2}{(z - 0.5 - j0.5)(z - 0.5 + j0.5)}$$

$$\therefore \frac{F(z)}{z} = \frac{z}{(z - 0.5 - j0.5)(z - 0.5 + j0.5)}$$

The roots of the quadratic

$$z^2 - z + 0.5 = 0 \text{ are}$$

$$z = \frac{1 \pm \sqrt{1 - 4 \times 0.5}}{2}$$

$$= 0.5 \pm j0.5$$

By partial fraction expansion, we can write,

$$\frac{F(z)}{z} = \frac{A}{z - 0.5 - j0.5} + \frac{A^*}{z - 0.5 + j0.5}$$

$$A = \frac{F(z)}{z} (z - 0.5 - j0.5) \Big|_{z=0.5+j0.5}$$

$$= \frac{z}{(z - 0.5 - j0.5)(z - 0.5 + j0.5)} (z - 0.5 - j0.5) \Big|_{z=0.5+j0.5}$$

$$\therefore A^* = (0.5 - j0.5)^* = 0.5 + j0.5$$

$$\therefore \frac{F(z)}{z} = \frac{0.5 - j0.5}{z - 0.5 - j0.5} + \frac{0.5 + j0.5}{z - 0.5 + j0.5}$$

$$F(z) = \frac{(0.5 - j0.5)z}{z - (0.5 + j0.5)} + \frac{(0.5 + j0.5)z}{z - (0.5 - j0.5)}$$

$$\mathcal{Z}\{a^k\} = \frac{z}{z-a}$$

We know that

On taking inverse z-transform of F(z) we get,

$$\begin{aligned} f(k) &= (0.5 - j0.5)(0.5 + j0.5)^k + (0.5 + j0.5)(0.5 - j0.5)^k \\ &= -j \left(\frac{0.5}{-j} + 0.5 \right) (0.5 + j0.5)^k + j \left(\frac{0.5}{j} + 0.5 \right) (0.5 - j0.5)^k \\ &= -j(0.5 + j0.5)(0.5 + j0.5)^k + j(0.5 - j0.5)(0.5 - j0.5)^k \\ &= -j(0.5 + j0.5)^{(k+1)} + j(0.5 - j0.5)^{(k+1)} \end{aligned}$$

(c) Given that, $F(z) = \frac{1+z^{-1}}{1-z^{-1}+0.5z^{-2}}$

$$F(z) = \frac{1+z^{-1}}{1-z^{-1}+0.5z^{-2}} = \frac{1+1/z}{1-\frac{1}{z}+\frac{0.5}{z^2}}$$

$$= \frac{\frac{z+1}{z}}{\frac{z^2-z+0.5}{z^2}} = \frac{z(z+1)}{(z^2-z+0.5)} = \frac{z(z+1)}{(z-0.5-j0.5)(z-0.5+j0.5)}$$

The roots of the quadratic

$$z^2 - z + 0.5 = 0 \text{ are}$$

$$z = \frac{1 \pm \sqrt{1-4 \times 0.5}}{2} \\ = 0.5 \pm j0.5$$

By partial fraction expansion, we can write.

$$\frac{F(z)}{z} = \frac{(z+1)}{(z-0.5-j0.5)(z-0.5+j0.5)} = \frac{A}{z-0.5-j0.5} + \frac{A^*}{z-0.5+j0.5}$$

$$A = \frac{F(z)}{z} (z-0.5-j0.5) \Big|_{z=0.5-j0.5}$$

$$= \frac{(z+1)}{(z-0.5-j0.5)(z-0.5+j0.5)} (z-0.5-j0.5) \Big|_{z=0.5-j0.5}$$

$$= \frac{0.5+j0.5+1}{0.5+j0.5-0.5+j0.5} = \frac{1.5+j0.5}{j1} = -j1.5+0.5 = 0.5-j1.5$$

$$A^* = (0.5-j1.5)^* = 0.5+j1.5$$

$$\therefore \frac{F(z)}{z} = \frac{0.5-j1.5}{z-0.5-j0.5} + \frac{0.5+j1.5}{z-0.5+j0.5}$$

$$F(z) = (0.5-j1.5) \frac{z}{z-(0.5-j0.5)} + (0.5+j1.5) \frac{z}{z-(0.5+j0.5)}$$

We know that $\mathcal{Z}\{a^k\} = \frac{z}{z-a}$

On taking inverse z-transform of F(z) we get,

$$f(k) = (0.5-j1.5)(0.5+j0.5)^k + (0.5+j1.5)(0.5-j0.5)^k ; \text{ for } k \geq 0$$

(d) Given that, $F(z) = \frac{1}{(1+z^{-1})(1-z^{-1})^2}$

$$F(z) = \frac{1}{(1+z^{-1})(1-z^{-1})^2} = \frac{1}{\left(1+\frac{1}{z}\right)\left(1-\frac{1}{z}\right)^2}$$

$$= \frac{1}{\frac{(z+1)}{z} \left(\frac{z-1}{z}\right)^2} = \frac{z^3}{(z+1)(z-1)^2}$$

$$\therefore \frac{F(z)}{z} = \frac{z^2}{(z+1)(z-1)^2}$$

By partial fraction expansion, we can write,

$$\begin{aligned} \frac{F(z)}{z} &= \frac{A_1}{z+1} + \frac{A_2}{(z-1)^2} + \frac{A_3}{z-1} \\ A_1 &= \left. \frac{F(z)}{z} (z+1) \right|_{z=-1} = \left. \frac{z^2}{(z+1)(z-1)^2} (z+1) \right|_{z=-1} = \left. \frac{z^2}{(z-1)^2} \right|_{z=-1} = \frac{(-1)^2}{(-1-1)^2} = 0.25 \\ A_2 &= \left. \frac{F(z)}{z} (z-1)^2 \right|_{z=1} = \left. \frac{z^2}{(z+1)(z-1)^2} (z-1)^2 \right|_{z=1} \\ &= \left. \frac{z^2}{z+1} \right|_{z=1} = \frac{1}{1+1} = 0.5 \\ A_3 &= \left. \frac{d}{dz} \left[\frac{F(z)}{z} (z-1)^2 \right] \right|_{z=1} = \left. \frac{d}{dz} \left[\frac{z^2}{(z+1)(z-1)^2} (z-1)^2 \right] \right|_{z=1} \\ &= \left. \frac{d}{dz} \left[\frac{z^2}{z+1} \right] \right|_{z=1} = \left. \frac{(z+1)2z - z^2}{(z+1)^2} \right|_{z=1} = \frac{(1+1) \times 2 - 1}{(1+1)^2} = \frac{3}{4} = 0.75 \\ \therefore \frac{F(z)}{z} &= \frac{0.25}{z+1} + \frac{0.5}{(z-1)^2} + \frac{0.75}{z-1} \end{aligned}$$

$$\begin{aligned} F(z) &= 0.25 \frac{z}{z+1} + 0.5 \frac{z}{(z-1)^2} + 0.75 \frac{z}{z-1} \\ &= 0.25 \frac{z}{z-(-1)} + 0.5 \frac{z}{(z-1)^2} + 0.75 \frac{z}{z-1} \end{aligned}$$

$$\text{We know that } \mathcal{Z}\{a^k\} = \frac{z}{z-a} ; \mathcal{Z}\left\{\frac{az}{(z-a)^2}\right\} = ka^k \text{ and } \mathcal{Z}\{u(k)\} = \frac{z}{z-1}$$

On taking inverse z-transform of F(z) we get,

$$\begin{aligned} f(k) &= 0.25(-1)^k + 0.5k(1)^k + 0.75 u(k) \\ f(k) &= 0.25(-1)^k + 0.5k + 0.75 u(k) ; \text{ for } k \geq 0 \end{aligned}$$

EXAMPLE 4.7

Determine the inverse z-transform of the following z-domain functions.

$$\begin{aligned} \text{(a)} \quad F(z) &= \frac{3z^2 + 2z + 1}{z^2 - 3z + 2} & \text{(b)} \quad F(z) &= \frac{3z^2 + 2z + 1}{z^2 + 3z + 2} \\ \text{(c)} \quad F(z) &= \frac{z - 0.4}{z^2 + z + 2} & \text{(b)} \quad F(z) &= \frac{z - 4}{(z-1)(z-2)^2} \end{aligned}$$

SOLUTION

(a) Given that, $F(z) = \frac{3z^2 + 2z + 1}{z^2 - 3z + 2}$

$$\begin{aligned} F(z) &= \frac{3z^2 + 2z + 1}{z^2 - 3z + 2} = 3 + \frac{11z - 5}{z^2 - 3z + 2} \\ &= 3 + \frac{11z - 5}{(z-1)(z-2)} \end{aligned}$$

$\begin{array}{r} 3 \\ z^2 - 3z + 2 \overline{) 3z^2 + 2z + 1} \\ \underline{3z^2 - 9z + 6} \\ 11z - 5 \end{array}$

By partial fraction expansion we get, $F(z) = 3 + \frac{A_1}{z-1} + \frac{A_2}{z-2}$

$$A_1 = \frac{11z-5}{(z-1)(z-2)} (z-1) \Big|_{z=1} = \frac{11z-5}{(z-2)} \Big|_{z=1} = \frac{11-5}{1-2} = -6$$

$$A_2 = \frac{11z-5}{(z-1)(z-2)} (z-2) \Big|_{z=2} = \frac{11z-5}{(z-1)} \Big|_{z=2} = \frac{11 \times 2 - 5}{2-1} = 17$$

$$\begin{aligned} \therefore F(z) &= 3 - \frac{6}{z-1} + \frac{17}{z-2} \\ &= 3 - 6 \frac{1}{z} \frac{z}{z-1} + 17 \frac{1}{z} \frac{z}{z-2} \\ &= 3 - 6 z^{-1} \frac{z}{z-1} + 17 z^{-1} \frac{z}{z-2} \end{aligned}$$

We know that, $\mathcal{Z}\{\delta(k)\} = 1$; $\mathcal{Z}\{u(k)\} = \frac{z}{z-1}$ and $\mathcal{Z}\{a^k\} = \frac{z}{z-a}$

By time shifting property we get,

$$\mathcal{Z}\{u(k-1)\} = z^{-1} \frac{z}{z-1} \text{ and } \mathcal{Z}\{a^{(k-1)}\} = z^{-1} \frac{z}{z-a}$$

On taking inverse z-transform of $F(z)$ we get,

$$f(k) = 3 \delta(k) - 6 u(k-1) + 17 \times 2^{(k-1)} u(k-1) ; \text{ for } k \geq 0$$

Note: The term $2^{(k-1)}$ is multiplied by $u(k-1)$, because this term have samples only after $k \geq 1$.

(b) Given that, $F(z) = \frac{3z^2 + 2z + 1}{z^2 + 3z + 2}$

$$\begin{aligned} F(z) &= \frac{3z^2 + 2z + 1}{z^2 + 3z + 2} = 3 - \frac{7z+5}{z^2 + 3z + 2} \\ &= 3 - \frac{7z+5}{(z+1)(z+2)} \end{aligned}$$

$\begin{array}{r} 3 \\ \hline z^2 + 3z + 2 \overline{) 3z^2 + 2z + 1} \\ \underline{3z^2 + 9z + 6} \\ -7z - 5 \end{array}$
--

By partial fraction expansion we get, $F(z) = 3 - \frac{A_1}{z+1} - \frac{A_2}{z+2}$

$$A_1 = \frac{7z+5}{(z+1)(z+2)} (z+1) \Big|_{z=-1} = \frac{7z+5}{(z+2)} \Big|_{z=-1} = \frac{7 \times (-1) + 5}{-1+2} = -2$$

$$A_2 = \frac{7z+5}{(z+1)(z+2)} (z+2) \Big|_{z=-2} = \frac{7z+5}{z+1} \Big|_{z=-2} = \frac{7 \times (-2) + 5}{-2+1} = 9$$

$$\begin{aligned} \therefore F(z) &= 3 + \frac{2}{z+1} - \frac{9}{z+2} \\ &= 3 + 2 \frac{1}{z} \frac{z}{z-(-1)} - 9 \frac{1}{z} \frac{z}{z-(-2)} \\ &= 3 + 2z^{-1} \frac{z}{z-(-1)} - 9z^{-1} \frac{z}{z-(-2)} \end{aligned}$$

We know that, $\mathcal{Z}\{\delta(k)\} = 1$ and $\mathcal{Z}\{a^k\} = \frac{z}{z-a}$

By time shifting property,

$$\mathcal{Z}\{a^{(k-1)}\} = z^{-1} \frac{z}{z-a}$$

On taking inverse z-transform of F(z) we get,

$$f(k) = 3\delta(k) + 2(-1)^{(k-1)} u(k-1) + 9(-2)^{(k-1)} u(k-1) ; \text{ for } k \geq 0$$

Note: The term $d^{(k-1)}$ is multiplied by $u(k-1)$, because these terms have samples only after $k \geq 1$.

(c) Given that, $F(z) = \frac{z-0.4}{z^2+z+2}$

$$\begin{aligned} F(z) &= \frac{z-0.4}{z^2+z+2} \\ &= \frac{z-0.4}{(z+0.5-j\sqrt{7}/2)(z+0.5+j\sqrt{7}/2)} \end{aligned}$$

The roots of the quadratic

$$z^2 + z + 2 = 0 \text{ are,}$$

$$z = \frac{-1 \pm \sqrt{1-4 \times 2}}{2}$$

$$= -0.5 \pm j\sqrt{7}/2$$

By partial fraction expansion we get, $F(z) = \frac{A}{z+0.5-j\sqrt{7}/2} + \frac{A^*}{z+0.5+j\sqrt{7}/2}$

$$\begin{aligned} A &= \frac{z-0.4}{(z+0.5-j\sqrt{7}/2)(z+0.5+j\sqrt{7}/2)} (z+0.5-j\sqrt{7}/2) \Big|_{z=-0.5+j\sqrt{7}/2} \\ &= \frac{z-0.4}{(z+0.5+j\sqrt{7}/2)} \Big|_{z=-0.5+j\sqrt{7}/2} = \frac{-0.5+j\sqrt{7}/2-0.4}{-0.5+j\sqrt{7}/2+0.5+j\sqrt{7}/2} \\ &= \frac{-0.9+j\sqrt{7}/2}{j\sqrt{7}} = \frac{-0.9}{j\sqrt{7}} + \frac{j\sqrt{7}/2}{j\sqrt{7}} = 0.5 + j\frac{0.9}{\sqrt{7}} = 0.5 + j0.34 \end{aligned}$$

$$\therefore A^* = (0.5 + j0.34)^* = 0.5 - j0.34$$

$$\begin{aligned} \therefore F(z) &= \frac{0.5 + j0.34}{z+0.5-j\sqrt{7}/2} + \frac{0.5 - j0.34}{z+0.5+j\sqrt{7}/2} \\ &= (0.5 + j0.34) \frac{1}{z} \frac{z}{z+0.5-j\sqrt{7}/2} + (0.5 - j0.34) \frac{1}{z} \frac{z}{z+0.5+j\sqrt{7}/2} \\ &= (0.5 + j0.34) z^{-1} \frac{z}{z-(-0.5+j\sqrt{7}/2)} + (0.5 - j0.34) z^{-1} \frac{z}{z-(-0.5-j\sqrt{7}/2)} \end{aligned}$$

We know that, $\mathcal{Z}\{a^k\} = \frac{z}{z-a}$

By time shifting property we get, $\mathcal{Z}\{a^{(k-1)}\} = z^{-1} \frac{z}{z-a}$

On taking inverse z-transform of F(z) we get,

$$f(k) = (0.5 + j0.34) (-0.5 + j\sqrt{7}/2)^{(k-1)} u(k-1) \\ + (0.5 - j0.34) (-0.5 - j\sqrt{7}/2)^{(k-1)} u(k-1) ; \text{ for } k \geq 0$$

Note: Since the term $a^{(k-1)}$ is valid only for $k \geq 1$, it is multiplied by $u(k-1)$.

(d) Given that, $F(z) = \frac{z-4}{(z-1)(z-2)^2}$

By partial fraction expansion we get,

$$F(z) = \frac{z-4}{(z-1)(z-2)^2} = \frac{A_1}{z-1} + \frac{A_2}{(z-2)^2} + \frac{A_3}{(z-2)}$$

$$A_1 = \left. \frac{z-4}{(z-1)(z-2)^2} (z-1) \right|_{z=1} = \left. \frac{z-4}{(z-2)^2} \right|_{z=1} = \frac{1-4}{(1-2)^2} = -3$$

$$A_2 = \left. \frac{z-4}{(z-1)(z-2)^2} (z-2)^2 \right|_{z=2} = \left. \frac{z-4}{z-1} \right|_{z=2} = \frac{2-4}{2-1} = -2$$

$$A_3 = \left. \frac{d}{dz} \left[\frac{z-4}{(z-1)(z-2)^2} (z-2)^2 \right] \right|_{z=2} = \left. \frac{d}{dz} \left[\frac{z-4}{z-1} \right] \right|_{z=2}$$

$$= \left. \frac{(z-1) - (z-4)}{(z-1)^2} \right|_{z=2} = \left. \frac{3}{(z-1)^2} \right|_{z=2} = \frac{3}{(2-1)^2} = 3$$

$$\therefore F(z) = \frac{-3}{z-1} - \frac{2}{(z-2)^2} + \frac{3}{z-2} = -3 \frac{1}{z} \frac{z}{z-1} - \frac{1}{z} \frac{2z}{(z-2)^2} + 3 \frac{1}{z} \frac{z}{z-2}$$

$$= -3z^{-1} \frac{z}{z-1} - z^{-1} \frac{2z}{(z-2)^2} + 3z^{-1} \frac{z}{z-2}$$

We know that, $\mathcal{Z}\{u(k)\} = \frac{z}{z-1}$; $\mathcal{Z}\{a^k\} = \frac{z}{z-a}$ and $\mathcal{Z}\{ka^k\} = \frac{az}{(z-a)^2}$

By time shifting property we get,

$$\mathcal{Z}\{u(k-1)\} = z^{-1} \frac{z}{z-1} ; \mathcal{Z}\{a^{(k-1)}\} = z^{-1} \frac{z}{z-a} \text{ and } \mathcal{Z}\{(k-1)a^{(k-1)}\} = z^{-1} \frac{az}{(z-a)^2}$$

On taking inverse z-transform of $F(z)$ we get,

$$f(k) = -3 u(k-1) - (k-1) 2^{(k-1)} u(k-1) + 3 \times 2^{(k-1)} u(k-1)$$

Note: Since the term $a^{(k-1)}$ is valid only for $k \geq 1$, it is multiplied by $u(k-1)$.

EXAMPLE 4.8

Determine the inverse z-transform of $F(z) = \frac{1}{1 - \frac{3}{2}z^{-1} + \frac{1}{2}z^{-2}}$

When (a) ROC : $|z| > 1.0$ and (b) ROC : $|z| < 0.5$.

SOLUTION

Since the ROC is the exterior of a circle, we expect $f(k)$ to be causal signal. Hence we can express $F(z)$ as a power series expansion in negative powers of z . On dividing the numerator of $F(z)$ by its denominator we get,

$$F(z) = \frac{1}{1 - \frac{3}{2}z^{-1} + \frac{1}{2}z^{-2}} = 1 + \frac{3}{2}z^{-1} + \frac{7}{4}z^{-2} + \frac{15}{8}z^{-3} + \frac{31}{16}z^{-4} + \dots$$

...4.8.1

$$\begin{array}{r}
 1 + \frac{3}{2}z^{-1} + \frac{7}{4}z^{-2} + \frac{15}{8}z^{-3} + \frac{31}{16}z^{-4} + \dots \\
 1 - \frac{3}{2}z^{-1} + \frac{1}{2}z^{-2} \overline{) \begin{array}{l} 1 \\ 1 - \frac{3}{2}z^{-1} + \frac{1}{2}z^{-2} \\ \hline \frac{3}{2}z^{-1} - \frac{1}{2}z^{-2} \\ \frac{3}{2}z^{-1} - \frac{9}{4}z^{-2} + \frac{3}{4}z^{-3} \\ \hline \frac{7}{4}z^{-2} - \frac{3}{4}z^{-3} \\ \frac{7}{4}z^{-2} - \frac{21}{8}z^{-3} + \frac{7}{8}z^{-4} \\ \hline \frac{15}{8}z^{-3} - \frac{7}{8}z^{-4} \\ \frac{15}{8}z^{-3} - \frac{45}{16}z^{-4} + \frac{15}{16}z^{-5} \\ \hline \frac{31}{16}z^{-4} - \frac{15}{16}z^{-5} \\ \vdots \end{array}}
 \end{array}$$

If $F(z)$ is z -transform of $f(k)$ then, by the definition of z -transform we get,

$$F(z) = \mathcal{Z}\{f(k)\} = \sum_{k=-\infty}^{\infty} f(k) z^{-k}$$

For a causal signal,

$$F(z) = \sum_{k=0}^{\infty} f(k) z^{-k}$$

On expanding the summation we get,

$$F(z) = f(0)z^{-0} + f(1)z^{-1} + f(2)z^{-2} + f(3)z^{-3} + f(4)z^{-4} + \dots$$

....4.8.2

On comparing the two power series of $F(z)$ [i.e., equ (4.8.1) & (4.8.2)], we get,

$$\begin{aligned}
 f(0) &= 1; \quad f(1) = \frac{3}{2}; \quad f(2) = \frac{7}{4}; \quad f(3) = \frac{15}{8}; \quad f(4) = \frac{31}{16}; \dots \\
 f(k) &= \left\{ \frac{1}{2}, \frac{3}{4}, \frac{7}{8}, \frac{15}{16}, \dots \right\}; \text{ for } k \geq 0
 \end{aligned}$$

(b) Since the ROC is the interior of a circle, we expect $f(k)$ to be anticausal signal. Hence we can express $F(z)$ as a power series expansion in positive powers of z . Therefore,

rewrite the denominator polynomial of $F(z)$ in the reverse order and then the numerator, is divided by the denominator as shown below.

$$\begin{array}{r} 2z^2 + 6z^3 + 14z^4 + 30z^5 + 62z^6 + \dots \\ \frac{1}{2}z^{-2} - \frac{3}{2}z^{-1} + 1 \overline{) 1} \\ \underline{1 - 3z + 2z^2} \\ 3z - 2z^2 \\ \underline{3z - 9z^2 + 6z^3} \\ 7z^2 - 6z^3 \\ \underline{7z^2 - 21z^3 + 14z^4} \\ 15z^3 - 14z^4 \\ \underline{15z^3 - 45z^4 + 30z^5} \\ 31z^4 - 30z^5 \\ \vdots \end{array}$$

$$\begin{aligned} \therefore F(z) &= \frac{1}{1 - \frac{3}{2}z^{-1} + \frac{1}{2}z^{-2}} = \frac{1}{\frac{1}{2}z^{-2} - \frac{3}{2}z^{-1} + 1} \\ &= 2z^2 + 6z^3 + 14z^4 + 30z^5 + 62z^6 + \dots \end{aligned}$$

...4.8.3

If $F(z)$ is z -transform of $f(k)$ then, by the definition of z -transform we get,

$$F(z) = \mathcal{Z}\{f(k)\} = \sum_{k=-\infty}^{\infty} f(k) z^{-k}$$

$$\text{For an anticausal signal, } F(z) = \sum_{k=-\infty}^0 f(k) z^{-k}$$

On expanding the summation we get,

$$F(z) = \dots f(-6) z^6 + f(-5) z^5 + f(-4) z^4 + f(-3) z^3 + f(-2) z^2 + f(-1) z + f(0) \quad \dots 4.8.4$$

On comparing the two power series of $F(z)$ [i.e., equ (4.8.3) & (4.8.4)], we get,

$$\begin{aligned} f(-6) &= 62 ; \quad f(-5) = 30 ; \quad f(-4) = 14 ; \quad f(-3) = 6; \\ f(-2) &= 2 ; \quad f(-1) = 0 \quad \text{and} \quad f(0) = 0 \end{aligned}$$

$$\therefore f(k) = \left\{ \dots, 62, 30, 14, 6, 2, 0, 0 \right\}$$

4.7 LINEAR DISCRETE TIME SYSTEMS

A discrete-time system is a device or algorithm that operates on a discrete-time signal called the input or excitation, according to some well-defined rule, to produce another discrete-time signal called the output or the response of the system. We can say that the input signal $r(k)$ is transformed by the system into a signal $c(k)$ and expressed as

$$c(k) = H[r(k)]$$

where H denotes the transformation (also called as operator)

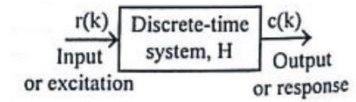
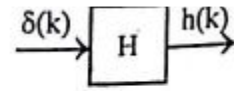


Figure 4.17

A discrete time system is linear if it obeys the principle of superposition and it is time invariant if its input-output relationships do not change with time.



When the input to a discrete time system is unit impulse, $\delta(k)$ then the output is called impulse response of the system and denoted by $h(k)$.

Figure 4.18

$$h(k) = H[\delta(k)] \quad \dots 4.28$$

A linear-time invariant discrete time system is characterized by its impulse response $h(k)$ and so the impulse response $h(k)$ is also called weighting sequence.

The input-output description of a discrete-time system consists of mathematical expression or a rule, which explicitly defines the relation between the input and output signals (input-output relationship). It is denoted by

$$r(k) \xrightarrow{H} c(k) \quad \dots 4.29$$

The input-output relationship of a linear-time invariant discrete time system, (LDS) can be expressed by N^{th} order constant coefficient difference equation given below.

$$c(k) = - \sum_{m=1}^N a_m c(k-m) + \sum_{m=0}^M b_m r(k-m) \quad \dots 4.30$$

The integer N is called the order of the system and $M \leq N$.

Here $c(k-m)$ are past outputs, $r(k-m)$ are past inputs, $r(k)$ is present input and a_k and b_k are constant coefficients.

ANALYSIS OF LINEAR DISCRETE TIME SYSTEM (LDS)

There are two methods of analysing the behaviour or response of a LDS systems.

Method 1

The input-output relation of the LDS system is governed by the constant coefficient difference equation of the form shown in equ (4.30). Mathematically the direct solution of equation (4.30) can be obtained to analyse the performance of the system.

Method 2

The given input signal is first decomposed or resolved into a sum of elementary signals. Then using the linearity property of the system, the responses of the system to the elementary signals are added to obtain the total response of the system to the given input signals.

Resolution of discrete time signal (or sequence) into impulses

Let $r(k)$ = Discrete time signal

$\delta(k)$ = Unit impulse signal

and $\delta(k-m)$ = Delayed unit impulse signal

Consider the product of $r(k)$ and $\delta(k-m)$

$$r(k) \delta(k-m) = \begin{cases} r(m) \delta(k-m) & ; \text{ at } k = m \\ 0 & ; \text{ for } k \neq m \end{cases} \quad \dots 4.41$$

$$\therefore r(k) \delta(k-m) = \begin{cases} r(m) & ; \text{ at } k = m \quad (\because \delta(k-m) = 1 \text{ at } k = m) \\ 0 & ; \text{ for } k \neq m \end{cases} \quad \dots 4.42$$

The product $r(k) \delta(k-m)$ has zero everywhere except at $k = m$. The value of the signal at $k = m$ is the m^{th} sample of the signal $r(k)$ and it is denoted by $r(m)$. Therefore each multiplication of the signal $r(k)$ by an unit impulse at some delay m , in essence picks out the signal value $r(m)$ of the signal $r(k)$ at $k = m$, where the unit impulse is non zero. Consequently if we repeat this multiplication over all possible delays in the range of, $0 \leq m < \infty$ and sum all the product sequences, the result will be a sequence that is equal to the sequence $r(k)$. Hence $r(k)$ can be expressed as

$$r(k) = \sum_{m=0}^{\infty} r(m) \delta(k-m) \quad \dots (4.53)$$

Note: Each product $r(k) \delta(k-m)$ is an impulse and the summation of impulses give $r(k)$. Here $r(k)$ is considered as one sided sequence. If $r(k)$ is two sided sequence then the range of m is $-\infty$ to $+\infty$.

RESPONSE OF LDS SYSTEM TO ARBITRARY INPUT – THE CONVOLUTION SUM

In a LDS system the response $c(k)$ of the system for arbitrary input $r(k)$ is given by convolution of the input $r(k)$ with the impulse response $h(k)$ of the system. It is expressed as

$$c(k) = r(k) * h(k) \quad \dots 4.34$$

where the symbol $*$ represents convolution operation.

Proof

Let $c(k)$ be the response of the H for an input $r(k)$. [Let $r(k)$ be a one sided sequence].

$$\therefore c(k) = H[r(k)] \quad \dots 4.35$$

The signal $r(k)$ can be expressed as a summation of impulses as,

$$r(k) = \sum_{m=0}^{\infty} r(m) \delta(k-m) \quad \dots 4.36$$

where $\delta(k-m)$ is the delayed unit impulses signal.

From equation (4.35) and (4.36) we get,

$$c(k) = H \left[\sum_{m=0}^{\infty} r(m) \delta(k-m) \right] \quad \dots 4.37$$

The system H is a function of k and not a function of m . Hence by linearity property the equ (4.37) can be written as,

$$c(k) = \sum_{m=0}^{\infty} r(m) H[\delta(k-m)] \quad \dots 4.38$$

Let the response of the LDS system to the unit impulse input $\delta(k)$ be denoted by $h(k)$.

$$\therefore h(k) = H[\delta(k)] \quad \dots 4.39$$

Then by time invariance property the response of the system to the delayed unit impulse input $\delta(k-m)$ is

$$h(k-m) = H[\delta(k-m)] \quad \dots 4.40$$

Using equ (4.40), the equ (4.38) can be expressed as

$$c(k) = \sum_{m=0}^{\infty} r(m) h(k-m) \quad \dots 4.41$$

The equation of $c(k)$ [equ(4.41)] is called convolution sum. We can say that the input $r(k)$ is convoluted with the impulse response $h(k)$ to yield the output $c(k)$.

$$\therefore c(k) = \sum_{m=0}^{\infty} r(m) h(k-m) = r(k) * h(k) \quad \dots 4.42$$

PROPERTIES OF CONVOLUITON

- Commutative property : $r(k) * h(k) = h(k) * r(k)$
- Associative property : $[r(k) * h_1(k)] * h_2(k) = r(k) * [h_1(k) * h_2(k)]$
- Distributive property : $r(k) * [h_1(k) + h_2(k)] = [r(k) * h_1(k)] + [r(k) * h_2(k)]$

4.8 TRANSFER FUNCTION OF LDS SYSTEM (PULSE TRANSFER FUNCDTION)

The transfer function of LDS system is given by z-transform of its impulse response. The transfer function of LDS system is also called z-transfer function or pulse transfer function.

Let $h(k)$ = Impulse response of a LDS system

Now, z-transform of $h(k) = Z\{h(k)\} = H(z)$

$$\therefore \text{Transfer function of LDS system} = H(z) \quad \dots 4.43$$

The input-output relationship of a LDS system is governed by a convolution sum of equ (4.42). By taking z-transform of this convolution sum it can be shown that, $H(z)$ is given by the ratio by $C(z)/R(z)$, where $C(z)$ is the z-transform of output $c(k)$ of LDS system and $R(z)$ is the z-transform of input $r(k)$ to the LDS system.

Proof

By the definition of one sided z-transform.

$$C(z) = \mathcal{Z}\{c(k)\} = \sum_{k=0}^{\infty} c(k) z^{-k} \quad \dots 4.44$$

From equ (4.42), we get $c(k) = \sum_{m=0}^{\infty} r(m) h(k-m)$

On substituting this convolution sum in equ (4.44) we get,

$$C(z) = \sum_{k=0}^{\infty} \left[\sum_{m=0}^{\infty} r(m) h(k-m) \right] z^{-k} \quad \dots 4.45$$

The order of summation in equ (4.45) can be interchanged. Therefore equ (4.45) can be written as

$$C(z) = \sum_{m=0}^{\infty} r(m) \sum_{k=0}^{\infty} h(k-m) z^{-k} \quad \dots 4.46$$

Let, $p = (k - m)$, \therefore when $k = 0$, $p = -m$
 and when $k = \infty$, $p = \infty$
 Also, $k = p + m$

On replacing $(k - m)$ by p in equ (4.46) we get

$$\begin{aligned} C(z) &= \sum_{m=0}^{\infty} r(m) \sum_{p=-m}^{\infty} h(p) z^{-(p+m)} \\ &= \sum_{m=0}^{\infty} r(m) \sum_{p=0}^{\infty} h(p) z^{-p} z^{-m} \quad (\because h(p) = 0 \text{ ; for } p < 0) \\ &= \sum_{m=0}^{\infty} r(m) z^{-m} \sum_{p=0}^{\infty} h(p) z^{-p} \end{aligned} \quad \dots 4.47$$

By the definition of one sided z-transform,

$$\sum_{m=0}^{\infty} r(m) z^{-m} = R(z) \text{ and } \sum_{p=0}^{\infty} h(p) z^{-p} = H(z)$$

Hence equation (4.47) can be written as

$$C(z) = R(z) H(z) \quad (\text{or}) \quad H(z) = \frac{C(z)}{R(z)} \quad \dots 4.48$$

From equ (4.48) we can conclude that the transfer function of the system is given by the ratio $C(z) / R(z)$.

From the above analysis we can define the transfer function of the LDS system as the ratio of the z-transform of the output of a system to the z-transform of the input to the system with zero initial conditions.

Let $r(k)$ = Input of LDS system

and $c(k)$ = Output of a LDS system

Now, $\mathcal{Z}\{r(k)\} = R(z)$ and $\mathcal{Z}\{c(k)\} = C(z)$

$$\therefore \text{Transfer function of LDS system} = \frac{C(z)}{R(z)} \quad \dots 4.49$$

The input-output relation of LDS system is governed by the constant coefficient difference equation.

$$c(k) = - \sum_{m=1}^N a_m c(k-m) + \sum_{m=0}^M b_m r(k-m) \quad \dots 4.50$$

where N is the order of the system and $M \leq N$.

On taking z-transform of equ (4.50) we get,

[By time shifting property, $\mathcal{Z}\{c(k-m)\} = z^{-m} C(z)$ and $\mathcal{Z}\{r(k-m)\} = z^{-m} R(z)$]

$$\begin{aligned} C(z) &= - \sum_{m=1}^N a_m z^{-m} C(z) + \sum_{m=0}^M b_m z^{-m} R(z) \\ \therefore C(z) + \sum_{m=1}^N a_m z^{-m} C(z) &= \sum_{m=0}^M b_m z^{-m} R(z) \end{aligned} \quad \dots 4.51$$

On expanding the equ (4.51) with $M = N$, we get,

$$\begin{aligned} C(z) + a_1 z^{-1} C(z) + a_2 z^{-2} C(z) + \dots + a_N z^{-N} C(z) &= b_0 R(z) + b_1 z^{-1} R(z) + b_2 z^{-2} R(z) + \dots + b_N z^{-N} R(z) \\ C(z) [1 + a_1 z^{-1} + a_2 z^{-2} + \dots + a_N z^{-N}] &= R(z) [b_0 + b_1 z^{-1} + b_2 z^{-2} + \dots + b_N z^{-N}] \\ \therefore \frac{C(z)}{R(z)} &= \frac{b_0 + b_1 z^{-1} + b_2 z^{-2} + \dots + b_N z^{-N}}{1 + a_1 z^{-1} + a_2 z^{-2} + \dots + a_N z^{-N}} \end{aligned} \quad \dots 4.52$$

From the above discussions it is evident that the transfer function of the LDS system can be obtained by taking z-transform of the difference equation governing the system.

EXAMPLE 4.9

The input-output relation of a sampled data system is described by the equation

$$c(k+2) + 3c(k+1) + 4c(k) = r(k+1) - r(k).$$

Determine the z-transfer function. Also obtain the weighting sequence of the system.

SOLUTION

Let $R(z) = \mathcal{Z}\{r(k)\}$ and $C(z) = \mathcal{Z}\{c(k)\}$

By time shifting property, when initial conditions are zero, we get,

$$\mathcal{Z}\{c(k+m)\} = z^m C(z) \text{ and } \mathcal{Z}\{r(k+m)\} = z^m R(z)$$

Given that, $c(k+2) + 3c(k+1) + 4c(k) = r(k+1) - r(k)$

On taking z-transform of the above equation we get,

$$z^2 C(z) + 3z C(z) + 4 C(z) = z R(z) - R(z)$$

$$(z^2 + 3z + 4) C(z) = (z - 1) R(z)$$

$$\therefore \frac{C(z)}{R(z)} = \frac{z - 1}{z^2 + 3z + 4}$$

The transfer function of the system, $H(z) = \frac{C(z)}{R(z)} = \frac{z - 1}{z^2 + 3z + 4}$

The weighting sequence is the impulse response, $h(k)$ of the system. It is given by inverse z-transform of $H(z)$.

$$H(z) = \frac{z - 1}{z^2 + 3z + 4} = \frac{z - 1}{\left(z + \frac{3}{2} + j\frac{\sqrt{7}}{2}\right)\left(z + \frac{3}{2} - j\frac{\sqrt{7}}{2}\right)}$$

By partial fraction technique $H(z)$ can be expressed as

$$H(z) = \frac{A}{z + \frac{3}{2} + j\frac{\sqrt{7}}{2}} + \frac{A^*}{z + \frac{3}{2} - j\frac{\sqrt{7}}{2}}$$

$$A = \frac{z - 1}{\left(z + \frac{3}{2} + j\frac{\sqrt{7}}{2}\right)\left(z + \frac{3}{2} - j\frac{\sqrt{7}}{2}\right)} \left(z + \frac{3}{2} + j\frac{\sqrt{7}}{2}\right) \bigg|_{z = -\frac{3}{2} - j\frac{\sqrt{7}}{2}} = \frac{z - 1}{z + \frac{3}{2} - j\frac{\sqrt{7}}{2}} \bigg|_{z = -\frac{3}{2} - j\frac{\sqrt{7}}{2}}$$

$$= \frac{-\frac{3}{2} - j\frac{\sqrt{7}}{2} - 1}{-\frac{3}{2} - j\frac{\sqrt{7}}{2} + \frac{3}{2} - j\frac{\sqrt{7}}{2}} = \frac{-\frac{5}{2} - j\frac{\sqrt{7}}{2}}{-j\sqrt{7}} = -j\frac{5}{2\sqrt{7}} + \frac{1}{2} = \frac{1}{2} - j\frac{5}{2\sqrt{7}}$$

$$A^* = \left(\frac{1}{2} - j\frac{5}{2\sqrt{7}}\right)^* = \frac{1}{2} + j\frac{5}{2\sqrt{7}}$$

$$H(z) = \frac{\left(\frac{1}{2} - j\frac{5}{2\sqrt{7}}\right)}{z + \frac{3}{2} + j\frac{\sqrt{7}}{2}} + \frac{\frac{1}{2} + j\frac{5}{2\sqrt{7}}}{z + \frac{3}{2} - j\frac{\sqrt{7}}{2}}$$

$$H(z) = \left(\frac{1}{2} - j\frac{5}{2\sqrt{7}}\right) z^{-1} \frac{z}{z - \left(-\frac{3}{2} - j\frac{\sqrt{7}}{2}\right)} + \left(\frac{1}{2} + j\frac{5}{2\sqrt{7}}\right) z^{-1} \frac{z}{z - \left(-\frac{3}{2} + j\frac{\sqrt{7}}{2}\right)}$$

The roots of the quadratic

$$z^2 + 3z + 4 = 0 \text{ are}$$

$$z = \frac{-3 \pm \sqrt{9 - 4 \times 4}}{2}$$

$$= -\frac{3}{2} \pm j\frac{\sqrt{7}}{2}$$

We know that $\mathcal{Z}\{a^k\} = \frac{z}{z-a}$

By time shifting property, $\mathcal{Z}\{a^{(k-1)}\} = z^{-1} \frac{z}{z-a}$

On taking inverse z-transform of $H(z)$ we get,

$$h(k) = \left(\frac{1}{2} - j\frac{5}{2\sqrt{7}}\right) \left(-\frac{3}{2} - j\frac{\sqrt{7}}{2}\right)^{(k-1)} u(k-1) + \left(\frac{1}{2} + j\frac{5}{2\sqrt{7}}\right) \left(-\frac{3}{2} + j\frac{\sqrt{7}}{2}\right)^{(k-1)} u(k-1)$$

EXAMPLE 4.10

Solve the difference equation $c(k+1) + 3c(k+1) + 2c(k) = u(k)$

Given that $c(0) = 1$; $c(1) = -3$; $c(k) = 0$ for $k < 0$

SOLUTION

Let $\mathcal{Z}\{c(k)\} = C(z)$ and $\mathcal{Z}\{u(k)\} = U(z)$

Since $u(k)$ is unit step signal, $U(z) = \frac{z}{z-1}$

We know that, if $F(z) = \mathcal{Z}\{f(k)\}$ then

$$\mathcal{Z}\{f(k+m)\} = z^m F(z) - \sum_{i=0}^{m-1} f(i) z^{m-i}$$

Given that, $c(k+2) + 3c(k+1) + 2c(k) = u(k)$

On taking z-transform of the above equation we get,

$$\mathcal{Z}\{c(k+2)\} + \mathcal{Z}\{3c(k+1)\} + \mathcal{Z}\{2c(k)\} = \mathcal{Z}\{u(k)\}$$

$$z^2 C(z) - z^2 c(0) - z c(1) + 3[z C(z) - z c(0)] + 2 C(z) = \frac{z}{z-1}$$

On substituting the initial conditions, $c(0) = 1$ and $c(1) = -3$ we get,

$$z^2 C(z) - z^2 + 3z + 3z C(z) - 3z + 2 C(z) = \frac{z}{z-1}$$

$$(z^2 + 3z + 2) C(z) - z^2 = \frac{z}{z-1}$$

$$(z^2 + 3z + 2) C(z) = \frac{z}{z-1} + z^2$$

$$(z+1)(z+2) C(z) = \frac{z + z^2(z-1)}{(z-1)}$$

$$\therefore C(z) = \frac{z[1 + z^2 - z]}{(z-1)(z+1)(z+2)}$$

$$\therefore \frac{C(z)}{z} = \frac{z^2 - z + 1}{(z-1)(z+1)(z+2)}$$

By partial fraction expansion technique we can write $C(z)/z$ as,

$$\begin{aligned} \frac{C(z)}{z} &= \frac{A_1}{z-1} + \frac{A_2}{z+1} + \frac{A_3}{z+2} \\ A_1 &= \left. \frac{z^2 - z + 1}{(z-1)(z+1)(z+2)} (z-1) \right|_{z=1} = \left. \frac{z^2 - z + 1}{(z+1)(z+2)} \right|_{z=1} = \frac{1-1+1}{(1+1)(1+2)} = \frac{1}{6} \\ A_2 &= \left. \frac{z^2 - z + 1}{(z-1)(z+1)(z+2)} (z+1) \right|_{z=-1} = \left. \frac{z^2 - z + 1}{(z-1)(z+2)} \right|_{z=-1} = \frac{(-1)^2 - (-1) + 1}{(-1-1)(-1+2)} = -\frac{3}{2} \\ A_3 &= \left. \frac{z^2 - z + 1}{(z-1)(z+1)(z+2)} (z+2) \right|_{z=-2} = \left. \frac{z^2 - z + 1}{(z-1)(z+1)} \right|_{z=-2} = \frac{(-2)^2 - (-2) + 1}{(-2-1)(-2+1)} = \frac{7}{3} \\ \therefore \frac{C(z)}{z} &= \frac{1}{6} \frac{1}{z-1} - \frac{3}{2} \frac{1}{z+1} + \frac{7}{3} \frac{1}{z+2} \\ C(z) &= \frac{1}{6} \frac{z}{z-1} - \frac{3}{2} \frac{z}{z-(-1)} + \frac{7}{3} \frac{z}{z-(-2)} \end{aligned}$$

We know that $\mathcal{Z}\{u(k)\} = \frac{z}{z-1}$ and $\mathcal{Z}\{a^k\} = \frac{z}{z-a}$

On taking inverse z-transform of $C(z)$ we get,

$$c(k) = \frac{1}{6} u(k) - \frac{3}{2} (-1)^k + \frac{7}{3} (-2)^k ; k \geq 0$$

The above equation of $c(k)$ is the solution of the given difference equation.

4.9 ANALYSIS OF SAMPLER AND ZERO – ORDER HOLD

Consider a pulse sampler with zero-order hold (ZOH) shown in Figure 4.19. Let the output of sampler be a pulse train of pulse width Δ . For each input pulse, the ZOH produces a pulse of duration T , where T is the sampling period.

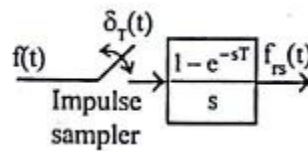
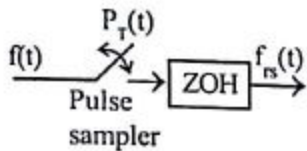


Figure 4.19 Pulse sampler with ZOH Figure 4.20 Equivalent representation pulse sampler with ZOH

It can be proved that the output of pulse sampler with ZOH can be produced by impulse sampled $f(t)$ when passed through a transfer function.

$$G_0(s) = \frac{1 - e^{-sT}}{s} \quad \dots 4.53$$

Hence the pulse sampler with ZOH can be replaced by an equivalent system consisting of an impulse sampler and a block with transfer function, $(1 - e^{-sT})/s$ as shown in Figure 4.20. This equivalent representation offers easier analysis of sampled data control systems.

FREQUENCY RESPONSE CHARACTERISTICS ZERO ORDER HOLDING DEVICE

The sinusoidal transfer function of ZOH can be obtained from $G_0(s)$ by replacing s by $j\omega$.

$$\therefore G_0(j\omega) = \frac{1 - e^{-j\omega T}}{j\omega} \quad \dots 4.54$$

$$\text{We know that, } \frac{-j\omega T}{e^{\frac{-j\omega T}{2}}} \frac{+j\omega T}{e^{\frac{+j\omega T}{2}}} = 1 \quad \dots 4.55$$

Hence from equation (4.54) and (4.55) we get,

$$\begin{aligned} G_0(j\omega) &= \frac{e^{\frac{-j\omega T}{2}} e^{\frac{j\omega T}{2}} - e^{-j\omega T}}{j\omega} = \frac{e^{\frac{-j\omega T}{2}} e^{\frac{j\omega T}{2}} - e^{\frac{-j\omega T}{2}} e^{\frac{-j\omega T}{2}}}{j\omega} \\ &= \left(\frac{e^{\frac{j\omega T}{2}} - e^{\frac{-j\omega T}{2}}}{j\omega} \right) e^{\frac{-j\omega T}{2}} = \frac{2}{\omega} \sin \frac{\omega T}{2} e^{\frac{-j\omega T}{2}} \\ &= T \frac{\sin \frac{\omega T}{2}}{\frac{\omega T}{2}} e^{\frac{-j\omega T}{2}} \end{aligned} \quad \dots 4.56$$

Note : $\sin \theta = \frac{e^{j\theta} - e^{-j\theta}}{2j}$

We know that, sampling frequency, $\omega_s = \frac{2\pi}{T}$

$$\therefore T = \frac{2\pi}{\omega_s}$$

On substituting $T = 2\pi/\omega_s$ in equ (4.56) we get,

$$G_0(j\omega) = \frac{2\pi}{\omega_s} \frac{\sin(\pi\omega / \omega_s)}{(\pi\omega / \omega_s)} e^{\frac{-j\pi\omega}{\omega_s}}$$

$$\text{Magnitude of } G_0(j\omega) = |G_0(j\omega)| = \frac{2\pi}{\omega_s} \frac{\sin(\pi\omega / \omega_s)}{(\pi\omega / \omega_s)} \quad \dots 4.57$$

$$\text{Argument (or phase) of } G_0(j\omega) = \angle G_0(j\omega) = \frac{-\pi\omega}{\omega_s} \quad \dots 4.58$$

The frequency response characteristics consists of magnitude response and phase response characteristics. The magnitude and phase response of ZOH device are given by equations (4.57) and (4.58) respectively. The Figure (4.21) shows the frequency response curve of ZOH device. From the frequency response curve we can conclude that ZOH device has low pass filtering characteristics.

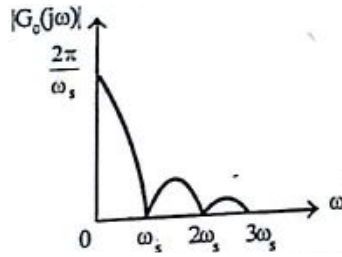


Fig.a. Magnitude response of ZOH device

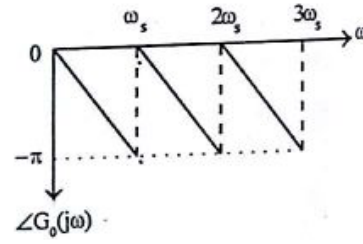


Fig.b. Phase response of ZOH device

Figure 4.21 Frequency response of ZOH device

4.10 ANALYSIS OF SYSTEM WITH IMPULSE SAMPLING

Consider a linear continuous time system fed from an impulse sampler as shown in Figure 4.22a. Let $H(s)$ be the transfer function of the system in s-domain. In such a system we are interested in reading the output at sampling instants. This can be achieved by means of a mathematical sampler or read-out sampler.

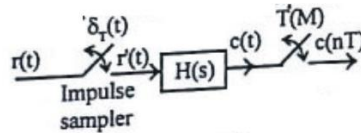


Fig. 4.22a

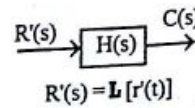
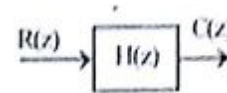


Fig. 4.22b

Figure 4.22 Linear continuous time system with impulse sampled input

For the system shown in Figure 4.22b, it can be shown that the z-domain transfer function $H(z)$ can be directly obtained from s-domain transfer function by taking z-transform of $H(s)$

$$\text{i.e.}, H(z) = Z\{H(s)\} \quad \dots 4.59$$



The Figure 4.23 shows the z-transform equivalent of the s-domain system of Figure 4.22b.

Figure 4.23: The z-transform equivalent of the system shown in Figure 3.22b

$$\text{The output in z-domain is given by, } C(z) = H(z) R(z) \quad \dots 4.60$$

Procedure to find z-transfer function from s-domain transfer function

1. Determine $h(t)$ from $H(s)$, where $h(t) = L^{-1}\{H(s)\}$
2. Determine the discrete sequence $h(kT)$ by replacing t by kT in $h(t)$
3. Take z-transform of $h(kT)$, which is z-transform function of the system (i.e., $H(z) = Z\{h(kT)\}$).

Table 4.4 Laplace and Z-transformations

H(s)	H(z)
$\frac{1}{s}$	$\frac{z}{z-1}$
$\frac{1}{s^2}$	$\frac{Tz}{(z-1)^2}$
$\frac{1}{s^3}$	$\frac{T^2 z (z+1)}{2 (z-1)^3}$
$\frac{1}{s+a}$	$\frac{z}{z-e^{-aT}}$
$\frac{1}{(s+a)^2}$	$\frac{T z e^{-aT}}{(z-e^{-aT})^2}$
$\frac{n}{s(s+a)}$	$\frac{z(1-e^{-aT})}{(z-1)(z-e^{-aT})}$
$\frac{\omega}{s^2+\omega^2}$	$\frac{z \sin \omega T}{z^2 - 2z \cos \omega T + 1}$
$\frac{s}{s^2+\omega^2}$	$\frac{z(z-\cos \omega T)}{z^2 - 2z \cos \omega T + 1}$
$\frac{\omega}{(s+a)^2+\omega^2}$	$\frac{z e^{-aT} \sin \omega T}{z^2 - 2z e^{-aT} \cos \omega T + e^{-2aT}}$
$\frac{s+a}{(s+a)^2+\omega^2}$	$\frac{z^2 - z e^{-aT} \cos \omega T}{z^2 - 2z e^{-aT} \cos \omega T + e^{-2aT}}$

Alternatively, by partial fraction technique if $H(s)$ can be expressed as a summation of first order terms then using standard transform pairs listed in Table 4.4, the z-transform of $H(s)$ can be directly obtained.

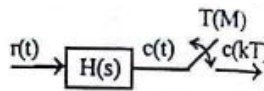


Fig. 3.24a

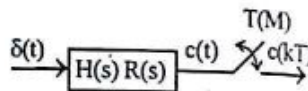


Fig. 3.24b

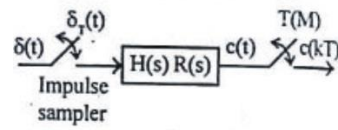


Fig. 3.24c

Fig. 3.24

Consider a continuous time system with transfer function $H(s)$ as shown in Figure 4.24a. Let the input $r(t)$ be a continuous time input. To read the continuous output at sampling instants, let us image a mathematical sampler at the output stage.

The system shown in Figure 4.24a can be equivalently represented by a block of $H(s)R(s)$ with impulse input $\delta(t)$ as shown in Figure 4.24b. Now the input and so the output does not change by imaging a fictitious impulse sampler through which $\delta(t)$ is applied to $H(s)R(s)$ as shown in Figure 3.24c. For such a system we can prove that

$$C(z) = Z\{H(s)R(s)\} \quad \dots 4.61$$

$$\text{Hence, if } C(s) = H(s)R(s) \text{ then } C(z) = Z\{H(s)R(s)\} = HR(z) \quad \dots 4.62$$

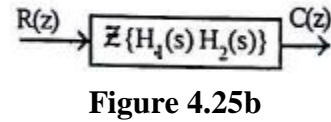
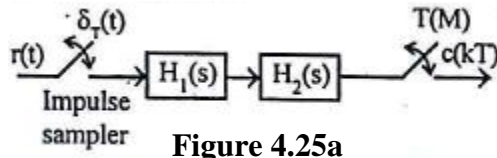
The function $Z\{H(s)R(s)\}$ is also denote das $HR(z)$.

When the impulse sampled input is applied to two or more s-domain transfer function in cascade as shown in Figure 4.25a, then z-transfer function of the system is given by

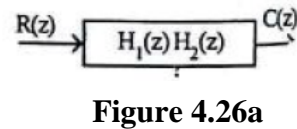
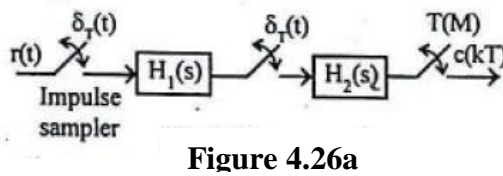
$$H(z) = \mathcal{Z}\{H_1(s) H_2(s)\} \quad \dots 4.63$$

$$\text{and } C(z) = \mathcal{Z}\{H_1(s) H_2(s)\} R(z) \\ \text{where } R(z) = \mathcal{Z}\{R(s)\} \text{ and } R(s) = \mathcal{L}[r(t)] \quad \dots 4.64$$

The function $\mathcal{Z}\{H_1(s) H_2(s)\}$ is also denoted as $H_1 H_2(z)$. The equivalent z-domain system is shown in Figure 4.25b.



Consider a system in which impulse sampler is introduced at the input of each block as shown in Figure 4.26a.



Now the z-transfer function of the system is given by,

$$H(z) = H_1(z) H_2(z)$$

where $H_1(z) = \mathcal{Z}\{H_1(s)\}$ and $H_2(z) = \mathcal{Z}\{H_2(s)\}$

and $C(z) = H_1(z) H_2(z) R(z)$

where $R(z) = \mathcal{Z}\{R(s)\}$ and $R(s) = \mathcal{L}[r(t)]$.

The equivalent z-domain system is shown in Figure 4.26b.

EXAMPLE 4.11

Determine the z-domain transfer function for the following s-domain transfer functions.

$$(a) \quad H(s) = \frac{n}{(s+a)^2}$$

$$(b) \quad H(s) = \frac{n}{s^2 + \omega^2}$$

$$(c) \quad H(s) = \frac{n}{s^2 - a^2}$$

$$(d) \quad H(s) = \frac{n}{(s+b)^2 + a^2}$$

$$(e) \quad H(s) = \frac{n}{(s+b)^2 + a^2}$$

SOLUTION

(a) Given that, $H(s) = \frac{n}{(s+a)^2}$

$$h(t) = \mathcal{L}^{-1}[H(s)] = \mathcal{L}^{-1}\left[\frac{n}{(s+a)^2}\right] = n t e^{-at}$$

The discrete sequence $h(kT)$ is obtained by letting $t = kT$ in $h(t)$

$$\therefore h(kT) = a k T c^{-a k T}$$

z-transfer function, $H(z) = \mathcal{Z}\{h(kT)\}$

Let $f(k) = e^{-a k T}$, $\therefore F(z) = \mathcal{Z}\{f(k)\}$

By the definition of z-transform,

$$F(z) = \sum_{k=0}^{\infty} f(k) z^{-k} = \sum_{k=0}^{\infty} e^{-a k T} z^{-k} = \sum_{k=0}^{\infty} \left(e^{-a T} z^{-1} \right)^k = \frac{1}{1 - e^{-a T} z^{-1}}$$

$$\left(\because \sum_{k=0}^{\infty} C^k = \frac{1}{1-C} \text{ if } |C| < 1 \right) \quad \left(\begin{array}{l} \text{Infinite geometric} \\ \text{series sum formula} \end{array} \right)$$

$$\therefore F(z) = \frac{1}{1 - e^{-a T} / z} = \frac{z}{z - e^{-a T}}$$

By the property of z-transform we get, $\mathcal{Z}\{k f(k)\} = -z \frac{d}{dz} F(z)$

$$\therefore \mathcal{Z}\{k e^{-a k T}\} = \mathcal{Z}\{k f(k)\} = -z \frac{d}{dz} \frac{z}{z - e^{-a T}} = -z \times \frac{(z - e^{-a T}) - z}{(z - e^{-a T})^2} = \frac{z e^{-a T}}{(z - e^{-a T})^2}$$

$$\therefore H(z) = a T \times \mathcal{Z}\{k e^{-a k T}\} = a T \times \frac{z e^{-a T}}{(z - e^{-a T})^2} = \frac{a T z e^{-a T}}{(z - e^{-a T})^2}$$

(b) Given that, $H(s) = \frac{s}{s^2 + \omega^2}$

$$h(t) = \mathbf{L}^{-1}[H(s)] = \mathbf{L}^{-1}\left[\frac{s}{s^2 + \omega^2}\right] = \cos \omega t$$

The discrete sequence $h(kT)$ is obtained by letting $t = kT$ in $h(t)$

$$\therefore h(kT) = \cos \omega k T$$

$$\text{z-transfer function, } H(z) = \mathcal{Z}\{h(kT)\} = \mathcal{Z}\{\cos \omega k T\} = \frac{z(z - \cos \omega T)}{z^2 - 2z \cos \omega T + 1}$$

[Refer Table 4.3 and example 4.4(c)]

(c) Given that, $H(s) = \frac{n}{s^2 - a^2}$

$$h(t) = \mathbf{L}^{-1}[H(s)] = \mathbf{L}^{-1}\left[\frac{n}{s^2 - a^2}\right] = \mathbf{L}^{-1}\left[\frac{n}{(s+a)(s-a)}\right]$$

By partial fraction expansion,

$$\frac{n}{(s+a)(s-a)} = \frac{A_1}{s+a} + \frac{A_2}{s-a}$$

$$A_1 = \frac{n}{(s+a)(s-a)} (s+a) \Big|_{s=-a} = \frac{n}{s-a} \Big|_{s=-a} = \frac{n}{-a-a} = \frac{-n}{2a} = -\frac{1}{2}$$

$$A_2 = \frac{a}{(s+a)(s-a)} (s-a) \Big|_{s=a} = \frac{a}{s+a} \Big|_{s=a} = \frac{a}{a+a} = \frac{a}{2a} = \frac{1}{2}$$

$$\therefore h(t) = \mathbf{L}^{-1} \left[-\frac{1}{2} \frac{1}{(s+a)} + \frac{1}{2} \frac{1}{(s-a)} \right] = -\frac{1}{2} e^{-at} + \frac{1}{2} e^{at}$$

The discrete sequence $h(kT)$ is obtained by letting $t = kT$ in $h(t)$

$$\therefore h(kT) = -\frac{1}{2} e^{-akT} + \frac{1}{2} e^{akT}$$

By the definition of one sided z-transform,

$$\begin{aligned} H(z) &= \sum_{k=0}^{\infty} h(kT) z^{-k} = \sum_{k=0}^{\infty} \left[-\frac{1}{2} e^{-akT} + \frac{1}{2} e^{akT} \right] z^{-k} \\ &= -\frac{1}{2} \sum_{k=0}^{\infty} e^{-akT} z^{-k} + \frac{1}{2} \sum_{k=0}^{\infty} e^{akT} z^{-k} \\ &= -\frac{1}{2} \sum_{k=0}^{\infty} \left(e^{-aT} z^{-1} \right)^k + \frac{1}{2} \sum_{k=0}^{\infty} \left(e^{aT} z^{-1} \right)^k \end{aligned}$$

From infinite geometric sum series formula we know that,

$$\sum_{k=0}^{\infty} C^k = \frac{1}{1-C} \quad ; \quad \text{when } |C| < 1$$

$$\begin{aligned} \therefore H(z) &= -\frac{1}{2} \frac{1}{1 - e^{-aT} z^{-1}} + \frac{1}{2} \frac{1}{1 - e^{aT} z^{-1}} = -\frac{1}{2} \frac{1}{1 - e^{-aT} / z} + \frac{1}{2} \frac{1}{1 - e^{aT} / z} \\ &= -\frac{1}{2} \frac{z}{z - e^{-aT}} + \frac{1}{2} \frac{z}{z - e^{aT}} = \frac{z}{2} \left[\frac{-(z - e^{aT}) + z - e^{-aT}}{(z - e^{-aT})(z - e^{aT})} \right] \\ &= \frac{z}{2} \left[\frac{-z + e^{aT} + z - e^{-aT}}{z^2 - z e^{aT} - z e^{-aT} + e^{-aT} e^{aT}} \right] = \frac{z}{2} \left[\frac{e^{aT} - e^{-aT}}{z^2 - z(e^{aT} + e^{-aT}) + 1} \right] \end{aligned}$$

$$\text{Since, } \cosh \theta = \frac{e^{\theta} + e^{-\theta}}{2} \quad \text{and} \quad \sinh \theta = \frac{e^{\theta} - e^{-\theta}}{2}$$

$$H(z) = \frac{z}{2} \left(\frac{2 \sinh aT}{z^2 - 2z \cosh aT + 1} \right) = \frac{z \sinh aT}{z^2 - 2z \cosh aT + 1}$$

(d) **Given that,** $H(s) = \frac{(s+b)}{(s+b)^2 + a^2}$

$$h(t) = \mathbf{L}^{-1}[H(s)] = \mathbf{L}^{-1} \left[\frac{(s+b)}{(s+b)^2 + a^2} \right] = e^{-bt} \cos at$$

The discrete sequence $h(kT)$ is obtained by letting $t = kT$ in $h(t)$

$$\therefore h(kT) = e^{-bkT} \cos akT$$

z-transfer function, $H(z) = Z\{h(kT)\} = Z\{e^{-bkT} \cos akT\}$

For example 4.5(a) we get

$$H(z) = \frac{z e^{bT} (z e^{bT} - \cos aT)}{z^2 e^{2bT} - 2 z e^{bT} \cos aT + 1}$$

(e) Given that, $H(s) = \frac{a}{(s+b)^2 + a^2}$

$$h(t) = \mathcal{L}^{-1}[H(s)] = \mathcal{L}^{-1}\left[\frac{a}{(s+b)^2 + a^2}\right] = e^{-bt} \sin at$$

The discrete sequence $h(kT)$ is obtained by letting $t = kT$ in $h(t)$

$$\therefore h(kT) = e^{-bkT} \sin akT$$

z-transfer function, $H(z) = Z\{h(kT)\} = Z\{e^{-bkT} \sin akT\}$

$$H(z) = \frac{z e^{bT} \sin aT}{z^2 e^{2bT} - 2 z e^{bT} \cos aT + 1}$$

From example 4.5(b) we get,

4.11 ANALYSIS OF SAMPLED DATA CONTROL SYSTEMS USING Z-TRANSFORM

The analysis of sampled data control systems are performed using the concepts developed in section 4.9 and 4.10. The following points serve as guidelines to determine the output in z-domain and hence the z-transfer function of the sampled data control systems.

1. The pulse sampling is approximated as impulse sampling.
2. The ZOH is replaced by a block with transfer function, $G_0(s) = (1 - e^{-sT})/s$.
3. When the input to a block is impulse sampled signal then the z-transform of the output of the block can be obtained from the z-transform of the input and z-transform of the s-domain transfer function of the block. In determining the output of a block one may come across the following cases.

Case (i) The impulse sampler is located at the input of a block as shown in Figure 4.27.

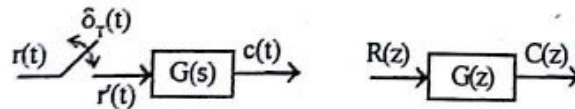


Figure 4.27

In this case, $C(z) = G(z) R(z)$...4.67

Here, $G(z) = Z\{G(s)\}$; $R(z) = Z\{R'(s)\}$ and $R'(s) = \mathcal{L}[r'(t)]$

Case (ii) The impulse sampler is located at the input of two s-domain cascaded blocks as shown in Figure 4.28.

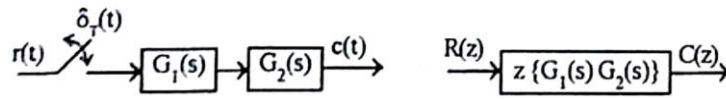


Figure 4.28

In this case, $C(z) = Z\{G_1(s) G_2(s)\} R(z)$

Case (iii) The impulse sampler is located at the input of each blocks as shown in Figure 4.29.

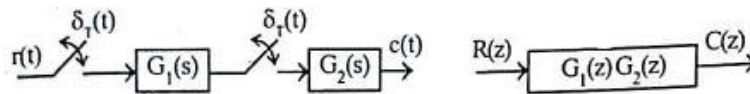


Figure 4.29

In this case, $C(z) = G_1(z) G_2(z) R(z)$...4.69

Here, $G_1(z) = Z\{G_1(s)\}$ and $G_2(z) = Z\{G_2(s)\}$

Case (iv) The impulse sampler is located at the input of ZOH in cascade with G(s) as shown in Figure 4.30.

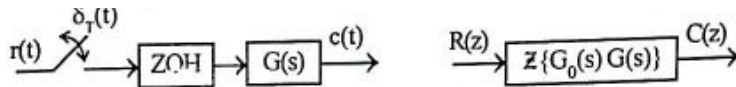


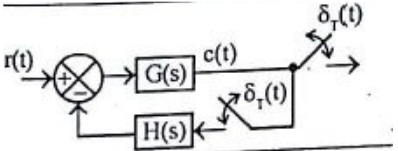
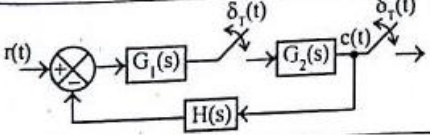
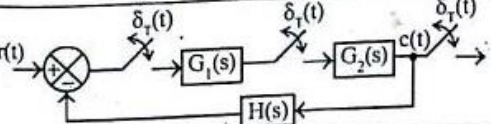
Figure 4.30

In this case, $C(z) = Z\{G_0(s) G(s)\} R(z) = (1-z^{-1}) Z\{G(s) / s\} R(z)$...4.70

The Table 4.5 shows some configurations of the closed loop sampled data control systems and their corresponding z-domain outputs.

Table 3.5

Closed loop sampled data control system	Output z-domain
	$C(z) = \frac{G(z) R(z)}{1 + Z\{G(s) H(s)\}}$ $= \frac{G(z) R(z)}{1 + GH(z)}$
	$C(z) = \frac{G(z) R(z)}{1 + G(z) H(z)}$

	$C(z) = \frac{Z\{G(s) R(s)\}}{1 + Z\{G(s) H(s)\}}$ $= \frac{GR(z)}{1 + GH(z)}$
	$C(z) = \frac{Z\{G_1(s) R(s)\} G_2(z)}{1 + Z\{G_1(s) G_2(s) H(s)\}}$ $= \frac{G_1 R(z) G_2(z)}{1 + G_1 G_2 H(z)}$
	$C(z) = \frac{G_1(z) G_2(z) R(z)}{1 + G_1(z) Z\{G_2(s) H(s)\}}$ $= \frac{G_1(z) G_2(z) R(z)}{1 + G_1(z) G_2 H(z)}$

EXAMPLE 4.12

Find $C(z) / R(z)$ for the following closed loop sampled data control systems. Assume all the samplers to be of impulse type.

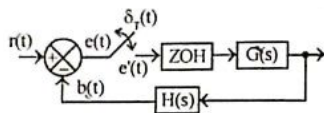


Figure 4.12a

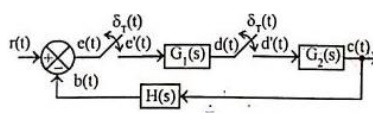


Figure 4.12b

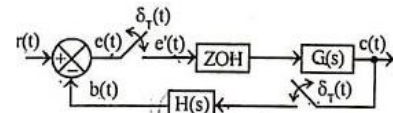


Figure 4.12c

SOLUTION

- (a) The ZOH in the system is replaced by $G_0(s)$ as shown in Figure 4.12.1, where $G_0(s) = (1 - e^{-sT})/s$

Let $e(t)$ = Error signals
 $e'(t)$ = Impulse sampled error signal
 $b(t)$ = Feedback signal

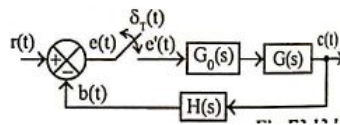


Figure 4.12.1

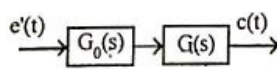


Figure 4.12.2a

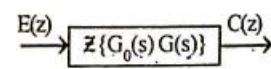


Figure 4.12.2b

The input to the cascaded blocks of $G_0(s)$ and $G(s)$ is an impulse sampled signal as shown in Figure 4.12.2a. Its z-domain equivalent is shown in Figure 4.12.2b.

From Figure 4.12.2b we get, $C(z) = Z\{G_0(s) G(s)\} E(z)$...4.12.1

Here, $C(z) = Z\{C(s)\}$; $E(z) = Z\{E'(s)\}$; $C(s) = L[c(t)]$ and $E'(s) = L[e'(t)]$

The input to the cascaded blocks of $G_0(s)$, $G(s)$ and $H(s)$ is an impulse sampled signal as shown in Figure 4.12.3a. Its z-domain equivalent is shown in Figure 4.12.3b.

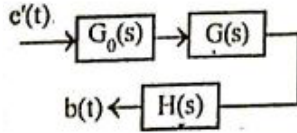


Figure 4.12.3a

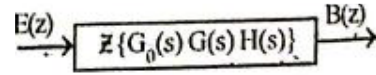


Figure 4.12.3b

From Figure 4.12.2b we get, $B(z) = Z\{G_0(s) G(s) H(s)\} E(z)$...4.12.2

Here, $B(z) = Z\{B(s)\}$ and $B(s) = L[b(t)]$

With reference to Figure 4.12.1, at the summing point we get,

$$e(t) = r(t) - b(t) \quad \dots 4.12.3$$

Since $e'(t) = e(kT)$ is an impulse sampled signal, by superposition principle the equation (4.12.3) can be written as,

$$e(kT) = r(kT) - b(kT) \quad \dots 4.12.4$$

where $e(kT)$, $r(kT)$ and $b(kT)$ are impulse sampled signals of $e(t)$, $r(t)$ and $b(t)$ respectively.

On taking z-transform of equ (4.12.4) we get,

$$\begin{aligned} \underline{E(z)} &= R(z) - B(z) \\ \therefore R(z) &= E(z) + B(z) \end{aligned} \quad \dots 4.12.5$$

Where $R(z) = Z\{R(s)\}$ and $R(s) = L[r(t)]$

On substituting for $B(z)$ from equ (4.12.2) in equ (4.12.5) we get,

$$\begin{aligned} R(z) &= E(z) + Z\{G_0(s) G(s) H(s)\} E(z) \\ &= [1 + Z\{G_0(s) G(s) H(s)\}] E(z) \end{aligned} \quad \dots 4.12.6$$

From equations (4.12.1) and (4.12.6) the z-transfer function or pulse transfer function, $C(z)/R(z)$ can be written as,

$$\frac{C(z)}{R(z)} = \frac{Z\{G_0(s) G(s)\}}{1 + Z\{G_0(s) G(s) H(s)\}} = \frac{G_0 G(z)}{1 + G_0 G H(z)} \quad \dots 4.12.7$$

Here, $Z\{G_0(s) G(s)\}$ is denoted as $G_0 G(z)$ and $Z\{G_0(s) G(s) H(s)\}$ is denoted as $G_0 G H(z)$.

- (b) The input to the block $G_2(s)$ is an impulse sampled signal as shown in Figure 4.12.4a. Its z-domain equivalent is shown in Figure 4.12.4b.

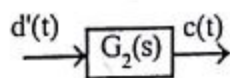


Figure 4.12.4a

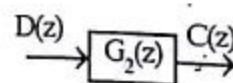


Figure 4.12.4b

From Figure 4.12.4b we get, $C(z) = G_2(z) D(z)$...4.12.8

where $C(z) = \mathcal{Z}\{C(s)\}$; $G_2(z) = \mathcal{Z}\{G_2(s)\}$; $D(z) = \mathcal{Z}\{D'(s)\}$; $C(s) = \mathcal{L}[c(t)]$ and $D'(s) = \mathcal{L}[d'(t)]$

The input to the block $G_1(s)$ is an impulse sampled signal as shown in Figure 4.12.5a. Its z-domain equivalent is shown in Figure 3.12.5b.

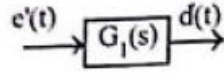


Figure 4.12.5a

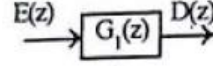


Figure 4.12.5b

From Figure 4.12.5b we get, $D(z) = G_1(z) E(z)$...4.12.9

From equations (4.12.8) and (4.12.9) we get,

$$C(z) = G_2(z)G_1(z) E(z) \quad \dots 4.12.10$$

where $G_1(z) = \mathcal{Z}\{G_1(s)\}$; $E(z) = \mathcal{Z}\{E'(s)\}$ and $E'(s) = \mathcal{L}[e'(t)']$

The input to the cascaded blocks $G_2(s)$ and $H(s)$ is an impulse sampled signal as shown in Figure 4.12.6a. Its z-domain equivalent is shown in Figure 4.12.6b.

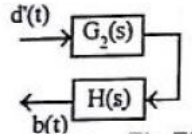


Figure 4.12.6a

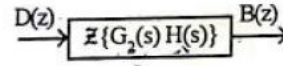


Figure 4.12.6b

From Figure 4.12.6b we get,

$$B(z) = \mathcal{Z}\{G_2(s) H(s)\} D(z) \quad \dots 3.12.11$$

On substituting for $D(z)$ from equ (4.12.9). in equ (4.12.11) we get,

$$B(z) = \mathcal{Z}\{G_2(s) H(s)\} G_1(z) E(z) \quad \dots 3.12.12$$

With reference to Figure 3.12b, at the summing point we get,

$$e(t) = r(1) - b(t) \quad \dots 4.12.13$$

Since $e'(t) = e(Kt)$ is an impulse sampled signal, by superposition principle the equation (4.12.13) can be written as,

$$e(kT) = r(kT) - b(kT) \quad \dots 4.12.14$$

where $e(kT)$, $r(kT)$ and $b(kT)$ are impulse sampled signals of $e(t)$, $r(t)$ and $b(t)$ respectively.

On taking z-transform of equ (4.12.14) we get.

$$E(z) = R(z) - B(z)$$

$$\therefore R(z) = E(z) + B(z) \quad \dots 4.12.15$$

On substituting, for B(z) from equ (4.12.2) in equ (4.12.15) we get,

$$\begin{aligned} R(z) &= E(z) + Z \{G_2(s) H(s)\} G_1(z) E(z) \\ &= [1 + Z \{G_2(s) H(s)\} G_1(z)] E(z) \end{aligned} \quad \dots 4.12.16$$

From equation (4.12.10) and (4.12.16) the z-transfer function or pulse transfer function $C(z)/R(z)$ can be written as,

$$\frac{C(z)}{R(z)} = \frac{G_1(z) G_2(z)}{1 + Z \{G_2(s) H(s)\} G_1(z)} = \frac{G_1(z) G_2(z)}{1 + G_2 H(z) G_1(z)} \quad \dots 4.12.17$$

Here $Z \{G_2(s) H(s)\}$ is denoted as $G_2H(z)$

- (c) The ZOH in the system is replaced by $G_0(S)$ as shown in Figure 4.12.7, where $G_a(s) = (1 - e^{-sT})/s$.

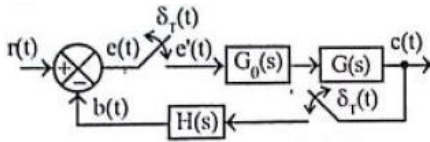


Figure 4.12.7

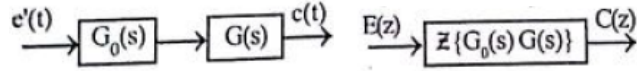


Figure 4.12.8a

Figure 4.12.8b

The input to the cascaded blocks of $G_0(s)$ and $G(s)$ is an impulse sampled signal as shown in Figure 4.12.8a. Its z-domain equivalent is shown in Figure 4.12.8b.

$$\text{From 4.12.8b, we get } C(z) = Z\{G_0(s) G(s)\} E(z) \quad \dots 4.12.18$$

where, $C(z) = Z\{C(s)\}$; $E(z) = Z\{E'(s)\}$; $C(s) = L[c(t)]$ and $E'(s) = L[e'(t)]$.

The input to the block $H(s)$ is an impulse sampled signal as shown in Figure 4.12.9a. Its z-domain equivalent is shown in Figure 4.12.9b.

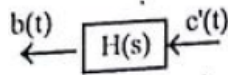


Figure 4.12.9a

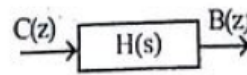


Figure 4.12.9b

$$\text{From Figure 4.12.9b, we get, } B(z) = H(z) C(z) \quad \dots 4.12.19$$

with reference to Figure 4.12.7, at the summing point we get,

$$e(t) = r(t) - b(t) \quad \dots 4.12.20$$

Since $e'(t) = e(kT)$ is an impulse sampled signal, by principle of superposition the equ (4.12.20) can be written as,

$$e(kT) = r(kT) - b(kT) \quad \dots 4.12.21$$

where $e(kT)$, $r(kT)$ and $b(kT)$ are impulse sampled signals of $e(t)$, $r(t)$ and $b(t)$ respectively.

On taking z-transform of equ (4.12.21) we get,

$$E(z) = R(z) - B(z) \quad \dots 4.12.22$$

On substituting for $B(z)$ from equ (4.12.19) in equ (4.12.22) we get,

$$E(z) = R(z) - H(z) C(z) \quad \dots 4.12.23$$

On substituting for $E(z)$ from equ (4.12.23) in equ (4.12.18) we get,

$$\begin{aligned} C(z) &= Z\{G_0(s) G(s)\} [R(z) - H(z) C(z)] \\ C(z) &= Z\{G_0(s) G(s)\} R(z) - Z\{G_0(s) G(s)\} H(z) C(z) \\ C(z) + Z\{G_0(s) G(s)\} H(z) C(z) &= Z\{G_0(s) G(s)\} R(z) \\ C(z) [1 + Z\{G_0(s) G(s)\} H(z)] &= Z\{G_0(s) G(s)\} R(z) \\ \therefore \frac{C(z)}{R(z)} &= \frac{Z\{G_0(s) G(s)\}}{1 + Z\{G_0(s) G(s)\} H(z)} = \frac{G_0 G(z)}{1 + G_0 G(z) H(z)} \end{aligned} \quad \dots 4.12.24$$

The equation (4.12.24) is the z-transfer function of the system.

Here $Z\{G_0(s) G(s)\}$ is denoted as $G_0 G(z)$.

EXAMPLE 4.13

Find the output $C(z)$ in z-domain for the closed loop sampled data control system shown in Figure 4.13.

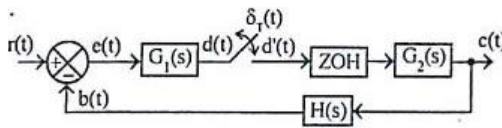


Figure 4.13

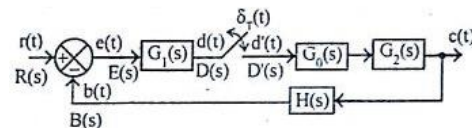


Figure 4.13.1

SOLUTION

The ZOH in Figure 4.13 is replaced by a block with transfer function $G_0(s)$ as shown in Figure 4.13.1, where $G_0(s) = (1 - e^{-sT}) / s$.

Here, $d'(t)$ = Impulse sampled signal of $d(t)$.

The input to the cascaded blocks of $G_0(s)$ and $G_2(s)$ is an impulse sampled signal as shown in Figure 4.13.2a. It's z-domain equivalent is shown in Figure 4.13.2b.

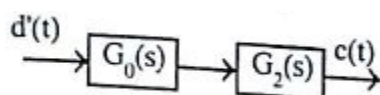


Figure 4.13.2a

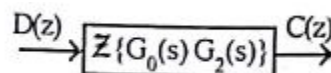


Figure 4.13.2b

From Figure 4.13.12b we get, $C(z) = Z\{G_0(s) G_2(s)\} D(z)$...4.13.1

Where $C(z) = Z\{C(s)\}$; $D(z) = Z\{D'(s)\}$; $C(s) = L[c(t)]$ and $D'(s) = L[d'(t)]$.

With reference to Figure 4.13.1 the following s-domain equations can be obtained.

$$E(s) = R(s) - B(s) \quad \dots 4.13.2$$

$$D(s) = E(s) G_1(s) \quad \dots 4.13.3$$

$$B(s) = G_0(s) G_2(s) H(s) D'(s) \quad \dots 4.13.4$$

On substituting for $E(s)$ from equ (4.13.2) in equ (4.13.3) we get,

$$D(s) = [R(s) - B(s)] G_1(s) = G_1(s) R(s) - G_1(s) B(s) \quad \dots 4.13.5$$

On substituting for $B(s)$ from equ (4.13.4) in equ (4.13.5) we get,

$$D(s) = G_1(s) R(s) - G_1(s) G_0(s) G_2(s) H(s) D'(s) \quad \dots 4.13.6$$

On taking z-transform of equ (4.13.6) we get,

$$D(z) = \mathcal{Z}\{G_1(s) R(s)\} - \mathcal{Z}\{G_1(s) G_0(s) G_2(s) H(s)\} D(z)$$

$$D(z) + \mathcal{Z}\{G_0(s) G_1(s) G_2(s) H(s)\} D(z) = \mathcal{Z}\{G_1(s) R(s)\}$$

$$D(z) [1 + \mathcal{Z}\{G_0(s) G_1(s) G_2(s) H(s)\}] = \mathcal{Z}\{G_1(s) R(s)\}$$

$$\therefore D(z) = \frac{\mathcal{Z}\{G_1(s) R(s)\}}{1 + \mathcal{Z}\{G_0(s) G_1(s) G_2(s) H(s)\}} \quad \dots 4.13.7$$

Note: The term $G_0(s) G_1(s) G_2(s) H(s) D'(s)$ represents the output of a block with transfer from $G_0(s) G_1(s) G_2(s) H(s)$ when the input is $D'(s)$.

On substituting for $D(z)$ from equ (4.13.7) in equ (4.13.1) we get,

$$\text{Output in z-domain, } C(z) = \frac{\mathcal{Z}\{G_0(s) G_2(s)\} \mathcal{Z}\{G_1(s) R(s)\}}{1 + \mathcal{Z}\{G_0(s) G_1(s) G_2(s) H(s)\}} = \frac{G_0 G_2(z) G_1 R(z)}{1 + G_0 G_1 G_2 H(z)}$$

Where $\mathcal{Z}\{G_0(s) G_2(s)\}$ is represented as $G_0 G_2(z)$,

$\mathcal{Z}\{G_1(s) R(s)\}$ is represented as $G_1 R(z)$ and

$\mathcal{Z}\{G_0(s) G_1(s) G_2(s) H(s)\}$ is represented as $G_0 G_1 G_2 H(z)$

EXAMPLE 4.14

For the sampled data control system shown in Figure 4.14, find the response to unit step input, where $G(s) = 1/(s+t)$,

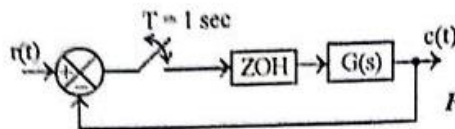


Figure 4.14

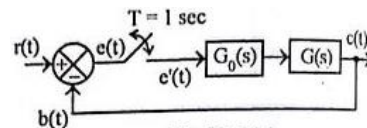


Figure 4.14.1

SOLUTION

The ZOH in the system is replaced by $G_0(s)$ as shown in fig 4.14.1, where $G_0(s) = (1 - e^{-sT})/s$.

The input to the cascaded blocks of $G_0(s)$ and $G(s)$ is an impulse sampled signal as shown in fig 4.14.2a. Its z-domain equivalent is shown in fig 4.14.2b.

From fig 4.14.2b we get, $C(z) = Z\{G_0(s)G(s)\}E(z)$... (4.14.1)

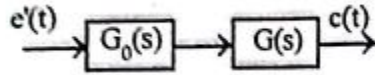


Figure 4.14.2a

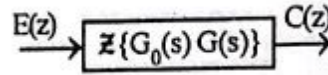


Figure 4.14.2b

From Figure 4.14.2b we get, $C(z) = Z\{G_0(s)G(s)\}E(z)$... 4.14.1

With reference to Figure 4.14.1, at the summing point we get,

$$e(t) = r(t) - c(t) \quad \dots 4.14.2$$

Since $e'(t) = e(kT)$ is an impulse sampled signal, the equation (4.14.2) can be written as,

$$E(kT) = r(kT) - c(kT) \quad \dots 4.14.3$$

where $e(kT)$, $r(kT)$ and $c(kT)$ are impulse sampled signals of $e(t)$, $r(t)$ and $c(t)$ respectively.

On taking z-transform of equ (4.14.3) we get,

$$E(z) = R(z) - C(z) \quad \dots 4.14.4$$

On substituting for $E(z)$ from equ (4.14.4) in equ (4.14.1) we get,

$$\begin{aligned} C(z) &= Z\{G_0(s)G(s)\} [R(z) - C(z)] \\ C(z) &= Z\{G_0(s)G(s)\} R(z) - Z\{G_0(s)G(s)\} C(z) \\ C(z) + Z\{G_0(s)G(s)\} C(z) &= Z\{G_0(s)G(s)\} R(z) \\ C(z) [1 + Z\{G_0(s)G(s)\}] &= Z\{G_0(s)G(s)\} R(z) \\ \therefore C(z) &= \frac{Z\{G_0(s)G(s)\} R(z)}{1 + Z\{G_0(s)G(s)\}} \quad \dots 4.14.5 \end{aligned}$$

$$\text{We know that, } Z\{G_0(s)G(s)\} = (1 - z^{-1}) Z\left\{\frac{G(s)}{s}\right\}$$

$$\text{Here, } G(s) = \frac{1}{s+1} \text{ and } \frac{G(s)}{s} = \frac{1}{s(s+1)}$$

By partial fraction expansion,

$$\frac{G(s)}{s} = \frac{1}{s(s+1)} = \frac{A}{s} + \frac{B}{s+1}$$

$$A = \frac{1}{s(s+1)} s \Big|_{s=0} = \frac{1}{s+1} \Big|_{s=0} = 1$$

$$B = \frac{1}{s(s+1)} \bigg|_{s=-1} = \frac{1}{s} \bigg|_{s=-1} = -1$$

$$\mathcal{Z}\left\{\frac{G(s)}{s}\right\} = \mathcal{Z}\left\{\frac{1}{s} - \frac{1}{s+1}\right\} = \mathcal{Z}\left\{\frac{1}{s}\right\} - \mathcal{Z}\left\{\frac{1}{s+1}\right\}$$

From standard laplace and z-transform pairs we get,

$$\mathcal{Z}\left\{\frac{1}{s}\right\} = \frac{z}{z-1} \quad \text{and} \quad \mathcal{Z}\left\{\frac{1}{s+a}\right\} = \frac{z}{z-e^{-aT}}$$

Here $a = 1$ and $T = 1$

$$\therefore \mathcal{Z}\left\{\frac{G(s)}{s}\right\} = \frac{z}{z-1} - \frac{z}{z-e^{-1}}$$

$$\begin{aligned} \text{Now, } \mathcal{Z}\{G_0(s) G(s)\} &= (1-z^{-1}) \mathcal{Z}\left\{\frac{G(s)}{s}\right\} = (1-z^{-1}) \left(\frac{z}{z-1} - \frac{z}{z-e^{-1}}\right) \\ &= \left(1 - \frac{1}{z}\right) \left(\frac{z(z-e^{-1}) - z(z-1)}{(z-1)(z-e^{-1})}\right) \\ &= \left(\frac{z-1}{z}\right) \left(\frac{z(z-e^{-1} - z + 1)}{(z-1)(z-e^{-1})}\right) = \frac{1-e^{-1}}{z-e^{-1}} = \frac{0.632}{z-0.368} \end{aligned} \quad \dots 4.14.6$$

Given that input is unit step

$$\therefore R(z) = U(z) = \frac{z}{z-1} \quad \dots 4.14.7$$

From equation (4.14.5), (4.14.6) and (4.14.7) we get,

$$\begin{aligned} C(z) &= \frac{\mathcal{Z}\{G_0(s) G(s)\} R(z)}{1 + \mathcal{Z}\{G_0(s) G(s)\}} = \frac{\left(\frac{0.632}{z-0.368}\right) \frac{z}{z-1}}{1 + \left(\frac{0.632}{z-0.368}\right)} \\ &= \frac{\frac{0.632 z}{(z-1)(z-0.368)}}{\frac{(z-0.368) + 0.632}{(z-0.368)}} = \frac{0.632 z}{(z-1)(z-0.368+0.632)} = \frac{0.632 z}{(z-1)(z+0.264)} \\ \therefore \frac{C(z)}{z} &= \frac{0.632}{(z-1)(z+0.264)} \end{aligned}$$

By partial fraction expansion,

$$\begin{aligned} \frac{C(z)}{z} &= \frac{A}{z-1} + \frac{B}{z+0.264} \\ A &= \frac{0.632}{(z-1)(z+0.264)} (z-1) \bigg|_{z=1} = \frac{0.632}{z+0.264} \bigg|_{z=1} = \frac{0.632}{1+0.264} = 0.5 \end{aligned}$$

$$\begin{aligned}
B &= \frac{0.632}{(z-1)(z+0.264)} (z+0.264) \Big|_{z=-0.264} = \frac{0.632}{z-1} \Big|_{z=-0.264} \\
&= \frac{0.632}{-0.264-1} = -0.5 \\
\therefore \frac{C(z)}{z} &= \frac{0.5}{z-1} - \frac{0.5}{z+0.264} \\
C(z) &= 0.5 \frac{z}{z-1} - 0.5 \frac{z}{z-(-0.264)} \quad \dots 4.14.8
\end{aligned}$$

We know that

$$\mathcal{Z}\{1\} = \frac{z}{z-1} \quad \text{and} \quad \mathcal{Z}\{a^k\} = \frac{z}{z-a}$$

On taking inverse z-transform of equ (4.14.8) we get,

$$c(k) = 0.5 - 0.5(-0.264)^k = 0.5 [1 - (-0.264)^k] \quad \dots 4.14.9$$

The equation (4.14.9) is the response of given system for unit step input.

4.12 THE z AND s-DOMAIN RELATONSHIP

Let $r(kT)$ be a discrete sequence which has been obtained by sampling $r(t)$ at a sampling rate of $1/T$. On taking z-transform of $r(kT)$ we get,

$$\mathcal{Z}\{r(kT)\} = R(z) = \sum_{k=0}^{\infty} r(kT) z^{-k} \quad \dots 4.71$$

Let, $r'(t)$ = Impulse sampled signal of $r(t)$ at the sampling rate of $1/T$ and $R'(s) = \mathcal{L}[r'(t)]$ = Laplace transform of $r'(t)$.

$$\text{Now, } r'(t) = \sum_{k=0}^{\infty} r(kT) \delta(t - kT) \quad \dots 4.72$$

On taking laplace transform of equ (4.72) we get,

$$R'(s) = \sum_{k=0}^{\infty} r(kT) e^{-ksT} \quad \dots 4.73$$

Let us choose a transformation such that,

$$z = e^{sT} \quad \dots 4.74$$

$$\therefore \ln z = sT \quad (\text{or}) \quad s = \frac{1}{T} \ln z \quad \dots 4.75$$

On substituting for s from equ (4.75) in equ (4.73) we get,

$$R'(s) = \sum_{k=0}^{\infty} r(kT) e^{(-kT \cdot \frac{1}{T} \ln z)}$$

$$= \sum_{k=0}^{\infty} r(kT) e^{(\ln z)^{-k}} = \sum_{k=0}^{\infty} r(kT) z^{-k} = R(z) \quad \dots 4.76$$

From equ (4.76) it is obvious that z-transform of a discrete sequence can be obtained from the laplace transform of its impulse sampled version, by choosing a transformation, $s=(1/T)\ln z$ (or $z=e^{sT}$).

The transformation, $s=(1/T)\ln z$, maps the s-plane into the z-plane. It can be shown that every section of $j\omega$ -axis of length, $N\omega_s$, maps into the unit circle in the anticlockwise direction where N is an integer and ω_s is the sampling frequency and it can be shown that every strip in the left half s-plane of width ω_s , maps into the interior of the unit circle as shown in fig 4.31.

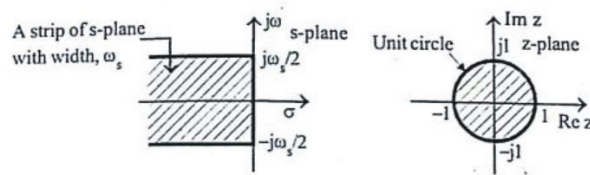


Figure Mapping of s-plane into z-plane

The above mapping helps in extending the s-plane stability criterion to z-plane. For stability of a system in s-plane the poles of s-domain transfer function should lie on the left half of s-plane. In this transformation the left half of s-plane maps into interior of unit circle. Hence for the stability of the system in z-domain, the poles of the z-transfer function should lie inside the unit circle.

4.13 STABILITY ANALYSIS OF SAMPLED DATA CONTROL SYSTEMS

The sampled data control system is stable if all the poles of the z-transfer function of the system lie inside the unit circle in z-plane. The poles of the transfer function are given by the roots of the characteristic equation. Hence the system stability can be determined from the roots of the characteristic equation.

The z-transfer function of the sampled data control system can be expressed as a ratio of two polynomials in z as shown below.

$$z - \text{transfer function, } H(z) = \frac{C(z)}{R(z)} = A_0 \frac{P(z)}{Q(z)} \quad \dots 4.77$$

Where, $A_0 = \text{constant}$

$P(z) = \text{Numerator polynomial}$

$Q(z) = \text{Denominator polynomial}$

The characteristic equation is the denominator polynomial of $H(z)$. [i.e., characteristic equation is given by $Q(z) = 0$].

Consider the system shown in Figure 4.32. For this system, the z-transfer function is given by,

$$H(z) = \frac{C(z)}{R(z)} = \frac{Z\{G_0(s) G(s)\}}{1 + Z\{G_0(s) G(s) H(s)\}} \quad \dots 4.78$$

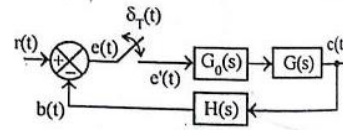


Figure 4.32

and the characteristic equation is,

$$1 + Z\{G_0(s) G(s) H(s)\} = 0 \quad (4.79)$$

The following methods are available for the stability analysis of sampled data control system using the characteristic equation

1. Jury's stability test
2. Bilinear transformation
3. Root locus technique

The Jury's stability test and bilinear transformation are presented in this book.



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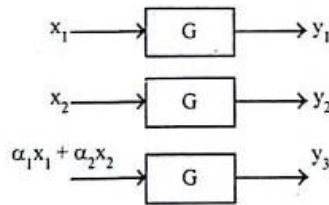
UNIT – V – Advanced Control Systems – SEEA1602

NON LINEAR SYSTEMS

5.1 INTRODUCTION TO NON LINEAR SYSTEMS

The non-linear systems are which does not obey the principle of superposition. The linear systems are systems which satisfy that principle of superposition.

The principle of superposition implies that if a system has responses $y_1(t)$ and $y_2(t)$ any two inputs $x_1(t)$ and $x_2(t)$ respectively then the system response to the linear combination of these inputs $\alpha_1 x_1(t) + \alpha_2 x_2(t)$ is given by the linear combination of the individual outputs, i.e. $\alpha_1 y_1(t) + \alpha_2 y_2(t)$, where α_1 and α_2 are constants.



To satisfy the principle of superposition, $y_3 = \alpha_1 y_1 + \alpha_2 y_2$

Example of linear system : $y = ax + b \frac{dx}{dt}$

Example of nonlinear system : $y = ax^2 + e^{bx}$

EXAMPLE 5.1

The response of a system is, $y = ax + b \frac{dx}{dt}$. Test whether the system is linear or non linear.

SOLUTION

Let x_1 and x_2 be the two inputs to the system and y_1 and y_2 be their responses, respectively.

Given that $y = ax + b \frac{dx}{dt}$

When $x = x_1$, $y = y_1$, $\therefore y_1 = ax_1 + b \frac{dx_1}{dt}$

When $x = x_2$, $y = y_2$, $\therefore y_2 = ax_2 + b \frac{dx_2}{dt}$

Consider a linear combination of inputs $\alpha_1 x_1 + \alpha_2 x_2$ and let the response of the system for this linear combination of inputs be y_3 .

When $x = \alpha_1 x_1 + \alpha_2 x_2$, $y = y_3$

$$\begin{aligned} \therefore y_3 &= a(\alpha_1 x_1 + \alpha_2 x_2) + b \frac{d}{dt}(\alpha_1 x_1 + \alpha_2 x_2) \\ &= \alpha_1 ax_1 + \alpha_2 ax_2 + \alpha_1 b \frac{dx_1}{dt} + \alpha_2 b \frac{dx_2}{dt} \end{aligned}$$

Consider the same linear combination of output, $\alpha_1 y_1 + \alpha_2 y_2$.

$$\begin{aligned}\alpha_1 y_1 + \alpha_2 y_2 &= \alpha_1 \left[ax_1 + b \frac{dx_1}{dt} \right] + \alpha_2 \left[ax_2 + b \frac{dx_2}{dt} \right] \\ &= \alpha_1 ax_1 + \alpha_2 ax_2 + \alpha_1 b \frac{dx_1}{dt} + \alpha_2 b \frac{dx_2}{dt}\end{aligned}$$

It is observed that $y_3 = \alpha_1 y_1 + \alpha_2 y_2$. Hence the system is linear.

EXAMPLE 5.2

The response of a system is $y = ax^2 + e^{bx}$. Test whether the system is linear or nonlinear.

SOLUTION

Let x_1 and x_2 be two inputs to the system and y_1 and y_2 be their responses respectively.

Given that $y = ax^2 + e^{bx}$

$$\text{When } x = x_1, y = y_1, \therefore y_1 = ax_1^2 + e^{bx_1}$$

$$\text{When } x = x_2, y = y_2, \therefore y_2 = ax_2^2 + e^{bx_2}$$

Consider a linear combination of inputs $\alpha_1 x_1 + \alpha_2 x_2$ and let the response of the system for this linear combination of inputs be y_3 .

$$\begin{aligned}\text{When } x &= \alpha_1 x_1 + \alpha_2 x_2, y = y_3 \\ \therefore y_3 &= a (\alpha_1 x_1 + \alpha_2 x_2)^2 + e^{b(\alpha_1 x_1 + \alpha_2 x_2)} \\ &= a (\alpha_1^2 x_1^2 + \alpha_2^2 x_2^2 + 2\alpha_1 x_1 \alpha_2 x_2) + e^{\alpha_1 bx_1} \cdot e^{\alpha_2 bx_2} \\ &= a\alpha_1^2 x_1^2 + a\alpha_2^2 x_2^2 + 2a\alpha_1 \alpha_2 x_1 x_2 + e^{\alpha_1 bx_1} \cdot e^{\alpha_2 bx_2}\end{aligned}$$

Consider the same linear combination of output, $\alpha_1 y_1 + \alpha_2 y_2$

$$\begin{aligned}\alpha_1 y_1 + \alpha_2 y_2 &= \alpha_1 [ax_1^2 + e^{bx_1}] + \alpha_2 [ax_2^2 + e^{bx_2}] \\ &= a\alpha_1 x_1^2 + \alpha_1 e^{bx_1} + a\alpha_2 x_2^2 + \alpha_2 e^{bx_2}\end{aligned}$$

It is observed that $y_3 \neq \alpha_1 y_1 + \alpha_2 y_2$. Hence the system is nonlinear.

In all practical engineering systems, there will be always some nonlinearity due to friction, inertia, stiffness, backflash, hysteresis, saturation and dead-zone. The effect of the non linear components can be avoided by restricting the operation of the component over a narrow limited range. Moreover most of the automatic control systems operate within a narrow range, e.g. the speed controller of an electric drive for constant speed operation of 1500 rpm will be required to operate between 1450 to 1550 rpm. Similarly, automatic voltage controller will be operating within $\pm 5\%$ of the specified voltage. Thus the characteristics of components may be considered as linear over this limited range.

Further, some components behave linearly over its working range, e.g., a spring when loaded, gets extended. As the load is being increased the load-displacement curve is linear within the working range. However, when the load is increased beyond the maximum of the working

range, the spring material starts to yield and it becomes permanently deformed. It can be concluded that the spring behaves linearly over its working range and beyond this range it is nonlinear.

Although nonlinearities in systems may generally be due to imperfections of a physical device, some times we deliberately introduce nonlinear device or operate the linear devices in nonlinear regions with a view to improve system performance.

The characteristics of nonlinear system are given below.

1. The response of nonlinear system to a particular test signal is no guide to their behaviour to other inputs, since the principle of superposition does not hold good for nonlinear systems.
2. The nonlinear system response may be highly sensitive to input amplitude. The stability study of nonlinear systems requires the information about the type and amplitude of the anticipated inputs, initial conditions, etc., in addition to the usual requirement of the mathematical model.
3. The nonlinear systems may exhibit cycles which are self sustained oscillations of fixed frequency and amplitude.
4. The nonlinear systems may have jump resonance in the frequency response.
5. The output of a nonlinear system will have harmonics and sub-harmonics when excited by sinusoidal signals.
6. The nonlinear systems will exhibit phenomena like frequency entrainment and asynchronous quenching.

BEHAVIOUR OF NONLINEAR SYSTEMS

In nonlinear systems, the response (output) depends on the magnitude and type of input signal. The principle of superposition will not hold good for nonlinear systems. The nonlinear systems may exhibit various phenomena like jump resonance, sub harmonic oscillation, limit cycles, frequency entrainment and asynchronous quenching. The various phenomena that occur in nonlinear system are explained in this section.

Frequency-amplitude dependence

The frequency-amplitude dependence is one of the most fundamental characteristics of the oscillations of nonlinear systems. The frequency-amplitude dependence can be best studied by considering the mechanical system shown in Figure 5.1 in which the spring is nonlinear. The differential equation governing the dynamic of the system may be written as

$$M \ddot{x} + B \dot{x} + Kx + K'x^3 = 0 \quad \dots 5.1$$

where $Kx + K'x^3$ – Opposing force due to nonlinear spring.

The parameters M , B and K are positive constants. The parameters K' may be positive or negative. If K' is positive, the spring is called hard spring and if K' is negative the spring is called soft spring. The equation (5.1) is nonlinear differential equation and it also called Duffing's equation.

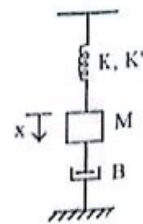


Figure 2.1 Mechanical system with nonlinear spring

When the system of Figure 5.1 has non zero initial conditions, the free response (i.e., solution of equ 5.1) is damped oscillatory. The frequency of free oscillations depends on the amplitude of oscillations. When $K' < 0$ (soft spring) the frequency decreases with decreasing amplitude. When $K' > 0$ (hard spring) the frequency increases with decreasing amplitude. When $K' = 0$ (corresponding to linear system) the frequency remains unchanged as the amplitude of free oscillation decreases. The frequency-amplitude dependence characteristic of nonlinear mechanical system of Fig. 5.1 is shown in Fig. 5.2

Jump resonance

In the frequency response of nonlinear systems, the amplitude of the response (output) may jump from one point to another for increasing or decreasing values of frequency, ω . This phenomenon is called jump resonance and it can be observed in the frequency response of the system shown in Fig. 5.1., when it is subjected to sinusoidal input.

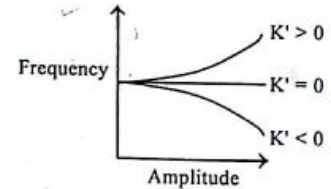


Figure 5.2 Amplitude vs frequency curves for free oscillations in the system described by equation 5.1

Let the mechanical system of Fig. 5.1, be subjected to an input of type $A \cos \omega t$. Now the differential equation governing the mechanical system is

$$M \ddot{x} + B \dot{x} + Kx + K'x^3 = A \cos \omega t \quad \dots 5.2$$

Let X be the amplitude of the response or output of the system. In frequency response studies, the amplitude, A of the input is held constant, while its ω is varied and the amplitude, X of the output is observed. The frequency response curve is plotted between X and ω . The frequency response curves of the mechanical system of fig 5.1 are shown in fig 5.3a and 5.3b for hard and soft springs respectively.

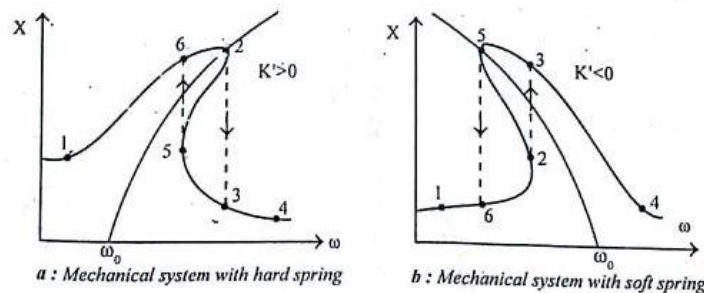


Figure 5.3 Frequency response curves showing jump resonance

In the frequency response curve shown in fig 5.3a and b, as the frequency ω is increased, the amplitude X increases, until point-2 is reached, A further increase in frequency will cause a jump from point-2 to point-3. This phenomenon is called jump resonance. As the frequency is increased further, the amplitude X follows the curve from point-3 towards point-4.

When the frequency is reduced starting from a high value corresponding to point-4, the amplitude X slowly increases through point-3, until point-5 is reached. A further decrease in ω will cause another jump from point-5 to point-6. This phenomenon is called jump resonance. After this jump, the amplitude X decreases with ω and follows the curve from point-6 towards point-1.

For jump resonance to take place, it is necessary that the damping term be small and the amplitude of the forcing function be large enough to drive the system into a region of appreciably nonlinear operation.

Subharmonic oscillations

When an nonlinear system is excited by a sinusoidal signal, the response or output will have steady-state oscillation whose frequency is an integral submultiple of the forcing frequency. These oscillations are called sub harmonic oscillations. The generation of sub harmonic oscillations depends on the system parameters and initial conditions. It also depends on amplitude and frequency of the forcing functions.

Limit cycles

The response (or output) of nonlinear systems may exhibit oscillations with fixed amplitude and frequency. These oscillations are called limit cycles. Consider a mechanical system with nonlinear damping and described by the equation,

$$M \ddot{x} + B(1-x^2) \dot{x} + Kx = 0 \quad \dots 5.3$$

where M , B and K are positive constants. The equation (5.3) is called the van der pol equation. For small values of x the damping will be negative which implies the stored energy in the damper is fed to the system. For large values of x the damping is positive which implies that it absorbs energy from the system. Thus, it can be expected that such a system may exhibit a sustained oscillation. Since the system explained above is not a forced system, this oscillation is called a self-excited oscillation or zero input limit cycle.

Frequency entrainment

The phenomena of frequency entrainment is observed in the frequency response of nonlinear systems that exhibit limit cycles. Consider a system capable of exhibiting a limit cycle of frequency ω_1 . If a periodic input of frequency ω is applied to this system then the phenomenon of beats is observed. [The beat is the oscillation whose frequency is the difference between ω_1 and ω . This frequency is also called beat frequency]. In linear systems, the beat frequency decreases indefinitely as ω approaches ω_1 . But in nonlinear systems, the frequency ω_1 of the limit cycle falls in synchronistically with or is entrained by the forcing frequency, ω within a certain band of frequencies. This phenomenon is called frequency entrainment. The band of frequency in which entrainment occurs is called the zone of frequency entrainment. In this zone, the frequencies ω and ω_1 coalesce and only one frequency, ω exists. The relationship between $|\omega - \omega_1|$ and ω is shown in Figure 5.4.

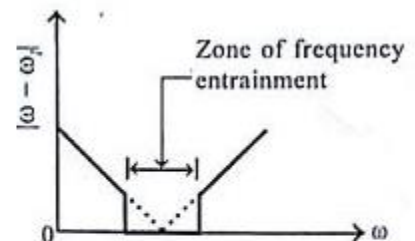


Figure 5.4 $|\omega - \omega_1|$ vs ω curve showing the zone of frequency entrainment

Asynchronous quenching

In a nonlinear system that exhibits a limit cycle of frequency, ω_1 it is possible to quench (stop or eliminate) the limit cycle oscillation by forcing the system of a frequency ω_q , where ω_q and ω_1 are not related to each other. The phenomenon is called signal stabilization or asynchronous quenching.

INVESTIGATION OF NONLINEAR SYSTEMS

For analysis, the nonlinear system can be approximated by a linear model in the entire operating region. The nonlinear systems can be piecewise approximated. Each piece can be analysed by a differential equation governing the systems.

The two popular methods of analysing nonlinear systems are phase-plane method and describing function method.

The phase plane method is basically a graphical method from which information about transient behaviour and stability is easily obtained by constructing phase trajectories. This method is restricted to second order systems. Higher order systems may first be approximated by their second-order equivalent for investigation by the phase plane method.

The Describing function method is based on harmonic linearization. Here the input to nonlinear component is sinusoidal and depending upon the filtering properties of the linear part of the overall system, the output is adequately represented by the fundamental frequency term in fourier series.

The phase-plane and describing function methods use complimentary approximations. The phase-plane method retains the nonlinearity as such and uses the second-order approximation of a higher-order linear part, while on the other hand, the describing function method retains the linear part and harmonically linearizes the nonlinearity.

COMMON PHYSICAL NONLINEARITIES

The nonlinearities can be classified as incidental and intentional.

The incidental nonlinearities are those which are inherently present in the system. Common examples of incidental nonlinearities are saturation, dead-zone, coulomb friction, stiction, backlash, etc.

The intentional nonlinearities are those which are deliberately inserted in the system to modify system characteristics. The most common example of this type of nonlinearity is a relay.

SATURATION: In this type nonlinearity the output is proportional to input for a limited range of input signals. When the input exceeds this range, the output tends to become nearly constant as shown in Figure 5.5.

All devices when driven by sufficiently large signals, exhibit the phenomenon of saturation due to limitations of their physical capabilities. Saturation in the output of electronic, rotating and flow (hydraulic and pneumatic) amplifiers, speed and torque saturation in electric and hydraulic motors, saturation in the output of sensors for measuring position, velocity, temperature etc., are the well known examples.

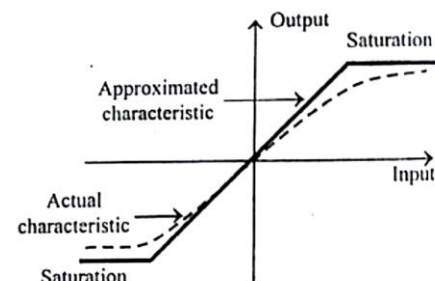


Figure 5.5 Saturation

DEADZONE: The deadzone is the region in which the output is zero for a given input. Many physical devices do not respond to small signals, i.e., if the input amplitude is less than

some small value, there will be no output. The region in which the output is zero is called deadzone. When the input is increased beyond this deadzone value, the output will be linear.

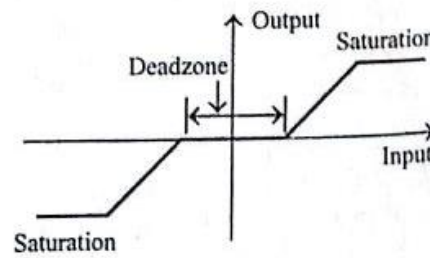
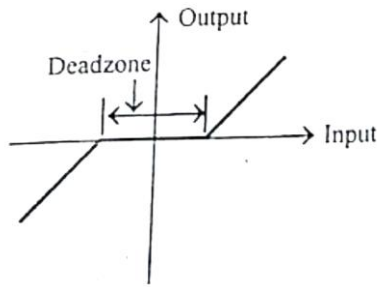


Figure 5.6: Dead zone nonlinearity **Figure 5.7: Dead zone and saturation nonlinearity**

The Figure 5.6 shows the deadzone nonlinearity and the Figure 5.7 shows the combination of dead zone and saturation nonlinearity.

FRICTION: Friction exists in any system when there is relative motion between contacting surfaces. The different types of friction are viscous friction, coulomb friction and stiction.

The viscous friction is linear in nature and the frictional force is directly proportional to relative velocity of the sliding surfaces.

The coulomb friction and stiction are nonlinear frictions. The coulomb friction offers a constant retarding force only when the motion is initiated. Due to interlocking of surface irregularities, more force is required to move an object from rest than to maintain it in motion. Hence the force of stiction is always greater than that of coulomb friction.

In actual practice, the stiction force gradually decreases with velocity and changes over to coulomb friction at reasonably low velocities as shown in Figure 5.10. The composite characteristics of various frictions are shown in Figure 5.8 to 5.11.

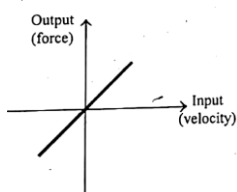


Figure 5.8: Viscous friction

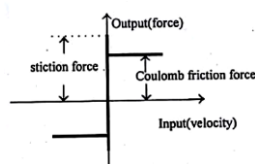


Figure 5.9: Ideal stiction and coulomb friction

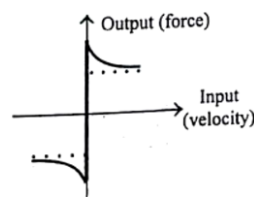


Figure 5.10: Actual stiction and coulomb friction

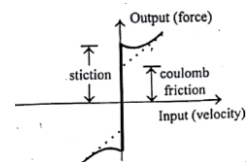


Figure 2.11: Stiction, coulomb friction and viscous friction

5.2 DESCRIBING FUNCTION

Consider the block diagram of the nonlinear system shown in Figure 5.12

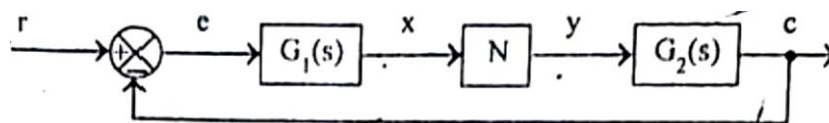


Figure 5.12: A nonlinear system

In the above system the block $G_1(s)$ and $G_2(s)$ represents linear elements and the block N represent nonlinear element.

Let $x = X \sin \omega t$ be the input to nonlinear element. Now the output y of the nonlinear element will be in general a nonsinusoidal periodic function. The fourier series representation of the output y can be expressed as (by assuming that the nonlinearity does not generate sub harmonics).

$$y = A_0 + A_1 \sin \omega t + B_1 \cos \omega t + A_2 \sin 2\omega t + B_2 \cos 2\omega t + \dots \quad \dots 5.4$$

If the nonlinearity is symmetrical the average value of y is zero and hence the output y is given by

$$y = A_1 \sin \omega t + B_1 \cos \omega t + A_2 \sin 2\omega t + B_2 \cos 2\omega t + \dots \quad \dots 5.5$$

In the absence of an external input (i.e., when $r = 0$) the output y of the nonlinearity N is feedback to its input through the linear element $G_2(s)$ and $G_1(s)$ in tandem. If $G_1(s) G_2(s)$ has low-pass characteristics, then all the harmonics of y are filtered, so that the input x to the nonlinear element N is mainly contributed by the fundamental component of y and hence x remains sinusoidal. Under such conditions the harmonics of the output are neglected and the fundamental components of y alone considered for the purpose of analysis.

$$\therefore y = y_1 = A_1 \sin \omega t + B_1 \cos \omega t = Y_1 \angle \phi_1 = Y_1 \sin (\omega t + \phi_1) \quad \dots 5.6$$

$$\text{where, } Y_1 = \sqrt{A_1^2 + B_1^2} \quad \dots 5.7$$

$$\text{and } \phi_1 = \tan^{-1} \frac{B_1}{A_1} \quad \dots 5.8$$

Y_1 = Amplitude of the fundamental harmonic component of the output.

ϕ_1 = Phase shift of the fundamental harmonic component of the output with respect to the input.

The coefficient A_1 and B_1 of the fourier series are given by

$$A_1 = \frac{2}{2\pi} \int_0^{2\pi} y \sin \omega t \, d(\omega t) \quad \dots 5.9$$

$$B_1 = \frac{2}{2\pi} \int_0^{2\pi} y \cos \omega t \, d(\omega t) \quad \dots 5.10$$

When the input, x to the nonlinearity is sinusoidal (i.e., $x = X \sin \omega t$) the describing function of the nonlinearity is defined as,

$$K_N(X, \omega) = \frac{Y_1}{X} \angle \phi_1 \quad \dots 5.11$$

The nonlinear element N in the system can be replaced by the describing function as shown in Figure 5.13.

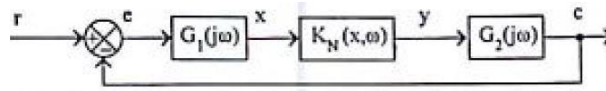


Figure 5.13: Nonlinear system with non linearity replaced by describing function

If the nonlinearity is replaced by a describing function then all linear theory frequency domain technique can be used for the analysis of the system. The describing functions are used only for stability analysis and it is not directly applied to the optimization of system design. The describing function is a frequency domain approach and no general correlation is possible between time and frequency responses.

5.3 DESCRIBING FUNCTION OF DEAD-ZONE AND SATURATION NONLINEARITY

The input and the output relationship of nonlinearity with dead-zone and saturation is shown in Figure 5.14.

The dead-zone region is from $x = -D/2$ to $+D/2$ and in this region the output is zero. The input-output relation is linear for $x = \pm D/2$ to $\pm S$ and when the input, $x > S$, the output reaches a saturated value of $\pm K(S-D/2)$.

The output equation for the linear region can be obtained from the general equation of straight line as shown below.

The equation of straight lines is, $y = mx + c$...5.12

In the linear region, when $x = D/2$, $y = 0$. On substituting this values of x and y in equ (5.12) we get,

$$0 = mD/2 + c \quad \dots 5.13$$

In the linear region, when $x = S$, $y = K(S-D/2)$. On substituting this values of x and y in equ (5.12) we get,

$$K(S-D/2) = mS + c \quad \dots 5.14$$

Equ (5.14) – equ (5.13) yields,

$$K(S - \frac{D}{2}) = mS + c - m\frac{D}{2} - c$$

$$K(S - \frac{D}{2}) = m(S - \frac{D}{2})$$

$$\therefore m = K$$

...5.15

$$\text{Put } m = K \text{ in eqn(5.13), } \therefore 0 = K\frac{D}{2} + c \quad (\text{or}) \quad c = -K\frac{D}{2}$$

...5.16

From equations (5.12), (5.15) and (5.16) the output equation for the linear region can be written as,

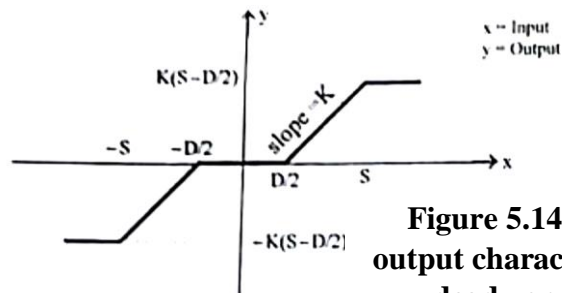


Figure 5.14 Input-output characteristic of dead-zone and

$$y = mx + c = Kx - K\frac{D}{2} = K(x - \frac{D}{2}) \quad \dots 2.17$$

The response or output of the non linearity when the input is sinusoidal signal ($x = X \sin \omega t$) is shown in Figure 5.15.

$$\text{The input } x \text{ is sinusoidal } \therefore x = X \sin \omega t \quad \dots 2.18$$

Where X = Maximum value of input.

In fig 2.15, when, $\omega t = \alpha$, $x = D/2$

Hence from equ (2.18) we get

$$D/2 = X \sin \alpha$$

$$\sin \alpha = D/2X$$

$$\therefore \alpha = \sin^{-1} \frac{D}{2X} \quad \dots (2.19)$$

In fig 2.15 when $\omega t = \beta$, $x =$

Hence from equ (2.18) we get

$$S = X \sin \beta$$

$$\sin \beta = S/X$$

$$\therefore \beta = \sin^{-1} \frac{S}{X} \quad \dots (2.20)$$

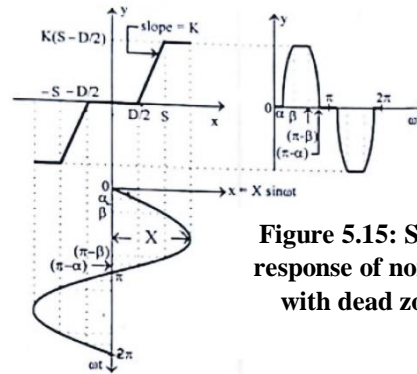


Figure 5.15: Sinusoidal response of nonlinearity with dead zone and

The output y of the nonlinearity can be divided five regions in a period of π and the output equation for the five regions are given below.

$$y = \begin{cases} 0 & ; 0 \leq \omega t \leq \alpha \\ K(x - \frac{D}{2}) & ; \alpha \leq \omega t \leq \beta \\ K(S - \frac{D}{2}) & ; \beta \leq \omega t \leq (\pi - \beta) \\ K(x - \frac{D}{2}) & ; (\pi - \beta) \leq \omega t \leq (\pi - \alpha) \\ 0 & ; (\pi - \alpha) \leq \omega t \leq \pi \end{cases}$$

Let Y_1 = Amplitude of the fundamental harmonic component of the output.

ϕ_1 = Phase shift of the fundamental harmonic component of the output with respect to the input

The describing function is given by

$$K_N(X, \omega) = \frac{Y_1}{X} \angle \phi_1$$

$$\text{where, } Y_1 = \sqrt{A_1^2 + B_1^2} \quad \text{and} \quad \phi_1 = \tan^{-1} \frac{B_1}{A_1}$$

$$A_1 = \frac{2}{2\pi} \int_0^{2\pi} y \sin \omega t d(\omega t) \quad \text{and} \quad B_1 = \frac{2}{2\pi} \int_0^{2\pi} y \cos \omega t d(\omega t)$$

On substituting for D/2 and S from equations (5.26) and (5.27) in equ (5.25) we get,

$$\begin{aligned}
 A_1 &= \frac{4K}{\pi} \left[\frac{X}{2} \left(\beta - \alpha - \frac{\sin 2\beta}{2} + \frac{\sin 2\alpha}{2} \right) - X \sin \alpha \cos \alpha + X \sin \beta \cos \beta \right] \\
 &= \frac{4KX}{\pi} \left[\frac{\beta}{2} - \frac{\alpha}{2} - \frac{\sin 2\beta}{4} + \frac{\sin 2\alpha}{4} - \frac{\sin 2\alpha}{2} + \frac{\sin 2\beta}{2} \right] \\
 &= \frac{4KX}{\pi} \left[\frac{1}{2}(\beta - \alpha) + \frac{\sin 2\beta}{4} - \frac{\sin 2\alpha}{4} \right] \\
 &= \frac{2KX}{\pi} \left[(\beta - \alpha) + \frac{\sin 2\beta}{2} - \frac{\sin 2\alpha}{2} \right] \\
 &= \frac{KX}{\pi} [2(\beta - \alpha) + \sin 2\beta - \sin 2\alpha] \quad \dots 5.28
 \end{aligned}$$

$$\begin{aligned}
 Y_1 &= \sqrt{A_1^2 + B_1^2} = \sqrt{A_1^2 + 0} = A_1 \\
 \therefore Y_1 &= A_1 = \frac{KX}{\pi} [2(\beta - \alpha) + \sin 2\beta - \sin 2\alpha] \quad \dots 5.29
 \end{aligned}$$

$$\phi_1 = \tan^{-1} \frac{B_1}{A_1} = \tan^{-1} 0 = 0 \quad \dots 5.30$$

$$\text{The describing function } K_N(X, \omega) = \frac{Y_1}{X} \angle \phi_1 \quad \dots 5.31$$

On substituting for Y_1 and ϕ_1 from equations (5.29) and (5.30) in equ (5.31) we get

$$K_N(X, \omega) = \frac{Y_1}{X} \angle \phi_1 = \frac{K}{\pi} [2(\beta - \alpha) + \sin 2\beta - \sin 2\alpha] \angle 0^\circ \quad \dots 5.32$$

Depending on the maximum value of input, X the describing function of equ (5.32) can be written as,

$$\text{If } X < \frac{D}{2}, \text{ then } \alpha = \beta = \frac{\pi}{2} \text{ and } K_N(X, \omega) = 0 \quad \dots 5.33$$

$$\text{If } \frac{D}{2} < X < S, \text{ then } \beta = \frac{\pi}{2} \text{ and } K_N(X, \omega) = K \left[1 - \frac{2}{\pi} (\alpha + \sin \alpha \cos \alpha) \right] \quad \dots 5.34$$

$$\text{If } X > S, \quad K_N(X, \omega) = \frac{K}{\pi} [2(\beta - \alpha) + \sin 2\beta - \sin 2\alpha] \quad \dots 5.35$$

5.4 DESCRIBING FUNCTION OF SATURATION NONLINEARITY

The input-output relationship of saturation nonlinearity is shown in Figure 5.16.

The input-output relation is linear for $x = 0$ to S . When the input $x > S$, the output reaches a saturated value of KS .

The response of the nonlinearity when the input is sinusoidal signal ($x = X \sin \omega t$) is shown in Figure 5.17.

The input x is sinusoidal,

$$\therefore x = X \sin \omega t \quad \dots (5.36)$$

Where X is the maximum value of input.

In Figure (5.17), when $\omega t = \beta$, $x = S$.

Hence equ (5.36) can be written as, $S = X \sin \beta$
...5.37

$$\therefore \sin \beta = \frac{S}{X} \quad (\text{or}) \quad \beta = \sin^{-1} \left(\frac{S}{X} \right) \quad \dots 5.38$$

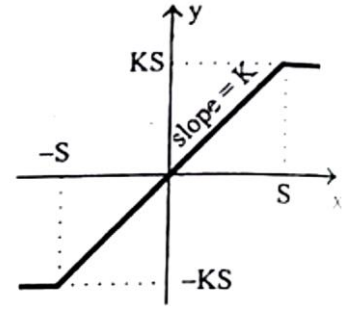


Figure 5.16 Input-output characteristics of saturation nonlinearity

The output y of the nonlinearity can be divided into three regions in a period of π . The output equation for the three regions are given by equ (5.39).

$$y = \begin{cases} Kx & ; 0 \leq \omega t \leq \beta \\ KS & ; \beta \leq \omega t \leq (\pi - \beta) \\ Kx & ; (\pi - \beta) \leq \omega t \leq \pi \end{cases} \quad \dots 5.39$$

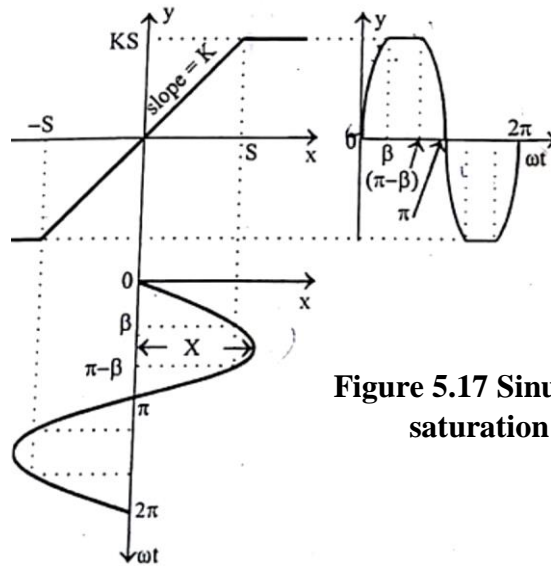


Figure 5.17 Sinusoidal response of saturation nonlinearity

Let Y_1 = Amplitude of the fundamental harmonic component of the output.

ϕ = Phase shift of the fundamental harmonic component of the output with respect to the input.

The describing function is given by, $K_N(X, \omega) = (Y_1 / X) \angle \phi_1$

where $Y_1 = \sqrt{A_1^2 + B_1^2}$ and $\phi_1 = \tan^{-1} (B_1 / A_1)$

The output y has half wave and quarter wave symmetries

$$\therefore B_1 = 0 \quad \text{and} \quad A_1 = \frac{2}{\pi/2} \int_0^{\pi/2} y \sin \omega t \, d(\omega t) \quad \dots 2.40$$

The output, y is given by two different expressions in the period 0 to $\pi/2$. Hence equ (5.40) can be written as shown in equ (5.41).

$$A_1 = \frac{4}{\pi} \int_0^\beta y \sin \omega t \, d(\omega t) + \frac{4}{\pi} \int_\beta^{\pi/2} y \sin \omega t \, d(\omega t) \quad \dots 2.41$$

On substituting the values of y from equ (5.39) in equ (5.41) we get,

$$A_1 = \frac{4}{\pi} \int_0^\beta Kx \sin \omega t \, d(\omega t) + \frac{4}{\pi} \int_\beta^{\pi/2} KS \sin \omega t \, d(\omega t)$$

On substituting $x = X \sin \omega t$, we get

$$\begin{aligned} A_1 &= \frac{4K}{\pi} \int_0^\beta X \sin \omega t \times \sin \omega t \, d(\omega t) + \frac{4KS}{\pi} \int_\beta^{\pi/2} \sin \omega t \, d(\omega t) \\ &= \frac{4KX}{\pi} \int_0^\beta \sin^2 \omega t \, d(\omega t) + \frac{4KS}{\pi} \int_\beta^{\pi/2} \sin \omega t \, d(\omega t) \\ &= \frac{4KX}{\pi} \int_0^\beta \frac{1 - \cos 2\omega t}{2} \, d(\omega t) + \frac{4KS}{\pi} \int_\beta^{\pi/2} \sin \omega t \, d(\omega t) \\ &= \frac{2KX}{\pi} \left[\omega t - \frac{\sin 2\omega t}{2} \right]_0^\beta + \frac{4KS}{\pi} [-\cos \omega t]_\beta^{\pi/2} \\ &= \frac{2KX}{\pi} \left[\beta - \frac{\sin 2\beta}{2} \right] + \frac{4KS}{\pi} \left[-\cos \frac{\pi}{2} + \cos \beta \right] \\ &= \frac{2KX}{\pi} \left[\beta - \frac{\sin 2\beta}{2} \right] + \frac{4KS}{\pi} \cos \beta \end{aligned} \quad \dots 5.42$$

On substituting for S , (i.e., $S = X \sin \beta$) from equ (5.37) in equ (5.42) we get,

$$\begin{aligned} A_1 &= \frac{2KX}{\pi} \left[\beta - \frac{\sin 2\beta}{2} \right] + \frac{4K}{\pi} X \sin \beta \cos \beta \\ &= \frac{2KX}{\pi} \left[\beta - \frac{2\sin \beta \cos \beta}{2} \right] + \frac{4KX}{\pi} \sin \beta \cos \beta \\ &= \frac{2KX}{\pi} [\beta - \sin \beta \cos \beta + 2\sin \beta \cos \beta] \\ &= \frac{2KX}{\pi} [\beta + \sin \beta \cos \beta] \end{aligned} \quad \dots 5.43$$

$$Y_1 = \sqrt{A_1^2 + B_1^2} = \sqrt{A_1^2 + 0} = A_1 = \frac{2KX}{\pi} [\beta + \sin \beta \cos \beta] \quad \dots 5.44$$

$$\phi_1 = \tan^{-1} \frac{B_1}{A_1} = \tan^{-1} 0 = 0 \quad \dots 5.45$$

$$\text{The describing function } K_N(X, \omega) = \frac{Y_1}{X} \angle \phi_1 \quad \dots 5.46$$

Using equations (5.44) and (5.45), the describing function of equ (5.46) can be written as,

$$K_N(X, \omega) = \frac{Y_1}{X} \angle \phi_1 = \frac{2K}{\pi} [\beta + \sin \beta \cos \beta] \angle 0^\circ \quad \dots 5.47$$

Depending on the maximum value of input X, the describing function can be written as,

$$\text{If } X < S, \text{ then } \beta = \frac{\pi}{2}, \quad K_N(X, \omega) = K \quad \dots 5.48$$

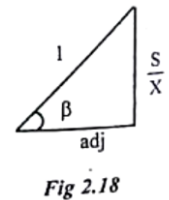
$$\text{If } X > S, \quad K_N(X, \omega) = \frac{2K}{\pi} [\beta + \sin \beta \cos \beta] \quad \dots 5.49$$

The equation (5.49) can be expressed in another form as shown below.

$$\text{From equ(2.37) we get, } S = X \sin \beta, \quad \therefore \sin \beta = \frac{S}{X} \quad \dots 5.50$$

On constructing right angle triangle with unity hypotenuse is shown in Figure 5.18, $\cos \beta$ can be evaluated. From Figure 5.28 we get.

$$\text{adj} = \sqrt{1 - \left(\frac{S}{X}\right)^2} \quad \therefore \cos \beta = \frac{\text{adj}}{\text{hyp}} = \sqrt{1 - \left(\frac{S}{X}\right)^2} \quad \dots 5.51$$



In the describing function of equ (5.49), substitute for β , $\sin \beta$ and $\cos \beta$ from equations (5.38), (5.50), and (5.51)

$$\therefore K_N(X, \omega) = \frac{2K}{\pi} \left[\sin^{-1} \left(\frac{S}{X} \right) + \left(\frac{S}{X} \right) \sqrt{1 - \left(\frac{S}{X} \right)^2} \right] \quad \text{for } X > S \quad \dots 5.52$$

5.5 DESCRIBING FUNCTION OF DEAD-ZONE NONLINEARITY

The input-output relationship of dead-zone nonlinearity is shown in Figure 5.19. The output is zero, when the input is less than $D/2$. The input-output relationship is linear when the input is greater than $D/2$. The response of the nonlinearity when input is sinusoidal signal ($x = X \sin \omega t$) is shown in Figure 5.20.

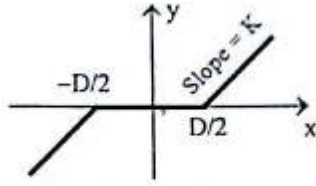


Figure 5.19 Input-output characteristic of dead-zone

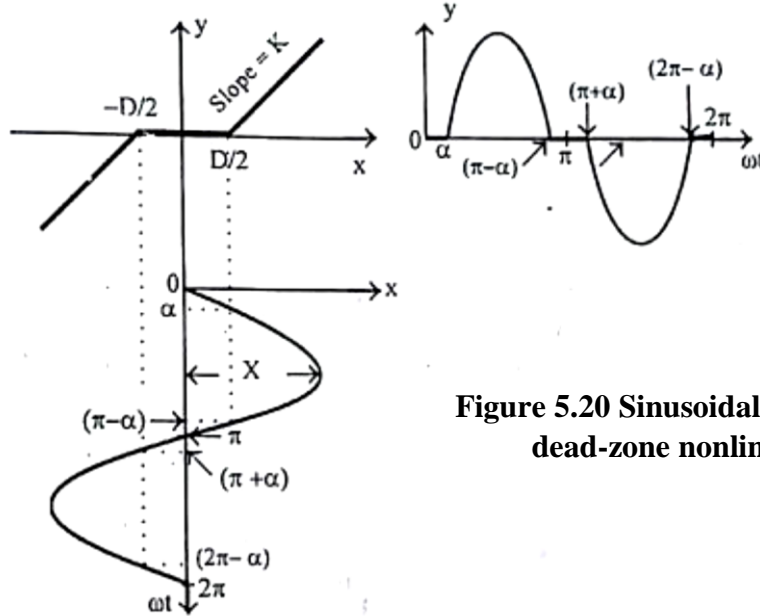


Figure 5.20 Sinusoidal response of dead-zone nonlinearity

The input x is sinusoidal, $\therefore x = X \sin \omega t$...5.53

Where X is the maximum value of input

In Figure 5.20, when $\omega t = \alpha$, $x = D/2$

Hence when $\omega t = \alpha$, the equ (5.53) can be written as, $D/2 = X \sin \alpha$

$$\therefore \sin \alpha = \frac{D}{2X} \quad \dots 5.55$$

$$\text{and } \alpha = \sin^{-1} \frac{D}{2X} \quad \dots 5.56$$

The output y can be divided into three regions in a period of π . The output equation for the three regions are given by equ (5.27).

$$y = \begin{cases} 0 & ; 0 \leq \omega t \leq \alpha \\ K(x - \frac{D}{2}) & ; \alpha \leq \omega t \leq (\pi - \alpha) \\ 0 & ; (\pi - \alpha) \leq \omega t \leq \pi \end{cases} \quad \dots 5.57$$

Let Y_1 = Amplitude of the fundamental harmonic component of the output.

ϕ_1 = Phase shift of the fundamental harmonic component of the output with respect to the input.

The describing function is given by, $K_N(X, \omega) = (Y_1 / X) \angle \phi_1$

where $Y_1 = \sqrt{A_1^2 + B_1^2}$ and $\phi_1 = \tan^{-1} (B_1 / A_1)$

The output y has half wave and quarter wave symmetries

$$\therefore B_1 = 0 \quad \text{and} \quad A_1 = \frac{2}{\pi} \int_0^{\pi/2} y \sin \omega t \, d(\omega t) \quad \dots 5.58$$

Since the output, y is zero in the range, $0 \leq \omega t \leq \alpha$, the limits on integration in equ (5.58) can be changed to, α to $\pi/2$ instead of, 0 to $\pi/2$.

$$\therefore A_1 = \frac{4}{\pi} \int_{\alpha}^{\pi/2} K(x - \frac{D}{2}) \sin \omega t \, d(\omega t) \quad \dots 5.59$$

Put $x = X \sin \omega t$ in equ (5.59)

$$\begin{aligned} \therefore A_1 &= \frac{4K}{\pi} \left[\int_{\alpha}^{\pi/2} \left(X \sin \omega t - \frac{D}{2} \right) \sin \omega t \, d(\omega t) \right] \\ &= \frac{4K}{\pi} \left[\int_{\alpha}^{\pi/2} X \sin^2 \omega t \, d(\omega t) - \frac{D}{2} \int_{\alpha}^{\pi/2} \sin \omega t \, d(\omega t) \right] \\ &= \frac{4K}{\pi} \left[\frac{X}{2} \int_{\alpha}^{\pi/2} (1 - \cos 2\omega t) \, d(\omega t) - \frac{D}{2} \int_{\alpha}^{\pi/2} \sin \omega t \, d(\omega t) \right] \\ &= \frac{4K}{\pi} \left[\frac{X}{2} \left[\omega t - \frac{\sin 2\omega t}{2} \right]_{\alpha}^{\pi/2} - \frac{D}{2} \left[-\cos \omega t \right]_{\alpha}^{\pi/2} \right] \\ &= \frac{4K}{\pi} \left[\frac{X}{2} \left(\frac{\pi}{2} - \frac{\sin \pi}{2} - \alpha + \frac{\sin 2\alpha}{2} \right) - \frac{D}{2} (-\cos \frac{\pi}{2} + \cos \alpha) \right] \\ &= \frac{4K}{\pi} \left[\frac{X}{2} \left(\frac{\pi}{2} - \alpha + \frac{\sin 2\alpha}{2} \right) - \frac{D}{2} (\cos \alpha) \right] \quad \dots 5.60 \end{aligned}$$

From equ (5.55) we get, $\sin \alpha = \frac{D}{2X} \therefore D = 2 \sin \alpha \quad \dots 5.61$

On substituting for D from equ (5.61) in equ (5.60) we get,

$$A_1 = \frac{4K}{\pi} \left[\frac{X}{2} \left(\frac{\pi}{2} - \alpha + \frac{\sin 2\alpha}{2} \right) - X \sin \alpha \cos \alpha \right] = \frac{4KX}{\pi} \left[\frac{\pi}{4} - \frac{\alpha}{2} + \frac{2 \sin \alpha \cos \alpha}{4} - \sin \alpha \cos \alpha \right]$$

$$= \frac{4KX}{\pi} \left[\frac{\pi}{4} - \frac{\alpha}{2} - \frac{\sin \alpha \cos \alpha}{2} \right] = KX \left[\frac{4}{\pi} \times \frac{\pi}{4} - \frac{4}{\pi} \times \frac{1}{2} (\alpha + \sin \alpha \cos \alpha) \right]$$

$$= KX \left[1 - \frac{2}{\pi} (\alpha + \sin \alpha \cos \alpha) \right]$$

$$Y_1 = \sqrt{A_1^2 + B_1^2} = \sqrt{A_1^2 + 0} = A_1 = KX \left[1 - \frac{2}{\pi} (\alpha + \sin \alpha \cos \alpha) \right]$$

$$\phi_1 = \tan^{-1} \frac{B_1}{A_1} = \tan^{-1} 0 = 0$$

$$\text{The describing function, } K_N(X, \omega) = \frac{Y_1}{X} \angle \phi_1 \quad \dots 5.65$$

Using equations (5.63) and (5.64) the describing function of equ (5.65) can be written as,

$$K_N(X, \omega) = K \left[1 - \frac{2}{\pi} (\alpha + \sin \alpha \cos \alpha) \right] \angle 0^\circ \quad \dots 5.66$$

Depending on the maximum value of input X, the describing function can be written as,

$$\text{If } X < \frac{D}{2}, \quad K_N(X, \omega) = 0 \quad \dots 5.67$$

$$\text{If } X > \frac{D}{2}, \quad K_N(X, \omega) = K \left[1 - \frac{2}{\pi} (\alpha + \sin \alpha \cos \alpha) \right] \quad \dots 5.68$$

The equation (5.68) can be expressed in another form as shown below.

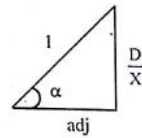


Fig 2.21

From equ (5.55), we get, $\sin \alpha = D/2X$

On constructing right able triangle with unity hypotenuse as shown in Fig. 5.21 $\cos \alpha$ can be evaluated.

From Figure 5.21, we get,

$$\text{adj} = \sqrt{1 - \left(\frac{D}{2X} \right)^2} \quad \therefore \cos \alpha = \frac{\text{adj}}{\text{hyp}} = \sqrt{1 - \left(\frac{D}{2X} \right)^2} \quad \dots 5.69$$

In the describing function of equ (5.68), substitute fro α , $\sin \alpha$ and $\cos \alpha$ from equations (5.26) (5.55) and (5.69) respectively.

$$\therefore K_N(X, \omega) = K \left[1 - \frac{2}{\pi} \left(\sin^{-1} \left(\frac{D}{2X} \right) + \left(\frac{D}{2X} \right) \sqrt{1 - \left(\frac{D}{2X} \right)^2} \right) \right] \quad \text{for } X > \frac{D}{2} \quad \dots 5.70$$

5.6 DESCRIBING FUNCTION OF RELAY WITH DEAD-ZONE AND HYSTERESIS

The input and the output relationship of a relay with dead-zone and hysteresis shown in Figure 5.22.

Due to dead-zone the relay will respond only after a definite value of input. Due to hysteresis the output follows a different paths for increasing and decreasing values of input. When the input x is increased from zero, the output follows the path ABCD and when the input is decreased from a maximum value, the output follows the path DCEA.

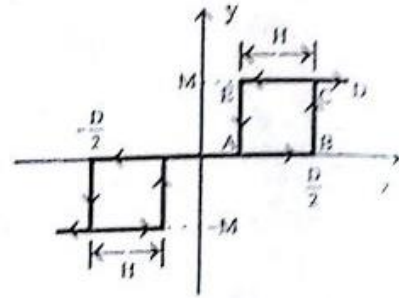


Figure 5.22 Input output characteristics of relay with dead-zone and hysteresis

For increasing values of input, the output is zero when $x < (D/2)$ and the output is M when $x > (D/2)$. For decreasing values of input the output is M when $x > (D/2 - H)$ and output is zero when $x < (D/2 - H)$.

The response or output of the relay when the input is sinusoidal signal ($x = X \sin \omega t$) is shown in fig 5.23.

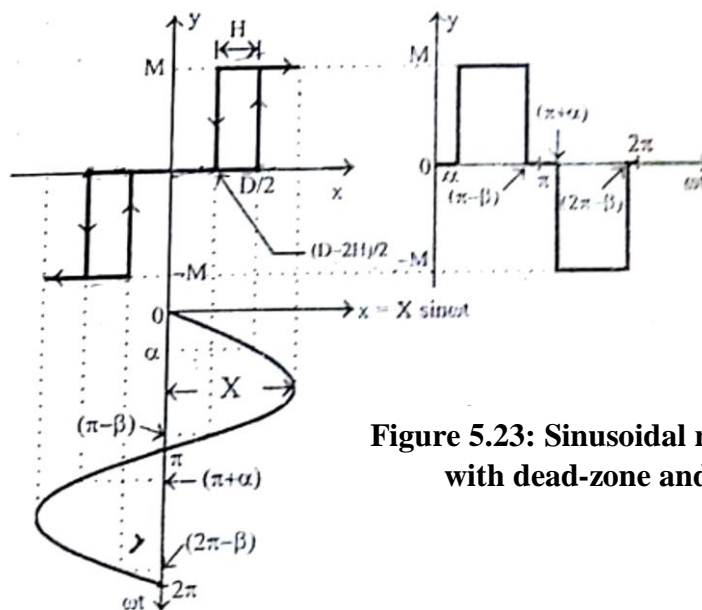


Figure 5.23: Sinusoidal response of relay with dead-zone and hysteresis

The input x is sinusoidal, $\therefore x = X \sin \omega t$...5.71

Where X = maximum value of input.

In fig. 5.23, when $\omega t = \alpha$, $x = D/2$

Hence equ (5.71) can be writte as $D/2 = \sin \alpha$

$$\therefore \sin \alpha = D / 2X \quad \dots 5.72$$

$$\text{and } \alpha = \text{sib}^{-1} D / 2X \quad \dots 5.73$$

In Figure 5.23, when $\omega t = \pi - \beta$, $x = D/2 - H$

Hence eqn (5.71) can be written as

$$D/2 - H = X \sin(\pi - \beta)$$

$$D/2 - H = X \sin \beta$$

$$\sin \beta = \frac{1}{X} \left(\frac{D}{2} - H \right) \quad \dots 5.74$$

$$\beta = \sin^{-1} \left(\frac{1}{X} \left(\frac{D}{2} - H \right) \right) \quad \dots 5.75$$

The output can be divided into five regions in a period of 2π and the output equation for the five regions are given by eqn(5.76).

$$y = \begin{cases} 0 & ; 0 \leq \omega t \leq \alpha \\ M & ; \alpha \leq \omega t \leq (\pi - \beta) \\ 0 & ; (\pi - \beta) \leq \omega t \leq (\pi + \alpha) \\ -M & ; (\pi + \alpha) \leq \omega t \leq (2\pi - \beta) \\ 0 & ; (2\pi - \beta) \leq \omega t \leq 2\pi \end{cases} \quad \dots 5.76$$

Let Y_1 = Amplitude of the fundamental harmonic component of the output.

ϕ_1 = Phase shift of the fundamental harmonic component of the output with respect to the input.

The describing function is given by, $K_N(X, \omega) = (Y_1 / X) \angle \phi_1$

where $Y_1 = \sqrt{A_1^2 + B_1^2}$ and $\phi_1 = \tan^{-1} (B_1 / A_1)$

$$\begin{aligned} A_1 &= \frac{2}{\pi} \int_0^{\pi} y \sin \omega t \, d(\omega t) = \frac{2}{\pi} \int_{\alpha}^{\pi - \beta} M \sin \omega t \, d(\omega t) \\ &= \frac{2M}{\pi} [-\cos \omega t]_{\alpha}^{\pi - \beta} = \frac{2M}{\pi} [-\cos(\pi - \beta) + \cos \alpha] \\ &= \frac{2M}{\pi} (\cos \alpha + \cos \beta) \end{aligned}$$

Note : $\cos(\pi - \beta) = -\cos \beta$

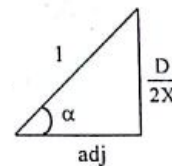


Figure 5.24

From eqn(5.72) we get, $\sin \alpha = D/2X$

On constructing right angle triangle with unity hypotenuse as shown in fig 5.24, $\cos \alpha$ can be evaluated

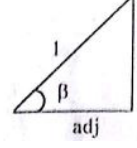
$$\text{adj} = \sqrt{1 - (D/2X)^2} \quad \therefore \cos \alpha = \frac{\text{adj}}{\text{hyp}} = \sqrt{1 - (D/2X)^2} \quad \dots 5.78$$

From equ (5.74) we get $\sin \beta = \left(\frac{D}{2X} - \frac{H}{X} \right)$.

On constructing right angle triangle with unity hypotenuse as shown in fig 5.25, $\cos \beta$ can be evaluated

$$\text{adj} = \sqrt{1 - \left(\frac{D}{2X} - \frac{H}{X} \right)^2}$$

$$\cos \beta = \frac{\text{adj}}{\text{hyp}} = \sqrt{1 - \left(\frac{D}{2X} - \frac{H}{X} \right)^2}$$



...5.79 **Figure 5.25**

On substituting for $\cos \alpha$ and $\cos \beta$ from equations (5.78) and (5.79) in equ(5.77) we get,

$$A_1 = \frac{2M}{\pi} \left(\sqrt{1 - \left(\frac{D}{2X} \right)^2} + \sqrt{1 - \left(\frac{D}{2X} - \frac{H}{X} \right)^2} \right) \quad \dots 5.80$$

$$B_1 = \frac{2}{\pi} \int_0^{\pi} y \cos \omega t d(\omega t) = \frac{2}{\pi} \int_{\alpha}^{\pi - \beta} M \cos \omega t d(\omega t)$$

Note : $\sin(\pi - \beta) = \sin \beta$

$$= \frac{2M}{\pi} [\sin \omega t]_{\alpha}^{\pi - \beta} = \frac{2M}{\pi} [\sin(\pi - \beta) - \sin \alpha] = \frac{2M}{\pi} (\sin \beta - \sin \alpha)$$

On substituting for $\sin \alpha$ and $\sin \beta$ from equ (2.72) and equ (2.74) we get,

$$B_1 = \frac{2M}{\pi} \left[\frac{D}{2X} - \frac{H}{X} - \frac{D}{2X} \right] = \frac{2M}{\pi} \left(-\frac{H}{X} \right) = \frac{2M}{\pi} \left(-\frac{H}{X} \right) \quad \dots 5.81$$

$$\therefore Y_1 = \sqrt{A_1^2 + B_1^2} = [A_1^2 + B_1^2]^{\frac{1}{2}}$$

$$Y_1 = \left[\frac{4M^2}{\pi^2} \left\{ \sqrt{1 - \left(\frac{D}{2X} \right)^2} + \sqrt{1 - \left(\frac{D}{2X} - \frac{H}{X} \right)^2} \right\}^2 + \frac{4M^2}{\pi^2} \left(\frac{H^2}{X^2} \right) \right]^{\frac{1}{2}} \quad \dots 5.82$$

$$\phi_1 = \tan^{-1} \frac{B_1}{A_1} = \tan^{-1} \left(\frac{\frac{2M}{\pi} \left(-\frac{H}{X} \right)}{\frac{2M}{\pi} \left[\sqrt{1 - \left(\frac{D}{2X} \right)^2} + \sqrt{1 - \left(\frac{D}{2X} - \frac{H}{X} \right)^2} \right]} \right) \quad \dots 5.83$$

The describing function of the relay with dead-zone and hysteresis is given by

$$K_N(X, \omega) = \frac{Y_1}{X} \angle \phi_1 \quad \dots 5.84$$

Where Y_1 is given by equ (5.82) and ϕ_1 is given by equ(5.83).

From the equ(5.84), the describing functions of the following three cases of relay can be obtained.

1. Ideal relay
2. Relay with dead-zone
3. Relay with hysteresis

1. IDEAL RELAY

In this case $D = H = 0$,

On substituting $D = H = 0$, in equ (5.82) and equ (5.83) we get,

$$Y_1 = \frac{2M}{\pi} \text{ and } \phi_1 = 0$$

Hence the describing function of the ideal relay is given by,

$$K_N(X, \omega) = \frac{Y_1}{X} \angle \phi_1 = \frac{2M}{\pi X} \quad \dots 5.85$$

2. RELAY WITH DEAD-ZONE

In this case $H = 0$

On substituting $H = 0$, in equ (5.82) and (5.83) we get,

$$\begin{aligned} Y_1 &= \left[\frac{4M^2}{\pi^2} \left\{ 2 \sqrt{1 - \left(\frac{D}{2X} \right)^2} \right\}^2 \right]^{\frac{1}{2}} \\ &= \frac{4M}{\pi} \sqrt{1 - \left(\frac{D}{2X} \right)^2} \\ \phi_1 &= 0 \end{aligned}$$

Hence the describing function of relay with dead-zone is given by

$$K_N(X, \omega) = \frac{Y_1}{X} \angle \phi_1 = \begin{cases} 0 & ; X < \frac{D}{2} \\ \frac{4M}{\pi X} \sqrt{1 - \left(\frac{D}{2X} \right)^2} & ; X > \frac{D}{2} \end{cases} \quad \dots 5.86$$

3. RELAY WITH HYSTERESIS

In this case $D = H$

On substituting $D = H$ in equ (5.82) we get,

$$Y_1 = \left[\frac{4M^2}{\pi^2} \left\{ \sqrt{1 - \left(\frac{H}{2X} \right)^2} + \sqrt{1 - \left(\frac{-H}{2X} \right)^2} \right\}^2 + \frac{4M^2}{\pi^2} \left(\frac{H^2}{X^2} \right) \right]^{\frac{1}{2}}$$

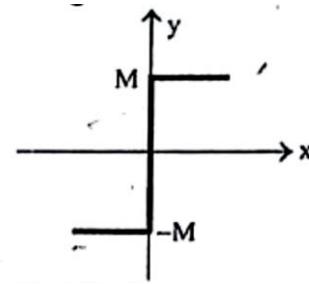


Figure 5.26 : Input – Output characteristics of ideal relay

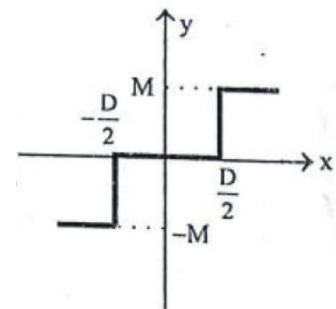


Figure 5.27 Input-Output characteristics of relay with dead-zone

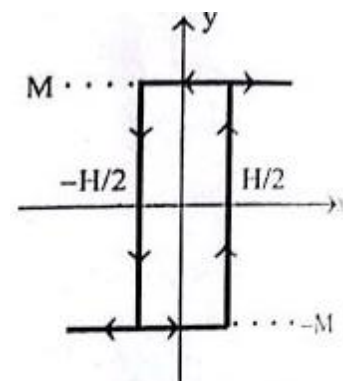


Figure 5.28 Input – output characteristics of relay with hysteresis

$$\begin{aligned}
&= \frac{2M}{\pi} \left[4 \left(1 - \frac{H^2}{4X^2} \right) + \left(\frac{H^2}{X^2} \right) \right]^{\frac{1}{2}} \\
&= \frac{2M}{\pi} \left[4 - \frac{H^2}{X^2} + \frac{H^2}{X^2} \right]^{\frac{1}{2}} \\
&= \frac{4M}{\pi}
\end{aligned}
\tag{5.87}$$

On substituting $D = H$ in equ (5.83) we get,

$$\begin{aligned}
\phi_1 &= \tan^{-1} \frac{\frac{2M}{\pi} \left(-\frac{H}{X} \right)}{\frac{2M}{\pi} \left[\sqrt{1 - \left(\frac{H}{2X} \right)^2} + \sqrt{1 - \left(\frac{-H}{2X} \right)^2} \right]} = \tan^{-1} \frac{-\frac{H}{X}}{2\sqrt{1 - \frac{H^2}{4X^2}}} \\
\phi_1 &= -\tan^{-1} \frac{\frac{H}{2X}}{\sqrt{1 - \frac{H^2}{4X^2}}} \\
\therefore -\phi_1 &= \tan^{-1} \frac{\frac{H}{2X}}{\sqrt{1 - \frac{H^2}{4X^2}}} \\
\tan(-\phi_1) &= \frac{\frac{H}{2X}}{\sqrt{1 - \frac{H^2}{4X^2}}}
\end{aligned}
\tag{5.88}$$

Using the numerator and denominator of equ (5.88) as two sides, we can construct a right angle triangle as shown in Figure 5.29.

From Figure 5.29 we get, $\sin(-\phi_1) = \frac{H}{2X}$

$$\begin{aligned}
\therefore -\phi_1 &= \sin^{-1} \frac{H}{2X} \\
\text{(or)} \quad \phi_1 &= -\sin^{-1} \frac{H}{2X}
\end{aligned}$$

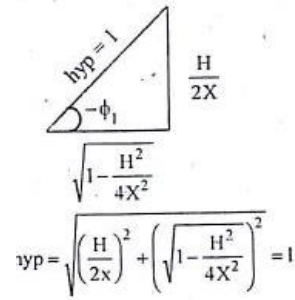


Figure 5.29

Using equations (5.87) and (5.89), the describing function of relay with hysteresis can be written as,

$$K_N(X, \omega) = \frac{Y_1}{X} \angle \phi_1 = \begin{cases} 0 & ; X < \frac{H}{2} \\ \frac{4M}{\pi X} \angle (-\sin^{-1} \frac{H}{2X}) & ; X > \frac{H}{2} \end{cases}
\tag{5.90}$$

5.7 DESCRIBING FUNCTION OF BACKLASH NONLINEARITY

The input-output relationship of Backlash nonlinearity is shown in fig 5.30.

The response of the nonlinearity when the input is sinusoidal signal ($x = X \sin \omega t$) is shown in fig 5.31.

In Fig 5.31, when $\omega t = (\pi - \beta)$, $x = X - b$

On substituting this value of x and ωt in the input signal, $x = X \sin \omega t$ we get

$$X - b = X \sin (\pi - \beta)$$

$$X - b = X \sin \beta$$

$$\therefore \sin \beta = \frac{X - b}{X} = 1 - \frac{b}{X}$$

...5.91

$$\text{and } \beta = \sin^{-1} \left(1 - \frac{b}{X} \right)$$

...5.92

The output can be divided into five regions in a period of 2π and the output equation for the five regions are given by equ(5.93).

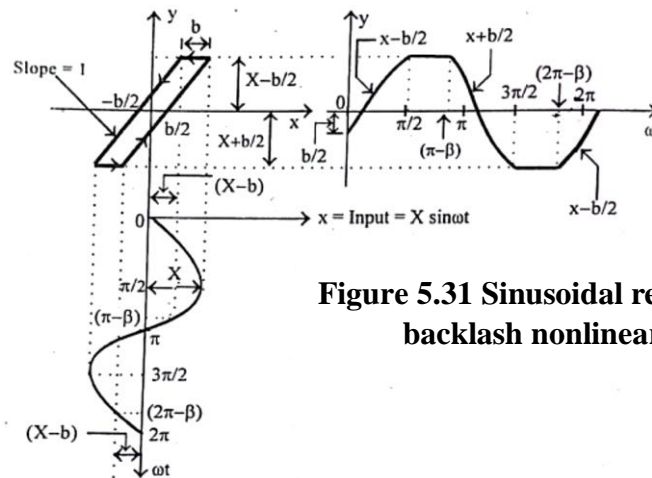


Figure 5.30: Input-Output characteristic of backlash nonlinearity

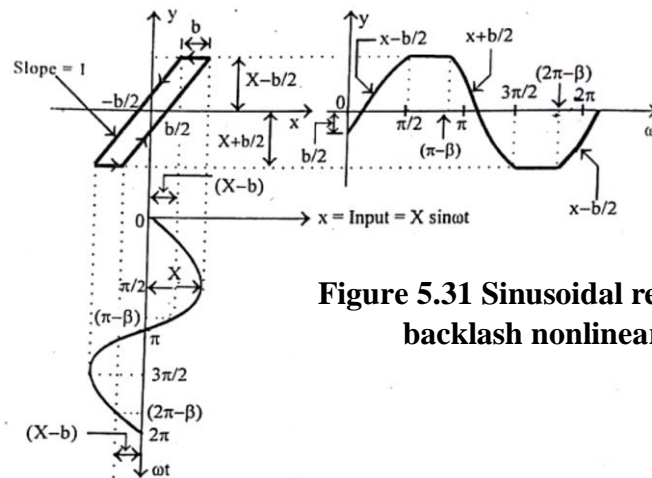


Figure 5.31 Sinusoidal response of backlash nonlinearity

$$y = \begin{cases} x - b/2 & ; 0 \leq \omega t \leq \frac{\pi}{2} \\ X - b/2 & ; \frac{\pi}{2} \leq \omega t \leq (\pi - \beta) \\ x + b/2 & ; (\pi - \beta) \leq \omega t \leq 3\frac{\pi}{2} \\ -X + b/2 & ; 3\frac{\pi}{2} \leq \omega t \leq (2\pi - \beta) \\ x - b/2 & ; (2\pi - \beta) \leq \omega t \leq 2\pi \end{cases}$$

....5.93

Let Y_1 = Amplitude of the fundamental harmonic component of the output.

ϕ_1 = Phase shift of the fundamental harmonic component of the output with respect to the input.

The describing function is given by, $K_N(X, \omega) = (Y_1 / X) \angle \phi_1$

where $Y_1 = \sqrt{A_1^2 + B_1^2}$ and $\phi_1 = \tan^{-1} (B_1 / A_1)$

$$A_1 = \frac{2}{\pi} \int_0^{\pi} y \sin \omega t \, d(\omega t) \quad \dots 5.94$$

The output, y is given by three different equations in range 0 to π , hence equ (5.94) can be written as

$$A_1 = \frac{2}{\pi} \int_0^{\frac{\pi}{2}} \left(x - \frac{b}{2}\right) \sin \omega t \, d(\omega t) + \frac{2}{\pi} \int_{\frac{\pi}{2}}^{\pi-\beta} \left(X - \frac{b}{2}\right) \sin \omega t \, d(\omega t) + \frac{2}{\pi} \int_{\pi-\beta}^{\pi} \left(x + \frac{b}{2}\right) \sin \omega t \, d(\omega t) \quad \dots 5.95$$

Put $x = X \sin \omega t$ in equ (5.95)

$$\begin{aligned} A_1 &= \frac{2}{\pi} \int_0^{\frac{\pi}{2}} \left(X \sin \omega t - \frac{b}{2}\right) \sin \omega t \, d(\omega t) + \frac{2}{\pi} \int_{\frac{\pi}{2}}^{\pi-\beta} \left(X - \frac{b}{2}\right) \sin \omega t \, d(\omega t) \\ &\quad + \frac{2}{\pi} \int_{\pi-\beta}^{\pi} \left(X \sin \omega t + \frac{b}{2}\right) \sin \omega t \, d(\omega t) \\ &= \frac{2X}{\pi} \int_0^{\frac{\pi}{2}} \sin^2 \omega t \, d(\omega t) - \frac{b}{\pi} \int_0^{\frac{\pi}{2}} \sin \omega t \, d(\omega t) + \frac{2\left(X - \frac{b}{2}\right)}{\pi} \int_{\frac{\pi}{2}}^{\pi-\beta} \sin \omega t \, d(\omega t) \\ &\quad + \frac{2X}{\pi} \int_{\pi-\beta}^{\pi} \sin^2 \omega t \, d(\omega t) + \frac{b}{\pi} \int_{\pi-\beta}^{\pi} \sin \omega t \, d(\omega t) \end{aligned} \quad \dots 5.96$$

Put $\sin^2 \omega t = \frac{1 - \cos 2\omega t}{2}$ In equ (5.96)

$$\begin{aligned} \therefore A_1 &= \frac{X}{\pi} \int_0^{\frac{\pi}{2}} (1 - \cos 2\omega t) \, d(\omega t) - \frac{b}{\pi} \int_0^{\frac{\pi}{2}} \sin \omega t \, d(\omega t) + \frac{2\left(X - \frac{b}{2}\right)}{\pi} \int_{\frac{\pi}{2}}^{\pi-\beta} \sin \omega t \, d(\omega t) \\ &\quad + \frac{X}{\pi} \int_{\pi-\beta}^{\pi} (1 - \cos 2\omega t) \, d(\omega t) + \frac{b}{\pi} \int_{\pi-\beta}^{\pi} \sin \omega t \, d(\omega t) \\ &= \frac{X}{\pi} \left[\omega t - \frac{\sin 2\omega t}{2} \right]_0^{\frac{\pi}{2}} - \frac{b}{\pi} [-\cos \omega t]_0^{\frac{\pi}{2}} + \frac{2\left(X - \frac{b}{2}\right)}{\pi} [-\cos \omega t]_{\frac{\pi}{2}}^{\pi-\beta} \\ &\quad + \frac{X}{\pi} \left[\omega t - \frac{\sin 2\omega t}{2} \right]_{\pi-\beta}^{\pi} + \frac{b}{\pi} [-\cos \omega t]_{\pi-\beta}^{\pi} \end{aligned}$$

$$\begin{aligned}
&= \frac{X}{\pi} \left(\frac{\pi}{2} \right) - \frac{b}{\pi} (1) + \frac{2(X - \frac{b}{2})}{\pi} (-\cos(\pi - \beta) + \cos \frac{\pi}{2}) \\
&\quad + \frac{X}{\pi} \left[\pi - \frac{\sin 2\pi}{2} - (\pi - \beta) + \frac{\sin 2(\pi - \beta)}{2} \right] + \frac{b}{\pi} [-\cos \pi + \cos(\pi - \beta)] \\
&= \frac{X}{2} - \frac{b}{\pi} + \frac{2}{\pi} \left(X - \frac{b}{2} \right) \cos \beta + \frac{X}{\pi} \left(\beta - \frac{\sin 2\beta}{2} \right) + \frac{b}{\pi} (1 - \cos \beta) \\
&= \frac{X}{2} - \frac{b}{\pi} + \frac{2}{\pi} \left(X - \frac{b}{2} \right) \cos \beta + \frac{X\beta}{\pi} - \frac{X}{2\pi} \sin 2\beta + \frac{b}{\pi} - \frac{b}{\pi} \cos \beta \\
&= \frac{X}{2} + \frac{X\beta}{\pi} + \frac{2}{\pi} \left(X - \frac{b}{2} - \frac{b}{2} \right) \cos \beta - \frac{X}{2\pi} \sin 2\beta \\
&= \frac{X}{2} + \frac{X\beta}{\pi} + \frac{2X}{\pi} \left(1 - \frac{b}{X} \right) \cos \beta - \frac{X}{2\pi} \sin 2\beta \quad \dots 5.97
\end{aligned}$$

On substituting of $r(1-b/X)$ from equ (5.91) in equ (5.97) we get,

$$\begin{aligned}
A_1 &= \frac{X}{2} + \frac{X\beta}{\pi} + \frac{2X}{\pi} \sin \beta \cos \beta - \frac{X}{2\pi} \sin 2\beta = \frac{X}{2} + \frac{X\beta}{\pi} + \frac{X}{\pi} \sin 2\beta - \frac{X}{2\pi} \sin 2\beta \\
&= \frac{X}{2} + \frac{X\beta}{\pi} + \frac{X}{2\pi} \sin 2\beta = \frac{X}{\pi} \left(\frac{\pi}{2} + \beta + \frac{1}{2} \sin 2\beta \right) \quad \dots 5.98
\end{aligned}$$

$$B_1 = \frac{2}{\pi} \int_0^{\pi} y \cos \omega t \, d(\omega t) \quad \dots 5.99$$

The output, y is given by three different equations in the range 0 to π , hence equ (5.99) can be expressed as,

$$\begin{aligned}
B_1 &= \frac{2}{\pi} \int_0^{\frac{\pi}{2}} \left(X - \frac{b}{2} \right) \cos \omega t \, d(\omega t) + \frac{2}{\pi} \int_{\frac{\pi}{2}}^{\pi-\beta} \left(X - \frac{b}{2} \right) \cos \omega t \, d(\omega t) \\
&\quad + \frac{2}{\pi} \int_{\pi-\beta}^{\pi} \left(X + \frac{b}{2} \right) \cos \omega t \, d(\omega t) \quad \dots 5.100
\end{aligned}$$

Put $= X \sin \omega t$ in equ (5.100)

$$\begin{aligned}
\therefore B_1 &= \frac{2}{\pi} \int_0^{\frac{\pi}{2}} \left(X \sin \omega t - \frac{b}{2} \right) \cos \omega t \, d(\omega t) + \frac{2}{\pi} \int_{\frac{\pi}{2}}^{\pi-\beta} \left(X - \frac{b}{2} \right) \cos \omega t \, d(\omega t) \\
&\quad + \frac{2}{\pi} \int_{\pi-\beta}^{\pi} \left(X \sin \omega t + \frac{b}{2} \right) \cos \omega t \, d(\omega t) \\
&= \frac{X}{\pi} \int_0^{\frac{\pi}{2}} 2 \sin \omega t \cos \omega t \, d(\omega t) - \frac{b}{\pi} \int_0^{\frac{\pi}{2}} \cos \omega t \, d(\omega t) + \frac{2}{\pi} \left(X - \frac{b}{2} \right) \int_{\frac{\pi}{2}}^{\pi-\beta} \cos \omega t \, d(\omega t) \\
&\quad + \frac{X}{\pi} \int_{\pi-\beta}^{\pi} 2 \sin \omega t \cos \omega t \, d(\omega t) + \frac{b}{\pi} \int_{\pi-\beta}^{\pi} \cos \omega t \, d(\omega t)
\end{aligned}$$

$$\begin{aligned}
B_1 &= \frac{X}{\pi} \int_0^{\frac{\pi}{2}} \sin 2\omega t \, d(\omega t) - \frac{b}{\pi} \int_0^{\frac{\pi}{2}} \cos \omega t \, d(\omega t) + \frac{2}{\pi} \left(X - \frac{b}{2}\right) \int_{\frac{\pi}{2}}^{\pi-\beta} \cos \omega t \, d(\omega t) \\
&\quad + \frac{X}{\pi} \int_{\pi-\beta}^{\pi} \sin 2\omega t \, d(\omega t) + \frac{b}{\pi} \int_{\pi-\beta}^{\pi} \cos \omega t \, d(\omega t) \\
&= \frac{X}{\pi} \left[-\frac{\cos 2\omega t}{2} \right]_0^{\frac{\pi}{2}} - \frac{b}{\pi} [\sin \omega t]_0^{\frac{\pi}{2}} + \frac{2}{\pi} \left(X - \frac{b}{2}\right) [\sin \omega t]_{\frac{\pi}{2}}^{\pi-\beta} \\
&\quad + \frac{X}{\pi} \left[-\frac{\cos 2\omega t}{2} \right]_{\pi-\beta}^{\pi} + \frac{b}{\pi} [\sin \omega t]_{\pi-\beta}^{\pi} \\
&= \frac{X}{\pi} \left[-\frac{\cos \pi}{2} + \frac{\cos 0}{2} \right] - \frac{b}{\pi} \left[\sin \frac{\pi}{2} - 0 \right] + \frac{2}{\pi} \left(X - \frac{b}{2}\right) \left[\sin(\pi - \beta) - \sin \frac{\pi}{2} \right] \\
&\quad + \frac{X}{\pi} \left[-\frac{\cos 2\pi}{2} + \frac{\cos 2(\pi - \beta)}{2} \right] + \frac{b}{\pi} [\sin \pi - \sin(\pi - \beta)] \\
&= \frac{X}{\pi} \left(\frac{1}{2} + \frac{1}{2} \right) - \frac{b}{\pi} (1 - 0) + \frac{2}{\pi} \left(X - \frac{b}{2}\right) (\sin \beta - 1) + \frac{X}{\pi} \left(-\frac{1}{2} + \frac{\cos 2\beta}{2} \right) + \frac{b}{\pi} (0 - \sin \beta) \\
&= \frac{X}{\pi} - \frac{b}{\pi} + \frac{2X}{\pi} \sin \beta - \frac{b}{\pi} \sin \beta - \frac{2X}{\pi} + \frac{b}{\pi} - \frac{X}{2\pi} + \frac{X}{2\pi} \cos 2\beta - \frac{b}{\pi} \sin \beta \\
&= \frac{X}{\pi} \left(1 - 2 - \frac{1}{2} \right) + \frac{2X}{\pi} \sin \beta - \frac{2b}{\pi} \sin \beta + \frac{X}{2\pi} \cos 2\beta \\
&= -\frac{3X}{2\pi} + \frac{2X}{\pi} \sin \beta \left(1 - \frac{b}{X} \right) + \frac{X}{2\pi} \cos 2\beta \quad \dots 5.101
\end{aligned}$$

Since $(1 - b/X) = \sin \beta$ and $\cos 2\beta = (1 - 2 \sin^2 \beta)$, the equ (5.98) can be written as

$$\begin{aligned}
B_1 &= -\frac{3X}{2\pi} + \frac{2X}{\pi} \sin \beta (\sin \beta) + \frac{X}{2\pi} (1 - 2 \sin^2 \beta) \\
&= -\frac{3X}{2\pi} + \frac{2X}{\pi} \sin^2 \beta + \frac{X}{2\pi} - \frac{X}{\pi} \sin^2 \beta = -\frac{X}{\pi} + \frac{X}{\pi} \sin^2 \beta \\
&= -\frac{X}{\pi} + \frac{X}{\pi} (1 - \cos^2 \beta) = -\frac{X}{\pi} + \frac{X}{\pi} - \frac{X}{\pi} \cos^2 \beta \\
&= -\frac{X}{\pi} \cos^2 \beta \quad \dots 5.102
\end{aligned}$$

Note :
 $\sin^2 \beta + \cos^2 \beta = 1$
 $\therefore \sin^2 \beta = 1 - \cos^2 \beta$

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The Nyquist stability criterion can also be extended to the stability analysis of nonlinear systems. According to the Nyquist stability criterion the system will exhibit sustained oscillations or limit cycles when,

$$K_N G(j\omega) = -1 \quad \dots 5.108$$

The equation (5.108) implies that the sustained oscillations or limit cycles will occur if $K_N G(j\omega)$ locus pass through the critical point, $-1 + j0$, in the complex plane.

The equation (5.108) can be modified as shown below

$$G(j\omega) = -1/K_N \quad \dots 5.109$$

The equation (5.109) implies that the critical point, $-1 + j0$ becomes the critical locus which is the locus of $-1/K_N$. Hence the intersection point of $G(j\omega)$ locus and $-1/K_N$ locus will give the amplitude and frequency of limit cycles.

In the stability analysis, let us assume that the linear part of the system is stable. To determine the stability of the system due to nonlinearity sketch the $-1/K_N$ locus and $G(j\omega)$ locus (polar plot of $G(j\omega)$) in complex plane. (Use either a polar graph sheet or ordinary graph sheet) and from the sketches the following conclusions can be obtained.

1. If the $-1/K_N$ locus is not enclosed by the $G(j\omega)$ locus then the system is stable or there is no limit cycle at steady state.
2. If the $-1/K_N$ locus is enclosed by the $G(j\omega)$ locus then the system is unstable.
3. If the $-1/K_N$ locus and the $G(j\omega)$ locus intersect, then the system output may exhibit a sustained oscillation or a limit cycle. The amplitude of the limit cycle is given by the value of $-1/K_N$ locus at the intersection point. The frequency of the limit cycle is given by the frequency of $G(j\omega)$ corresponding to the intersection point.

CONCEPT OF ENCLOSURE

In a complex plane the $-1/K_N$ locus is said to be enclosed by $G(j\omega)$ locus if it lies in the region to the right of an observer travelling through $G(j\omega)$ locus in the direction of increasing ω , as shown in fig 5.33.

In a complex plane the $-1/K_N$ locus is not enclosed by $G(j\omega)$ locus if it lies in the region to the left of an observer travelling through $G(j\omega)$ locus in the direction of increasing ω , as shown in fig 5.34.

If the $-1/K_N$ locus and $G(j\omega)$ locus intersect as shown in fig 5.35, then for an observer travelling through $G(j\omega)$ locus in the direction of increasing ω , the region on the right is unstable region and the region on the left is stable region.

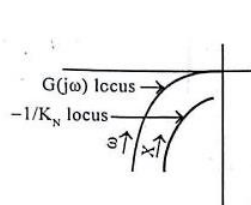


Figure 5.33 Figure showing enclosure of $-1/K_N$ locus by $G(j\omega)$ locus

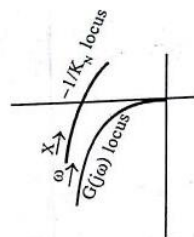


Figure 5.34 Figure showing non enclosure of $-1/K_N$ locus by $G(j\omega)$ locus

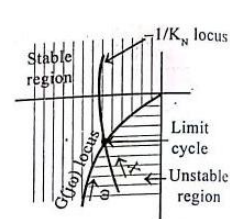


Figure 5.35 Figure showing intersection of $-1/K_N$ locus by $G(j\omega)$ locus

STABLE AND UNSTABLE LIMIT CYCLES

The $-1/K_N$ locus may intersect $G(j\omega)$ locus at one or more points. There exists a limit cycle at every intersecting point. These limit cycles can be either stable or unstable limit cycles, as shown in fig 5.36.

If $-1/K_N$ locus travels in unstable region and it intersect $G(j\omega)$ locus to enter stable region then the limit cycle corresponding to that intersection point is stable limit cycle.

If $-1/K_N$ locus travels in stable region and it intersect $G(j\omega)$ locus to enter unstable region then the limit cycle corresponding to that intersection point is unstable limit cycle.

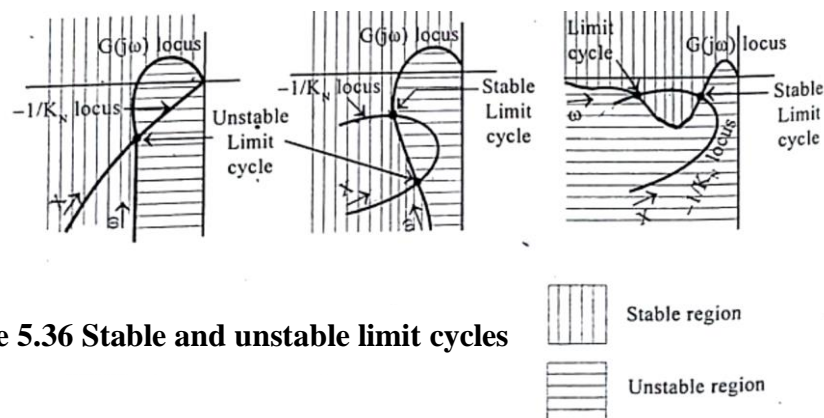


Figure 5.36 Stable and unstable limit cycles

Note: The concept of enclosure can be extended to db-phase angle plane (i.e. to Nichols plot) and it is same as that of complex plane.

5.9 REVIEW OF POLAR PLOT AND NICHOLS PLOT

POLAR PLOT

The polar plot of a sinusoidal transfer function, $G(j\omega)$ is a plot of the magnitude of $G(j\omega)$ versus the phase angle of $G(j\omega)$ on polar coordinates as ω is varied from zero to infinity. Thus the polar plot is the locus of vector $|G(j\omega)| \angle G(j\omega)$ as ω is varied from zero to infinity. The polar plot is also called Nyquist plot.

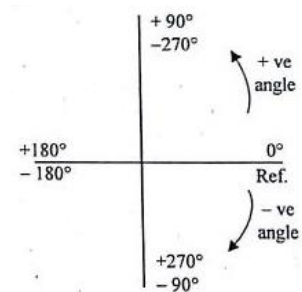


Figure 5.37 Polar graph

The polar plot is usually plotted on a polar graph sheet. The polar graph sheet has concentric circles and radial lines. The circles represent the magnitude and the radial lines represent the phase angles. Each point on the polar graph has a magnitude and phase angle. The magnitude of a point is given by the value of the circle passing through that point and the phase angle is given by the radial line passing through that point. In polar graph sheet a positive phase angle is measured in anticlockwise from the reference axis (0°) and a negative angle is measured clockwise from the reference axis (0°).

Alternatively, if $G(j\omega)$ can be expressed in rectangular coordinates as,

$$G(j\omega) = G_R(j\omega) + jG_I(j\omega)$$

Where, $G_R(j\omega)$ = Real part of $G(j\omega)$

and $G_I(j\omega)$ = Imaginary part of $G(j\omega)$

Then the polar plot can be plotted in ordinary graph sheet between $G_R(j\omega)$ and $G_I(j\omega)$ as ω is varied from 0 to ∞ .

To plot the polar plot, first compute the magnitude and phase of $G(j\omega)$ for various values of ω and tabulate them. Usually the choice of frequencies are corner frequencies and frequencies around corner frequencies. Choose proper scale the magnitude circles. Fix all the points on polar graph sheet and join the points by smooth curve. Write the frequency corresponding to each point of the plot.

To plot the polar plot on ordinary graph sheet, compute the magnitude and phase for various values of ω . Then convert the polar coordinates to rectangular coordinates using $P \rightarrow R$ conversion (polar to rectangular conversion) in the calculator. Sketch the polar plot using rectangular coordinates.

For minimum phase transfer function with only poles, the type number of the system determines at what quadrant the polar plot starts and the order of the system determines at what quadrant the polar plot ends.

Note: The minimum phase systems are systems with all poles and zeros on the left half of s-plane

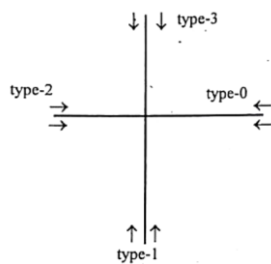


Figure 5.38
Start of polar plot

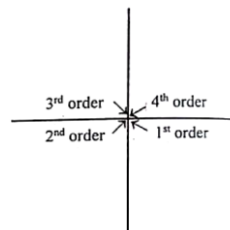


Figure 5.39
End of polar plot

NICHOLS PLOT

The Nichols plot is a frequency response plot of the open loop transfer function of a system. The Nichols plot is a graph between magnitude of $G(j\omega)$ in db and the phase of $G(j\omega)$ in degree, plotted on an ordinary graph sheet.

To plot the Nichols plot, first compute the magnitude of $G(j\omega)$ in db and phase of $G(j\omega)$ in deg for various values of ω and tabulate them. Usually the choice of frequencies are corner frequencies. Choose appropriate scales for magnitude on y-axis and phase on x-axis. Fix all the points on ordinary graph sheet and join the points by smooth curve. Write the frequency corresponding to each point of the plot.

In another method, first the Bode plot of $G(j\omega)$ is sketched. From the Bode plot the magnitude and phase for various values of frequency, ω are noted and tabulated. Using these values the Nichols plot is sketched as explained earlier.

In a system if the zero frequency gain K is varied then the magnitude of the transfer function alone will vary and there will not be any change in phase. This results in vertical shift of Nichols plot up or down. The constant K adds $20\log K$ to every point of the plot. If $20\log K$ is positive then the plot shifts upwards and if it is negative the plot shifts downwards.

EXAMPLE 5.2

A servo system used for positioning a load has backlash characteristics as shown in Fig 5.2.1. The block diagram of the system is shown in Fig 5.2.2. The magnitude and phase of the describing function of backlash nonlinearity for various values of b/X are listed in Table 5.2.1, where X = Maximum value of input sinusoidal signal to the nonlinearity.

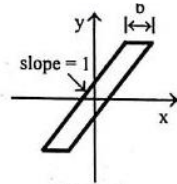


Figure 5.2.1

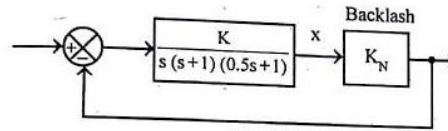


Figure 5.2.2

Table 5.2.1

b/X	0	0.2	0.4	1.0	1.4	1.6	1.8	1.9	2.0
$ K_N $	1	0.954	0.882	0.592	0.367	0.248	0.125	0.064	0
$\angle K_N$	0	-6.7°	-13.4°	-32.5°	-46.6°	-55.2°	-66°	-69.8°	-90°

Show that the system is stable if $K = 1$. Also show that limit cycle exists when $K = 2$. Investigate the stability of these limit cycles and determine their frequency and b/X .

SOLUTION

The describing function analysis of the system can be carried using either polar plot or using Nichols plot.

METHOD 1: USING POLAR PLOT

Polar plot of $G(j\omega)$ when $K = 1$

$$\text{Given that, } G(s) = \frac{K}{s(1+s)(1+0.5s)}$$

$$\text{Let } K = 1 \text{ and } s = j\omega$$

$$\begin{aligned} \therefore G(j\omega) &= \frac{1}{j\omega(1+j\omega)(1+j0.5\omega)} \\ &= \frac{1}{\omega \angle 90^\circ \sqrt{1+\omega^2} \angle \tan^{-1} \omega \sqrt{1+0.25\omega^2} \angle \tan^{-1} 0.5\omega} \\ |G(j\omega)| &= \frac{1}{\omega \sqrt{1+\omega^2} \sqrt{1+0.25\omega^2}} \\ \angle G(j\omega) &= -90^\circ - \tan^{-1} \omega - \tan^{-1} 0.5\omega \end{aligned}$$

The magnitude and phase of $G(j\omega)$ are calculated for various values of ω and tabulated in Table 5.2.2. Using polar to rectangular conversion the polar coordinates are converted to rectangular coordinates and listed in Table 5.2.2. The polar plot of $G(j\omega)$ when $K = 1$ drawn in an ordinary graph sheet, as shown in Figure 5.2.3.

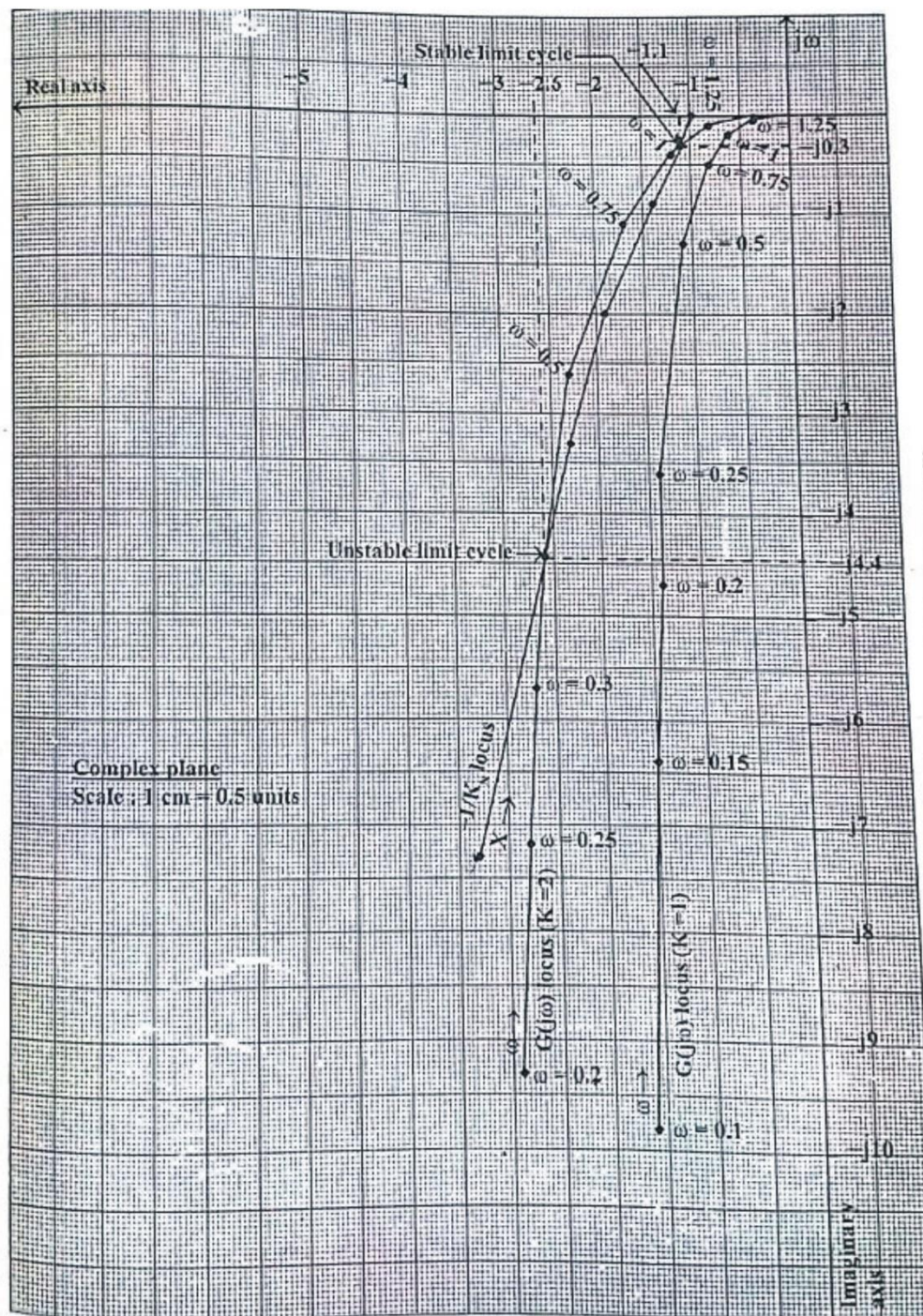


Figure 5.2.3 Polar plot of $G(j\omega)$ and $-1/K_N$

Table 5.2.2

ω rad/sec	0.1	0.15	0.2	0.25	0.5	0.75	1.0	1.25
$ G(j\omega) $	9.94	6.57	4.88	3.85	1.74	1.0	0.63	0.42
$\angle G(j\omega)$ deg.	-99	-103	-107	-111	-131	-147	-162	-173
$G_R(j\omega)$	-1.6	-1.5	-1.4	-1.4	-1.1	-0.8	-0.6	-0.4
$G_I(j\omega)$	-9.8	-6.4	-4.7	-3.6	-1.3	-0.5	-0.2	-0.05

Polar plot of $G(j\omega)$ when $K = 2$

The magnitude of $G(j\omega)$ when $K = 2$ is given by

$$|G(j\omega)| = \frac{2}{\omega \sqrt{1+\omega^2} \sqrt{1+0.25\omega^2}}$$

(The phase of $G(j\omega)$ will not change due to a change in the value of K)

The magnitude and phase of $G(j\omega)$ and the real part and imaginary part of $G(j\omega)$ $K = 2$ are calculated for various values of ω and listed in Table 5.2.3. The polar plot of 0 when $K = 2$, is drawn on the same graph sheet using the same scales as shown in Figure.

Table 5.2.3

ω rad/sec	0.2	0.25	0.3	0.5	0.75	1.0	1.25
$ G(j\omega) $	9.76	7.7	6.31	3.48	2.0	1.26	0.84
$\angle G(j\omega)$ deg.	-107	-111	-115	-132.1	-147	-162	-173
$G_R(j\omega)$	-2.9	-2.8	-2.7	-2.3	-1.7	-1.2	-0.8
$G_I(j\omega)$	-9.3	-7.2	-5.7	-2.6	-1.1	-0.4	-0.1

Polar plot of $-1/K_N$

The function $-1/K_N$ can be written as,

$$-1/K_N = -1 \times \frac{1}{K_N} = 1 \angle -180^\circ \times \frac{1}{|K_N| \angle K_N}$$

$$\therefore |-1/K_N| = \frac{1}{|K_N|} \quad \text{and} \quad \angle(-1/K_N) = -180^\circ - \angle K_N$$

The values of $|K_N|$ and $\angle K_N$ are given in the problem, in Table 5.2.1., for various values of b/X . Using the values if Table 5.2.1, the $|-1/K_N|$ and $\angle(-1/K_N)$ are calculated for various values of b/X and listed in Table 5.2.4. Then the real part and imaginary part of $-1/K_N$ are calculated using polar to rectangular conversion and listed in Table 5.2.4. The locus of $-1/K_N$ is sketched using rectangular coordinates in the same graph sheet as shown in Figure 5.2.3.

Table 5.2.4

b/X	0	0.2	0.4	1.0	1.4	1.6	1.8	1.9	2.0
$ K_N $	1	0.954	0.882	0.592	0.367	0.248	0.125	0.064	0
$\angle K_N$	0	-6.7°	-13.4°	-32.5°	-46.6°	-55.2°	-66°	-69.8°	-90°
$ -1/K_N $	1	1.05	1.13	1.69	2.72	4.03	8.0	15.63	∞
$\angle(-1/K_N)$	-180°	-173°	-166°	-148°	-133°	-125°	-114°	-110°	-90°
Real part of $-1/K$	-1.0	-1.04	-1.1	-1.4	-1.9	-2.3	-3.3	-5.3	0
Ima. Part of $1/K_N$	0	-0.1	-0.3	-0.9	-2.0	-3.3	-7.3	-14.7	∞

STABILITY ANALYSIS

Case (i) when $K = 1$

When $K = 1$, $G(j\omega)$ locus does not enclose $-1/K_N$ locus, hence the system is stable.

Case (ii) $K = 2$

When $K = 2$, the $G(j\omega)$ locus, intersects $-1/K_N$ locus at two points. From the polar plots, it is observed that at one intersection point, unstable limit cycle exists and at another intersection point stable limit cycle exists.

From Figure 5.2.3, Coordinates corresponding to unstable limit cycle = $-2.6 - j4.4 = 5.11 \angle -120^\circ$.

Let ω_{11} = Frequency corresponding to unstable limit cycle.

And b/X_1 = The value of b/X corresponding to unstable limit cycle

Now at $\omega = \omega_{11}$ $G(j\omega) = 5.11 \angle -120^\circ$

\therefore At $\omega = \omega_{11}$ $\angle G(j\omega) = -120^\circ$

By equating the expression for $\angle G(j\omega)$ to -120° , the frequency ω_{11} can be determined.

We know that, $\angle G(j\omega) = -90^\circ - \tan^{-1} \omega - \tan^{-1} 0.5\omega$

At $\omega = \omega_{11}$ $-90^\circ - \tan^{-1} \omega_{11} - \tan^{-1} 0.5\omega_{11} = -120^\circ$

$\therefore -90^\circ - \tan^{-1} \omega_{11} + \tan^{-1} 0.5\omega_{11} = 120^\circ$

$\tan^{-1} \omega_{11} + \tan^{-1} 0.5\omega_{11} = 120^\circ - 90^\circ = 30^\circ$

On taking \tan on either side we get,

$\tan(-\tan^{-1} \omega_{11} + \tan^{-1} 0.5\omega_{11}) = \tan 30^\circ$

$$\frac{\tan(\tan^{-1} \omega_{11}) + \tan(\tan^{-1} 0.5\omega_{11})}{1 - \tan(\tan^{-1} \omega_{11}) \times \tan(\tan^{-1} 0.5\omega_{11})} = 0.577$$

$$\frac{\omega_{11} + 0.5\omega_{11}}{1 - \omega_{11} \times 0.5\omega_{11}} = 0.577$$

$$\therefore 1.5\omega_{11} = 0.577 - 0.2885\omega_{11}^2$$

$$\therefore 0.2885\omega_{11}^2 + 1.5\omega_{11} - 0.577 = 0$$

On dividing by 0.2885 we get,

$$\omega_{11}^2 + \frac{1.5}{0.2885}\omega_{11} - \frac{0.577}{0.2885} = 0$$

$$\omega_{11}^2 + 5.2\omega_{11} - 2 = 0$$

$$\therefore \omega_{11} = \frac{-5.2 \pm \sqrt{5.2^2 - 4 \times (-2)}}{2} = \frac{-5.2 \pm 5.92}{2}$$

On taking only positive root we get,

$$\omega_{11} = \frac{-5.2 + 5.92}{2} = 0.36 \text{ rad/sec}$$

Note : $\tan(A + B) = \frac{\tan A + \tan B}{1 - \tan A \tan B}$

Also, at $\omega = \omega_{11}$, $-1/K_N = 5.11 \angle -120^\circ$

But $-1/K_N = 1 \angle -180^\circ \times \frac{1}{K_N}$, $\therefore 1 \angle -180^\circ \times \frac{1}{K_N} = 5.11 \angle -120^\circ$

$$\therefore K_N = \frac{1 \angle -180^\circ}{5.11 \angle -120^\circ} = \frac{1}{5.11} \angle -60^\circ = 0.196 \angle -60^\circ$$

Hence at $\omega = \omega_{11}$, $|K_N| = 0.196$ and $\angle K_N = -60^\circ$.

From the describing function of backlash nonlinearity we get,

$$\angle K_N = \tan^{-1} \left(\frac{-\cos^2 \beta}{(\pi/2) + \beta + (1/2) \sin 2\beta} \right)$$

At $\omega = \omega_{11}$, $\angle K_N = -60^\circ$, $b/X \rightarrow b/X_1$ and $\beta \rightarrow \beta_1$

$$\therefore \tan^{-1} \left(\frac{-\cos^2 \beta_1}{(\pi/2) + \beta_1 + (1/2) \sin 2\beta_1} \right) = -60^\circ$$

$$\frac{-\cos^2 \beta_1}{(\pi/2) + \beta_1 + (1/2) \sin 2\beta_1} = \tan(-60^\circ)$$

$$\therefore (\pi/2) + \beta_1 + (1/2) \sin 2\beta_1 = \frac{-\cos^2 \beta_1}{\tan(-60^\circ)} = 0.577 \cos^2 \beta_1$$

From the describing function of backlash nonlinearity we get,

$$|K_N| = \frac{1}{\pi} \left[((\pi/2) + \beta + (1/2) \sin 2\beta)^2 + \cos^4 \beta \right]^{1/2}$$

At, $\omega = \omega_{11}$, $|K_N| = 0.196$, $b/X \rightarrow b/X_1$ and $\beta \rightarrow \beta_1$

$$\therefore \frac{1}{\pi} \left[((\pi/2) + \beta_1 + \frac{1}{2} \sin 2\beta_1)^2 + \cos^4 \beta_1 \right]^{1/2} = 0.196$$

On substituting $((\pi/2) + \beta_1 + (1/2) \sin 2\beta_1) = 0.577 \cos^2 \beta_1$ and then squaring we get

$$(0.577 \cos^2 \beta_1)^2 + \cos^4 \beta_1 = (0.196\pi)^2$$

$$0.333 \cos^4 \beta_1 + \cos^4 \beta_1 = 0.379$$

$$1.333 \cos^4 \beta_1 = 0.379$$

$$\therefore \cos \beta_1 = \left(\frac{0.379}{1.333} \right)^{1/4} = 0.73 \quad ; \quad \beta_1 = \cos^{-1}(0.73) = 43.1^\circ$$

We know that, $\beta = \sin^{-1}(1 - b/X)$

$$\therefore \beta_1 = \sin^{-1}(1 - b/X_1) \text{ (or) } b/X_1 = 1 - \sin \beta_1 = 1 - \sin 43.1^\circ = 0.316.$$

From Figure 5.2.3,

Coordinates corresponding to stable limit cycle = $-1.1 - j0.3 = 1.14 \angle -165^\circ$.

Let ω_{12} = Frequency corresponding to stable limit cycle.

and b/X_2 = The value of b/X corresponding to stable limit cycle

Now at $\omega = \omega_{12}$ $G(j\omega) = 1.14 \angle -165^\circ$

$$\therefore \text{At } \omega = \omega_{12} \angle G(j\omega) = -165^\circ$$

By equating the expression for $\angle G(j\omega)$ to -165° , the frequency ω_{12} can be determined.

We know that, $\angle G(j\omega) = -90^\circ - \tan^{-1} \omega - \tan^{-1} 0.5\omega$

At $\omega = \omega_{12}$ $-90^\circ - \tan^{-1} \omega_{12} - \tan^{-1} 0.5\omega_{12} = -165^\circ$

$$\therefore -90^\circ - \tan^{-1} \omega_{12} + \tan^{-1} 0.5\omega_{12} = 165^\circ - 90^\circ = 75^\circ$$

On taking tan on either side we get,

$$\tan (\tan^{-1} \omega_{12} + \tan^{-1} 0.5\omega_{12}) = \tan 75^\circ$$

$$\frac{\tan (\tan^{-1} \omega_{12}) + \tan (\tan^{-1} 0.5\omega_{12})}{1 - \tan (\tan^{-1} \omega_{12}) \times \tan (\tan^{-1} 0.5\omega_{12})} = 3.732$$

$$\frac{\omega_{12} + 0.5\omega_{12}}{1 - \omega_{12} \times 0.5\omega_{12}} = 3.732$$

$$\therefore 1.5 \omega_{12} = 3.732 - 1.886\omega_{12}^2$$

$$\therefore \omega_{12}^2 + \frac{1.5}{1.866}\omega_{12} - \frac{3.732}{1.866} = 0 \quad ; \quad \omega_{12}^2 + 0.8\omega_{12} - 2 = 0$$

$$\therefore \omega_{12} = \frac{-0.8 \pm \sqrt{0.8^2 - 4 \times (-2)}}{2} = \frac{-0.8 \pm 2.94}{2}$$

On taking only positive root we get,

$$\omega_{12} = \frac{-0.8 + 2.94}{2} = 1.07 \text{ rad/sec}$$

$$\text{Also, at } \omega = \omega_{12}, \quad -1/K_N = 1.14 \angle -165^\circ$$

$$\text{But } -1/K_N = 1 \angle -180^\circ \times \frac{1}{K_N}, \quad \therefore 1 \angle -180^\circ \times \frac{1}{K_N} = 1.14 \angle -165^\circ$$

$$\therefore K_N = \frac{1 \angle -180^\circ}{1.14 \angle -165^\circ} = \frac{1}{1.14} \angle -15^\circ = 0.877 \angle -15^\circ$$

Hence at $\omega = \omega_{12}$, $|K_N| = 0.877$ and $\angle K_N = -15^\circ$.

From the describing function of backlash nonlinearity we get,

$$\angle K_N = \tan^{-1} \left(\frac{-\cos^2 \beta}{(\pi/2) + \beta + (1/2) \sin 2\beta} \right)$$

At $\omega = \omega_{12}$, $\angle K_N = -15^\circ$, $b/X \rightarrow b/X_2$ and $\beta \rightarrow \beta_2$

$$\therefore \tan^{-1} \left(\frac{-\cos^2 \beta_2}{(\pi/2) + \beta_2 + (1/2) \sin 2\beta_2} \right) = -15^\circ$$

$$\frac{-\cos^2 \beta_2}{(\pi/2) + \beta_2 + (1/2) \sin 2\beta_2} = \tan (-15^\circ)$$

$$\therefore (\pi/2) + \beta_2 + (1/2) \sin 2\beta_2 = \frac{-\cos^2 \beta_2}{\tan (-15^\circ)} = 3.732 \cos^2 \beta_2$$

From the describing function of backlash nonlinearity we get,

$$|K_N| = \frac{1}{\pi} \left[\left(\left(\frac{\pi}{2} \right) + \beta + \left(\frac{1}{2} \right) \sin 2\beta \right)^2 + \cos^4 \beta \right]^{\frac{1}{2}}$$

At, $\omega = \omega_{/2}$, $|K_N| = 0.877$, $b/X \rightarrow b/X_2$ and $\beta \rightarrow \beta_2$

$$\therefore \frac{1}{\pi} \left[\left(\left(\frac{\pi}{2} \right) + \beta_2 + \frac{1}{2} \sin 2\beta_2 \right)^2 + \cos^4 \beta_2 \right]^{\frac{1}{2}} = 0.877$$

On substituting $((\pi/2) + \beta_2 + (1/2)\sin 2\beta_2) = 3.732 \cos^2 \beta_2$ and then squaring we get,

$$(3.732 \cos^2 \beta_2)^2 + \cos^4 \beta_2 = (0.877\pi)^2$$

$$13.928 \cos^4 \beta_2 + \cos^4 \beta_2 = 7.59$$

$$14.928 \cos^4 \beta_2 = 7.59$$

$$\therefore \cos \beta_2 = \left(\frac{7.59}{14.928} \right)^{\frac{1}{4}} = 0.844 \quad ; \quad \therefore \beta_2 = \cos^{-1}(0.844) = 32.4^\circ$$

We know that, $\beta = \sin^{-1}(1 - b/X)$

$$\therefore \beta_2 = \sin^{-1}(1 - b/X_2) \quad (\text{or}) \quad b/X_2 = 1 - \sin \beta_2 = 1 - \sin 32.4^\circ = 0.464$$

RESULT

1. The unstable limit cycle exist when $b/X = 0.316$ and the frequency of oscillation is 0.36 rad / sec.
2. The stable limit cycle exist when $b/X = 0.464$ and the frequency of oscillation is 1.07 rad /sec.

METHOD 2 : USING NICHOLS PLOT

Nichols plot of $G(j\omega)$ when $K = 1$

Given that, $G(s) = K/s (1+s) (1+0.5s)$

Let $K = 1$ and put $s = j\omega$

$$\therefore G(j\omega) = \frac{1}{j\omega (1+j\omega) (1+j0.5\omega)} = \frac{1}{\omega \angle 90^\circ \sqrt{1+\omega^2} \angle \tan^{-1} \omega \sqrt{1+0.25\omega^2} \angle \tan^{-1} 0.5\omega}$$

$$\therefore |G(j\omega)| = \frac{1}{\omega \sqrt{1+\omega^2} \sqrt{1+0.25\omega^2}}$$

$$|G(j\omega)|_{\text{in db}} = 20 \log \left[\frac{1}{\omega \sqrt{1+\omega^2} \sqrt{1+0.25\omega^2}} \right]$$

$$\angle G(j\omega) = -90^\circ - \tan^{-1} \omega - \tan^{-1} 0.5\omega$$

The magnitude of $G(j\omega)$ in db and phase of $G(j\omega)$ are calculated for various values of ω and tabulated in Table 5.2.5. The Nichols plot of $G(j\omega)$ is sketched in an ordinary graph sheet as shown in Figure 5.2.4.

$$-1/K_N = -1 \times \frac{1}{K_N} = 1 \angle -180^\circ \times \frac{1}{|K_N| \angle K_N} = \frac{1}{|K_N|} \angle (-180^\circ - \angle K_N)$$

$$\therefore |1/K_N| = \frac{1}{|K_N|} \quad \text{and} \quad |1/K_N|_{\text{in db}} = 20 \log \frac{1}{|K_N|}$$

$$\angle(-1/K_N) = -180^\circ - \angle K_N$$

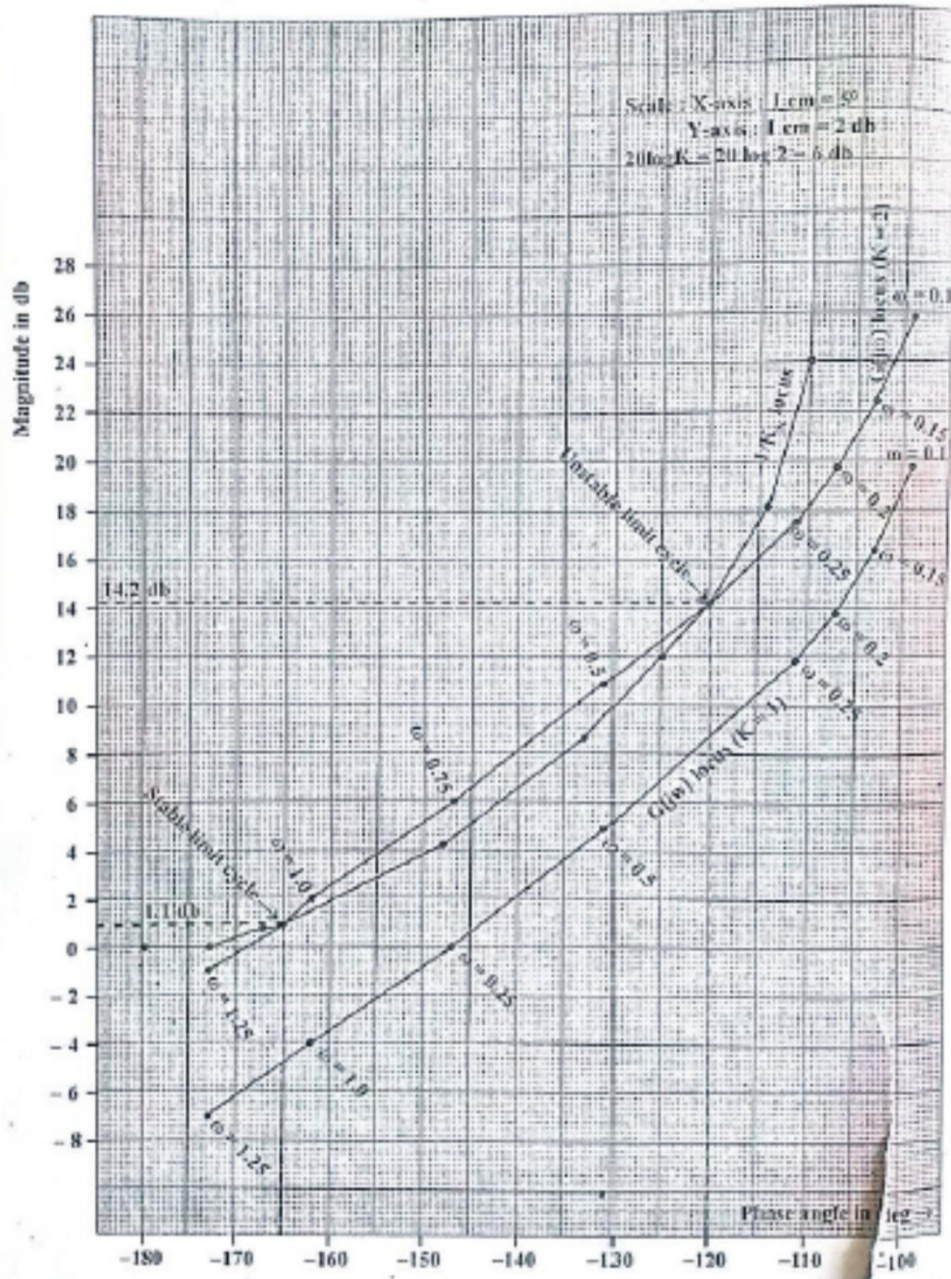


Figure 5.2.4 Nichols plot of $G(j\omega)$ and $-1/K_N$

Table 5.2.5

ω rad/sec	0.1	0.15	0.2	0.25	0.5	0.75	1.0	1.25
$ G(j\omega) $ db	19.9	16.4	13.8	11.7	4.8	0	-4	-7.5
$\angle G(j\omega)$ deg	-99	-103	-107	-111	-131	-147	-162	-173

Nichols plot of $G(j\omega)$ when $K = 2$

When $K = 2$, the magnitude of $G(j\omega)$ increases by an amount, $20 \log K = 20 \log 2 = 6$ db. The phase of $G(j\omega)$ is not altered.

The increase in magnitude independent of frequency. Hence $G(j\omega)$ locus when $K = 2$ is obtained by shifting the locus of $G(j\omega)$ when $K = 1$, by 6 db upwards as shown in Figure 5.2.4.

Nichols plot of $-1/K_N$

The function $-1/K_N$ can be written as,

The magnitude and phase of the describing function of backlash, K_N is listed in the problem in table 5.2.1 for various values of b/X . Using the values of $|K_N|$ and $\angle K_N$ given in table 5.2.1, the values of $|-1/K_N|$ in db and $\angle(-1/K_N)$ are calculated for various values of b/X and listed in table 5.2.6. Using these values the locus of $-1/K_N$ is sketched as shown in fig 5.2.4.

Table 5.2.6

b/X	0	0.2	0.4	1.0	1.4	1.6	1.8	1.9	2.0
$ K_N $	1	0.954	0.882	0.592	0.367	0.248	0.125	0.064	0
$\angle K_N$	0	-6.7°	-13.4°	-32.5°	-46.6°	-55.2°	-66°	-69.8°	-90°
$ -1/K_N $ in db	0	0.4	1.0	4.6	8.7	12.1	18.1	23.9	∞
$\angle(-1/K_N)$ in deg	-180°	-173°	-166°	-148°	-133°	-125°	-114°	-110°	-90°

STABILITY ANALYSIS

Case (i) when $K = 1$

From the Nichols plots it is observed that when $K = 1$, $G(j\omega)$ locus does not enclose $-1/K_N$ locus. Hence the system is stable.

Case (ii) when $K = 2$

From the Nichols plots it is observed that when $K = 2$, $G(j\omega)$ locus, intersects $-1/K_N$ locus at two points. At one intersection point unstable limit cycle exists and at another intersection point stable limit cycle exists.

The coordinates corresponding to unstable limit cycle } = (14.2 db, -120°) = $10^{14.2/20} \angle -120^\circ = 5.1 \angle -120^\circ$

The coordinates corresponding to stable limit cycle } = (11.2 db, -165°) = $10^{11.2/20} \angle -165^\circ = 1.14 \angle -165^\circ$

Note: It is observed that the coordinates corresponding to limit cycles are same as that obtained from polar plot, hence by an analysis similar to that of method-1. We can determine the frequency and b/X corresponding to limit cycles.

RESULT

1. The unstable limit cycle exist when $b/X = 0.316$ and the frequency of oscillation is 0.36 rad / sec .
2. The stable limit cycle exist when $b/X = 0.464$ and the frequency of oscillation is 1.07 rad /sec .

EXAMPLE 5.3

Consider a unity feedback system shown in Figure 5.3.1 having a saturating amplifier with gain K . Determine the maximum value of K for the system to stay stable. What would be the frequency and nature of limit cycle for a gain of $K = 2.59$.

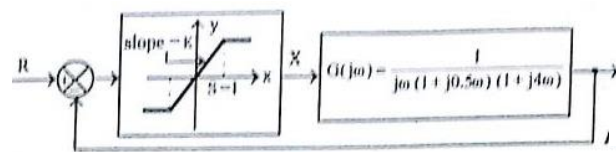


Figure 5.3.1

SOLUTION

The stability of the system can analysed using polar plot. The gain K of the saturating amplifier can be attached to $G(j\omega)$ and amplifier is considered to be an unity gain amplifier.

Polar plot of $G(j\omega)$ when $K = 1$

$$\begin{aligned} \text{Here, } G(j\omega) &= \frac{K}{j\omega (1 + j0.5\omega) (1 + j4\omega)} \\ \text{Let, } K &= 1, \therefore G(j\omega) = \frac{1}{j\omega (1 + j0.5\omega) (1 + j4\omega)} \\ &= \frac{1}{\omega \angle 90^\circ \sqrt{1 + 0.25\omega^2} \angle \tan^{-1} 0.5\omega \sqrt{1 + 16\omega^2} \angle \tan^{-1} 4\omega} \\ \therefore |G(j\omega)| &= \frac{1}{\omega \sqrt{1 + 0.25\omega^2} \sqrt{1 + 16\omega^2}} \\ \angle G(j\omega) &= -90^\circ - \tan^{-1} 0.5\omega - \tan^{-1} 4\omega \end{aligned}$$

The magnitude and phase of $G(j\omega)$ are calculated for various values of ω and listed in Table 5.3.1. Using polar to rectangular conversion the real part and imaginary part of $G(j\omega)$ are determined and listed in Table 5.3.1. The polar plot of $G(j\omega)$ is sketched in an ordinary graph sheet as shown in Figure 5.3.2.

Table 5.3.1

$\omega \text{ rad/sec}$	0.4	0.5	0.6	0.8	1.0	1.2
$ G(j\omega) $	1.299	0.868	0.614	0.346	0.216	0.145
$\angle G(j\omega)$	-159°	-167°	-174°	-184°	-192°	-199°
$G_R(j\omega)$	-1.21	-0.85	-0.61	-0.35	-0.21	-0.14
$G_I(j\omega)$	-0.47	-0.2	-0.06	0.02	0.04	0.05

Polar plot of $G(j\omega)$ when $K = 2.5$

$$\text{When } K = 2.5, |G(j\omega)| = \frac{2.5}{\omega \sqrt{1+0.25\omega^2} \sqrt{1+16\omega^2}}$$

The phase of $G(j\omega)$ is not altered by the term, K . The magnitude and phase of $G(j\omega)$ when $K = 2.5$ are calculated for various values of ω and listed in Table 5.3.2. Using plot to rectangular conversion the real part and imaginary part of $G(j\omega)$ when $K = 2.5$ are determined and listed in Table 5.3.2. The polar plot of $G(j\omega)$ when $K = 2.5$ is sketched in the same graph sheet using the same scale, as shown in Figure 5.3.2.

Table 5.3.2

ω rad/sec	0.6	0.65	0.75	0.8	1.0	1.2
$ G(j\omega) $	1.535	1.313	0.987	0.865	0.54	0.363
$\angle G(j\omega)$	-174	-177	-182	-184	-192	-199
$G_R(j\omega)$	-1.53	-1.31	-0.99	-0.87	-0.53	-0.34
$G_I(j\omega)$	-0.16	-0.07	0.03	0.06	0.11	0.12

Polar plot of $-1/K_N$

The function $-1/K_N$ can be expressed as,

$$\frac{-1}{K_N} = -1 \times \frac{1}{K_N} = 1 \angle -180^\circ \times \frac{1}{K_N}$$

We know that the describing function (K_N) of saturation nonlinearity is given by

$$K_N = \begin{cases} 1 & ; \text{ when } X < S \\ \frac{2K}{\pi} (\beta + \sin\beta \cos\beta) \angle 0^\circ & ; \text{ when } X > S \end{cases} \quad (\because K=1)$$

where, $\beta = \sin^{-1} (S/X)$

and X = Maximum value of input sinusoidal signal

Here, $K = 1$ and $S = 1$

$$\therefore -1/K_N = \begin{cases} 1 \angle -180^\circ & ; \text{ when } X < 1 \\ \frac{\pi}{2 (\beta + \sin\beta \cos\beta)} \angle -180^\circ & ; \text{ when } X > 1 \end{cases}$$

where, $\beta = \sin^{-1} (1/X)$

From the equation of $-1/K_N$ we can say that, the locus of $-1/K_N$ starts at $1 \angle -180^\circ$ (i.e., $=1+j0$) and travels along the negative real axis for increasing values of X as shown in Figure 5.3.2. The locus of $-1/K_N$ is shown as a bold line on the negative real axis.

$$\therefore |G(j\omega)| = 1 \text{ and } \angle G(j\omega) = -180^\circ$$

Let, ω_{11} = Frequency when $G(j\omega) = -1$

$$\therefore \text{At } \omega = \omega_{11}, \angle G(j\omega) = -90^\circ - \tan^{-1} 0.5 \omega_{11} - \tan^{-1} 4\omega_{11} = -180^\circ$$

$$\therefore \tan^{-1} 0.5 \omega_{11} + \tan^{-1} 4\omega_{11} = 90^\circ$$

On taking tan on either side we get,

$$\tan (\tan^{-1} 0.5\omega_{11} + \tan^{-1} 4\omega_{11}) = \tan 90^\circ$$

$$\frac{\tan (\tan^{-1} 0.5\omega_{11}) + \tan (\tan^{-1} 4\omega_{11})}{1 - \tan (\tan^{-1} 0.5\omega_{11}) \times \tan (\tan^{-1} 4\omega_{11})} = \tan 90^\circ$$

$$\frac{0.5 \omega_{11} + 4 \omega_{11}}{1 - 0.5 \omega_{11} \times 4 \omega_{11}} = \infty$$

For the above equation to be infinity, the denominator should be zero.

$$\therefore 1 - 2\omega_{11}^2 = 0 \quad ; \quad \omega_{11}^2 = 1/2 \quad (\text{or}) \quad \omega_{11} = \frac{1}{\sqrt{2}} \text{ rad / sec}$$

$$\text{At } \omega = \omega_{11}, \quad |G(j\omega)| = 1$$

$$\therefore \frac{K}{\omega_{11} \sqrt{1+0.25 \omega_{11}^2} \sqrt{1+16 \omega_{11}^2}} = 1 \quad (\text{or}) \quad K = \omega_{11} \sqrt{1+0.25 \omega_{11}^2} \sqrt{1+16 \omega_{11}^2}$$

$$K = \frac{1}{\sqrt{2}} \sqrt{(1+0.25 \times 0.5) (1+16 \times 0.5)} = 2.25$$

Therefore the system remains stable if, $K < 2.25$

Case (iii) when $K = 2.5$

When $K = 2.5$ the $G(j\omega)$ locus intersects, $-1/K_N$ locus at $-1.11 + j0$. At the intersection point stable limit cycle exists.

$$\text{Coordinate corresponding to stable limit cycle} = -1.11 + j0 = \angle -180^\circ$$

Let, ω_{12} = Frequency of stable limit cycle

$$\text{At } \omega = \omega_{12}, G(j\omega) = 1.11 \angle -180^\circ$$

$$\therefore \text{At } \omega = \omega_{12}, \angle G(j\omega) = -90^\circ - \tan^{-1} 0.5 \omega_{12} - \tan^{-1} 4\omega_{12} = -180^\circ$$

$$\therefore \tan^{-1} 0.5 \omega_{12} + \tan^{-1} 4\omega_{12} = 90^\circ$$

On taking tan on either side we get,

$$\tan (\tan^{-1} 0.5\omega_{12} + \tan^{-1} 4\omega_{12}) = \tan 90^\circ$$

$$\frac{\tan (\tan^{-1} 0.5 \omega_{12}) + \tan (\tan^{-1} 4\omega_{12})}{1 - \tan (\tan^{-1} 0.5\omega_{12}) \times \tan (\tan^{-1} 4\omega_{12})} = \tan 90^\circ$$

$$\frac{0.5 \omega_{12} + 4 \omega_{12}}{1 - 0.5 \omega_{12} \times 4 \omega_{12}} = \infty$$

For the above equation to be infinity, the denominator should be zero.

$$\therefore 1 - 2\omega_{f2}^2 = 0 \quad (\text{or}) \quad \omega_{f2}^2 = 1/2 \text{ rad/sec} \quad (\text{or}) \quad \omega_{f2} = 1/\sqrt{2} \text{ rad/sec}$$

$$\therefore \text{Frequency of limit cycle} = 1\sqrt{2} = 0.707 \text{ rad/sec}$$

RESULT

1. When $K = 1$, the system is stable
2. The system remains stable if $K < 2.25$
3. When $K = 2.5$, a stable limit cycle occurs, whose frequency of oscillation is 0.707 rad/sec.

5.10 PHASE PLANE AND PHASE TRAJECTORIES

The phase plane method of analysis is a graphical method for the analysis of linear and nonlinear systems. The analysis is carried by constructing phase trajectories. It gives an idea about the transient behaviour and stability of the system.

The phase plan analysis is usually restricted to second order systems excited by step or ramp inputs. This analysis technique can be extended to a higher order system if it is approximated as a second order system.

The dynamics of control systems can be represented by differential equations. A second order linear system can be represented by the differential equation.

$$\frac{d^2x}{dt^2} + 2\zeta\omega_n \frac{dx}{dt} + \omega_n^2 x = 0$$

where, x = One of the system variable (e.g. displacement in mechanical system, current in electrical system, etc.,)

ζ = Damping ratio

ω_n = Natural frequency of oscillation.

The state of the second order system represented by equ (5.11.0) can be described by choosing two state variables.

Note: Refer chapter 4 for state, state variables and state space modelling using phase variables.

In state space modelling using phase variables we choose one of the system variable and its derivatives as state variables. Let x_1 and x_2 be the state variables of the second order system.

$$\text{Here } x_1 = x \text{ and } x_2 = dx/dt \quad \dots 5.111$$

On substituting the state variables in equ (5.110) we get,

$$\dot{x}_2 + 2\zeta\omega_n x_2 + \omega_n^2 x_1 = 0 \quad \dots 5.112$$

The state equations of the system are obtained from equations (5.111) and (5.112). The state equations are,

$$\dot{x}_1 = x_2 \quad \dots 5.113$$

$$\dot{x}_2 = -\omega_n^2 x_1 - 2\zeta \omega_n x_2 \quad \dots 5.114$$

For linear systems the state equations are a set of first order linear differential equations and solutions of state equations can be easily obtained by integration. But for nonlinear systems, the state equations are a set of first-order nonlinear differential equations and solving the nonlinear differential equations will not be an easy task. Hence for nonlinear systems the phase plane method of analysis will be an useful tool.

QUESTON BANK

PART A

1. Formulate the choice of state variables?
2. Choose the basic elements used to construct the state diagram?
3. Create the general form of state model of n^{th} order system?
4. The drawback of transfer function model compare with state space model.
5. Compose the Phase variables of a linear time invariant system?
6. Estimate how the modal matrix can be determined
7. Construct the bush or companion form of state model
8. A system is characterized by the differential equation, $\frac{d^2y}{dt^2} + 10\frac{dy}{dt} + 7y - u = 0$
Formulate its transfer function.
9. Estimate the path to diagonalise a matrix
10. Estimate the Eigen values and Eigen vector?
11. Examine the solution of homogenous state solutions.
12. List the solution of non-homogenous state equations.
13. What is resolvent matrix?
14. List the different methods available for computing e^{At} ?
15. Enumerate the properties of state transition matrix.
16. What is state transition matrix?
17. Define the characteristic equation of a matrix.
18. State Cayley-Hamilton theorem.
19. List the disadvantage of state transition matrix using matrix exponential?
20. Illustrate the canonical form of state model?
21. Predict the condition for observability by Gilbert's method.
22. Predict the condition for controllability by Kalman's method.
23. Define observability
24. Define Controllability
25. What is pole placement by state feedback?
26. Write the Ackermann's formula to find the state feedback gain matrix, k .
27. Write the observable phase variable form of state model.
28. Write the controllable phase variable form of state model.
29. Correlate the duality between controllability and observability.
30. What is state observer?
31. Define periodic sampling?
32. Explain Shannon's sampling theorem.
33. Define pulse transfer function?
34. Define Zero order hold?
35. Compare analog and digital controller.
36. Discuss sampled data control systems?
37. Express one sided Z-transform.
38. Compute the infinite and finite geometric series sum formula.
39. Classify the different methods available for inverse Z-transform?
40. List the methods available for the stability analysis of sampled data control systems?
41. Compare the different kind of nonlinearities. Give examples.
42. List the properties of nonlinear systems.
43. Explain jump resonance?

44. Explain how limit cycles are formed?
45. Define a describing function?
46. List the different types of friction?
47. Explain hysteresis and backlash?
48. Classify the methods available for the analysis of nonlinear system?
49. Explain the non linearities that are introduced in the systems?
50. Trace the input-output characteristic of a relay with dead zone and hysteresis.

PART- B

1. Develop the state model of electro mechanical system whose speed can be controlled below the rated value.
2. Construct the canonical state model of the system, whose transfer function is

$$T(s) = \frac{2(s+5)}{[(s+2)(s+3)(s+4)]}$$
3. A feedback system has a closed-loop transfer function $\frac{Y(s)}{U(s)} = \frac{10(s+4)}{s(s+1)(s+3)}$ Construct state model for this system and give block diagram for the state model.

4. Develop the state model for Ward Leonard system
5. A linear time invariant system is described by the following state model.

$$\begin{bmatrix} \dot{X}_1 \\ \dot{X}_2 \\ \dot{X}_3 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -6 & -11 & -6 \end{bmatrix} \begin{bmatrix} X_1 \\ X_2 \\ X_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 2 \end{bmatrix} [U] \quad y = [1 \quad 0 \quad 0] \begin{bmatrix} X_1 \\ X_2 \\ X_3 \end{bmatrix}$$

Formulate this state model into a canonical state model.

6. A linear time invariant system is described by the following state model.

$$\begin{bmatrix} \dot{X}_1 \\ \dot{X}_2 \\ \dot{X}_3 \end{bmatrix} = \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix} \begin{bmatrix} X_1 \\ X_2 \\ X_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 2 \end{bmatrix} [U] \quad y = [1 \quad 0 \quad 0] \begin{bmatrix} X_1 \\ X_2 \\ X_3 \end{bmatrix}$$

Modify this state model into a canonical state model.

7. Given that $A_1 = \begin{bmatrix} \sigma & 0 \\ 0 & \sigma \end{bmatrix}$; $A_2 = \begin{bmatrix} 0 & \omega \\ -\omega & 0 \end{bmatrix}$; $A = \begin{bmatrix} \sigma & \omega \\ -\omega & \sigma \end{bmatrix}$ Inspect e^{At} .

8. A linear time invariant system is described by the following state model.

$$\begin{bmatrix} \dot{X}_1 \\ \dot{X}_2 \\ \dot{X}_3 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -6 & -11 & -6 \end{bmatrix} \begin{bmatrix} X_1 \\ X_2 \\ X_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 2 \end{bmatrix} [U] \quad y = [1 \quad 0 \quad 0] \begin{bmatrix} X_1 \\ X_2 \\ X_3 \end{bmatrix}$$

Compute the state transition matrix, e^{At} .

9. Discover the solution of Non Homogeneous state equations.

10. For a system represented by state equation $\dot{X}(t) = AX(t)$ The response is $X(t) = \begin{bmatrix} e^{-2t} \\ -2e^{-2t} \end{bmatrix}$ when $X(0) = \begin{bmatrix} 1 \\ -2 \end{bmatrix}$ and $X(t) = \begin{bmatrix} e^{-t} \\ -e^{-t} \end{bmatrix}$ when $X(0) = \begin{bmatrix} 1 \\ -1 \end{bmatrix}$ Examine the system matrix A and the state transition matrix.
11. For $A = \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix}$ Determine the state transition matrix e^{At} using Cayley-Hamilton theorem.
12. A linear time invariant system is characterised by homogeneous state equation.

$$\begin{bmatrix} \dot{X}_1 \\ \dot{X}_2 \end{bmatrix} = \begin{bmatrix} 1 & 0 \\ 1 & 1 \end{bmatrix} \begin{bmatrix} X_1 \\ X_2 \end{bmatrix}$$
Compute the solution of the homogeneous equation, assuming the initial state vector $X_0 = \begin{bmatrix} 1 \\ 0 \end{bmatrix}$
13. Consider a linear system described by the transfer function $\frac{Y(s)}{U(s)} = \frac{10}{s(s+1)(s+2)}$ Design a feedback controller with a state feedback so that the closed loop poles are placed at $-2, -1 + j1, -1 - j1$
14. The state model of a system is given by

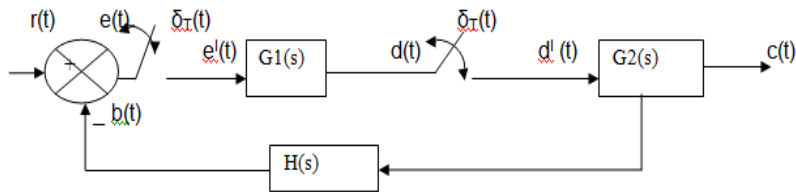
$$\begin{bmatrix} \dot{X}_1 \\ \dot{X}_2 \\ \dot{X}_3 \end{bmatrix} = \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix} \begin{bmatrix} X_1 \\ X_2 \\ X_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 2 \end{bmatrix} [U] \quad y = \begin{bmatrix} 1 & 0 & 0 \end{bmatrix} \begin{bmatrix} X_1 \\ X_2 \\ X_3 \end{bmatrix}$$
Formulate the state model to observable phase variable form.
15. Consider the system described by the state model
16. $\dot{X} = AX, Y = CX$ where $A = \begin{bmatrix} -1 & 1 \\ 1 & -2 \end{bmatrix}; C = \begin{bmatrix} 1 & 0 \end{bmatrix}$ Design a full-order state observer. The desired eigen values for the observer matrix are $\mu_1 = -5, \mu_2 = -5$
17. The state model of a system is given by

$$\begin{bmatrix} \dot{X}_1 \\ \dot{X}_2 \\ \dot{X}_3 \end{bmatrix} = \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix} \begin{bmatrix} X_1 \\ X_2 \\ X_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 2 \end{bmatrix} [U] \quad y = \begin{bmatrix} 1 & 0 & 0 \end{bmatrix} \begin{bmatrix} X_1 \\ X_2 \\ X_3 \end{bmatrix}$$
Formulate the state model to Controllable phase variable form.
18. The state model of a system is given by

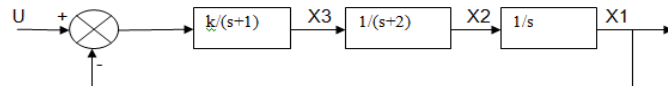
$$\begin{bmatrix} \dot{X}_1 \\ \dot{X}_2 \\ \dot{X}_3 \end{bmatrix} = \begin{bmatrix} 0 & 0 & 1 \\ -2 & -3 & 0 \\ 0 & 2 & -3 \end{bmatrix} \begin{bmatrix} X_1 \\ X_2 \\ X_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 2 \end{bmatrix} [U] \quad y = \begin{bmatrix} 1 & 0 & 0 \end{bmatrix} \begin{bmatrix} X_1 \\ X_2 \\ X_3 \end{bmatrix}$$
Test whether the system is completely controllable and observable by Kalman's Test.
19. A single-input system is described by the following state equation.

$$\begin{bmatrix} \dot{X}_1 \\ \dot{X}_2 \\ \dot{X}_3 \end{bmatrix} = \begin{bmatrix} -1 & 0 & 1 \\ 1 & -2 & 0 \\ 2 & 1 & -3 \end{bmatrix} \begin{bmatrix} X_1 \\ X_2 \\ X_3 \end{bmatrix} + \begin{bmatrix} 10 \\ 1 \\ 0 \end{bmatrix} [U]$$
Design a state feedback controller which will give closed-loop poles at $-1+j2, -1-j2, -6$

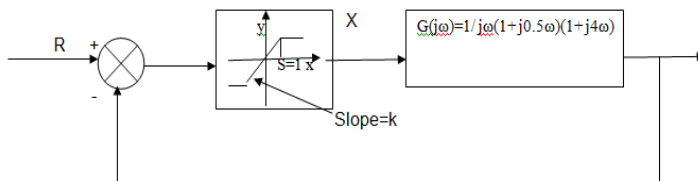
20. Estimate the analysis of sampling process in frequency domain.
21. Determine the Z-transform for the following discrete sequences (a) $f(k)=\{3,2,5,7\}$ (b) $(1/2)^k u(k)$ (c) $f(k)=K^2$
22. Determine $C(Z)/R(Z)$ for the given closed loop sampled data control systems. Assume the sampler to be of impulse type.



23. Evaluate the difference equation $c(k+2) + 3c(k+1) + 2c(k) = u(k)$ Given that $c(0)=1$; $c(1)=-3$; $c(k)=0$ for $k<0$
24. Estimate the stability of sampled data control systems represented by the following characteristic equation $z^4 - 1.7z^3 + 1.04z^2 + 0.024 = 0$
25. Determine the one sided z-transform of the discrete sequence generated by mathematically sampling the following continuous time functions $f(t) = \cos wt$
26. Assess the describing function. Derive the describing function of a relay with hysteresis and dead zone.
27. (a). Explain Liapunov stability and instability theorems.
(b). Determine the stability range for the gain 'k' of the system shown in the figure.



28. (a). Determine Krasovski's theorem of stability.
(b). Consider the nonlinear system
29. $\dot{x}_1 = -x_1 - x_2^2$, $\dot{x}_2 = -x_2$ Justify the stability of the equilibrium points using Krasovski's method.
30. Estimate the describing function of Dead-zone and saturation nonlinearity.
31. Consider a unity feedback system as shown in figure below having saturating amplifier with gain k. Determine the maximum value of k for which the system to stay stable.



32. Estimate the describing function of saturation nonlinearity.